

**COMBUSTION MODELLING OF
PULVERISED COAL BOILER FURNACES
FUELLED WITH
ESKOM COALS**

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**A dissertation submitted to the Faculty of Engineering, University of the Witwatersrand,
Johannesburg, in fulfilment of the requirements for the degree of Master in Science in
Engineering**

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DECLARATION

This dissertation is the original and independent work of the author except where acknowledged otherwise in the text. It is being submitted for the degree of Master of Science in Engineering at the University of the Witwatersrand, Johannesburg. It has not been submitted before for any degree or examination to any other University.

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ABSTRACT

Combustion modelling of utility furnace chambers provides a cost efficient means to extrapolate the combustion behaviour of pulverised fuel (pf) as determined from drop tube furnace (DTF) experiments to full scale plant by making use of computational fluid dynamics (CFD). The combustion model will be used to assimilate essential information for the evaluation and prediction of the effect of

- changing coal feedstocks
- proposed operational changes
- boiler modifications.

TRI commissioned a DTF in 1989 which has to date been primarily used for the comparative characterisation of coals in terms of combustion behaviour. An analysis of the DTF results allows the determination of certain combustion parameters used to define a mathematical model describing the rate at which the combustion reaction takes place. This model has been incorporated into a reactor model which can simulate the processes occurring in the furnace region of a boiler, thereby allowing the extrapolation of the DTF determined combustion assessment to the full scale. This provides information about combustion conditions in the boiler which in turn are used in the evaluation of the furnace performance.

Extensive furnace testwork of one of Eskom's wall fired plant (Hendrina Unit 9) during 1996, intended to validate the model for the applications outlined above, included the measurement of :

- gas temperatures
- O₂, CO₂, CO, NO_x and SO₂ concentrations
- residence time distributions
- combustible matter in combustion residues extracted from the furnace
- furnace heat fluxes.

The coal used during the tests was sampled and subjected to a series of chemical and other lab-scale analyses to determine the following :

- physical properties
- composition
- devolatilisation properties
- combustion properties

The same furnace was modelled using the University of Stuttgart's AIOLOS combustion code, the results of which are compared with the measured data.

A DTF derived combustion assessment of a coal sampled from the same site but from a different part of the beneficiation plant, which was found to burn differently, was subsequently used in a further simulation to assess the sensitivity of the model to char combustion rate data. The results of these predictions are compared to the predictions of the validation simulation.

It was found that the model produces results that compare well with the measured data. Furthermore, the model was found to be sufficiently sensitive to reactivity parameters of the coal. The model has thereby demonstrated that it can be used in the envisaged application of extrapolating DTF reactivity assessments to full scale plant. In using the model, it has become apparent that the evaluations of furnace modifications and assessments of boiler operation lie well within the capabilities of the model.

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I would like to dedicate this thesis to my family and fiancé, Véronique, whose patience and moral support made this work possible.

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LIST OF SYMBOLS

T	: temperature
T_g	: temperature of furnace gas
T_c	: wall temperature of the suction channel
t	: time
k	: thermal conductivity
k_{devol}	: devolatilisation reaction rate coefficient
k_{char}	: char combustion reaction rate coefficient
m	: mass
dp	: assumed particle diameter
\bar{d}	: diameter
x	: distance
F	: volumetric flow
P	: pressure
v	: velocity
g	: gravitational constant (9.81 m/s^2)
A	: ash content
A_{particle}	: assumed particle surface area
A_r	: area available for radiative heat transfer
A_c	: area available for convective heat transfer
VM	: volatile matter content of char (fractional)
V_x	: extent of devolatilisation
E_a	: activation energy
R	: Universal Gas Constant
C_p	: particle heat capacity
Nu	: Nusselt number
CE	: combustion extent
D	: diffusional coefficient
n_B	: swelling index
Δh	: enthalpy of formation
S_o	: source term of field variable
u	: cartesian component of velocity in the x-direction
KE_{turb}	: kinetic turbulent energy
C_μ	: turbulent viscosity model constant
X, Y	: parameters of hydrocarbon description C_xH_y

LIST OF GREEK SYMBOLS

λ	: convective heat exchange coefficient
ε	: emissivity
α	: convective heat transfer coefficient
σ	: Stefan Boltzmann constant
τ	: time constant
τ_d	: dead time
ρ	: density
μ	: viscosity
Φ	: general field variable
σ_Φ	: stress associated with field variable Φ

1. INTRODUCTION

Eskom utilised approximately 70 million tons of coal in 1995 in pulverised coal boilers. As coal usage increases and reserves are depleted, Eskom will face changing feedstocks. It is accepted that the conventional proximate analysis, calorific value, ash and petrographic analyses alone cannot adequately quantify the performance of steam raising coals¹. It is therefore difficult to predict the combustion behaviour of a coal without some other means of combustion behaviour assessment or prior operating experience with the coal in question.

In order to assess a coal's combustion behaviour, TRl commissioned a Drop Tube Furnace (DTF) in 1989 and has to date assimilated an extensive database of physical-, chemical- and combustion behaviour information of Eskom and other steaming coals. The combustion behaviour information of a coal includes a description for the rate of char combustion which, under pulverised fuel (pf) firing conditions, is the slowest of the combustion reactions. Ignition properties are estimated from devolatilisation characteristics.

DTF coal characterisations have been mainly used for the comparative assessment of the combustion behaviour of coals, with one coal generally being considered a reference coal whose combustion performance was considered acceptable.

A number of attempts have been made to correlate the DTF to full scale plant, and while the DTF can reproduce most of the combustion conditions typical in the full scale plant, this requires detailed measurements of conditions in the boiler, which can only be obtained from experiments on full-scale plant. These are both costly and time-consuming. However, the rapid development of numerical methods, physical- and chemical sub-models and computer hardware in the last 10 to 15 years has made computational fluid dynamics (cfd) based methods valuable for the engineering assessment of furnace processes. These processes include the combustion of pf and the formation of nitrogen pollutants.

This report presents the **ADVANCED SIMULATION CODE FOR FLOWS IN COMBUSTION SYSTEMS (JLOS)**, for the analysis of the flow field, combustion, heat transfer and nitrogen pollutants in multi-burner pulverised coal furnaces to extrapolate DTF results to full scale plant to provide essential information for the evaluation of

- changing coal feedstocks
- operational changes for optimisation purposes
- boiler modifications for NO_x reduction or optimised heat release.

2. METHODOLOGY AND THEORETICAL BACKGROUND

2.1 OVERVIEW

2.1.1 THEORETICAL OVERVIEW

Calculation based methods have been used in the design of fossil fuel fired combustion chambers to refine, to some extent, the design and operation optimisation experiences since the early 20th century. Particularly the advent and rapid development of computer technology has spurred the evolution of calculation based simulation tools. These are intended to help the engineer in designing and diagnosing combustion plant to improve combustor efficiencies and further, as a broader spectrum of fossil fuels is being considered, the capability of combustion plant to burn certain fuels. A number of models are available, some of which have particular applications. An attempt has been made to broadly classify these (Khalil⁴³, 1982) in terms of the information they yield on their application.

ZERO Dimensional Models

These models are used to predict average combustor heat transfer, furnace efficiency and other overall averaged properties such as wall emissivity, gas emissivity, gas composition and combustor temperature .

ONE Dimensional Models

These models allow the prediction of the aforementioned combustion environment properties in length or time. This type of model is restricted to flows with little or no recirculation although some recirculation effect may be taken into account by considering a combination of plug flow and well stirred reactors.

TWO and THREE Dimensional models

These models allow the determination of the spatial distribution of fluid and energy flow through the combustion system. Most models incorporate a variety of submodels for turbulence, reaction and heat transfer. The validity of these sub-models to various furnace and combustion chamber configurations has been assessed with

comparisons with experiments. Results from the above models require critical assessment since the models and their interactions are complex and relevant experimental data is limited. The prediction of trends of the effect of operational variables on the overall combustor performance has however improved dramatically with the development of computer technology which has provided the possibility to improve certain submodels, particularly the turbulence submodel.

2.1.2 OVERVIEW OF METHODOLOGY

Extensive testing of Hendrina's Unit 9 furnace was performed during February 1988 to obtain a set of furnace condition data that could be used to validate the AIOLOS advanced three dimensional combustion model for Eskom's coal fired plant. The furnace testwork included the measurement of :

- gas temperatures
- O₂, CO₂, CO, NO_x and SO₂ concentrations
- combustible matter in combustion residues extracted from the furnace
- residence time distributions
- furnace heat fluxes

Hendrina's boiler 9 coal feedstock was extensively sampled during the furnace tests, and characterised and analysed using the DTF and TRI's coal laboratory. The results of these tests, together with furnace design and operation information, were used to set up the AIOLOS combustion model, which was then run at the University of Stuttgart. The configuration of the model for the furnace in question was used to train an Eskom staff member in the use thereof, while the model predictions were compared to the measured data.

A DTF derived combustion assessment of a coal sampled from the same site but from a different part of the beneficiation plant, which was found to burn differently, was subsequently used in a further simulation to assess the sensitivity of the model to char combustion rate data. The results of these predictions are compared to the predictions of the validation simulation.

2.2 FURNACE TESTING

2.2.1 SELECTION OF BOILER TEST SITE

The reason for selecting Hendrina power station was that the furnace is relatively small (200 MWe) compared to the furnaces of Eskom's >500 MWe boilers, and hence the furnace could be probed to sufficient depth to obtain a set of data that could validate the AIOLOS combustion code.

2.2.2 FURNACE PROBING

Probing of the boiler's combustion chamber through various observation ports, situated on the side wall of the furnace, was performed by making use of water cooled suction probes ranging in length from 6 to 12m. The probes were used to measure furnace gas temperatures, O₂, CO, CO₂, NO_x and SO₂ concentrations, as well as extracting combustion residue samples at 0.5m intervals up to a furnace depth of around 6m, depending on what probe could be used at a particular access point. Figure 1 shows a schematic of the testing equipment used.

2.2.3 FURNACE GAS TEMPERATURE MEASUREMENT

Furnace gas temperatures are ideally directly measured using R-type (Platinum/Platinum -10% Rhodium) thermocouples supported by the aforementioned water cooled probes and protected at the junction from radiative heat exchange by a double ceramic sheath. Hot gas is drawn over the shielded junction at velocities high enough to render the temperature measurement independent of gas velocity (Laird et al² (1954)). This means of measurement did however fail frequently during the testing as a result of slag deposits in the ceramic shield, which in time blocked the flow of the hot gas. This condition became evident when the measured furnace temperature suddenly dropped several hundred degrees Celsius. Once evident, measurements were discarded, fouled components removed and replaced.

An indirect gas temperature measurement method (Idiatulin³ et al (1977)) was subsequently adopted, which uses three axially aligned K-Type (Nickel / Chrome-Nickel) thermocouples situated in the suction probe (see Figure 2).

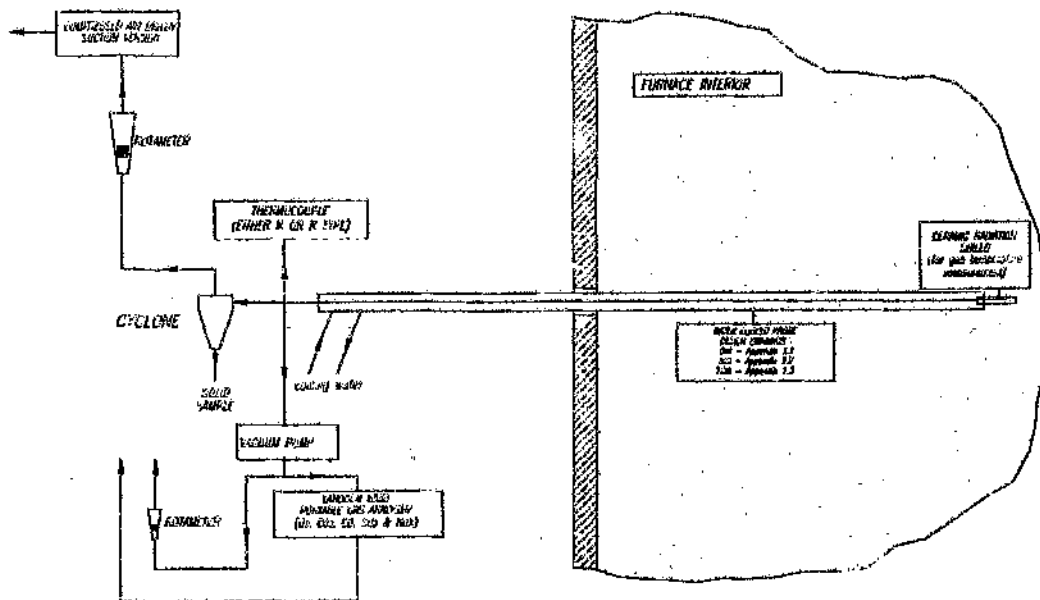


Figure 1 : Schematic of Furnace Probing Equipment
 (A4 copy of Figure 1 is attached in Appendix 1.1)

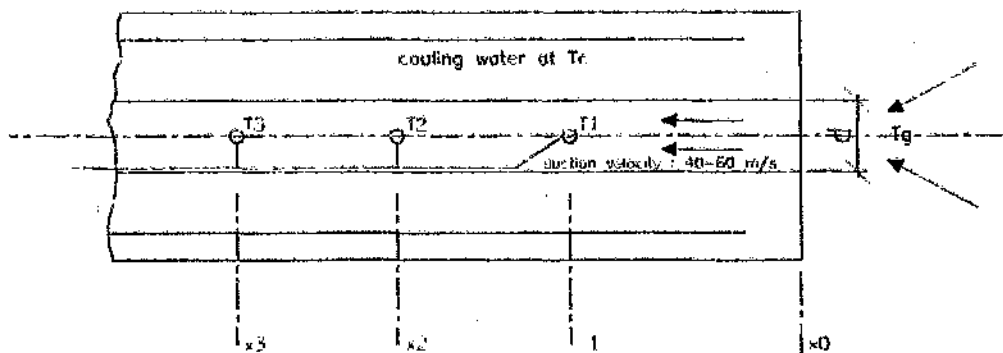


Figure 2 : Arrangement of Thermocouples in Suction Probe

Because the gas sample is quenched the instant it enters the probe (resulting in molten ash particles solidifying before they adhere to any probe surfaces) and because the probe surfaces are not as conducive to slag deposits forming on them as the ceramic surfaces are, this method of temperature measurement is considerably less problematic. However, the inlet gas temperature needs to be extrapolated from the temperatures measured using the thermocouples. The extrapolated temperature is determined as follows :

$$T_{g_i} = T_{x_i} + \frac{\epsilon \cdot \sigma}{\alpha} \left[(T_{x_i})^4 - T_c^4 \right] \quad (1)$$

$i = 1, 2, 3$

where

- ϵ = emissivity of the thermocouple junction
- α = convective heat transfer coefficient between the gas and the thermocouple junction
- σ = Stefan-Boltzmann constant

The wall temperature of the suction channel, T_c , can be determined with sufficient accuracy by measurement with the thermocouple without suction, when the thermocouple assumes the temperature of the surrounding walls. It was further found by the developers of this method that at a distance of $x = 2.5 d$ from the probe tip, the radiation from the furnace no longer affects the thermocouple. An extrapolated temperature can be determined by solving equation (1) simultaneously with two of the three thermocouple readings, assuming that ϵ remained constant, which can be reasonably assumed, as the temperature measured using the three thermocouples varied by only 200 - 400°C. The suction velocity through the probe (40-60 m/s) was sufficiently high to ensure turbulent flow past the thermocouples and hence α could be considered constant.

This temperature measurement system did however only function well when the thermocouples were in equilibrium and the furnace temperature remained reasonably constant over approx. 3s. When the three extrapolated temperatures differed significantly, the measurements were discarded. For the purpose of this investigation, a temperature variance of more than 20°C between the maximum and minimum predicted value was considered significant. Further analysis would however have to be performed to identify a steady state condition of the thermocouples.

Figure 3 shows a graph of the extrapolated temperatures as a function of furnace depth and time since start of measurement. Note the fluctuations of up to 70°C in gas temperature at a single furnace depth.

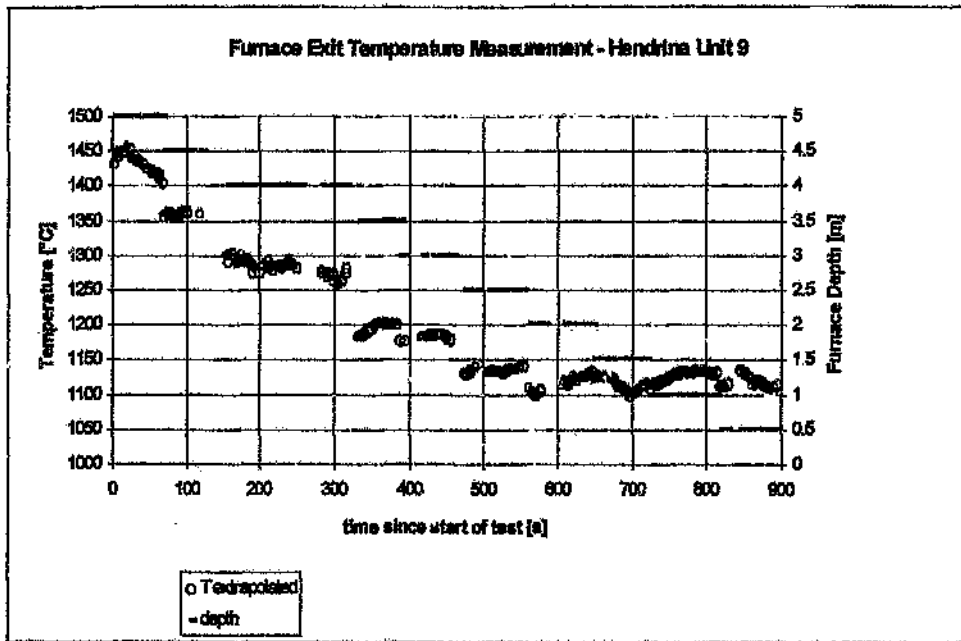


Figure 3 : Furnace Gas Temperature Measurements

2.2.4 GAS SPECIES CONCENTRATION MEASUREMENT

A furnace gas sample was drawn from the suction stream of the water cooled probe, as shown in Figure 1, using a suction pump. A portion of this stream was then continuously passed through a membrane dryer before entering the Landcom 6500 portable flue gas analyser. This flue gas analyser uses dedicated electrochemical cells for the determination of the following gases :

<u>GAS</u>	<u>RANGE</u>	<u>ACCURACY</u>	<u>RESOLUTION</u>
O ₂	0 - 25% vol.	± 1%	± 0.1% vol.
CO (low range)	0 - 2000 ppm	± 4%	± 1 ppm
CO (high range)	0 - 40000 ppm	-	± 100 ppm
SO ₂	0 - 2000 ppm	± 4%	± 1 ppm
NO	0 - 1000 ppm	± 4%	± 1 ppm
CO ₂	calculated	-	-

The analyser was calibrated using prepared standard certified sample gases prior to every test. The standard gases used were as follows :

<u>GAS</u>	<u>COMPOSITION</u>
O ₂	5.20%, balance N ₂
CO (low range)	120 ppm, 14% CO ₂ , balance N ₂
CO (high range)	not calibrated
SO ₂	768 ppm, balance N ₂
NO	474 ppm, balance N ₂

2.2.5 RESIDENCE TIME DISTRIBUTION MEASUREMENTS

For the purpose of this investigation, the residence time is defined as the time a pf particle has spent in the furnace region of the boiler. Due to the complex turbulent flowfield in the furnace, particularly in the burner regions, identical pf particles may spend different times in the furnace depending on the flow path they have taken through it. The residence time distribution (RTD) is by definition the statistical distribution of residence times that pf particles entering the furnace through a particular burner may spend in the furnace. The furnace exit plane does however have the same cross section as the furnace and the flue passing through it cannot be assumed to be perfectly mixed. In view of this, the RTD's from each of the three burner rows to a number of points on the furnace exit plane were determined.

It was however not practical to measure the RTD of a batch of particles that simultaneously enter the furnace through a burner, because this would require that the particle be identifiable based on some property unique to the batch in question and furthermore, the identification would have to be made at very short time increments throughout the test.

Because of the small size of pf particles (75% <75µm), it can be reasonably assumed that the RTD of particles will not differ much from the RTD of a gas molecule. Indeed, the free fall velocity of a 50µm particle would be only about 0.05 m/s compared to the gas velocities of around 10m/s at furnace exit.

Experiments with various tracers during a previous furnace RTD investigation at Matimba Power Station (Blenkinsop et al⁴) showed that sulphur powder could be used. A 5kg charge of sulphur was injected very rapidly (using air cannons) into the primary air / pf stream just before the burner inlet of one burner. The injected sulphur was sufficient to increase the furnace exit SO₂ concentration tenfold.

The sulphur sublimates at around 440°C, and then oxidises when in the gas phase. The rate of formation of SO₂ from the sulphur powder is therefore similar to the combustion rates of volatile products from the coal (diffusion limited), and for the purpose of the RTD estimations, was considered to occur instantaneously. Figure 4 shows a schematic of the testing equipment used.

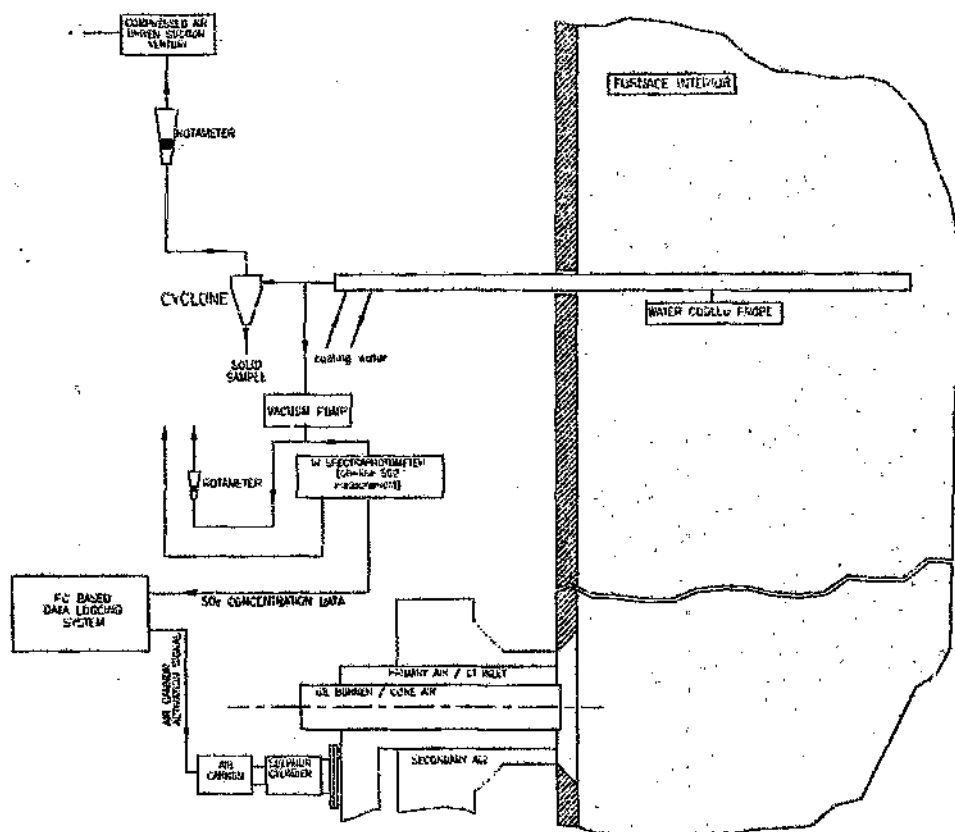


Figure 4 : Schematic of RTD Measurement Equipment
(A4 copy of Figure 4 is attached in Appendix 1.1)

The SO₂ concentration vs. time profile measured at furnace exit could therefore be considered to be a RTD of the combined furnace and probing system given an instantaneous SO₂ source just in front of the burners where sulphur was being injected.

Plot of experimental overall response to sulphur tracer injection

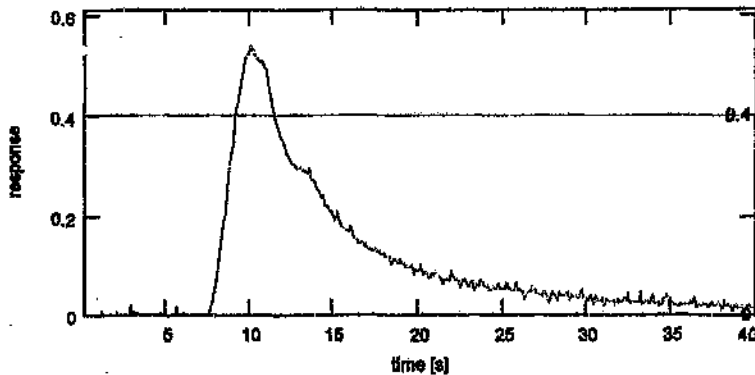


Figure 5 : Graph of overall RTD Response

The suction pyrometer used to sample the furnace exit does, however, have its own RTD (refer to Figure 6), necessitating that the overall measured RTD (refer to Figure 5) be corrected. In order to perform this correction, the RTD of the probe had to be determined. This was done measuring the SO_2 concentration vs. time profile through the probe given identical sampling conditions with a SO_2 sample being injected into the sample stream just in front of the probe using an injection device designed to disrupt the sample stream as little as possible. The design drawing of this device is presented in Appendix 1.1.5.

Plot of experimental probe response to SO_2 tracer injection

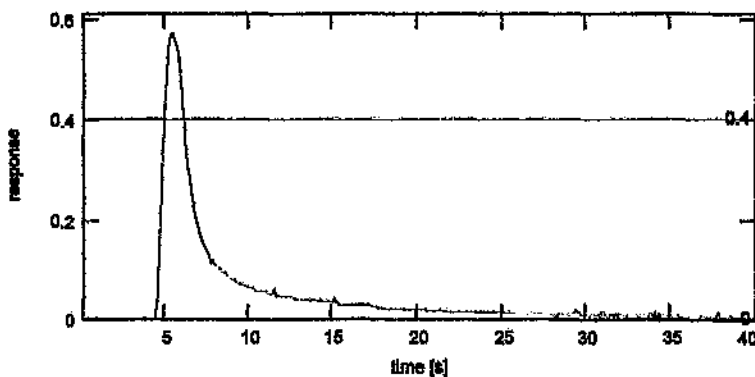


Figure 6 : Graph of Probe RTD Response

It was found that the experimentally determined concentration profiles determined by the tracer experiments could be satisfactorily described by fitting the data to a curve describing the response of a second order system with a lag or dead time to an impulse disturbance. The general form of the function in the time domain is given by :

$$f(t) = C \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[e^{-\tau_1(t - \tau_d)} - e^{-\tau_2(t - \tau_d)} \right] \cdot \phi(t - \tau_d) \quad (2)$$

where

τ_d = dead time

τ_1 = time constant 1

τ_2 = time constant 2

C = constant required to normalise the RTD so that $\int_0^{\infty} f(t) dt = 1$

$\phi(t - \tau_d)$ = Heavyside function, i.e. $\phi = 0$ for $t < \tau_d$
 $\phi = 1$ for $t \geq \tau_d$

The effect of the RTD on any disturbance (e.g. the tracer injection) entering the otherwise steady state system (i.e. with constant inputs and outputs) can be described by the convolution integral (Levenspiel⁶ (1972)).

$$C_{out}(t) = \int_0^t C_{in}(t-t') \cdot E_o(t') dt' \quad (3)$$

where

$E_o(t)$ = RTD of the system (i.e. the furnace and probe combination)

$C_{in}(t)$ = time domain description of the disturbance entering system (i.e. the sulphur injection resulting in a virtually instantaneous source of SO₂ at the burner mouth)

$C_{out}(t)$ = time domain description of a system output (e.g. SO₂) at the outlet of the system (i.e. the furnace exit)

The overall RTD, $E_o(t)$, of the furnace and probe combination can be considered as two independent systems in series, each with it's own RTD. This system is illustrated in Figure 7.

If the system input ($C_{in}(t)$) is given, the outlet concentration vs. time distribution function ($C_{out}(t)$) and the RTD of the probing system ($E_{probe}(t)$) is measured, then the RTD of the first system, i.e. the furnace ($E_{furnace}(t)$), can be determined.

Equation (4) is obtained by Laplace transforming equation (3).

$$\overline{C_{out}(s)} = \overline{C_{in}(s)} * \overline{E_o(s)} \quad (4)$$

The reason for implying that the RTD of the probing system ($E_{probe}(t)$) can be measured is that given an instant SO_2 injection at the probe inlet, the SO_2 concentration vs. time profile at the outlet has identical distribution and dead time characteristics as the RTD of the probing system because the Laplace transform of the impulse input is unity.

Levenspiel⁶ (1972) refers to C_{out} as the convolution of E_o with C_{in} . C_{out} is, however, also the convolution of the Laplace domain description of the furnace output with E_{probe} , which in turn is the convolution of $E_{furnace}$ with C_{in} .

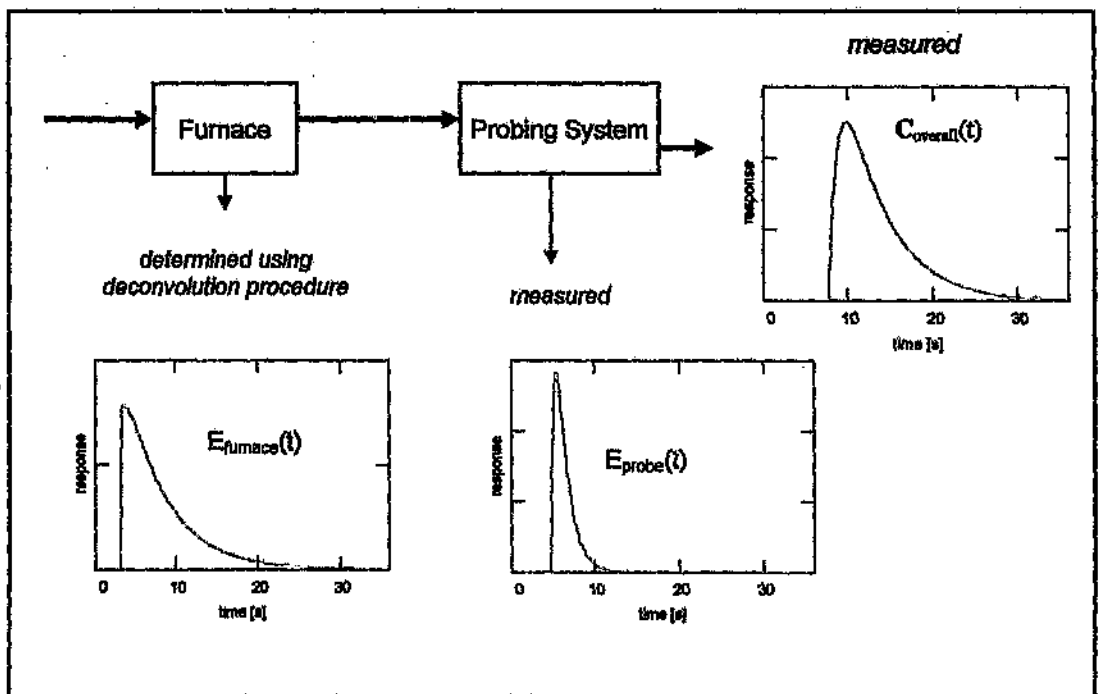


Figure 7 : Modification of an impulse tracer signal , $C_{in}(t)$, on passing through two systems in series

The Laplace domain description of the entire system is therefore :

$$\overline{C_{out}(s)} = \overline{C_{in}(s)} * \overline{E_{furnace}(s)} * \overline{E_{probe}(s)} \quad (5)$$

The mathematical properties of the convolution operator allow a deconvolution to obtain a Laplace domain description of E_{furnace} . The convolution operation becomes a multiplication in the Laplace domain, while a deconvolution a division operation.

$$\overline{E_{\text{furnace}}(s)} = \frac{\overline{C_{\text{out}}(s)}}{\left(\overline{C_{\text{in}}(s)} \cdot \overline{E_{\text{probe}}(s)}\right)} \quad (6)$$

In the Laplace domain, equation (2), the general form of the time domain distribution, becomes :

$$\overline{F(s)} = C \frac{e^{-\tau_d s}}{(1 + \tau_1 s)(1 + \tau_2 s)} \quad (7)$$

Similar expressions in the Laplace domain for the time domain overall (denoted by the subscript : *overall*) and probe (denoted by the subscript : *probe*) responses can be obtained :

$$\overline{E_{\text{probe}}(s)} = C_{\text{probe}} \frac{e^{-\tau_{d \text{ probe}} s}}{(1 + \tau_{\text{probe} 1} s)(1 + \tau_{\text{probe} 2} s)} \quad (8)$$

$$\overline{C_{\text{out}}(s)} = C_{\text{overall}} \frac{e^{-\tau_{\text{overall} d} s}}{(1 + \tau_{\text{overall} 1} s)(1 + \tau_{\text{overall} 2} s)} \quad (9)$$

The Laplace transform of the system input, which for the purpose of this investigation was considered an impulse input, is given by :

$$\overline{C_{\text{in}}(s)} = 1 \quad (10)$$

Expressions (8), (9) and (10) can now be substituted into (6) and by taking the inverse Laplace transform of the resulting expression, the RTD of the furnace ($E_{\text{furnace}}(t)$) in the time domain can be obtained :

$$E_{\text{furnace}}(t) = \frac{C_{\text{overall}}}{C_{\text{probe}}} \left[A \cdot e^{-\tau_{\text{overall}1} \cdot (t - \tau_{\text{furnace}})} + B \cdot e^{-\tau_{\text{overall}2} \cdot (t - \tau_{\text{furnace}})} \right] \cdot \Phi(t - \tau_d) \quad (11)$$

where

$$A = \frac{(\tau_{\text{overall}1} - \tau_{\text{probe}1}) \cdot (\tau_{\text{overall}1} - \tau_{\text{probe}2})}{(\tau_{\text{overall}1} - \tau_{\text{overall}2})}$$

$$B = \frac{(\tau_{\text{overall}2} - \tau_{\text{probe}1}) \cdot (\tau_{\text{overall}2} - \tau_{\text{probe}2})}{(\tau_{\text{overall}2} - \tau_{\text{overall}1})}$$

$$\tau_{\text{furnace}} = \tau_{\text{overall}d} - \tau_{\text{probe}d}$$

2.2.6 HEAT FLUX MEASUREMENTS

25 heat flux probes were installed into the left wall of the furnace and monitored on-line during the testing period. Figure 8 shows a schematic of a heat flux probe mounted in the furnace wall. The linearised temperature transmitters output a 4-20 mA signal to a centrally situated PC based data logger. The temperature measurements as well as the calculated heat fluxes were logged at 1 minute intervals. Figure 9 shows the positions of the heat flux probes

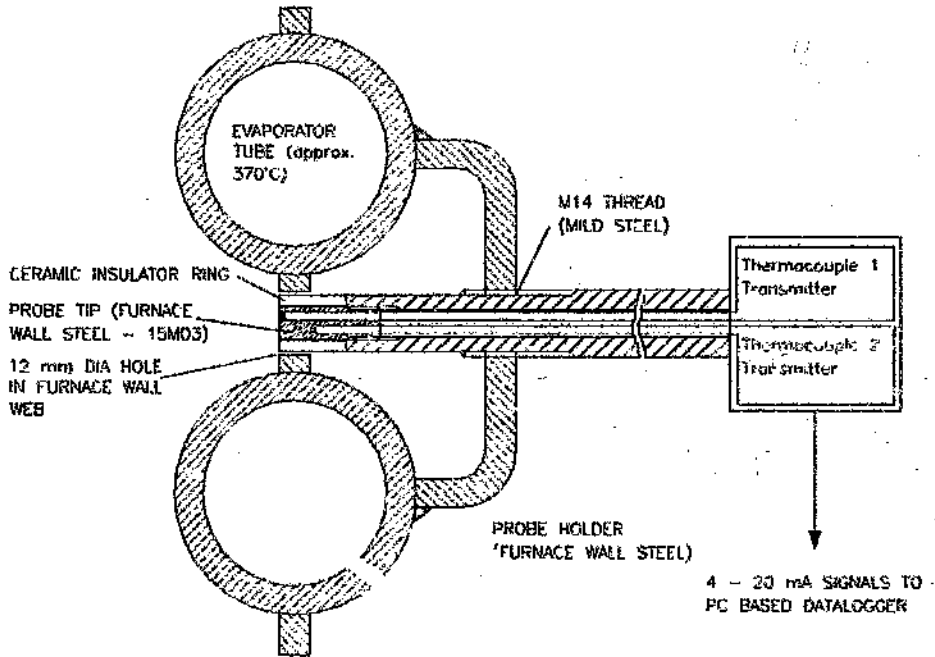


Figure 8 : Schematic of Heat Flux Probe

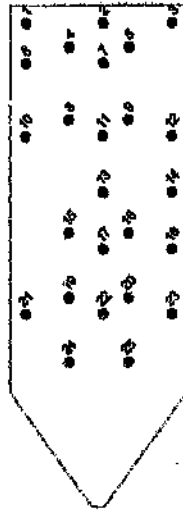


Figure 9 : Position of Heat Flux Probes on Hendrix Unit 9's Left Furnace Wall

The heat flux can be calculated from the temperature measurements obtained from the thermocouples. Let the temperature of the front and rear thermocouples be denoted by T_{front} and T_{rear} respectively. The heat flux through the probe tip is given by :

$$q = \frac{k}{x} (T_{\text{front}} - T_{\text{rear}}) \quad (12)$$

where k = thermal conductivity of 15MO3 steel

x = distance between the thermocouple beads

Figure 10 shows the thermal conductivity of the furnace wall steel (15MO3) as a function of metal temperature.

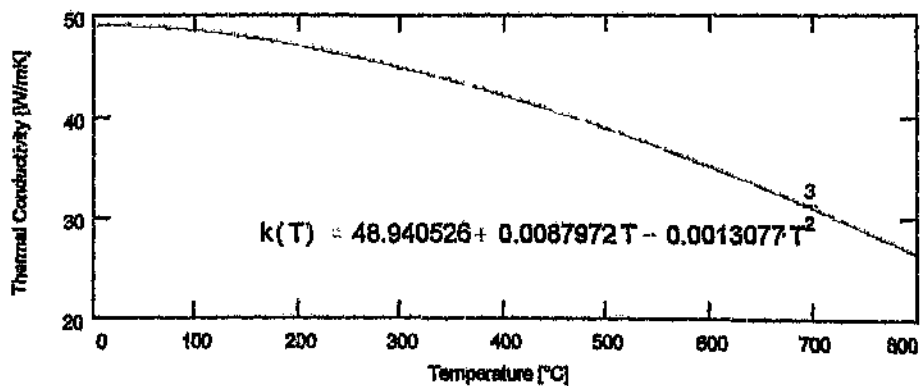


Figure 10 : Thermal Conductivity of 15MO3

Since the temperature difference between the thermocouples did not vary dramatically, an average thermal conductivity could be assumed. The average thermal conductivity was determined as follows :

$$k_{ave}(T_{front}, T_{rear}) = \frac{1}{T_{front} - T_{rear}} \int_{T_{rear}}^{T_{front}} k(T) dT \quad (13)$$

Figure 11 shows a typical 24 hour heat flux and temperature measurement log obtained during the furnace tests. Note the effect of wall blowing just after 02h00 and load changes from about 07h00 onwards.

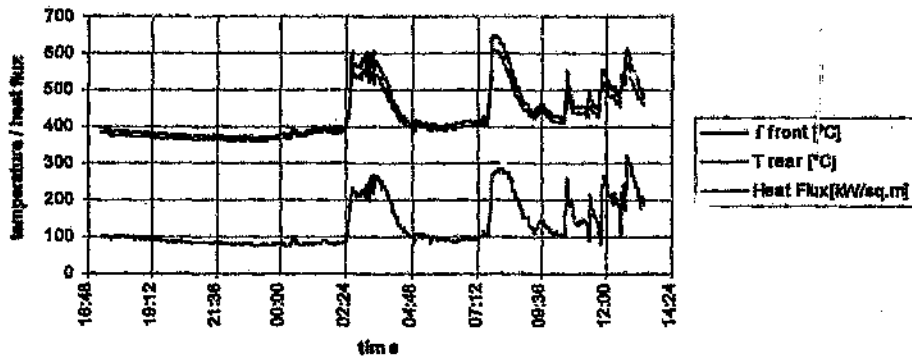


Figure 11 : Heat Flux / Temperature Log

2.3 COAL CHARACTERISATION

2.3.1 COAL SAMPLING AND OVERVIEW OF BASIC ANALYSES PERFORMED

Raw coal from the mill feeder systems of all operational mills was sampled on each day that furnace testwork was performed and composited to form a daily sample. Each sample was subjected to :

- moisture analysis (total and inherent)
- proximate analysis
- ultimate analysis
- calorific value determination.

The samples were subsequently composited to form a representative sample for the entire furnace testing program. This sample, hereafter referred to as '*R.C. Composite*', was analysed as follows :

- moisture analysis (total and inherent)
- proximate analysis
- ultimate analysis
- calorific value determination
- mineral matter analysis
- Hardgrove Index
- Abrasiveness Index
- ash fusion temperature determinations
- devolatilisation tests (using DTF)
- char combustion tests (using DTF)

Pulverised coal from each operational mill was sampled isokinetically on various days during the furnace tests. The following analyses were performed :

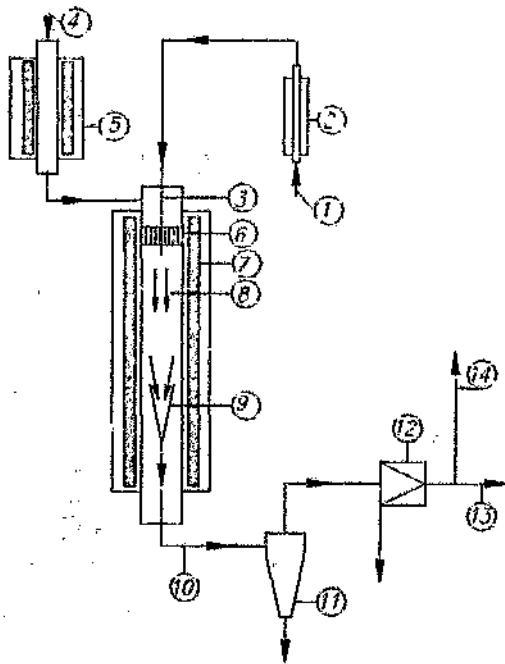
- moisture analysis (inherent)
- proximate analysis
- ultimate analysis
- calorific value determination
- particle size analyses

All analyses were performed according to standard Eskom methods. Detailed procedures of the DTF based devolatilisation and char combustion tests are described in 2.3.3 and 2.3.4 below.

2.3.2 DESCRIPTION OF THE DROP TUBE FURNACE (DTF)

The DTF is an electrically heated vertical tube reactor, in which coal particles are exposed to a controlled temperature and oxygen concentration environment for variable time periods in a hot laminar flow gas stream. The feed gas is preheated and enters the DTF's 70 mm ID alumina ceramic tube through a ceramic honeycomb flow straightener at a flowrate of approximately 20 l/min (STP). The reactor tube is heated by silicon carbide elements, which are controlled by dual thermocouples situated at the outside of the reactor tube wall. In furnace gas temperatures typically range between 700°C and 1400°C. About 1 g of dry coal/char sample is fed into the reactor over a period of about 10 minutes through a water cooled injection probe, the tip of which is situated approximately 450 mm from the top of the reactor tube. The burning particles are carried in the combustion gas axially down the furnace, the length of the reaction zone being variable up to 122 cm. The free fall velocity of the particles ranges between 0.01 m/s for a particle of approx. 25 µm to 0.04 m/s for a particle of approx. 48 µm, while the gas velocity through the reactor is about 0.4 m/s, depending on the reaction temperature. Partially reacted particles are caught in a water cooled collection probe, which quenches combustion. Field⁶ (1969) found the end effects of the feeder and collector combined to be small. Combustible matter analyses on the partially combusted product allow the determination of the extents of reaction, which vary according to combustion conditions and residence times. The DTF thus allows the control of three important variables of the combustion process, namely temperature, oxygen concentration and time. Based on the extent of combustion with respect to these variables, the kinetic behaviour during combustion of a particular coal can be quantified for a typical boiler combustion environment.

A schematic of the DTF is presented in Figure 12.



1	primary air
2	coal container and dosing hardware
3	water cooled injection probe
4	secondary air (with variable oxygen content)
5	secondary air preheater
6	honeycomb flow straightener
7	heating elements
8	70 ID alumina tube
9	movable watercooled collection probe
10	reaction product gas stream
11	cyclone
12	particulates filter
13	stream to gas analysers
14	stream to vacuum pump

Figure 12 : Schematic of Drop Tube Furnace

2.3.3 DETERMINATION OF THE DTF PARTICLE RESIDENCE TIME

The calculation of the time spent by a particle in the combustion environment of the DTF, viz. the particles residence time, is outlined below. The particle residence time is required for the calculation of the combustion kinetics.

The volumetric flow (F) of the gas must be corrected (from STP, inlet conditions) to the conditions in the combustion zone of the DTF. From the Ideal gas law, the following relationship can be derived.

$$F = F_0 \cdot \left(\frac{P_0}{P_1} \right) \cdot \left(\frac{T_1}{T_0} \right) \quad (14)$$

where

- F = corrected volumetric flow (m^3/s)
- F_0 = inlet volumetric flow (m^3/s - STP)
- P_0 = inlet pressure, T_0 = Inlet temperature (STP)
- P_1 = combustion zone pressure (atmospheric)
- T_1 = average combustion zone gas temperature

The linear velocity of the gas (v_g) is given by the quotient of the volumetric flow and the reactor volume per unit length.

The gas density (ρ_g) may be derived from the ideal gas law, which may be simplified to the following equation.

$$\rho_g = 3.36 \cdot 10^{-2} \cdot \frac{P}{T} \quad (15)$$

where

- ρ_g = gas density in combustion zone (kg/m^3)
- P = gas pressure (atm)
- T = gas temperature (K)

The gas viscosity⁴¹ (μ_g) has been found to be described satisfactorily by the following empirical relation.

$$\mu_g = 10^{\frac{\log T_g - 3.5019}{1.3733} + 4} \quad (16)$$

where

μ_g = gas viscosity (kg/m/s)

T_g = average gas temperature in combustion zone (K)

The free fall velocity of the particle (v_{ff}) can be determined by assuming that the velocity falls into the region governed by Stokes' Law⁴², then

$$v_{ff} = \frac{g \cdot d_p^2 \cdot (\rho_p - \rho_g)}{18 \mu_g} \quad (17)$$

where

v_{ff} = free fall velocity (m/s)

g = gravitational acceleration (m/s²)

d_p = particle diameter (m)

ρ_p, ρ_g = particle, gas density (kg/m³)

μ_g = gas viscosity (kg/m/s)

The residence time is given by

$$t_{res} = \frac{L}{(v_{ff} + v_g)} \quad (18)$$

where

t_{res} = residence time (s)

L = Length of particle path in the combustion zone

v_{ff}, v_g = free fall, gas velocities

The gas temperature was found to vary along the length of the combustion zone by up to 30K. In order to describe the physical parameters (i.e. density and viscosity) of the gas, the average temperature in the combustion zone was used for all

calculations. In order to describe the variation, the measured temperature data was curve fitted with a 6th order polynomial (P(x)). To determine the average gas temperature, the polynomial function was integrated over the length of the combustion zone as shown below.

$$T_{ave} = \frac{1}{L} \int_0^L P(x) dx \quad (19)$$

where T_{ave} = Average temperature in the combustion zone
 L = length of the particle trajectory
 $P(x)$ = 6th order polynomial describing the temperature distribution

2.3.4 DETERMINATION OF DEVOLATILISATION YIELDS AND RATES

The DTF is capable of reproducing particle heating rates of around 10^4 to 10^6 K/s, as found in pf combustors typical of steam raising plant and therefore can be used in the determination of devolatilisation rates and yields. For the purpose of the devolatilisation rate determination, the RC Composite sample was prepared as follows :

1. sample ground to PF consistency (viz. 75% < 75 μ m)
2. sample screened to remove all +150 μ m material, which was then reground and reintroduced into the sample
3. >75 μ m was reground to <75 μ m and reintroduced into the sample
4. sample was split into two size fractions, namely 38-75 μ m and <38 μ m
5. each size fraction underwent batch DTF testing at 5 residence times ranging between 0.05 and 1.1s for furnace setpoint temperatures of 1000, 1200 and 1400°C in N₂
6. solid residues collected from the above tests underwent proximate analyses to determine the extent of devolatilisation, from which the devolatilisation rates were determined

The volatile yield after DTF pyrolysis is determined by solving equation (20) given the ash content of a <150 μ m coal sample *before* and *after* it is passed through the DTF at 1400°C for 2.2s.

$$VM_{DTF} = \frac{1}{A_1} [A_1 \cdot (1 - A_0) - A_0 \cdot (1 - A_1)] \quad (20)$$

where A_0 = Ash content in DTF feed (dry basis) *before*

A_1 = Ash content in sampled residue (dry basis) *after*

This result is used in the assessment of the devolatilisation kinetics of the sample to suit the pyrolysis (i.e. devolatilisation) model of the combustion code. The combustion model uses a single step zero order Arrhenius reaction model by Badzioch and Hawksley⁷ according to which the pyrolysis rate is given by

$$\frac{d}{dt} V_x = -k_{\text{devol}} \exp\left(-\frac{\Delta E_{a_{\text{devol}}}}{R \cdot T_p}\right) \quad (21)$$

where V_x = extent of devolatilisation

k_{devol} = Arrhenius Factor for devolatilisation

$\Delta E_{a_{\text{devol}}}$ = Activation Energy for devolatilisation

R = Universal Gas Constant

T_p = Particle Temperature

The particle temperature, T_p , is determined from a simultaneous solution of the energy and mass balance around the devolatilising particle, taking into account radiative and convective heat transfer as the particle passes through the DTF. Assuming a constant external surface area of the coal / ash particles (viz. shrinking core), the radiative heat exchange between the particle and the DTF is given by :

$$\Gamma_{T_{rad}} = \frac{\epsilon \cdot \sigma \cdot A_{particle} \cdot [(T_{wall})^4 - (T_{particle})^4]}{Cp_{particle} \cdot m_{particle}} \quad (22)$$

- where
- ϵ = average emissivity between the particle and the wall
 - σ = Stefan Boltzman constant
 - $A_{particle}$ = assumed external surface particle area (m^2)
 - T_{wall} = DTF Wall Temperature (K)
 - $T_{particle}$ = Particle Temperature (K)
 - $Cp_{particle}$ = Particle Heat Capacity ($\frac{J}{kg \cdot K}$)
 - $m_{particle}$ = mass of particle (kg)

The convective heat exchange between the hot furnace gas and the particle is given by :

$$\Gamma_{T_{conv}} = \frac{Nu \cdot A_{particle} \cdot \lambda (T_{particle}, T_{gas}) \cdot (T_{boundary} - T_{particle})}{Cp_{particle} \cdot m_{particle} \cdot dp} \quad (23)$$

where

- Nu = Nusselt Number for boundary layer convective heat exchange
- $A_{particle}$ = external particle surface area (m^2)
- $T_{boundary}$ = Boundary Layer Temperature (assumed to be the average of the Gas and Particle Temperatures) (K)
- $T_{particle}$ = Particle Temperature (K)
- λ = $2.43 \cdot 10^{-5} \cdot \left(\frac{T_{boundary}}{273}\right)^{0.82}$ convective heat exchange coefficient ($\frac{W}{m \cdot K}$)
- $Cp_{particle}$ = Particle Heat Capacity ($\frac{J}{kg \cdot K}$)
- $m_{particle}$ = mass of particle (kg)
- dp = particle diameter (m)

The solution of the sum of the differential equations (22) and (23) gives an overall account of particle temperature with time. Figure 13 shows particle temperature vs. time plots for DTF set-point temperatures of 1000, 1200 and 1400°C for a 22 μ m coal particle.

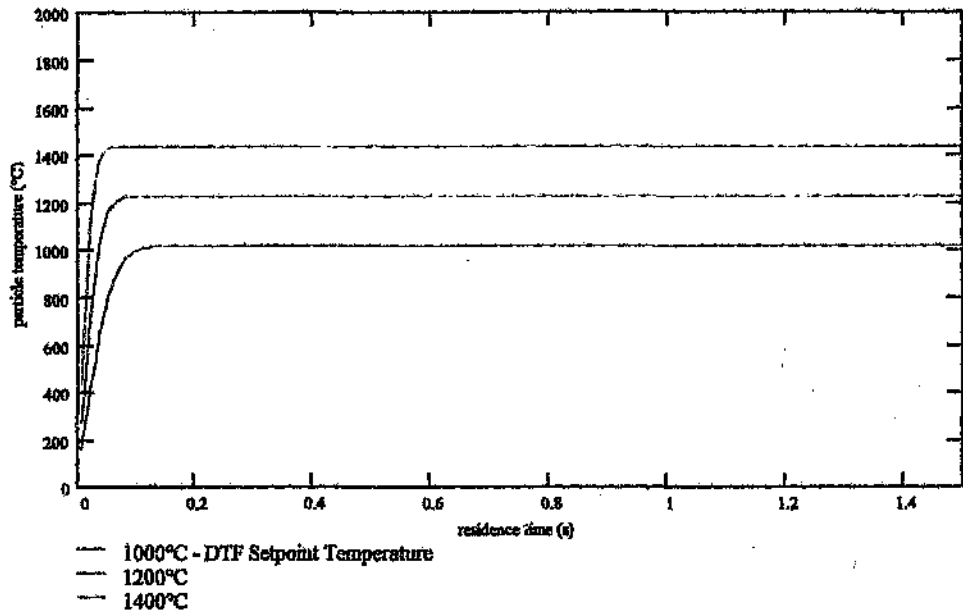


Figure 13 : Particle Temperature vs. time during Pyrolysis in the DTF

The corresponding devolatilisation vs. time graphs are shown in Figure 14 along with the DTF results.

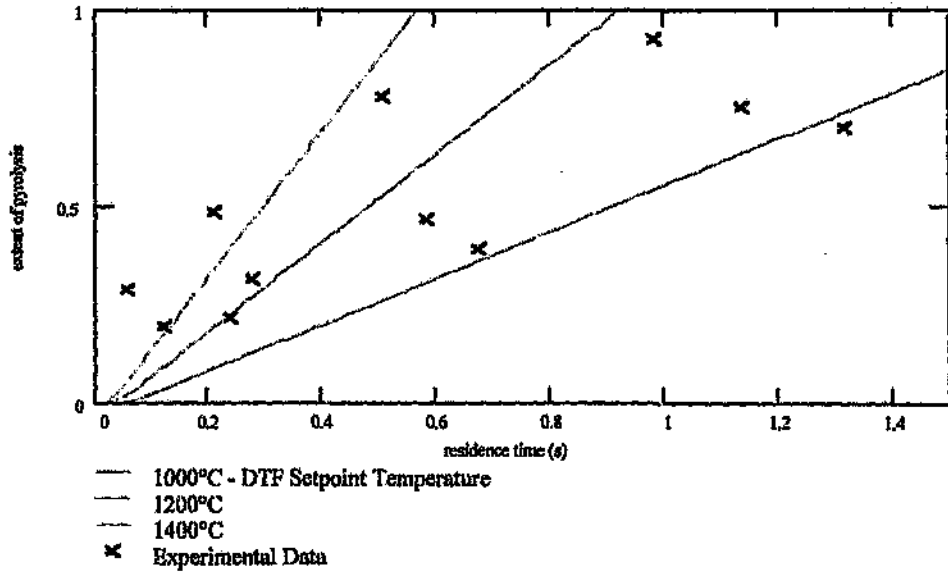


Figure 14 : Extent of Pyrolysis vs. time in the DTF

2.3.5 DETERMINATION OF CHAR COMBUSTION RATES

The standard ESKOM char combustion test procedure as performed on the RC Composite sample is outlined below :

1. sample ground to PF consistency (viz. 75% <75µm)
2. sample screened to remove all +150µm material, which was then reground and reintroduced into the sample
3. sample pyrolysed at 1400°C in N₂ for 2.2s
4. the resulting product was screened at 75µm and all >75µm material was reground to <75µm and reintroduced into the sample
5. the sample was split into a <38µm and a 38-75µm size fraction
6. each size fraction underwent DTF testing at 5 residence times ranging between 0.4 and 3.2 s for DTF setpoint temperatures of 1000,1100,1200,1300 and 1400°C at 3 vol.% O₂ with the balance being N₂ (This particular oxygen concentration is chosen since it is similar to the depleted bulk oxygen concentration in a typical utility boiler)
7. the solid combustion residues from the above tests were analysed for combustible matter and ash content, from which the extent of the combustion reaction is determined

As the mass fraction of ash increases with increasing residence time, the extent of combustion can be determined using the ash in the combustion residue as a tracer. The fractional extent of combustion or combustion efficiency (CE) is given by (on a dry ash free basis):

$$CE = \frac{A_0}{(1 - A_0)} \cdot \frac{(1 - A_1)}{A_1} \quad (24)$$

where

A_0 = mass fraction of ash in the raw char (dry basis)

A_1 = mass fraction of ash in the sampled residue (dry basis)

The experimentally determined combustion extent data along with its corresponding residence time in the DTF at a given DTF set-point temperature, is used to estimate the char combustion parameters required for the combustion sub-model of the combustion code. A pseudo steady-state assumption with regard to the particle temperature is required to arrive at estimations of the char combustion parameters, which are then refined by simulating the DTF combustion process. This simulation model is described later in this section. The method of arriving at the estimations of the char combustion parameters is described below :

Depending on the particle surface temperature, the rate of combustion can be limited either by the rate of O₂ diffusion to the surface of the particle or the rate of chemical reaction at the particle surface. In either situation, the rate of combustion depends on the partial pressure of O₂, the surface area available for char combustion and the combustion characteristics of the char in question. The overall rate of char combustion is mathematically modelled by equation (25) :

$$\frac{d}{dt} m_{C_{particle}} = k_{char\ overall} \cdot P_{O_2, \infty} \cdot A_{particle} \quad (25)$$

where

- $m_{C_{particle}}$ = mass of combustible matter in char particle
- $k_{char\ overall}$ = overall reaction rate coefficient
- $P_{O_2, \infty}$ = partial pressure of Oxygen
- $A_{particle}$ = apparent reactive surface area of the particle

Field⁹ (1970) proposed that the overall rate reaction coefficient comprise a rate coefficient description of oxygen diffusion to the reaction surface as well as description of the rate coefficient of chemical reaction at the surface according to equations (26) to (28). If the chemical reaction rate is fast compared to the rate of O₂ diffusion to the reacting char surface, the reaction is diffusion controlled. If the rate of chemical reaction is slow compared the rate of O₂ diffusion to the reacting surface, the reaction is chemically controlled.

$$\frac{1}{k_{\text{char overall}}} = \frac{1}{k_{\text{char diffusion}}} + \frac{1}{k_{\text{char reaction}}} \quad (26)$$

where

$$k_{\text{char diffusion}} = \frac{24 \cdot \phi \cdot m \cdot D_{\text{O}_2(1600\text{K})} \cdot \left(\frac{T_{\text{boundary}}}{1600\text{K}}\right)^{1.75}}{R \cdot 10^{-5} \cdot d_{\text{particle}} \cdot T_{\text{gas}}} \quad \left(\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}\right) \quad (27)$$

$$D_{\text{O}_2(1600\text{K})} = 3.3 \cdot 10^{-4} \quad \left(\frac{\text{m}^2}{\text{s}}\right)$$

ϕ = mechanism factor (2 if C oxidises to CO and 1 if C oxidises to CO₂)

and

$$k_{\text{char reaction}} = A_{\text{reaction}} \cdot \exp\left(\frac{-\Delta E_a}{R \cdot T_{\text{particle}}}\right) \quad \left(\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}\right) \quad (28)$$

$$A_{\text{char reaction}} = \text{Arrhenius factor} \quad \left(\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}\right)$$

$$\Delta E_a = \text{Activation Energy} \quad \left(\frac{\text{J}}{\text{mol}}\right)$$

The overall rate is determined from the experimental data by integration of differential equation (25). A description of the integration results is given in equations (29) and (30).

$$\int_{(m_{\text{C particle}})_{\text{initial}}}^{(m_{\text{C particle}})_{\text{final}}} \frac{m_{\text{C particle}}}{A_{\text{particle}}} dm_{\text{C particle}} = \int_0^{t_{\text{res}}} k_{\text{char overall}} \cdot P_{\text{O}_2} dt \quad (29)$$

where

$$m_{\text{C particle}} = (m_{\text{C particle}})_{\text{initial}} \cdot (1 - \text{CE})$$

similarly

$$A_{\text{particle}} = (A_{\text{particle}})^{\text{initial}} \cdot (1 - \text{CE})^{n_B} \quad (30)$$

where $(A_{\text{particle}})^{\text{initial}}$ = initial particle surface area

n_B : swelling index

The superscripts initial and final refer to the DTF feed and product respectively while t_{res} is the corresponding residence time in the DTF. Because the DTF is operated with a considerable oxygen excess (300 - 400%), the partial pressure of oxygen (P_{O_2}) remains effectively constant.

The initial particle surface area includes the active internal pore area created during the devolatilisation process and would depend primarily on the char structure and the heating rate. Laine et al⁸ (1964) found that for the same coal a higher heating rate resulted in a greater surface area for reaction and an increased combustion rate. Smoot et al¹⁰ noted that physical changes in the char structure resulted in changes in surface area and in turn changes in combustion rate. In order to account to some extent for the complex nature and structure of char forms and their transformations during the combustion process, an empirical swelling index, n_B , is used in equation (24). Table 1 summarises how this surface area model predicts the reactive surface area of char particle during the combustion process for various swelling indices.

Table 1 : Particle Surface Area as a function of Combustion Extent (CE)

Description	"non swelling"	"swelling"
Swelling Index n_B	2/3	1
$\frac{A_{\text{particle}}}{(A_{\text{particle}})^{\text{initial}}} = (1 - \text{CE})^{n_B}$		

As the combusting particle passes through the DTF, it is first heated by radiation from the hot furnace walls and by convection from the preheated gases passing through the DTF. An estimation of the particle surface temperature can be made by solving an energy balance around the particle which equates the heat lost/gained by radiation and convection to the heat released at the surface due to reaction. The heat release can be calculated from the measured overall reaction rate assuming that the surface reaction mechanism is known. For the purpose of this investigation, it is assumed that the oxidation on the particle surface results in the formation of CO. The overall energy balance is given in equation (31).

$$\Delta h \left[\frac{(m_{\text{C particle}})_{\text{final}} - (m_{\text{C particle}})_{\text{initial}}}{t_{\text{res}}} \right] = \varepsilon \cdot A_r \cdot \sigma \cdot [(T_{\text{wall}})^4 - (T_{\text{particle}})^4] + \frac{\text{Nu} \lambda}{d} \cdot A_c \cdot (T_{\text{boundary}} - T_{\text{particle}}) \quad (31)$$

where

- Δh = enthalpy of formation of CO from C(s) and O(g) $\left(\frac{\text{J}}{\text{mol}} \right)$
- A_r = surface area available for radiative heat exchange (m^2)
- A_c = surface area available for convective heat transfer (m^2)
- \bar{d} = average particle diameter (m)
- Nu = Nusselt Number (assumed = 2)
- T_{boundary} = temperature of the gas film around the particle (K)
- T_{wall} = temperature of the inside wall of the Drop Tube Furnace (K)
- T_{particle} = average temperature of the particle (K)

Figure 15 shows estimated particle temperatures for a variety of DTF residence times, wall and gas temperatures:

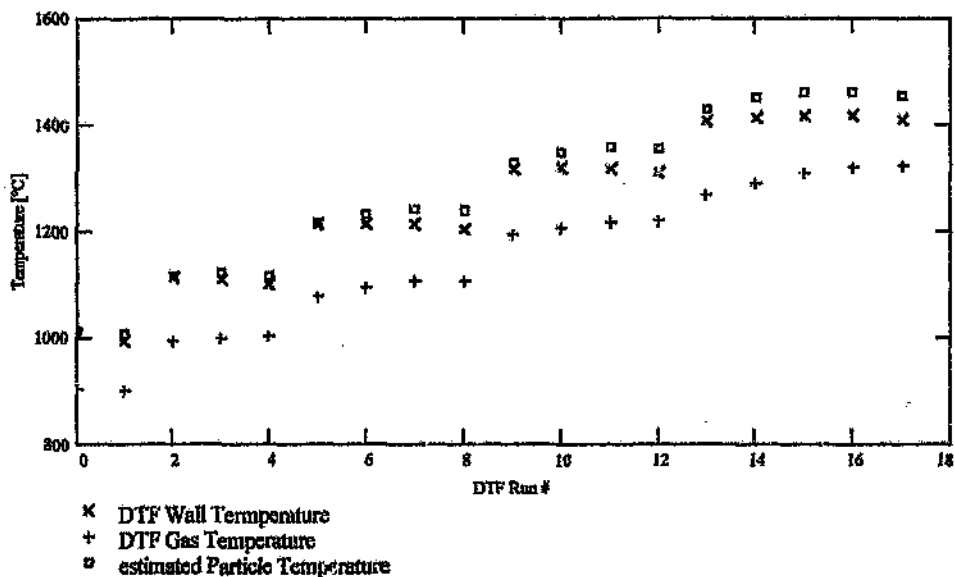


Figure 15 : estimated particle temperatures for various DTF experiments

The diffusion rate can be determined by solving equation (28) given the particle temperature estimate and a particle diameter assumed to be the mean initial particle diameter. Note, however, that when the diffusion rate is similar in magnitude to the experimentally derived overall reaction rate, the reaction may be diffusion controlled. In this event the reaction rate cannot be accurately determined for that particular DTF run because the value of the mechanism factor, ϕ_m , which cannot be determined accurately, becomes significant in the analysis of the results. Field⁶ (1969) suggests that only reaction rate coefficients from DTF experiments where

$$\frac{k_{char\ overall}}{k_{char\ diffusion}} < 0.5 \quad (32)$$

be used to obtain values for the chemical reaction rate coefficients.

When the surface reaction rate coefficients are plotted against the estimated surface temperatures, and a curve fit using the function form given in equation (22) of the data as reviewed by Field et al.¹¹ (1967) is performed, the function parameters define the kinetic parameters for the Arrhenius type reaction model. Figure 16 shows a

typical $\ln(k_{char,surface})$ vs. $10^4/T_p$ plot. The intercept and slope of the linear fit give the pre-exponential factor and activation energy respectively.

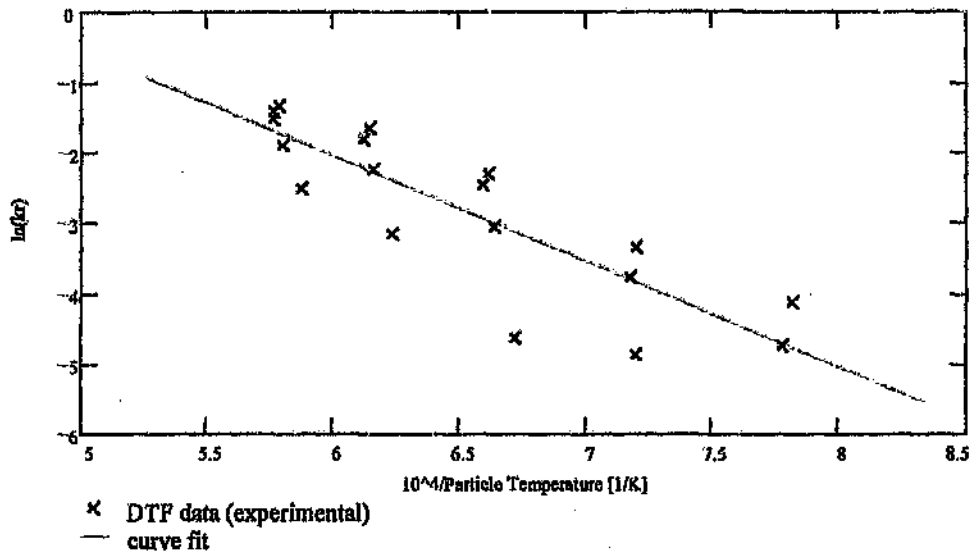


Figure 16 : Linear fit of DTF data to determine Arrhenius kinetic parameters

A model of the combustion process in the DTF was derived using the above process description. The model involves solving two simultaneous differential equations, one describing the combustion at the surface of the char, and the other, the surface temperature of the reacting char particle. The differential equations are given in equations (33) and (34).

$$\frac{d}{dt}C_c = -A_{char} \cdot \exp\left(\frac{-\Delta E_a}{R \cdot T_{particle}}\right) \cdot P_{O_2} \cdot A_{particle}(C_c) \quad (33)$$

$$\frac{d}{dt}T_p = \Gamma_{T_{rad}}(T_p, T_w) + \Gamma_{T_{conv}}(T_p, T_g) + \Gamma_{T_{gen}}(T_p, C_c, T_g) \quad (34)$$

The change in particle temperature by radiation and convection has been outlined in equations (22) and (23), while equation (35) describes the change in particle temperature due to reaction.

$$\Gamma_{T_{gen}}(T_p, C_c, T_g) = \frac{\Delta h}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot \frac{\delta}{\delta t} C_c \quad (35)$$

Figure 17 shows the char particle surface temperature of a particular coal in the DTF when operated at a set-point temperature of 1400°C and 3% (Vol.) O₂ in N₂.

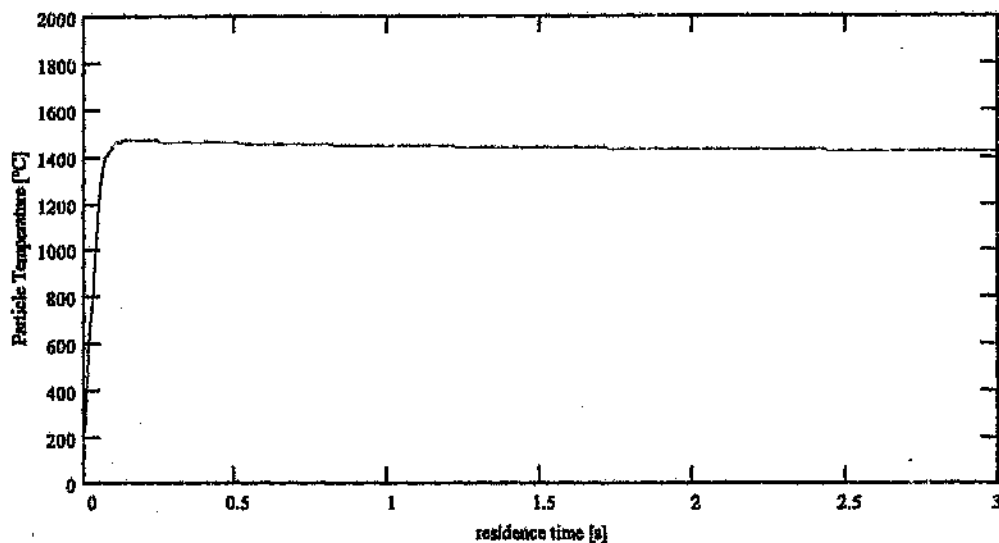


Figure 17 : Graph of Particle Temperature vs. time

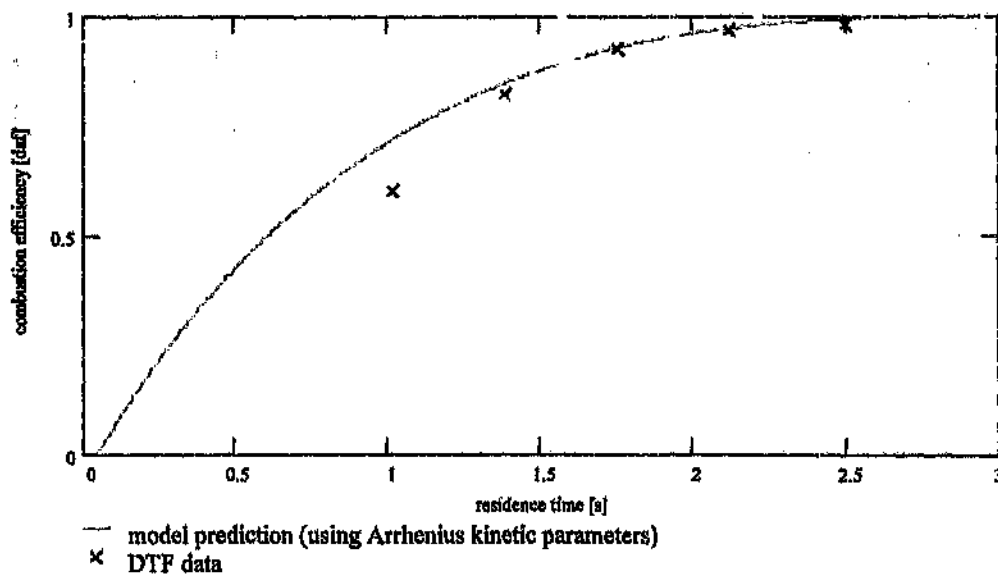


Figure 18 : Graph of Combustion Extent vs. time

2.4 THE AIOLOS COMBUSTION MODEL

A detailed account of the model's operational engine and its many physical, thermodynamic and chemical sub-models is available from Schnell¹² (1991), Epple¹³ (1993), Epple et al¹⁴ (1992), Epple et al¹⁵ (1993) and Schnell et al¹⁶ (1995). A brief outline of the model has been extracted from the aforementioned references for the purpose of a convenient review.

Turbulent flow in combustion chambers is predicted using the k-ε turbulence model formulated in partial differential equations compiled in equations (36) to (39), where the basic equation for a general variable ϕ is specified in equation (36).

The terms in equation (36) describe the local change of ϕ due to transient, convective, diffusive and source term contributions to a particular property assigned to this general variable. Adaptation of this equation leads to the derivation of equations (37) to (39).

Transport Equation :

$$\underbrace{\frac{\delta}{\delta t}(\rho \cdot \phi)}_{\text{transient}} + \underbrace{\frac{\delta}{\delta x_j}(\rho \cdot u_j \cdot \phi)}_{\text{convective}} = \underbrace{\frac{\delta}{\delta x_j} \left(\frac{\mu_{\text{eff}}}{\sigma_\phi} \frac{\delta \phi}{\delta x_j} \right)}_{\text{diffusive}} + \underbrace{S_\phi}_{\text{source}} \quad (36)$$

where

- t = time variable
- x_j = direction variable where j = 1,2,3
- u_j = velocity component
- ϕ = general field variable
- ρ = density
- μ_{eff} = effective viscosity
- σ_ϕ = stress associated with that general field variable
- S_ϕ = source term of field variable

Time-averaged Navier-Stokes Equation :

$$\frac{\delta}{\delta t} (\rho \cdot u_i) + \frac{\delta}{\delta x_j} (\rho \cdot u_j \cdot u_i) = \frac{\delta}{\delta x_j} \left(\Gamma_{ij} \cdot \frac{\delta u_i}{\delta x_j} \right) - \frac{\delta p}{\delta x_i} + \rho \cdot g_i + S_{\tau} \quad (37)$$

$$i = 1, 2, 3$$

where

$$\begin{aligned} \Gamma_{ij} &= \frac{\mu \text{ eff}}{\sigma_i} \\ p &= \text{pressure} \\ g_i &= \text{effective acceleration} \end{aligned}$$

Kinetic Turbulent Energy

$$\frac{\delta}{\delta t} (\rho \cdot KE_{\text{turb}}) + \frac{\delta}{\delta x_j} (\rho \cdot u_j \cdot k) = \frac{\delta}{\delta x_j} \left(\frac{\mu \text{ eff}}{\sigma_k} \cdot \frac{\delta k}{\delta x_j} \right) + G - \rho \varepsilon \quad (38)$$

where

$$KE_{\text{turb}} = \text{kinetic turbulent energy}$$

Dissipation Rate of of Turbulent Energy

$$\frac{\delta}{\delta t} (\rho \cdot \varepsilon) + \frac{\delta}{\delta x_j} (\rho \cdot u_j \cdot \varepsilon) = \frac{\delta}{\delta x_j} \left(\frac{\mu \text{ eff}}{\sigma_\varepsilon} \cdot \frac{\delta \varepsilon}{\delta x_j} \right) + \frac{\varepsilon}{k} (C_{\varepsilon_1} \cdot G - C_{\varepsilon_2} \cdot \rho \cdot \varepsilon) \quad (39)$$

$$\text{Generation Term : } G = \frac{\delta u_i}{\delta x_j} \left[\mu_t \left(\frac{\delta u_j}{\delta x_i} + \frac{\delta u_i}{\delta x_j} \right) \right]$$

$$\text{Turbulent Viscosity : } \mu_t = C_\mu \cdot \rho \cdot \frac{KE_{\text{turb}}^2}{\varepsilon}$$

where

$$\begin{aligned} \varepsilon &= \text{dissipation rate of turbulent energy} \\ C_\mu &= \text{constant of turbulent viscosity model} \end{aligned}$$

The time averaged Navier Stokes equation (also called the Reynolds equation) describes the momentum balance (velocities) around (into and out of) a control volume. The pressure, which can be considered as the source term in this momentum equation, is calculated by means of a pressure correction method by Epple et al¹⁷ (1991). The turbulence is characterised by the kinetic turbulent energy

KE_{turb} and its dissipation rate ϵ . More advanced and sophisticated turbulence models, which provide improved prediction accuracy, are available for use with AIOLOS. They do however require considerably more computation time (Schnell¹⁸, (1989)) and are therefore not generally used in three dimensional simulations. Their use in two dimensional AIOLOS applications has however been demonstrated with success.

The general variable ϕ may also stand for species that feature in the combustion of coal. Figure 19 shows how the combustion process is modelled and what species are considered. The circled numbers in Figure 19 refer to the column headed No of Table 2.

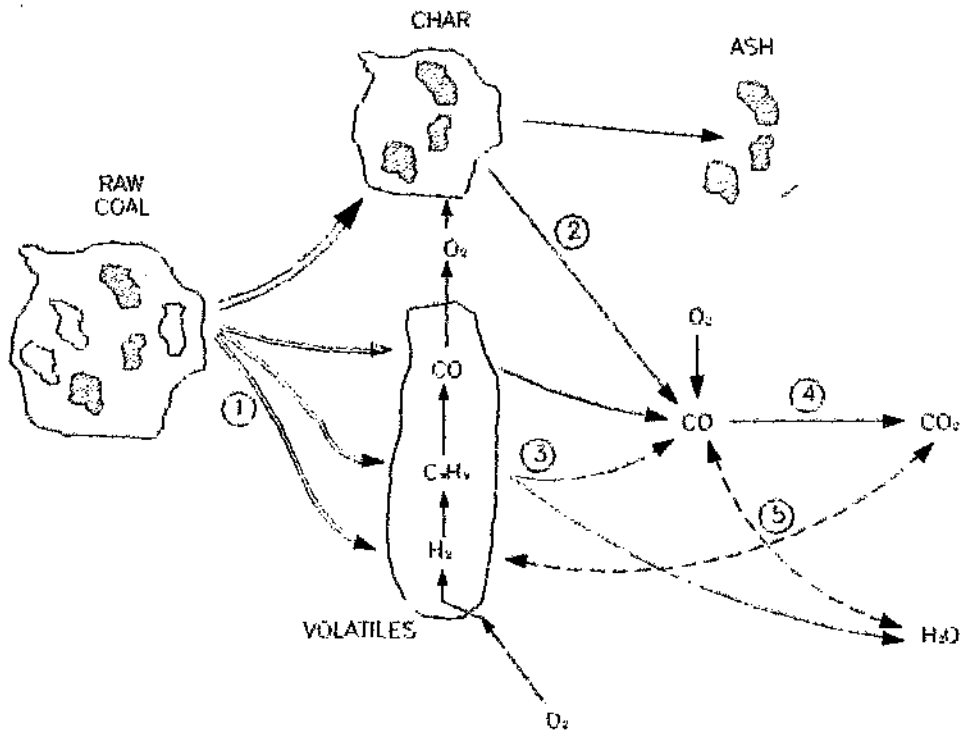


Figure 19 : Reaction Models of Pulverised Coal Combustion

Table 2 : Reaction Models of Pulverised Coal Combustion

No	Reaction Model	Reference
1	Devolatilisation	Section 2.3.3 of this report
2	Char Combustion	Section 2.3.4 of this report
3	Volatiles Combustion	Field et al ⁸ (pp 179)
4	Oxidation of CO	Field et al ⁸ (pp 175)
5	Water Gas Shift Reaction	Denn ²¹ , Schilling ²² (pp 10)

The AIOLOS coal combustion model (as outlined in Figure 19) considers coal to be devolatilised in a single step to form char and volatiles (reaction 1). The rate of devolatilisation for the Hendrina R.C. Composite was investigated using the DTF and analysed according to the method and analysis procedures described in part 2.3.3 of this report. The volatiles are assumed to consist of CO, H₂ and hydrocarbons C_xH_y. The hydrocarbon parameters X and Y are determined from the ultimate analysis of the coal. C_xH_y is assumed to oxidise in a two-step reaction yielding CO₂ and H₂O (reactions 3 and 4). These reactions are modelled with an Eddy-Dissipation-Model (Magnussen¹⁹ (1981) and Magnussen and Hjertager²⁰ (1976)). The residual char is heterogeneously oxidised to CO at the surface of the char (reaction 2). The rate of char oxidation depends on the local oxygen concentration, the reactive surface area and reactivity of the char itself. The reactivity of the char is experimentally determined using the DTF according to the method and analysis procedure outlined in section 2.3.4 of this report. Additionally, the water-gas-shift reaction adjusts the concentrations of CO, H₂, CO₂, and H₂O towards equilibrium (reaction 5). Information on this reaction is available from Denn²¹ (1987) and Schilling et al²² (1981). Particle size effects are also included by considering various size classes for the solid fuel. Ash is treated as an inert substance.

Besides the heat release due to the combustion reactions, the source of enthalpy in each control volume is dominated by radiative heat exchange. In pulverised coal fired combustors, radiative heat exchange in the furnace accounts for approx. 90% of the total heat transfer. AIOLOS allows the use of a number of radiative heat exchange models.

Viskanta²³ compiled an overview of radiation models and in the area of radiative heat transfer in pulverised coal combustion applications, these can be grouped into 'zone-', 'flux-', and statistical models.

Zone methods, such as proposed by Hottel et al²⁴ (1967), require a non-linear integro-differential equation for each zone, making the application of this method unsuitable for finely discretised furnace chambers in view of the computational effort required. The 'flux'-models are grouped according to the number of directions in which they model radiative heat transfer into and out of a control volume. 2-, 4- and 6-'flux' models have been proposed by Siddal and Seicuk^{25,26} (1979, 1976) and Filia and Maresa²⁷ (1975), which are coupled to resolve radiative exchange in three dimensions via the control volume temperature alone. A 6-'flux' model by De Marco and Lockwood²⁸ (1975) refined the 'flux' approach by coupling the radiation exchange in three dimensions. The 'flux'-radiation models can use the numerical solution

algorithms used by the flow solver. It should however be noted at this stage the 'Monte-Carlo-' and 'Discrete-Transfer-' radiation models considerably outperform the 'flux' radiation models with regard to solution accuracy.

The 'Discrete-Transfer-' model, proposed by Lockwood and Shah^{29,30} (1976), specifies a network of beams within the combustion chamber, along which radiative heat exchange occurs. An overall energy balance around the entire combustion chamber is required in the solution algorithm. The 'Discrete-Transfer-' model differs from the 'Monte-Carlo-' model in that no beam reflection from the furnace wall is considered. As the beams traverse the individual control volumes that the combustion chamber has been broken down into, a portion of the beams energy is transferred to the control volume according to its radiative properties. These properties are determined from the gas emissivity (which depends on the local gas composition) and more importantly the particulate loading. A comparison between the 'Monte-Carlo' radiation model (first proposed for cylindrical combustion chamber geometries by Stewart and Cannon³¹ (1971)) and the 'Discrete-Transfer-' model has been made by Guilbert³² (1989). A further radiation model, named the 'Discrete-Ordinates-' model as proposed by Fiveland^{33,34} (1987, 1984), is available in the AIOLOS combustion model.

A detailed outline of the application of the 'Monte-Carlo-' radiation exchange model in combustion modelling applications is available from Richter^{35,36} (1978, 1974). Whereas the Monte-Carlo method uses an entirely statistically determined array of beams to describe the radiative heat transfer, Richter has specified the beam numbers and directions at the source. Statistical treatment of the beam direction only takes place on beam reflections at the furnace walls. A previous Eskom combustion modelling investigation by Eichhorn et al³⁷ (1995) used a furnace model by Richter based on Monte-Carlo method for radiation exchange. It was found that despite having performed the simulation with a very coarse grid, reasonable predictions were made.

In order to analyse pollutant formation in pulverised coal combustion, the nitrogen oxides (NOx) formed from fuel bound nitrogen must be modelled. A fuel-NO- and thermal NO formation model are included in the AIOLOS combustion model. The thermal NO formation model is based on the well-known Zeldovich³⁸ mechanism, while the fuel N oxidation/reduction is assumed to occur according to the mechanisms described in Figure 20.

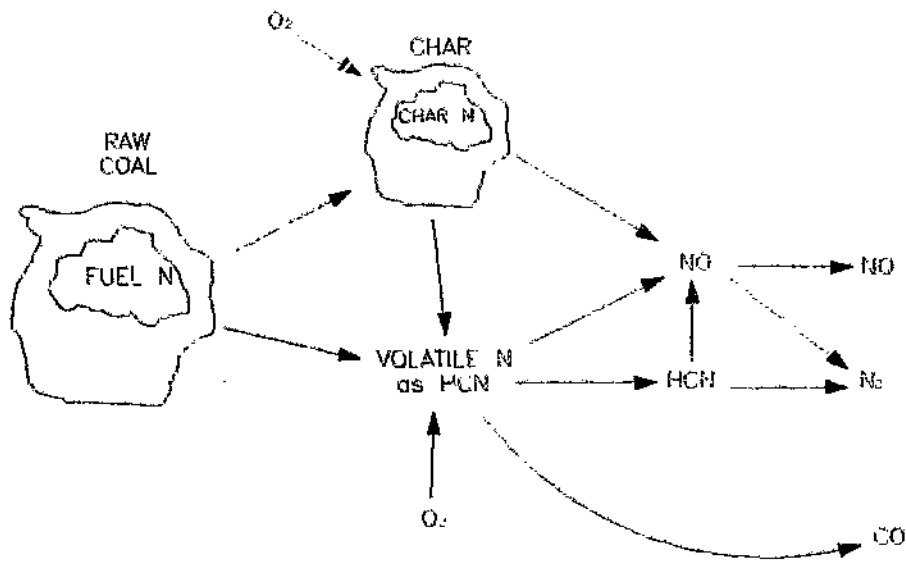


Figure 20 : Reaction Scheme of Fuel N Conversion

The fate of HCN and its successors is strongly dependant on the local flame conditions, particularly stoichiometry and temperature. De Soets³⁹ (1974) proposed a reduced scheme for NO formation from HCN, which has been adopted for use in AIOLOS. It must be noted that the reaction times of HCN in the flame are of the same order as the turbulent fluctuations in combustion conditions in the flame as is typical to many pollutant formation reactions which show high activation energies. It is assumed in the model that the species concentration fluctuations are not as marked as the temperature fluctuations in the burner regions as radiation is the dominant mechanism of heat transfer. A simplified probability density function (pdf) approach with a model for the instantaneous temperature fluctuations yields the HCN decay rate by oxidation to NO and reduction to N₂. Details of the model for the instantaneous temperature fluctuations are given by Schnell¹² (1991).

2.5 APPLICATION OF AIOLOS TO HENDRINA UNIT 9

2.5.1 DESCRIPTION OF THE HENDRINA UNIT 9 FURNACE

The furnace of Hendrina Unit 9 has 6 burner rows, each of which is fuelled by a dedicated vertical spindle mill of which five must be in operation at full load. Each mill supplies four burners, which are arranged in a single row on either the front or back wall of the furnace. A vertical section and view of the front wall of the furnace are shown in Figure 21.

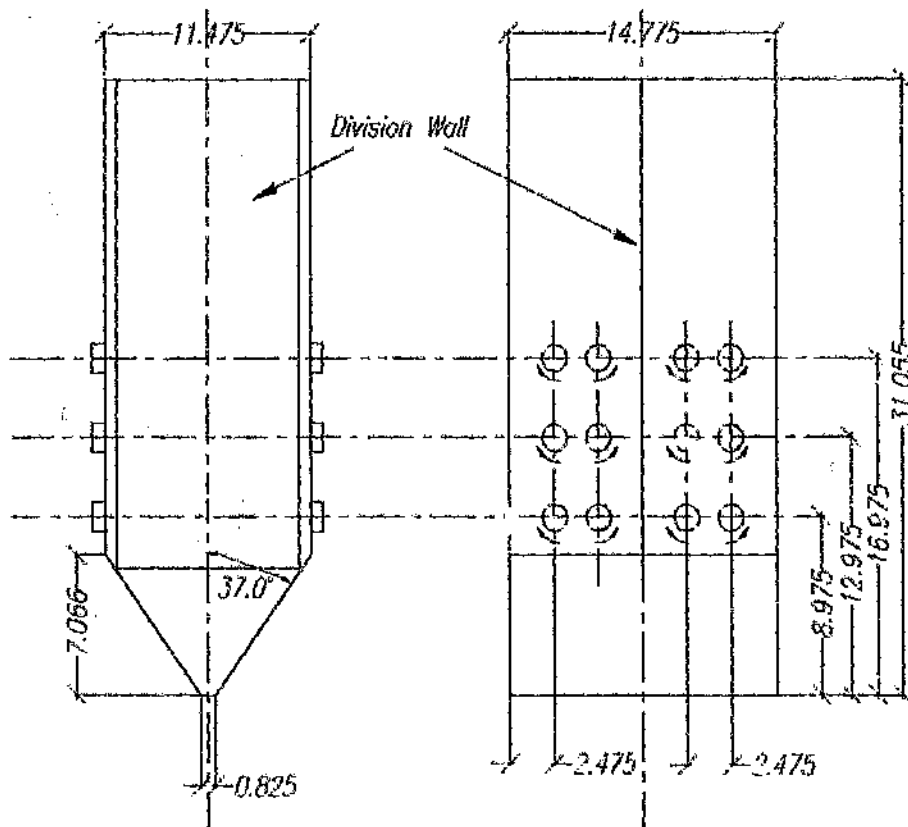


Figure 21 : Hendrina Unit 9 Furnace : Vertical Section and View of Front Wall

A summary of boiler specifications as required for the combustion model, sourced from the design specifications⁴⁰ by Steinmüller, the boiler's designers and manufacturers, has been compiled in Table 3. Note that the 97% maximum capacity rating (MCR) values have been obtained by linear interpolation of the 94% and 100% MCR specifications.

A section through the burner is shown in Figure 22. Note that the cylinder contained within the core air duct holds the oil burner assembly and that the primary air / pf mixture is not swirled. Swirl vanes have recently been installed into the core air ducts. Their effect on the combustion process has however not been modelled at this stage.

Table 3 : Furnace Specification as required for Combustion Model

Specification	94% MCR (202 kg/s evaporation)	100% MCR (215 kg/s evaporation)	97% MCR (208.5 kg/s evaporation)
Air Temperature at Air Heater Outlet	242	247	244.5
Temperature of air/fuel mix entering burners	80	80	80
Fuel Consumption (refer to Table in footnote ¹)	26.4	28.1	27.25
Air supplied to furnace as			
primary air (kg/s)	45.5	54.0	49.75
secondary air (kg/s)	177.5	185	181.25
Gas Temperature at Furnace Exit (°C)	1100	1125	1112

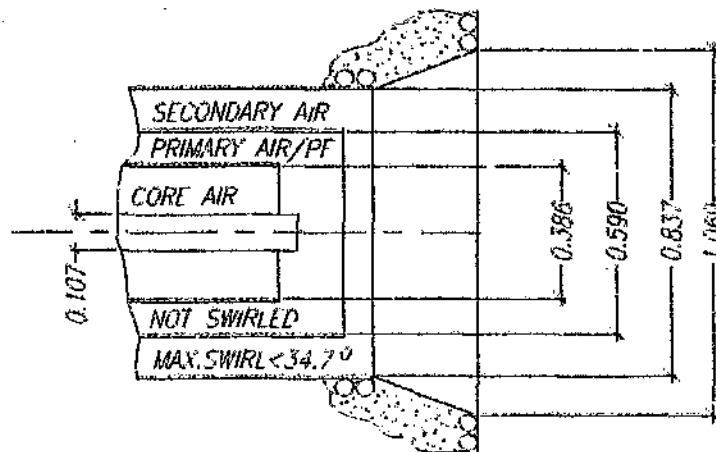


Figure 22 : Section through Hendrina Unit 9 Burner

¹ for typical coal properties from design specification :

	Typical Washed Fuel
Inherent Moisture (m% air dried)	4
Ash (m% air dried)	20
Volatile Matter (m% air dried)	24
Fixed Carbon (m% air dried)	53
Calorific Value (MJ/kg air dried)	25
Surface Moisture (m% as received)	6

2.5.2 FURNACE DISCRETISATION

The furnace was discretised into 276546 Cartesian grid based finite volumes (or cells) by dividing the height, width and depth of the furnace as follows :

height :	100
width (front / rear wall) :	76
depth (left / right wall) :	45

The divisions were specified to become gradually smaller as the burner regions were approached, to improve the resolution of near burner zone regions and to reduce the gradients in local combustion conditions between adjacent cells. A finer discretisation is necessary for the simulation of the volatiles combustion and NO formation which feature predominantly in this region. Figure 23 shows the discretised furnace. The AIOLOS code containing the discretisation information is attached in Appendix 3.1.

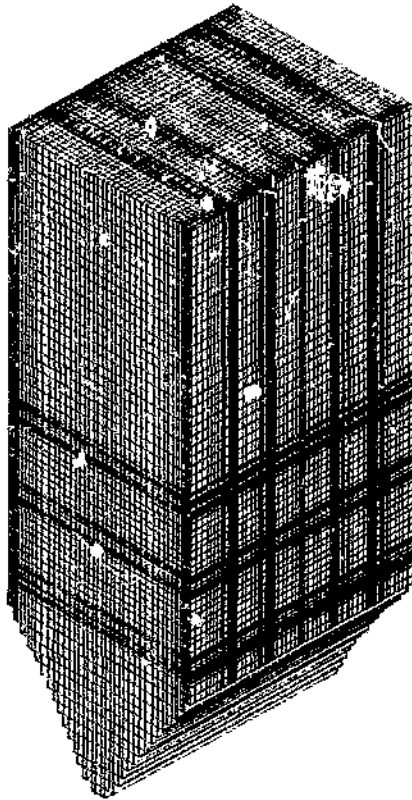


Figure 23 : Discretisation of Hendrina Unit 9 Furnace

2.5.3 SPECIFICATION OF BOUNDARY CONDITIONS

The specification of boundary conditions required that every cell inlet be allocated the mass flowrates of the species entering there as well as their respective inlet velocities and directions. For the purpose of defining the inlet flow patterns of the swirl burners, the centre-points of the burners were specified, around which, depending on the angle of rotation around the burner centre, the direction of flow of the inlet stream was described in terms of its velocity components.

Furthermore, the inlet stream temperature was specified for both the flow and radiation component of the combustion model. The AIOLOS code specifying the inlet conditions has been attached in Appendix 3.2.

2.5.4 SPECIFICATION OF THE COAL COMPOSITION, DEVOLATILISATION AND COMBUSTION PROPERTIES

The coal properties outlined in Table 4 require specification. Note that the bold typeface indicates the parameter specified.

Table 4 : Coal Specifications for AIOLOS

Likely Volatile Hydrocarbon	C_xH_y
Calorific Value (on dry ash free (daf) basis)	HUDEF
Volatile Matter (from DTF analysis, daf)	VMDAFA
Ultimate Analysis (as received basis)	XSI_i where i = C, O, H, S, N, A (Ash) and F (inherent moisture)
Devolatilisation Kinetic Parameters	Pre-exponential factor : RK01 Activation Energy : E1R
Char Combustion Kinetics	Pre-exponential factor : RK02 Activation Energy : E2R
Particle Size Distribution (3 size classes)	distribution (WSIZE_j) and average particle diameter (DPJJ) in size class where j = 1 .. 3
Density of Coal	RHOSO
Swelling Index	RN_B

The AIOLOS input file containing the aforementioned information has been attached in Appendix 3.3.

2.5.5 AIOLOS NO_x MODEL SPECIFICATIONS

The NO_x model is generally used in a post-processing mode since the quantities involved in NO formations are negligible compared to the coal combustion reactions and therefore do not affect the calculated combustion conditions. The model requires that the coal nitrogen on a daf basis be provided and if possible, the fraction of coal nitrogen reporting to volatile matter and char respectively. As little is known about the oxidation of char nitrogen, an arbitrary conversion factor of char nitrogen to NO can be specified. An AIOLOS input file containing the listed specifications required for the NO_x model has been attached in Appendix 3.4.

3. RESULTS AND DISCUSSION

3.1 FURNACE OPERATION ASSESSMENT

During the entire furnace testing program the operation of the furnace was logged according to standard Hendrina operation procedures. Figures 24 to 26 show logs of selected plant variables considered to be significant indicators of furnace operation.

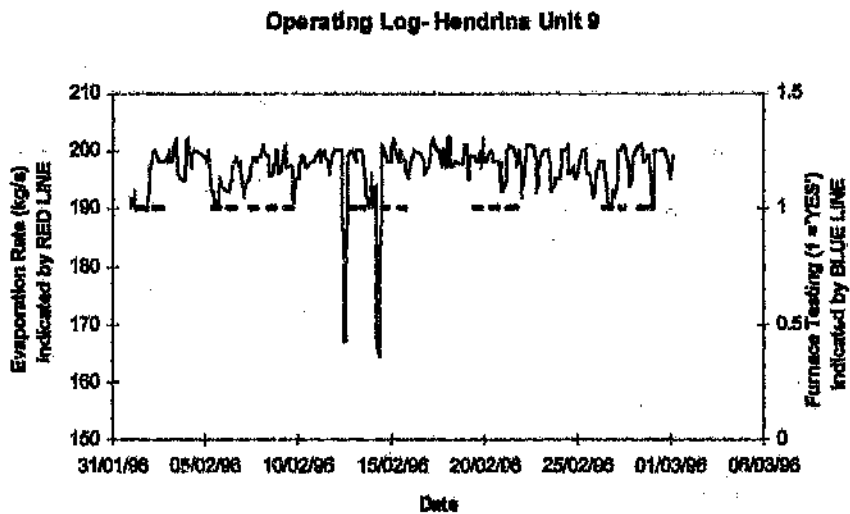


Figure 24 : Log of Evaporation Rate during Furnace Tests

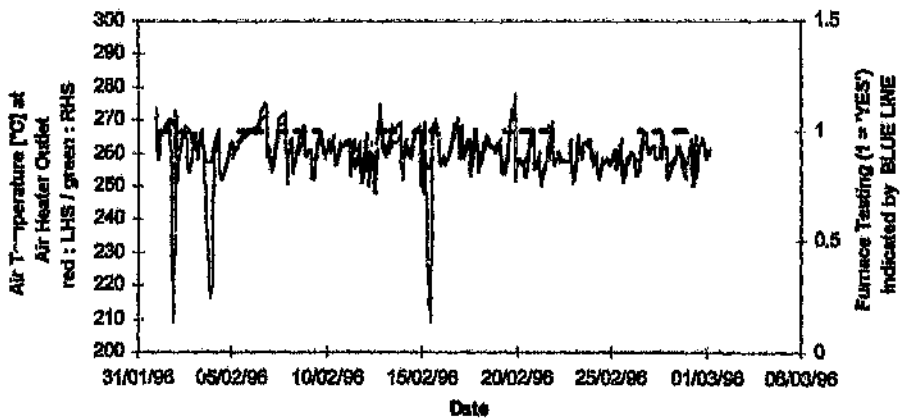


Figure 25 : Log of Air Temperatures (LHS/RHS) at Air Heater Outlet

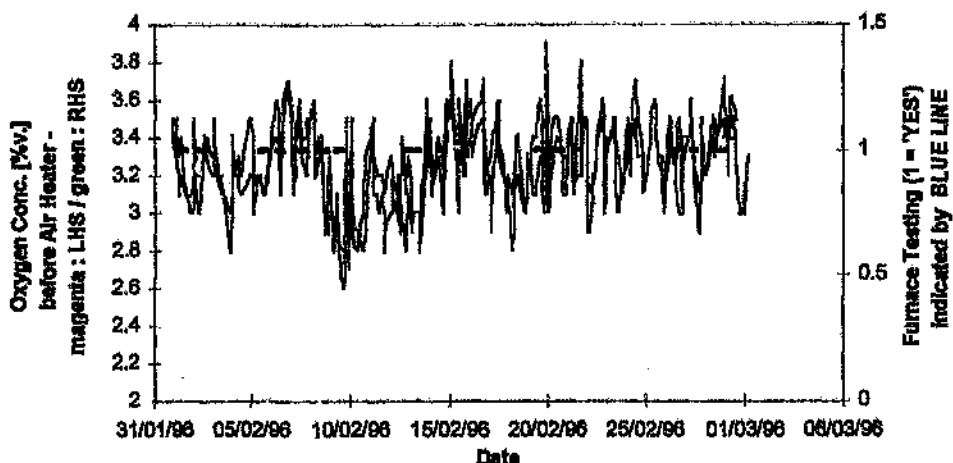


Figure 26 : Log of Oxygen Concentrations (LHS/RHS) before Airheater

The fluctuations in boiler operating parameters during the time of testing indicate that the furnace probing results require a critical review. It must however be noted that during the time it took to perform a furnace probing experiment, there were considerable variations in the property measured. This was particularly evident in the burner belt region of the boiler. The furnace exit conditions were considerably more stable.

The repeatability of furnace exit measurements was tested by means of furnace exit temperature measurements of boilers 9 and 10. These results of these measurements have been illustrated Figures 27 and 28.

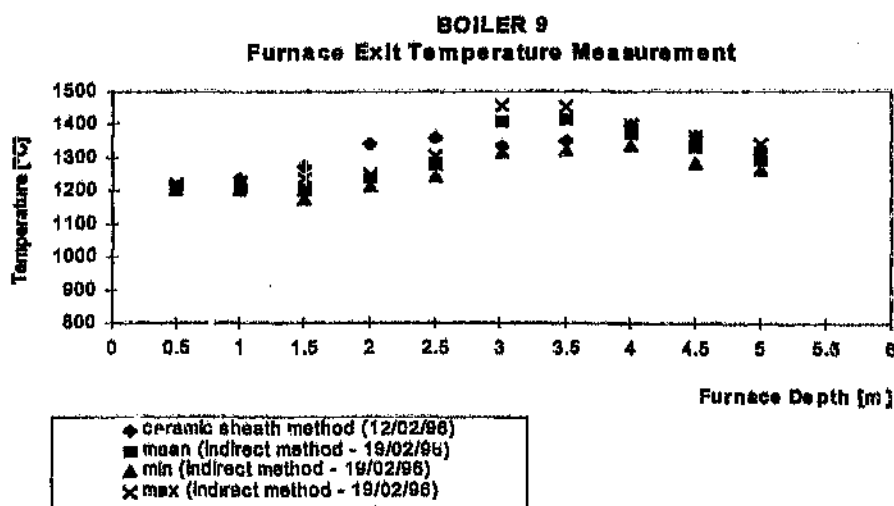


Figure 27 : Temperature Measurement - BOILER 9

**BOILER 10
Furnace Exit Temperature Measurement**

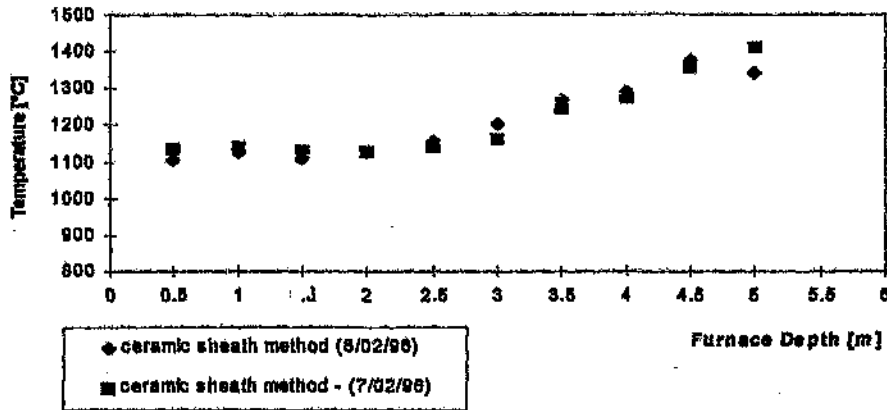


Figure 28 : Temperature Measurement - BOILER 10

The results depicted in Figure 27 show that most of the temperature measurements for 12/02/96 (ceramic sheath method) lie within the range observed on 19/02/96 (indirect method).

The results of the furnace tests did, however, show that furnace measurements are subject to considerable variations over short periods. This should be considered when comparing model predictions to measured data.

3.2 FURNACE RESIDENCE TIME DISTRIBUTIONS

The results of the RTD measurements have been presented in Table 5. In all RTD tests performed, all but one of the top burner rows was in operation.

Table 5 : RTD Measurement Results

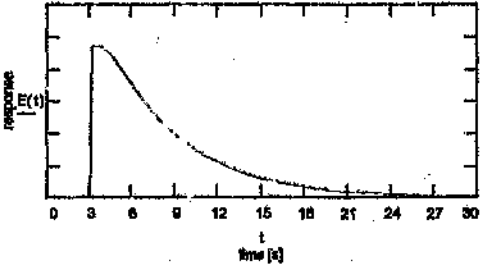
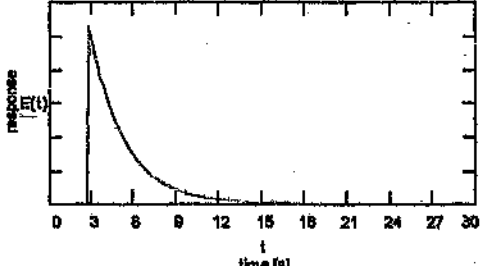
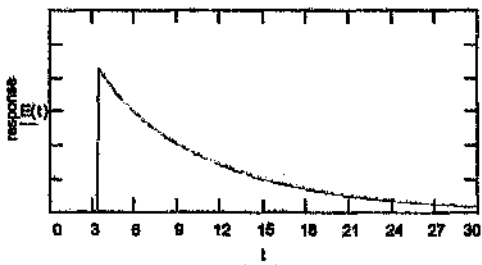
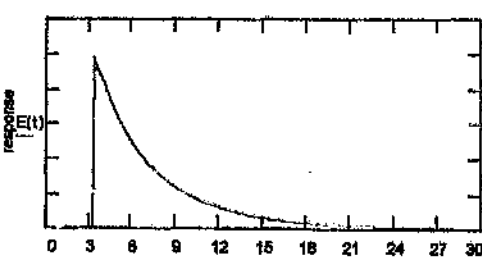
RTD from Burner Level	Insertion Depth of Probe [m]	Dead Time [s]	Furnace RTD
top, right	2	3.1	
top, right	3	2.4	
top, left	4	3.5	
top, right	4	3.4	

Table 5 : continued

RTD from Burner Level	Insertion Depth of Probe [m]	Dead Time [s]	Furnace RTD
middle, left	4	3.7	
middle, right	4	3.6	
bottom, left	4	5.4	
bottom, right	4	4.4	

The detailed determinations of the furnace RTD's are provided in Appendix 1.3. By summing the quotient of the vertical cell dimensions and upwards velocities predicted by AIOLOS at the furnace centre, an estimation of the minimum residence time (*viz.* the dead time) was made. These results are as presented in Table 6.

Table 6 : Minimum Residence Times predicted by AIOLOS

predicted minimum residence time	τ_d [s]
from top burner row	1.52
from centre burner row	2.42
from bottom burner row	4.01

It should be noted that the upwards velocities in the near burner regions would be less than at the furnace centre, which would explain why AIOLOS predicts considerably lower minimum residence times than were measured. An improved residence time prediction may be obtained if the vertical velocity components of a variety of furnace gas bundles were tracked. This would however require a dedicated sub-program within the post processor and was not attempted for the purpose of this investigation.

3.3 COAL PROPERTIES

As noted in section 2.3.1 of this report, both pf and raw coal from the mill feeder system were sampled. Proximate- and Ultimate analyses, as well as Calorific Value Determinations for the raw coal for various days of testing, are presented in Table 7.

Table 7 : Properties of Raw Coal

Property	31/01/96	01/02/96	05/02/96	07/02/96	08/02/96	17/02/96	25/02/96
[% by mass, air dried]							
Inherent Moisture	3.5	3.5	3.5	3.5	3.4	3.6	3.7
Ash	24.3	25.6	23.8	24.8	26.4	23.4	24.1
Volatile Matter	22.5	22.3	22.2	22.7	22.3	23.7	22.2
Fixed Carbon (by difference)	49.7	48.8	50.4	49.0	47.9	49.3	50.0
[% by mass, air dried]							
C	58.82	57.41	58.68	58.56	56.79	58.94	58.69
H	2.90	2.80	2.83	2.87	2.80	2.95	2.85
N	1.40	1.35	1.40	1.37	1.32	1.40	1.39
S	0.69	1.09	0.71	0.76	0.89	0.76	0.88
Carbonates	0.57	0.61	0.65	0.37	1.04	1.10	0.61
Oxygen (by difference)	7.82	7.64	8.33	7.77	7.36	7.85	7.98
Gross CV, [MJ/kg, air dried]	23.05	22.47	23.05	22.76	22.24	23.20	22.97

The basic chemical analyses of the pf samples taken as well as particle size analysis on each mill are shown in Table 8. Due to problems encountered by power station staff with the pf sampling equipment, pf samples could not be obtained after 9 February 1996.

Table 8 : Properties of Pulverised Coal

Property	31/01/96	01/02/96	05/02/96	07/02/96	08/02/96
[% by mass, air dried]					
Inherent Moisture	2.9	2.6	2.9	2.6	2.7
Ash	25.1	24.0	25.1	23.5	23.1
Volatile Matter	24.2	24.3	22.8	25.2	14.9
Fixed Carbon (by difference)	47.8	49.1	49.2	48.7	49.3
[% by mass, air dried]					
C	57.92	59.09	59.17	59.76	60.34
H	3.13	3.01	3.30	3.89	3.50
N	1.36	1.38	1.51	1.57	1.54
S	0.88	0.80	0.96	0.83	0.80
Carbonates	0.59	0.63	0.71	0.82	0.85
Oxygen (by difference)	8.12	8.49	6.35	7.03	7.17
Gross CV, [MJ/kg, air dried]					
	22.65	23.19	22.72	23.63	22.99
Particle Size Distribution					
	%	%	%	%	%
Mill A, <38 µm	64	25	63	64	43
38-75 µm	20	27	22	17	32
>75 µm	16	48	15	19	25
Mill B, <38 µm	47	74	31	78	-
38-75 µm	28	17	37	14	-
>75 µm	25	9	32	8	-
Mill C, <38 µm	-	-	-	-	44
38-75 µm	-	-	-	-	31
>75 µm	-	-	-	-	25
Mill D, <38 µm	49	65	39	78	72
38-75 µm	23	23	27	17	15
>75 µm	28	12	34	5	13
Mill E, <38 µm	32	-	40	44	42
38-75 µm	30	-	29	27	25
>75 µm	38	-	31	29	33
Mill F, <38 µm	46	54	74	73	49
38-75 µm	25	23	16	17	24
>75 µm	29	23	10	10	27

Detailed particle size analyses of the above pulverised coal samples have been attached in Appendix 2.3.

The R.C. Composite sample was subjected to a more extensive set of analyses, the results of which are presented in Table 9. Note that a kinetic summary for char combustion is included and will be referred to in section 3.5 of this report. Appendix 2.1 contains the detailed derivation of the kinetic parameters listed in Table 9.

Table 9 : Coal Properties Summary for R.C. Composite

Total Moisture : (as received)	9.8			
Proximate :	air dried	as received	dry	daf
Moisture	3.5	9.8		
Ash	25.0	23.4	25.9	
Volatiles	22.8	21.4	23.6	31.9
Fixed Carbon	48.7	45.4	50.5	68.1
Ultimate :				
Carbon	58.76	55.06	60.89	82.18
Hydrogen	3.02	2.83	3.13	4.22
Nitrogen	1.39	1.30	1.44	1.94
Sulphur	0.76	0.71	0.79	1.06
Carbonates	0.76	0.71	0.79	1.06
Oxygen	6.81	6.16	7.06	9.52
Oxygen + Carbonates		6.87		
Calorific Value	22.69	21.26	23.51	31.73
DTF DEVOLATILISATION				
Volatile Matter	28.025			
Q Factor ²	1.229			
DTF FULL TEST				
Size Fraction	<38 µm	38-75 µm	> 75 µm	Unit
Pre-exponential Factor	4269	70549	3212	kg/m ² .s.atm
	4213	69626	3170	kg/m ² .s.bar
Activation Energy	144.5	181.6	132.98	kJ/mol
Activation Energy /R	17380	21843	15995	K

² When a coal is devolatilised at higher temperatures (and heating rates) than would be typical for a Proximate analysis, the coal produces a higher yield of volatile matter. The Q Factor accounts for this difference and is the ratio of the volatiles produced under simulated furnace conditions and the volatiles produced in laboratory standard Proximate analysis.

Further furnace simulations were performed with coal sampled from the feeder belt of the run of mine Washing Plant at the colliery that provides Hendrina Power Station with coal. These simulations were performed to assess the effect of coal with differing char combustion kinetics on furnace conditions. This coal is referred to as 'Washing Plant Feed'. Its properties summary is presented in Table 10, while Appendix 2.2 contains the derivations of the kinetic parameters.

Table 10 : Coal Properties Summary for Washing Plant Feed

Total Moisture : (as received)	8.7			
Proximate :	air dried	as received	dry	daf
Moisture	2	8.70		
Ash	29.6	27.62	30.20	
Volatiles	24.4	22.77	24.90	35.67
Fixed Carbon	44	40.92	44.90	64.33
Ultimate :				
Carbon	58.24	52.47	57.39	82.22
Hydrogen	3.34	3.12	3.41	4.88
Nitrogen	1.4	1.31	1.43	2.05
Sulphur	0.84	0.76	0.86	1.23
Carbonates	0.53	0.49	0.54	0.77
Oxygen	6.05	5.51	6.17	8.85
Oxygen + Carbonates		6.01		
Calorific Value	22	20.53	22.45	32.16
DTF DEVOLATILISATION				
Volatile Matter	29.92			
Q Factor	1.226			
DTF FULL TEST				
Size Fraction	<38 μm	38-75 μm	> 75 μm	Unit
Pre-exponential Factor	208	1077	286	kg/m ² .s.atm
	205	1083	282	kg/m ² .s.bar
Activation Energy	101.66	124.9	99.45	kJ/mol
Activation Energy /R	12228	15623	11962	K

3.4 ASSESSMENT OF THE AIOLOS DEVOLATILISATION MODEL

The results of the DTF tests performed to characterise the devolatilisation behaviour of the Hendrina R.C. Composite sample and the analysis to determine the kinetic parameters of the single step, temperature dependant reaction model used in AIOLOS are attached in Appendix 2.1.

Figure 29 shows the graphs of extent of devolatilisation vs. residence time for both size fractions tested, while Table 11 lists the kinetic parameters of the devolatilisation of the R.C. Composite coal.

Table 11 : Kinetic Parameters of the Devolatilisation Process

Size Fraction	< 38 μm	38 - 75 μm	AIOLOS Default
Pre-exponential Factor [1/s]	61.1	4.63	1.5E5
Activation Energy [kJ/mol]	49.7	11.6	74

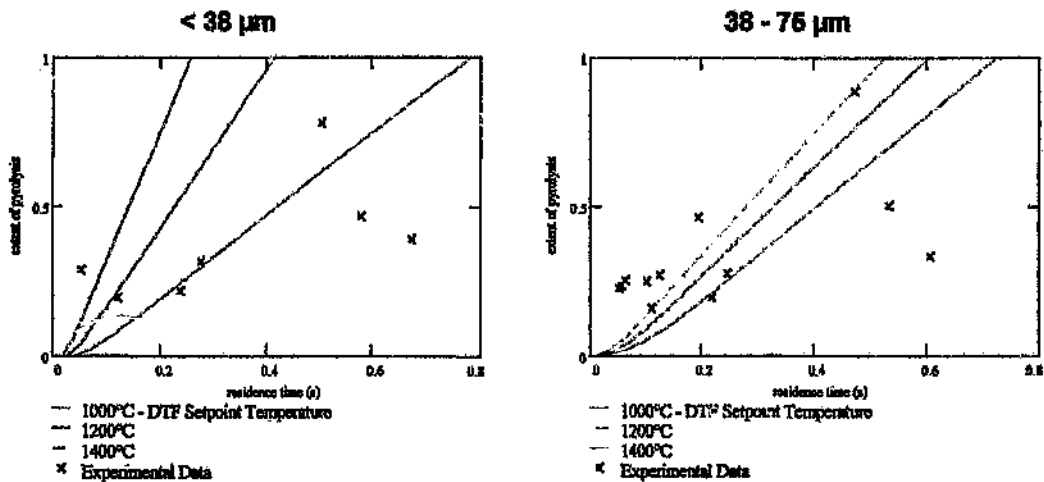


Figure 29 : Coal Devolatilisation in DTF

It was found that when the DTF devolatilisation kinetics were specified in the combustion model, the char combustion process was not activated. The AIOLOS default values were subsequently used.

In order to validate the DTF results, devolatilisation tests should be carried out on more Eskom coals. In the event of the presented results being confirmed, the devolatilisation model used in AIOLOS would have to be revisited and modified as in its current form the model clearly does not describe the devolatilisation process satisfactorily.

3.5 ASSESSMENT OF AIOLOS CHAR COMBUSTION MODEL

The char combustion model used in AIOLOS is based on the same model used in the determination of DTF kinetic parameters, and as such should be suitable for Eskom use. A model of the DTF utilising the char combustion model found in AIOLOS produces reasonable predictions of the experimental data. The kinetic parameters of

the R.C. Composite sample and the Washing Plant Feed have been summarised in Tables 9 and 10 respectively. Figures 30 and 31 show the predicted burnout profiles and DTF data of R.C. Composite sample used to determine the kinetic parameters, while Figures 32 and 33 do so for the Washing Plant Feed. The burnout profiles should ideally pass through the DTF data points given a certain DTF setpoint temperature.

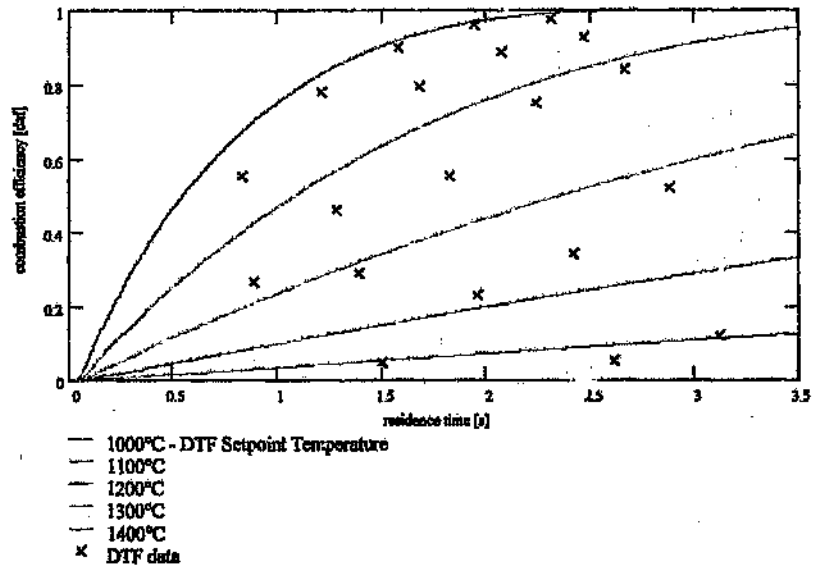


Figure 30 : Combustion Extent vs. time - R.C.Composite < 38 μ m

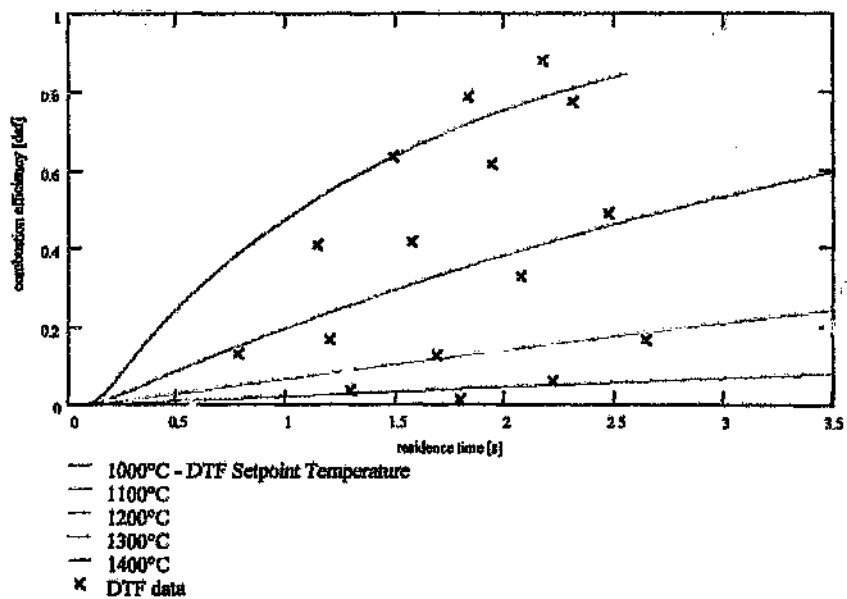


Figure 31 : Combustion Extent vs. time - R.C.Composite 38-75 μ m

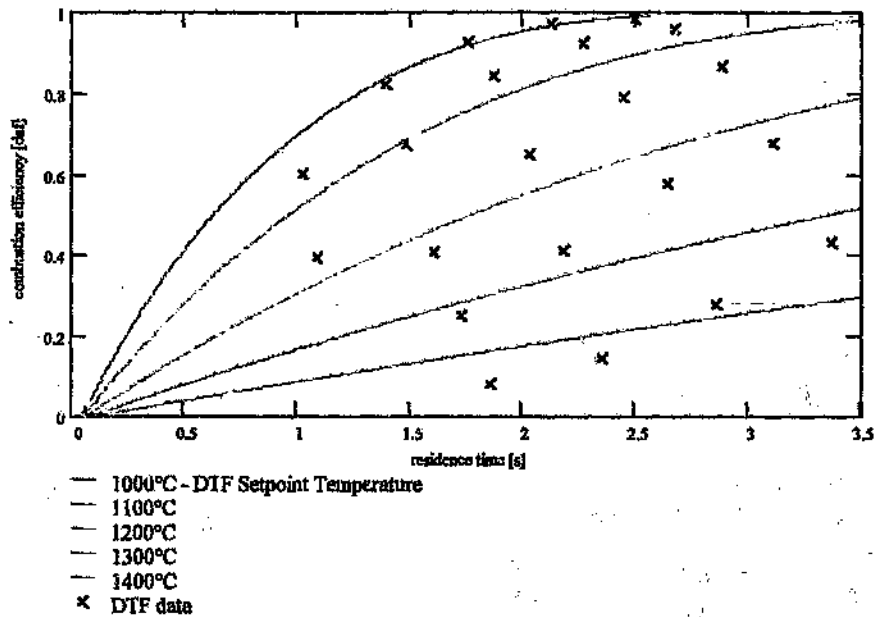


Figure 32: Combustion Extent vs. time - Washing Plant Feed. < 38 μm

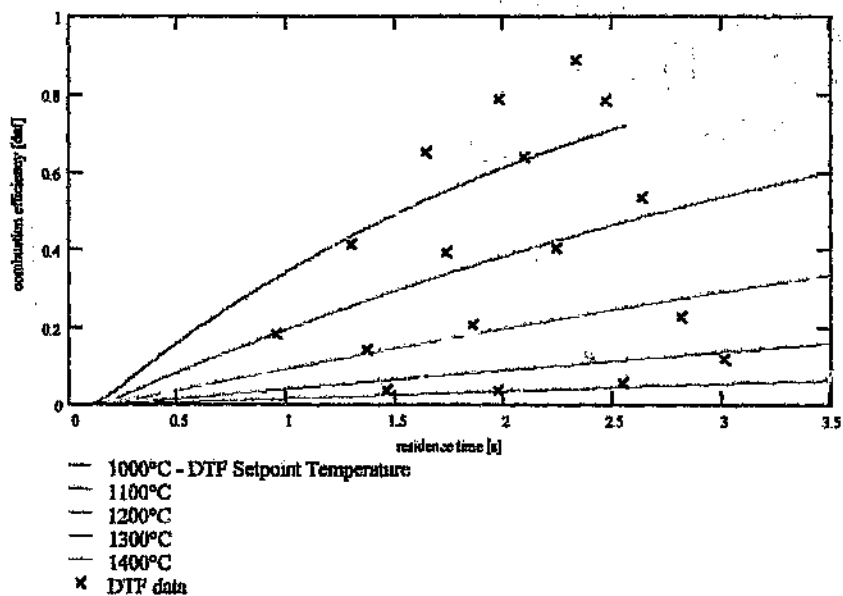


Figure 33 : Combustion Extent vs. time - Washing Plant Feed. 38-75 μm

3.6 COMPARISON OF PREDICTED AND MEASURED FURNACE DATA

Contour plots of furnace conditions are shown for various furnace levels as indicated on the isometric diagrams of the furnace. The contour plots also show the position and results of measurements. For the purpose of comparing experimental and measured data, graphs of experimental vs. measured data are shown as well.

3.6.1 Furnace Exit Conditions :

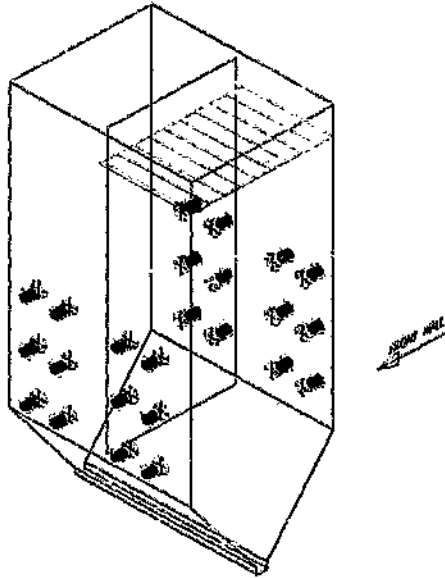


Figure 34 : Isometric View of Furnace showing the horizontal furnace plane at which data is presented

3.6.1.1 Gas Temperatures :

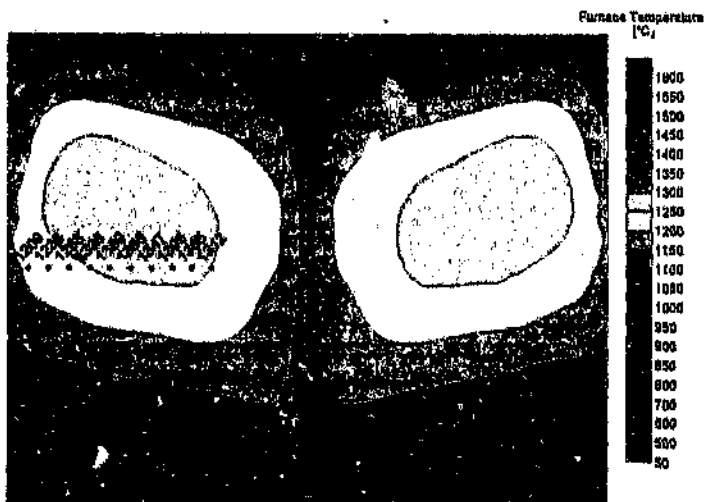


Figure 35 : Contour Plot of Furnace Exit Gas Temperature [°C]

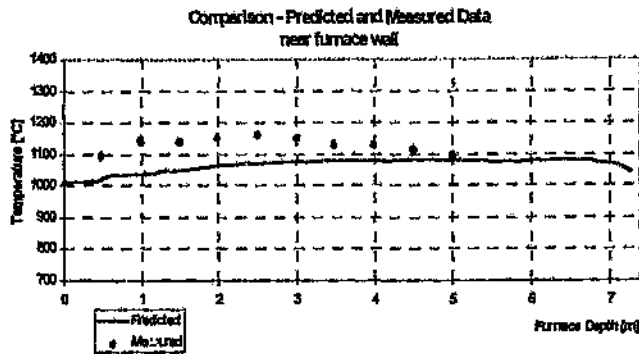


Figure 36 : Furnace Exit Temperatures [°C]

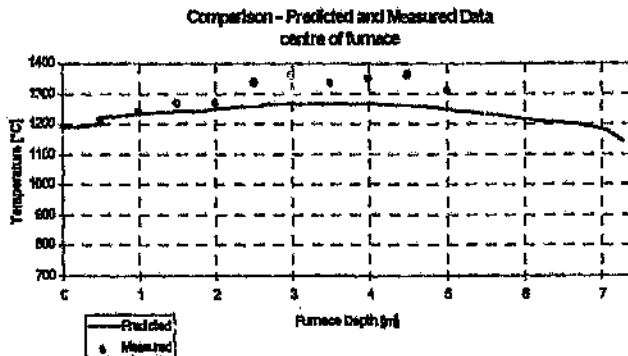


Figure 37 : Furnace Exit Temperatures [°C]

The furnace exit temperatures are an important indicator of furnace performance because high furnace exit temperatures suggest furnace under-performance which leads to high temperatures in the convective region of the boiler which, in turn, results in accelerated ageing of convective heat exchangers, fouling and possible control problems. High furnace exit temperatures may result from :

- delayed combustion in the furnace (typically from unreactive coal feedstocks or feedstocks that are not suited to that particular furnace design because the furnace residence times may be too short)
- over-firing (in which case some of the evaporator is used as a superheater)
- impaired radiative and convective heat transfer in the furnace region (caused primarily by insulating or reflective deposits on the furnace walls)

The predicted results tend to be lower than the measured temperatures. It should however be kept in mind that the fluctuations in temperature at one point in the furnace exceeded the variance between predicted and measured data. The uncertainty in furnace temperature measurement together with the uncertainty of

some of the boundary conditions, particularly the ash deposit emissivity and the coal properties, may make predicting absolute furnace gas temperatures difficult. The effect of changes in coal combustion properties on combustion conditions has been explored, the results of which are presented 3.7 of this report.

3.6.1.2 Gas Flow Velocities

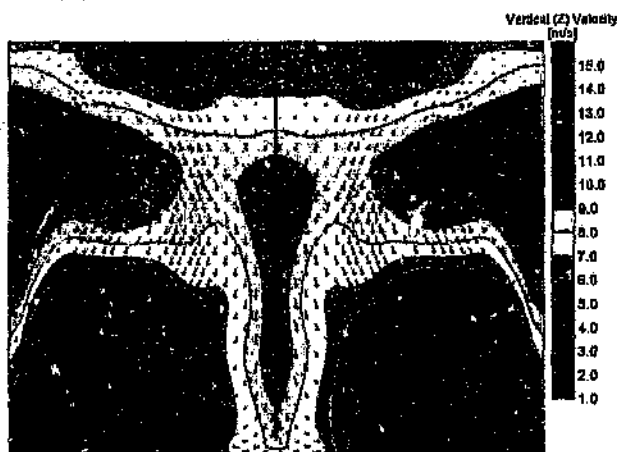


Figure 38 : Contour Plot of vertical velocities and Vector Plot of horizontal gas velocities. Peak horizontal velocity (represented by longest arrow) : 4.78 m/s

The velocity distributions at the furnace exit should ideally be uniform - i.e. little or no variation in vertical velocity as this is the ideal condition for a convective pass heat exchange- viz. no skew flow. The furnace exit velocities in Figure 38 show quite clearly the effect of having only one row of the top burner rows in operation and how this results in skewing the flow at furnace exit.

3.6.1.3 O₂ Concentrations

By the furnace exit, combustion of the pulverised coal should be complete. This means that the furnace exit oxygen concentration should be consistent with the oxygen concentration measured at the economiser outlet (except for tramp O₂). More importantly however, mixing of the combustion product should be good at furnace exit so that a fairly uniform O₂ distribution should be observed.

The predicted and measured O₂ concentrations show a very good agreement, suggesting that the model describes the combustion process with respect to O₂ consumption satisfactorily.

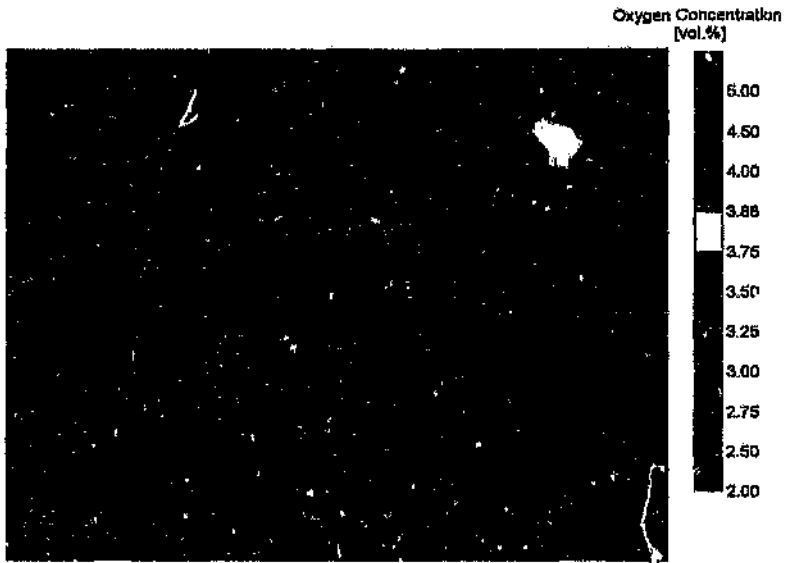


Figure 39 : Contour Plot of Furnace Exit Oxygen Concentrations

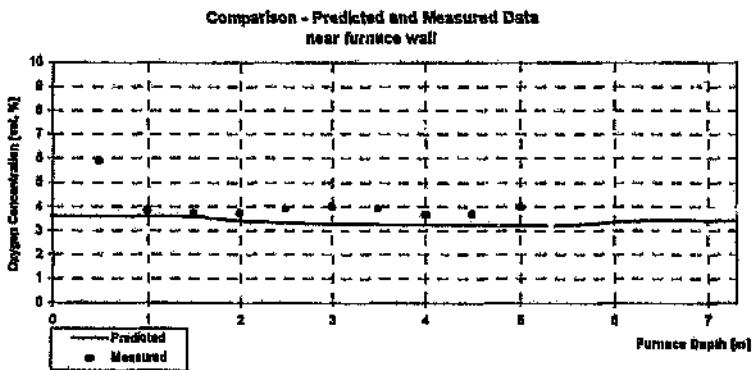


Figure 40 : Furnace Exit Oxygen Concentrations

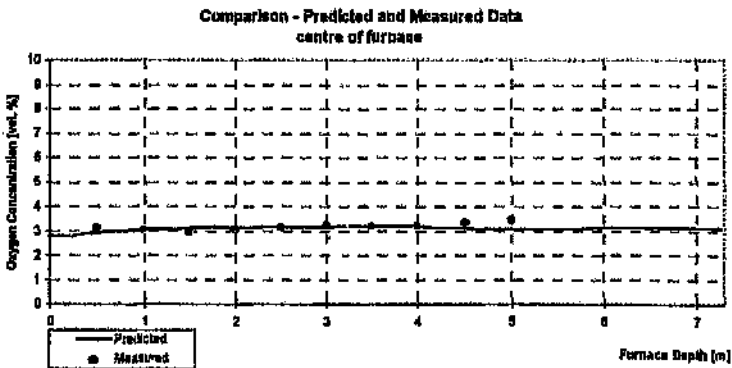


Figure 41 : Furnace Exit Oxygen Concentrations

One anomaly (@ $O_2 = 5.86\%$) is however evident in Figure 40, but is most likely attributable to an experimental error.

3.6.1.4 CO Concentrations

The CO concentrations predicted by the model were significantly higher than the measured values at furnace exit. Both the experimental technique used for the experimental measurement as well as the model should be revisited to determine the reason for these deviations.

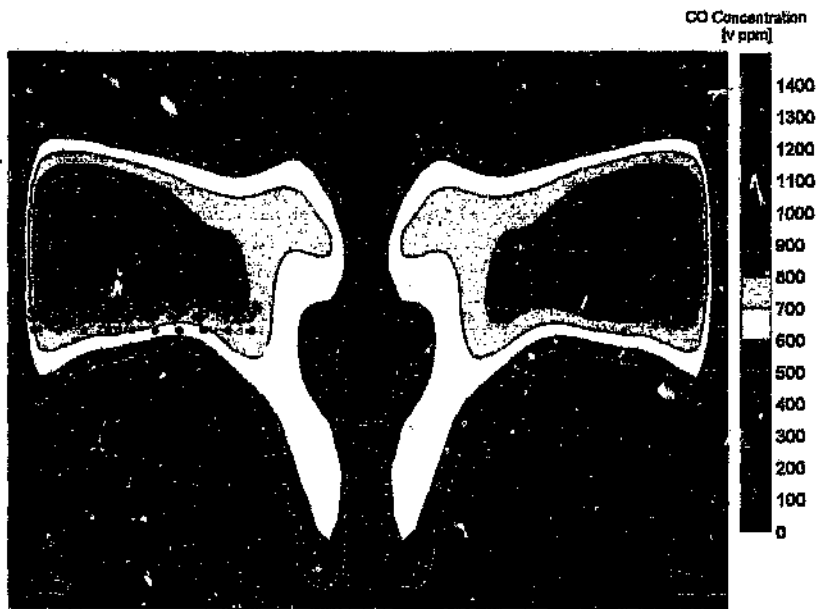


Figure 42 : Contour Plot of CO Concentrations [vol. ppm]

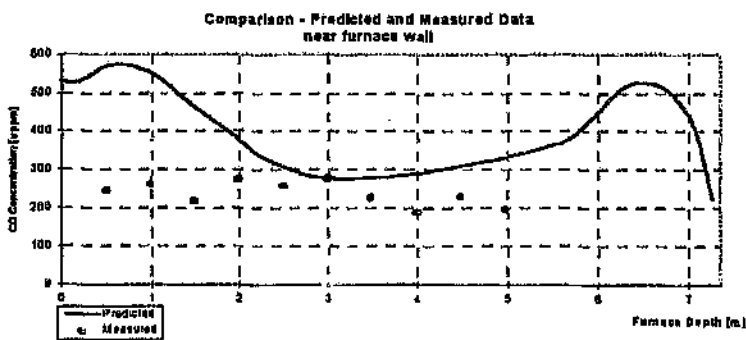


Figure 43 : CO Concentrations

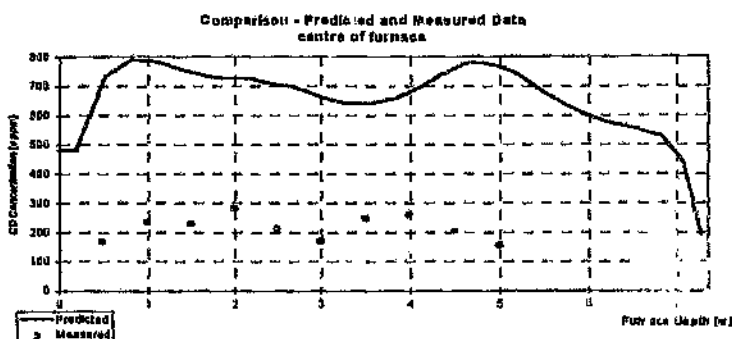


Figure 44 : CO Concentrations

3.6.1.5 NOx Concentrations

The NOx predictions and measurements correspond very well suggesting that the NOx model in the combustion code functions for over stoichiometric combustion. Further validation of the NOx code will be required for sub-stoichiometric conditions, which would be needed to evaluate possible low-NOx strategies. Further validation work is planned when the Australian Combustion Technology Centre's (ACTC) NOx test furnace is modelled with a variety of primary stoichiometries fueled by Eskom feedstocks.

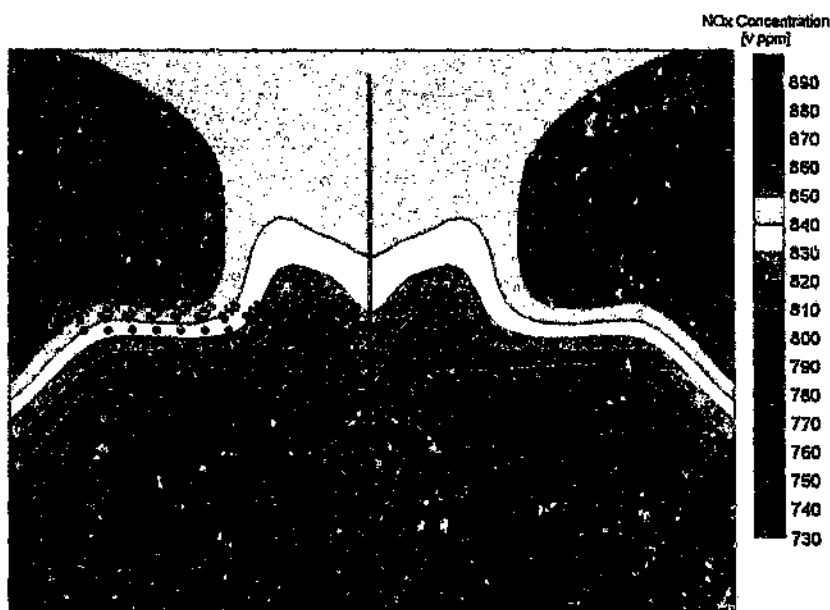


Figure 45 : Contour Plot of NOx Concentrations [vol. ppm]

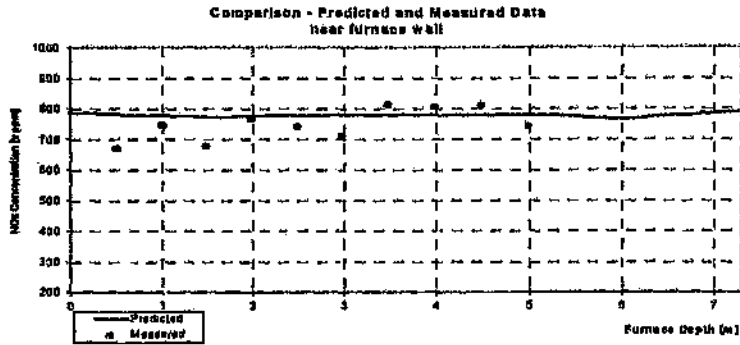


Figure 46 : NOx Concentrations

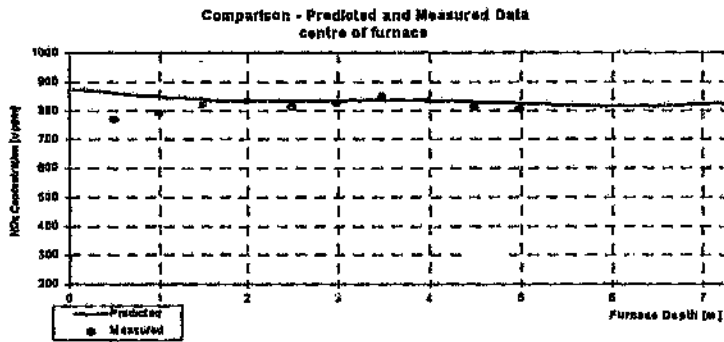


Figure 47 : NOx Concentrations

3.4.2 Top Burner Row

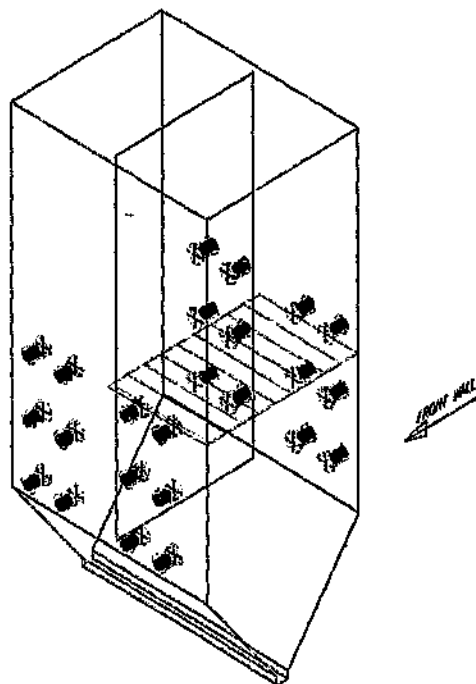


Figure 48: 3D view furnace : hatched horizontal plane indicates level at which data is presented
a. d measurements were made

3.6.2.1 Gas Temperatures

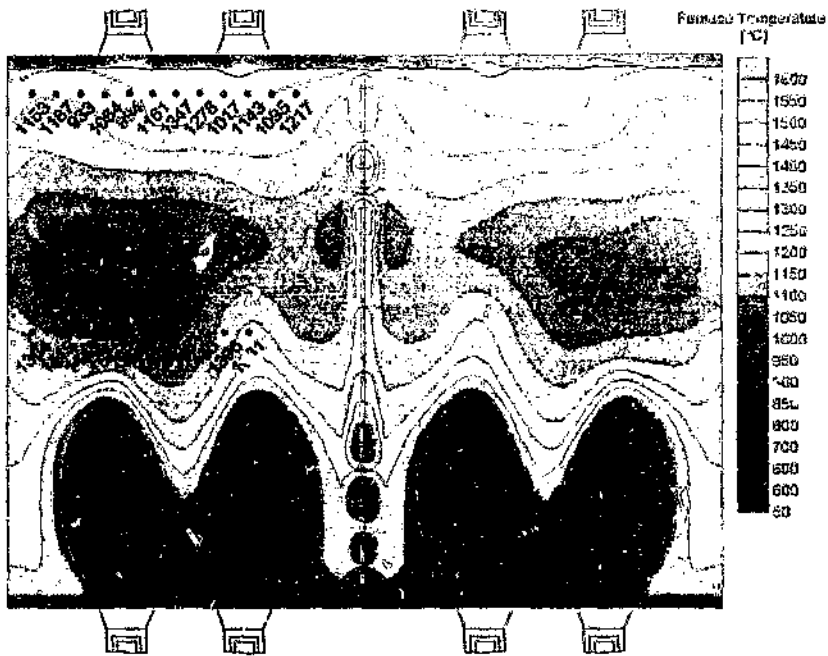


Figure 49 : Contour Plot of Furnace Temperatures on the top burner level

There is a good agreement between predicted and measured gas temperatures on the top burner level. The results do however suggest that an even hotter region exists than the measurements showed with temperatures exceeding 1500°C.

A concern is noted about the considerable penetration of the relatively cold inlet streams through the top burner row. This would lead to poor predictions of other combustion properties in the near burner regions.

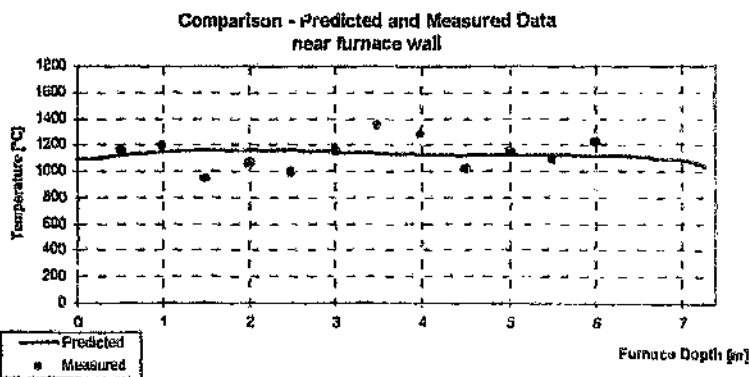


Figure 50 : Furnace Temperatures [°C]

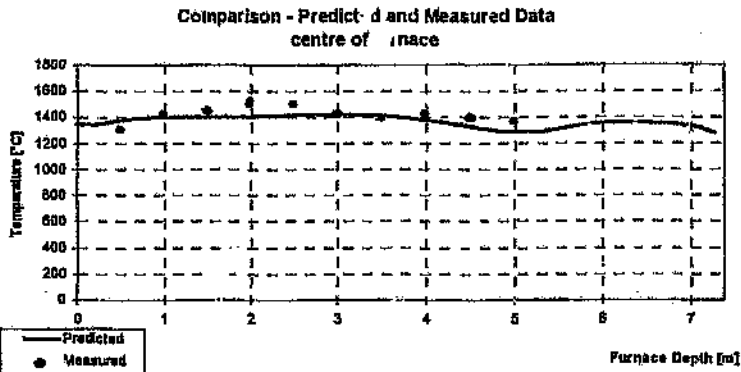


Figure 51 : Furnace Temperatures [°C]

3.6.2.2 Gas Flow Velocities

The contour plot of the vertical velocity component (refer to Figure 52) shows the downwards flow between the burner pairs clearly. The horizontal flow pattern shows the well defined recirculation zones on either side of the operational burner pairs. An internal recirculation zone is not apparent from the results due to the near burner zone not being discretised sufficiently finely. The combustion model's local grid refinement capability may improve the results so that investigations into burner stability and design may be performed. It must however be noted that the burner itself may be modelled with the combustion code in question. Note also the less defined recirculation zones on either side of the out of commission burner pairs. This recirculation zone is caused by the strong recirculation of the operational burners 4m below.

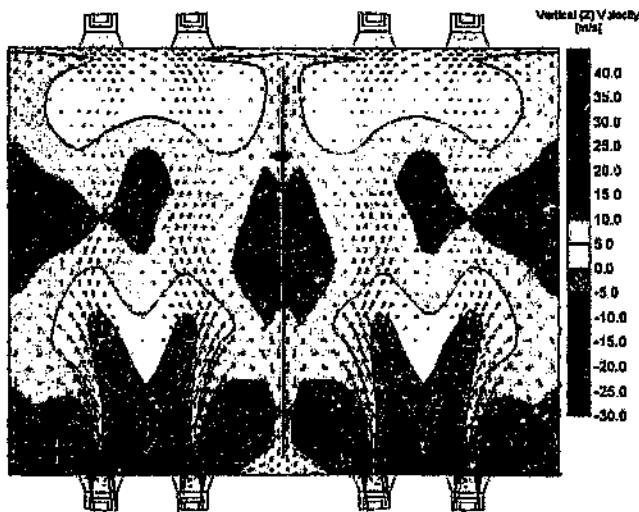


Figure 52 : Contour Plot of vertical velocities and Vector Plot of horizontal gas velocities. Peak horizontal velocity (represented by longest arrow) : 57.8m/s

3.6.3 Middle Burner Row

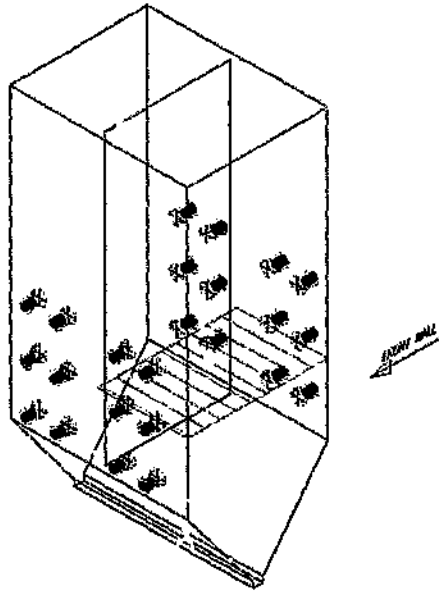


Figure 53: 3D view of furnace (hatched horizontal plane indicates level at which data is presented and measurements were made)

3.6.3.1 Gas Temperatures

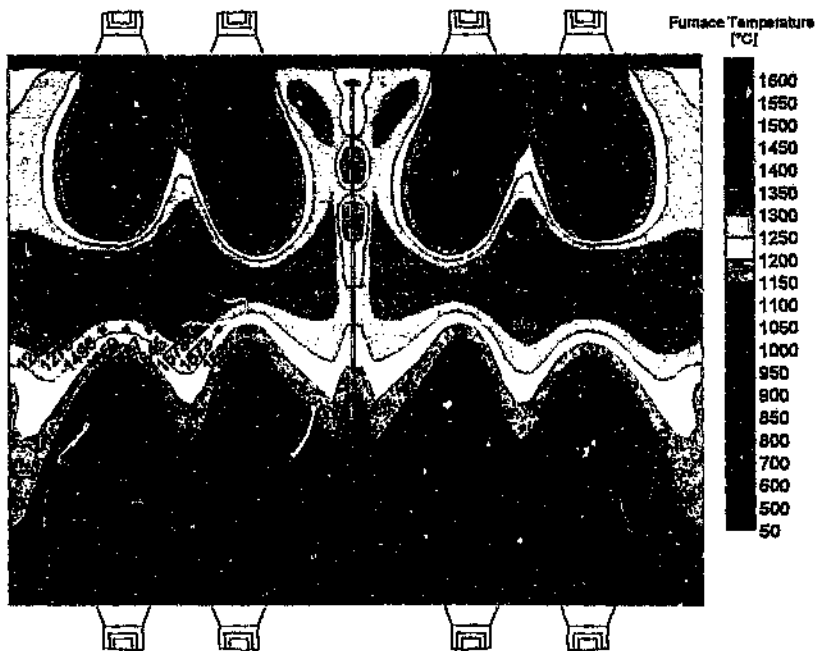


Figure 54 : Contour Plot of Furnace Temperatures on middle burner row

The concern noted in 3.2.1 (see Figure 52) about the relatively deep penetration into the furnace of inlet streams through the burners has in this case (refer Figure 54) a bearing on the predicted furnace centre temperature results which are lower than the measured ones. The contour plot (refer Figure 54) does however show a significantly hotter region away from the centre of the furnace, which could be caused by the mode of operation in which one of the top burner rows was out of commission. The fact that a considerable deviation in measured and predicted results exists suggest that the burner boundary conditions may have been specified incorrectly.

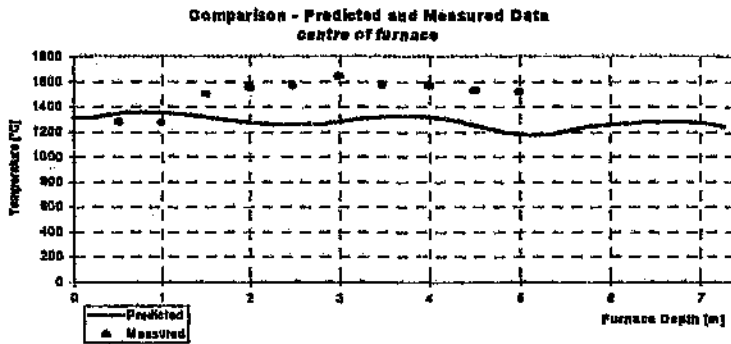


Figure 55 : Furnace Temperature [°C]

3.6.3.2 Gas Flow Velocities

Here again the contour plot of the vertical velocity component (refer to Figure 56) shows a well defined downwards flow between the burner pairs and the upward flow on either side thereof. Note also the recirculation zones on either side of the burner pairs and the flow directed towards the walls. Given high particle temperatures, there is a possibility of slag formation on the furnace walls. At the time of testing significant slag deposits were not however observed.

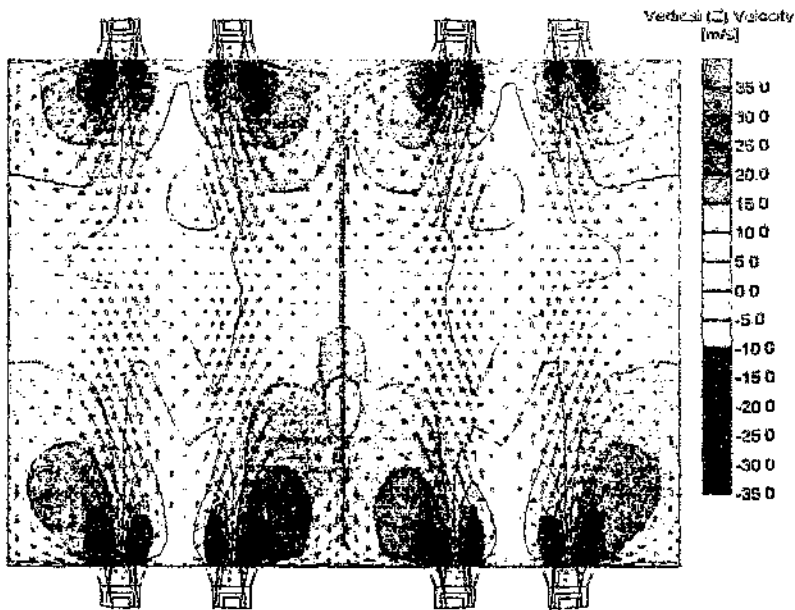


Figure 56 : Contour Plot of vertical velocities and Vector Plot of horizontal gas velocities. Peak horizontal velocity (represented by longest arrow) : 57.8m/s

3.6.4 Bottom Burner Row

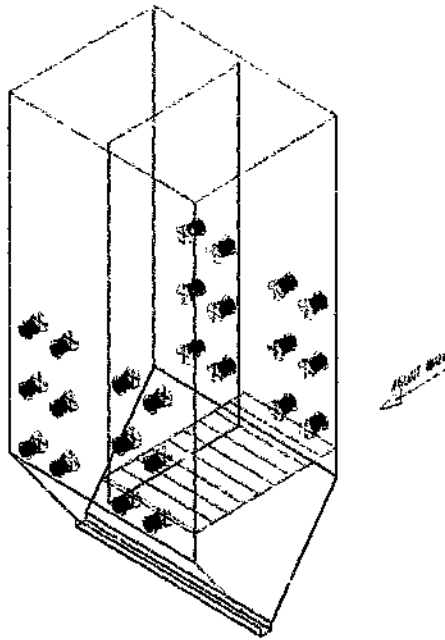


Figure 57 : 3D view of furnace : hatched horizontal plane indicates level at which data is presented and measurements were made

3.6.4.1 Gas Temperatures :

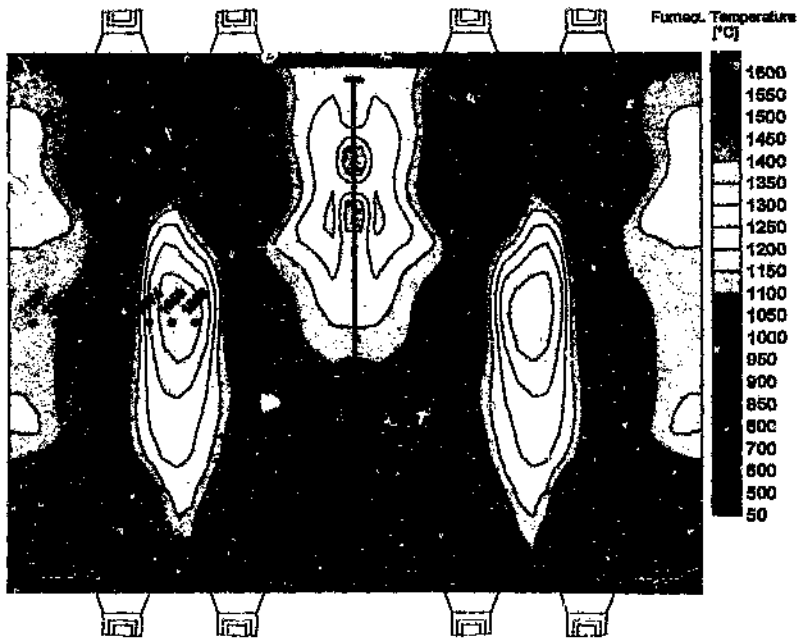


Figure 58 : Contour Plot of Furnace Gas Temperatures - bottom burner row

Note the skewness of the flame temperature contours (refer to Figure 58) on only one side of the furnace and further the difference in gas temperatures along the division wall of the furnace. This is due to the flow patterns having been established as a result of having one of the top burner rows out of operation (refer to Figures 52, 56 and 61).

Figures 59 and 60 show a good agreement between measured and predicted furnace temperatures. As mentioned before, the measurement of furnace gas temperatures near the burners was difficult in view of the fact that the conditions are not uniform and stable. Particularly the flame regions are highly turbulent and to obtain constant temperature readings is not possible.

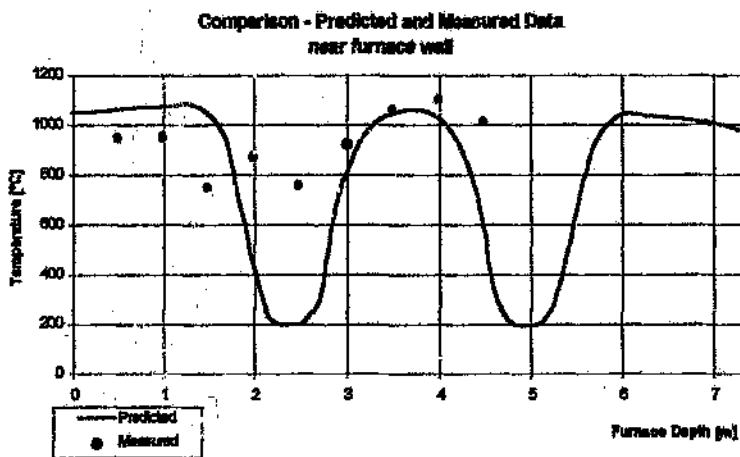


Figure 59 : Furnace Gas Temperatures [°C]

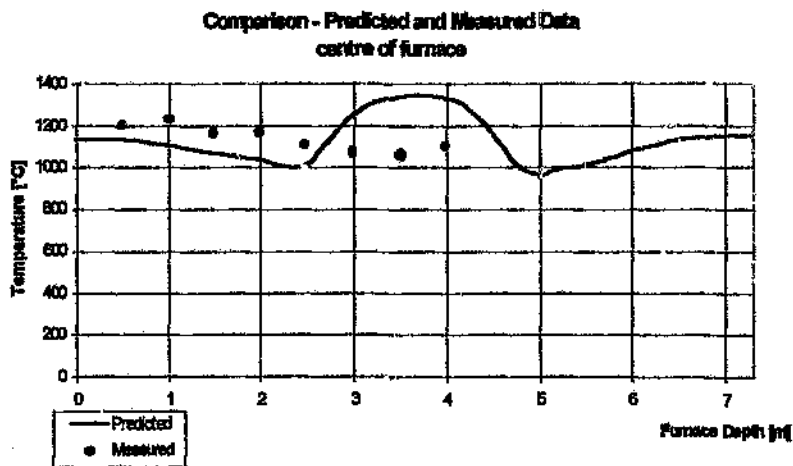


Figure 60 : Furnace Gas Temperatures [°C]

3.8.4.2 Gas Flow Velocities

Similar gas flow patterns to the middle burner level are predicted for this burner level with the exception of the large area of downwards flow between the burner pairs on the rear furnace wall (refer to Figure 61). These results suggest that the burner pairs on the rear wall penetrate the furnace through its centre while the front wall burners

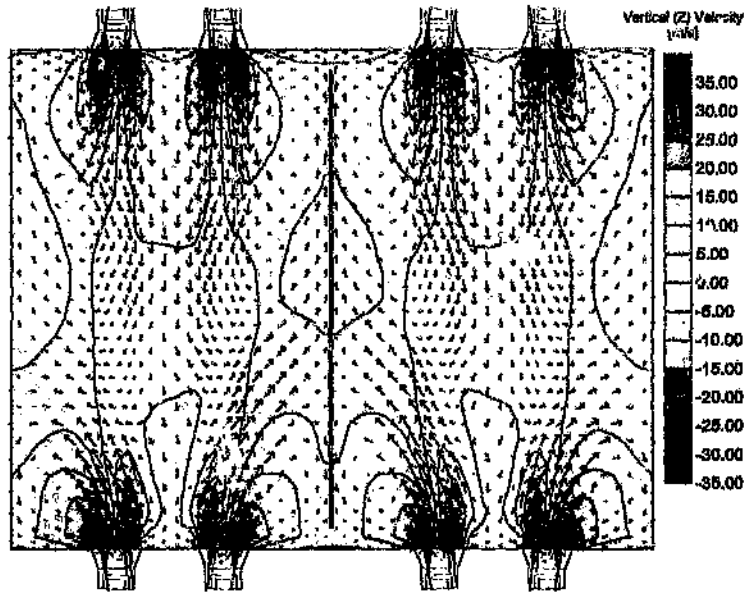


Figure 61 : Contour Plot of vertical velocities and Vector Plot of horizontal gas velocities. Peak horizontal velocity (represented by longest arrow) : 57.8m/s

have their swirling flow deflected to either side of the stream emanating from the rear wall burners toward the furnace walls. This may be caused by the fact that the top row on the rear wall was not in operation which would allow the establishment of the observed flow pattern.

3.6.5 Furnace Heat Fluxes

The heat fluxes through the furnace water wall were measured using heat flux probes situated on the left wall of the furnace. These were found to be generally low compared to heat fluxes measured at Matimba using identical devices. Further, the furnace temperatures at Hendrina were significantly higher than furnace temperatures measured at Matimba. Low heat fluxes and high furnace temperatures suggests that Hendrina's furnace heat transfer is impaired. Since approximately 90% of the heat transfer in the furnace occurs by radiation, a possible explanation for this may be that the ash deposits either reflect the incident radiation back into the furnace or the deposit layer acts as an excellent insulator resulting in a much higher furnace surface wall temperature and a resultant reduction in furnace heat transfer. In specifying the boundary conditions for the furnace simulation it was therefore decided to assume an emissivity of 0.5 for the entire furnace instead of an emissivity of 0.65 to 0.7 which is normally assumed for a clean, operational furnace. Experimental techniques for the determination of wall emissivities should be explored.

Figure 62 shows an incident heat flux contour plot of the left wall of the furnace, as predicted by the combustion model, while the heat flux measurement data (actual heat flux through the furnace wall), as logged on 13 February 1996, of selected heat flux meters are shown as heat flux vs. time graphs. The arrow between the heat flux vs. time plot and the contour plot of the incident heat flux indicates the position of the probe on the furnace wall. The saw tooth type behaviour of the measured heat fluxes (refer to graph 1 and 3) indicates that when a reflective and/or insulating build up on the probe tips either falls off or is removed, a sudden heat flux increase is observed. The heat flux vs. time pattern exhibited in graphs 4 and 6 has been observed previously at Matimba for heat flux probes situated near wall blowers, as was also the case at Hendrina's Unit 9 furnace. The time of wall blowing corresponds with the sudden increase in heat flux. Note, however, the different lengths of time that the heat flux remains high after wall blowing. This is most likely a result of the different nature of the deposits at different points in the boiler, due mainly to variations in deposition temperature and impact velocity of combusting particles with the furnace wall surface.

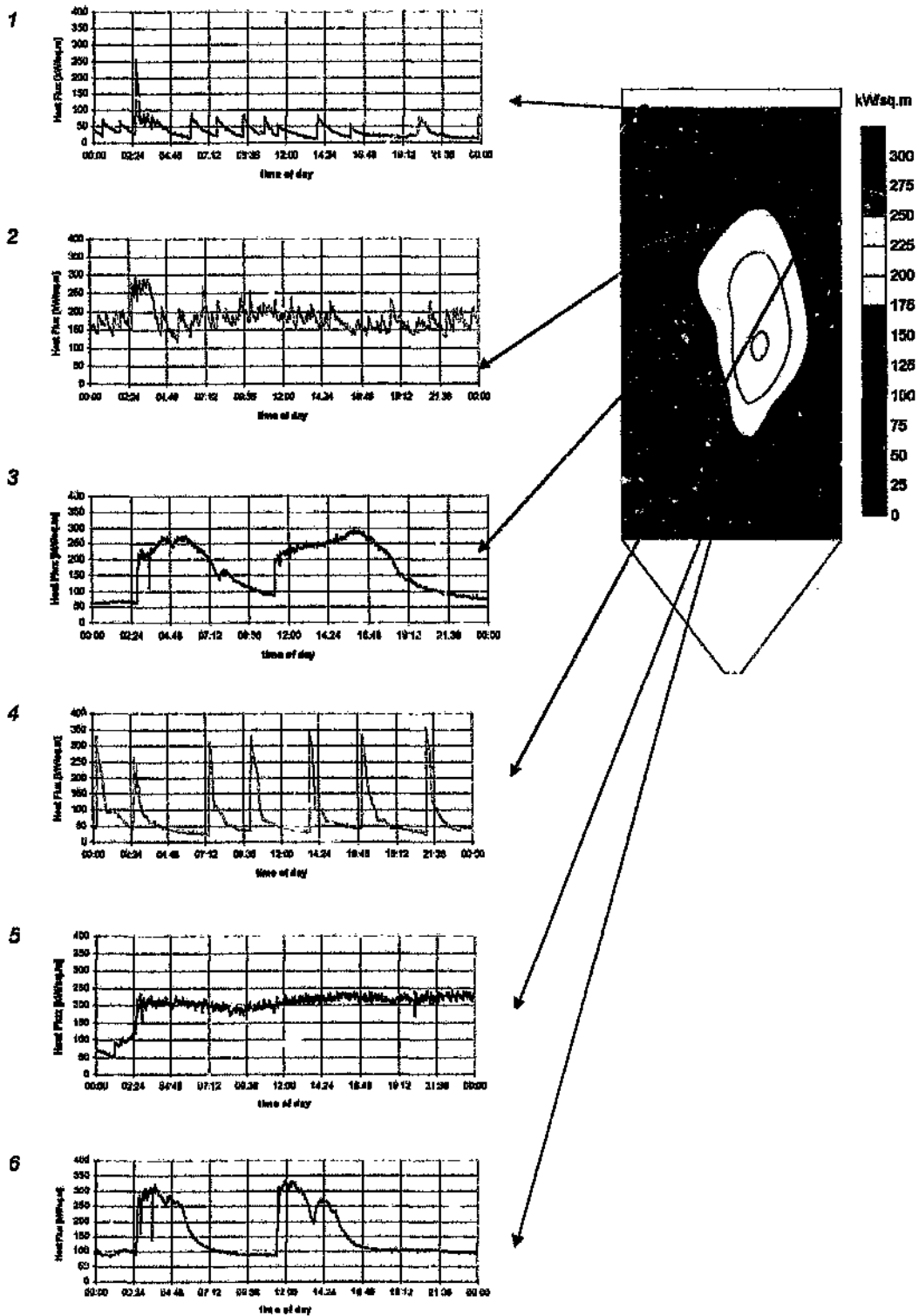


Figure 62: Predicted incident heat fluxes vs. measured heat fluxes through left furnace wall

Figure 63 shows the incident heat flux distribution to all the furnace walls as predicted by the combustion model. Note the low heat flux intensities on the front wall burner belt, which has all three burner rows in operation. This is attributable to the pf absorbing much of the radiation directed towards the front wall.

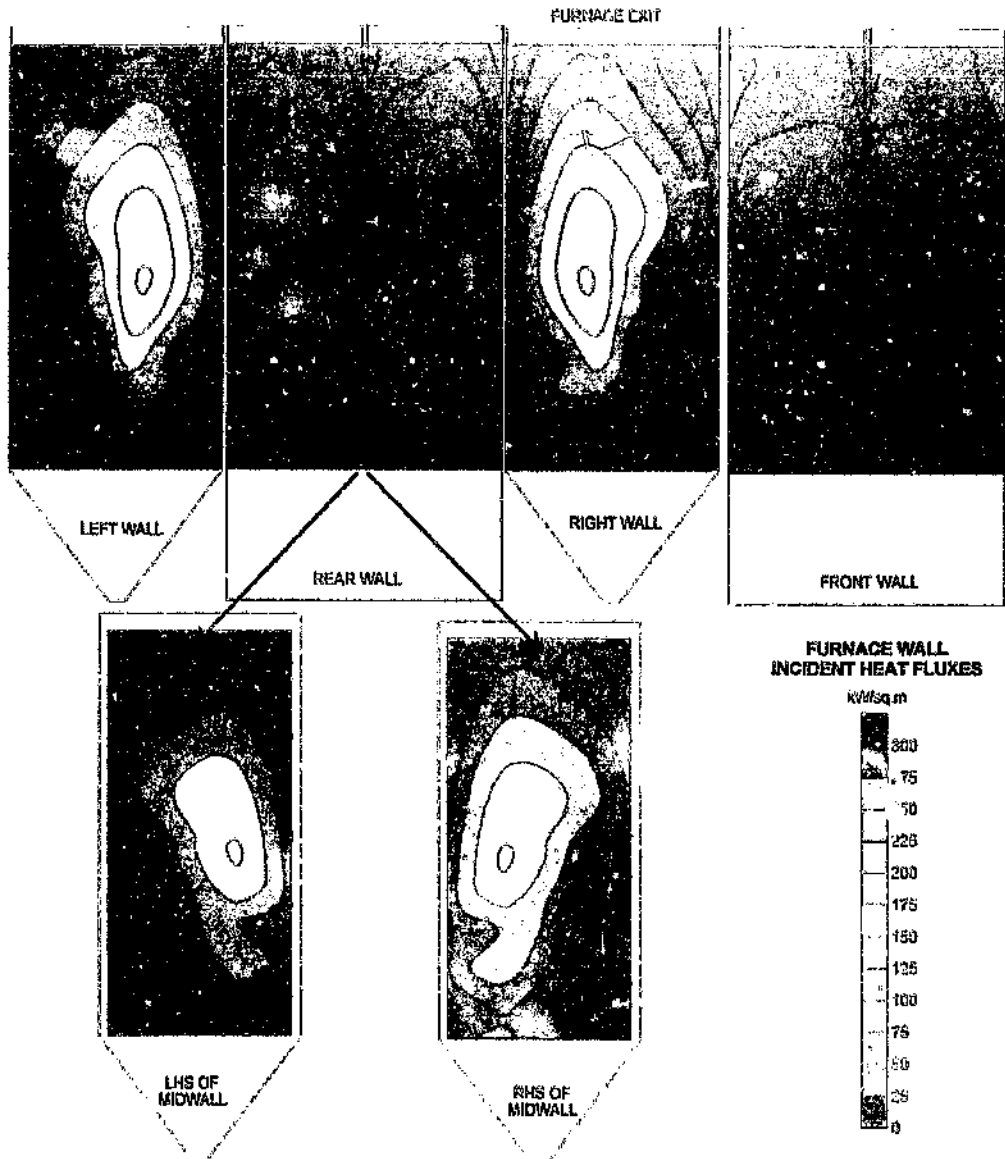


Figure 63: Incident heat flux to the furnace walls

3.7 COMPARISON OF MODEL PREDICTIONS FOR DIFFERENT COAL FEEDSTOCKS

Combustion simulations were performed using the results of DTF characterisation of the Hendrina R.C. Composite and the Washing Plant Feed coals, to examine the sensitivity of the model to changes in a coal's combustion behaviour. Figure 64 shows the particle size class averaged burnout profiles of the coals in question.

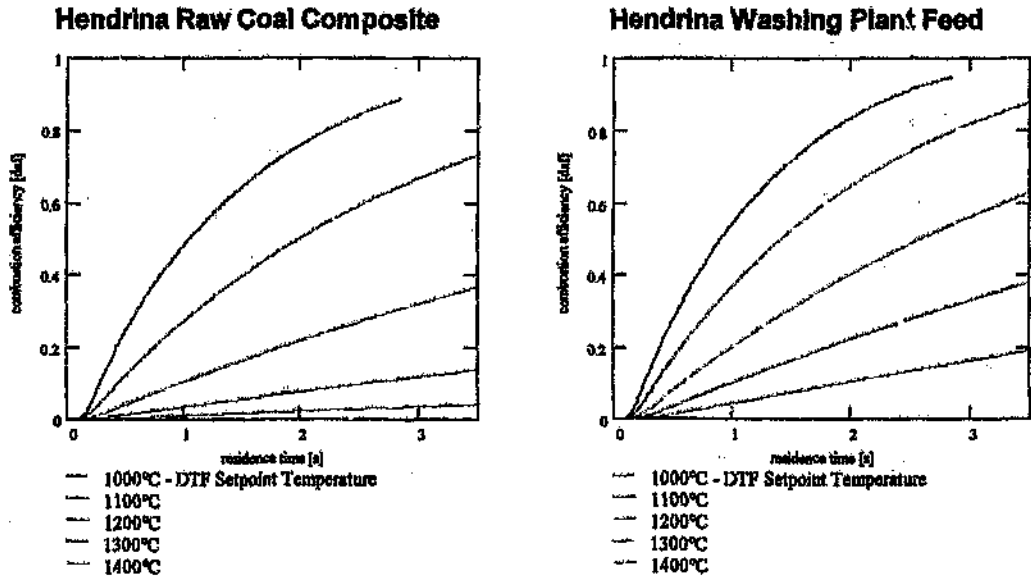


Figure 64 :
Burnout Profiles of Hendrina Feedstocks

The following simulation results show the effect of the different coals on furnace conditions.

3.7.1 Temperature Maps through the Furnace

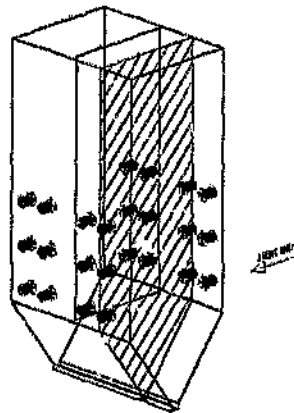


Figure 65 : Isometric View of Furnace showing the vertical furnace plane at which data is presented

In order to compare the effect of a change in coal reactivity realistically, the input rate of coal for both simulations was adjusted so that the furnace exit oxygen concentrations were similar at around 3 - 3.5 % by volume (refer to Figure 67). Given identical primary and secondary air inlet flowrates, the input rate of more reactive coal (W.P. Feed) was less than the input rate of the less reactive coal (R.C. Composite), despite the W.P. Feed having a higher ash content. This is attributed to the W.P. Feed achieving a greater burnout by furnace exit.

Hendrina Raw Coal Composite

Hendrina Washing Plant Feed

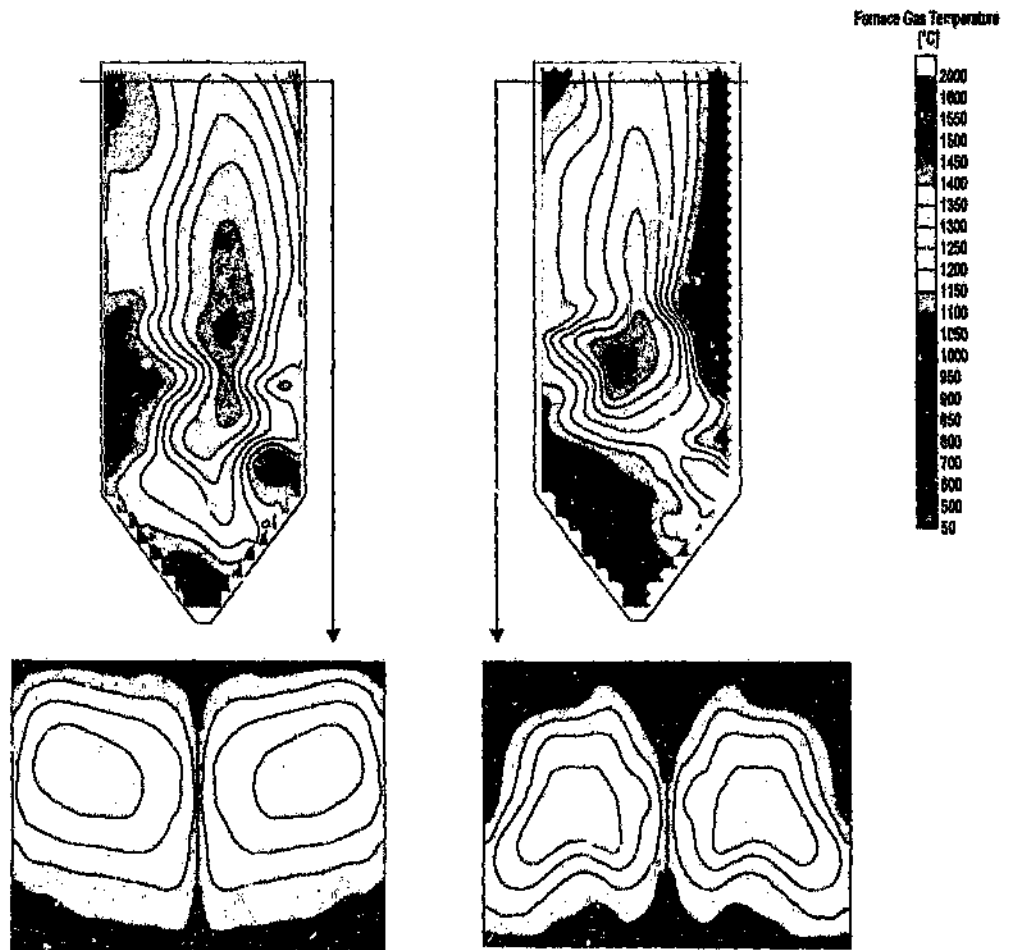


Figure 66 : Furnace Gas Temperature Maps

Note also how the region where the temperature is greater than 1400°C concentrates much lower in the furnace for the W.P. Feed than for the R.C. Composite (refer to vertical temperature contour plots in Figure 66). The furnace exit temperatures are however comparable (refer to furnace exit temperature contours in Figure 66). It

should be noted however, that the more reactive coal achieved a higher furnace efficiency in view of the reduced carbon in ash, i.e. coal consumption.

3.7.2 O₂ Concentrations at Furnace Exit

Hendrina Raw Coal Composite

Hendrina Washing Plant Feed

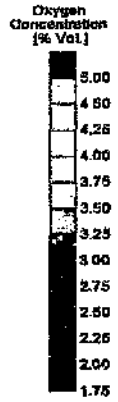


Figure 67 : Furnace Exit O₂ Maps

3.7.3 Heat Fluxes

Figure 68 shows heat flux contour plots onto the left wall for both feedstocks. Note how the more reactive feedstock shows high heatfluxes lower down in the furnace.

Hendrina Raw Coal Composite

Hendrina Washing Plant Feed

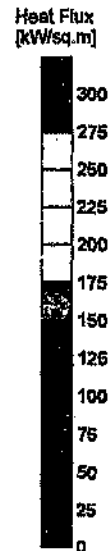
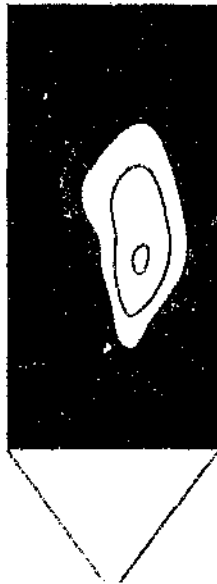


Figure 68 : Furnace Heat Fluxes

(onto left wall)

3.7.4 NO_x Production

Most of the NO_x originates from the oxidation of the nitrogen containing volatile species in the near burner zone as a direct result of the oxidising combustion conditions in this region. These conditions are typical for unstaged swirl burners. A significant difference in NO_x levels at furnace exit was therefore not expected for the two coals. The simulation results (refer to NO_x concentration contour plots in Figure 69) show this. Note, however, that the patterns of the contours differ. This is attributable to the differences in the flowfield, which depend strongly on the temperature distribution and, consequently, the density of combustion products in the furnace.

Hendrina Raw Coal Composite

Hendrina Washing Plant Feed

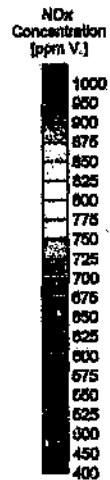
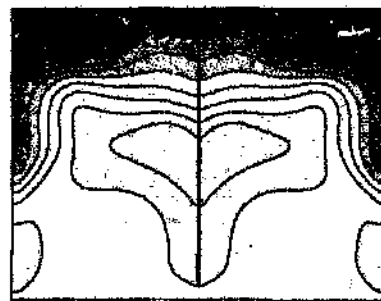
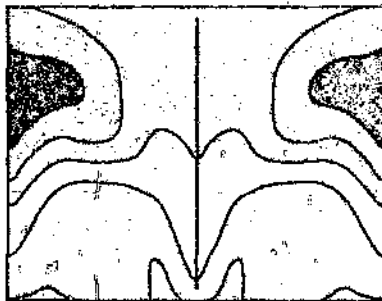


Figure 69 : Furnace Exit NO_x Maps

4. CONCLUSIONS

4.1 THE COMBUSTION MODEL VALIDATION

The combustion model predictions compare satisfactorily with furnace measurements. The use of the model is however at a developmental stage and therefore requires a critical review of the results. The model predictions depend to a great extent on accurate plant operation data which in many cases is not available. The use of reasonable estimates of this data is therefore required, and depending on the application of the model, should be borne in mind when reviewing combustion modelling results.

4.2 COMBUSTION MODELLING FOR COAL FEEDSTOCK ASSESSMENTS

The model showed that even a moderate difference in reactivity had a significant influence on the simulation results. The model appears therefore to be sufficiently sensitive to reactivity data for coal feedstock assessments.

4.3 THE COAL DEVOLATILISATION MODEL

The analysis of coal devolatilisation data obtained from DTF experiments suggested very low devolatilisation rates compared to what the University of Stuttgart assumes when this information is not available. When using the DTF experiment results in the combustion model, the model did not produce converged results. The University of Stuttgart default values were subsequently used as these produced acceptable results. The testing procedure designed to determine the pyrolysis kinetics does not produce results that can be used in setting the parameters of the pyrolysis sub-model used in the combustion code.

4.3 THE CHAR-COMBUSTION SUB-MODEL

The analysis of the DTF data for the coals in question suggests that char combustion model according to Field et al¹¹ (1967) was adequate. Refinements to the char combustion model may, however, improve the models capability to predict the effects of different coal feedstocks on furnace performance.

4.4 THE NO_x PRODUCTION MODEL

The model allows for the specification of the nitrogen partitioning within the coal (viz. fraction of coal nitrogen reporting to the volatile matter and char respectively) and for the simulations presented herein, the model produces plausible results.

4.5 RESULTS OF THE FURNACE INVESTIGATION

Given the experimental furnace data, as well as the kinetic and other characterisation of the feedstock used at the time of testing, it became apparent that Hendrina's Unit 9 furnace does not have too short a residence from the top burner row (approx. 2.2s) for the coal that was burnt during the testing period. High furnace exit temperatures, low furnace heat fluxes as well as very high furnace temperatures (up to 1645°C) were however observed, which suggests that heat transfer to the furnace walls was impaired. The cause of this could either be a higher than usual emissivity of ash deposits on the furnace wall, or the insulating nature of the ash, or a combination of the aforementioned. The heat flux measurements did show that wall blowing did increase the heat flux to the furnace walls considerably, suggesting that the furnace performance may be improved by ensuring that the furnace walls remain clean. The extent to which this can be achieved by additional or modified wall blowing hardware, or changes to wall blower operation, should be investigated.

4.6 NEW FURNACE TEST METHODOLOGIES

The indirect method of furnace gas temperature measurement proved to be considerably more robust and dependable than the traditional direct temperature measurement using ceramic radiation shields.

Further, the furnace residence time distribution measurement technique has now been demonstrated for wall fired plant as well. In conjunction with the combustion characterisation of the Hendrina feedstock, it could be concluded that coal reactivity was not the primary contributor to the high furnace exit temperatures observed.

6. RECOMMENDATIONS

5.1 RECOMMENDATIONS PERTAINING TO THE COMBUSTION MODEL

The AIOLOS furnace model should be validated for the Matimba Unit 6 furnace as well. The experimental data available from Matimba for feedstocks of varying reactivity would provide an ideal validation data set.

The NOx validation testwork and modelling results should be reviewed in order to assess the combustion model's potential application in evaluating NOx reduction hardware.

Refinements to the code's char combustion sub-model should be explored. These refinements should include specifying the combustion kinetics for all size fractions tested in the DTF, as well as introducing a description of the effect of combustion extent on the combustion kinetics of the coal.

The coal devolatilisation model should be reviewed and a suitable test methodology to experimentally determine a coal's devolatilisation rates should be researched.

As the effect of furnace wall emissivity has a significant effect on simulation predictions, the possibility of measuring this property should be explored.

5.2 RECOMMENDATION PERTAINING TO THE FURNACE TESTWORK AT HENDRINA

Improved furnace wall cleaning at Hendrina Power Station may help reduce furnace exit temperatures. It is recommended that this be explored. It should, however, be noted that burner hardware condition, primary and secondary air temperatures as well as coal properties (including grindsize and reactivity), may contribute to high furnace exit temperatures and reduced furnace performance.

5.3 RECOMMENDATION PERTAINING TO THE FURNACE TESTING METHODOLOGIES

The indirect furnace gas temperature measurement technique should be explored further to improve its accuracy and reliability.

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A4 copy of Figure 4 : Schematic of PTD Measurement Equipment

Appendix 1.1 FURNACE TESTING EQUIPMENT

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Appendix 1.1.2 12m probe - Design Drawings Design Calculations
Appendix 1.1.3 Sulphur Injection - Burner Door Design
Appendix 1.1.4 Air Cannon and Sulphur Canister - Assembly Sketch
Appendix 1.1.5 SO₂ Injector for probe RTD - Design Drawing
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Appendix 1.2 FURNACE TESTING PROCEDURES

Appendix 1.3 RESIDENCE TIME DISTRIBUTION MANIPULATIONS

Appendix 1.3.1 Top Right Burner to Furnace Exit - 2m into boiler
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Appendix 1.3.3 Top Right Burner to Furnace Exit - 3m into boiler
Appendix 1.3.4 Top Left Burner to Furnace Exit - 4m into boiler
Appendix 1.3.5 Top Right Burner to Furnace Exit - 4m into boiler
Appendix 1.3.6 Middle Left Burner to Furnace Exit - 4m into boiler
Appendix 1.3.7 Middle Right Burner to Furnace Exit - 4m into boiler
Appendix 1.3.9 Bottom Left Burner to Furnace Exit - 4m into boiler
Appendix 1.3.10 Bottom Right Burner to Furnace Exit - 4m into boiler

APPENDIX 2 COAL ANALYSES AND CHARACTERISATION

Appendix 2.1 Hendrina R.C. Composite

Appendix 2.1.1 Basic Analyses Datasheet
Appendix 2.1.2 Determination of Devolatilisation Kinetics
Appendix 2.1.3 Determination of Char Combustion Kinetics

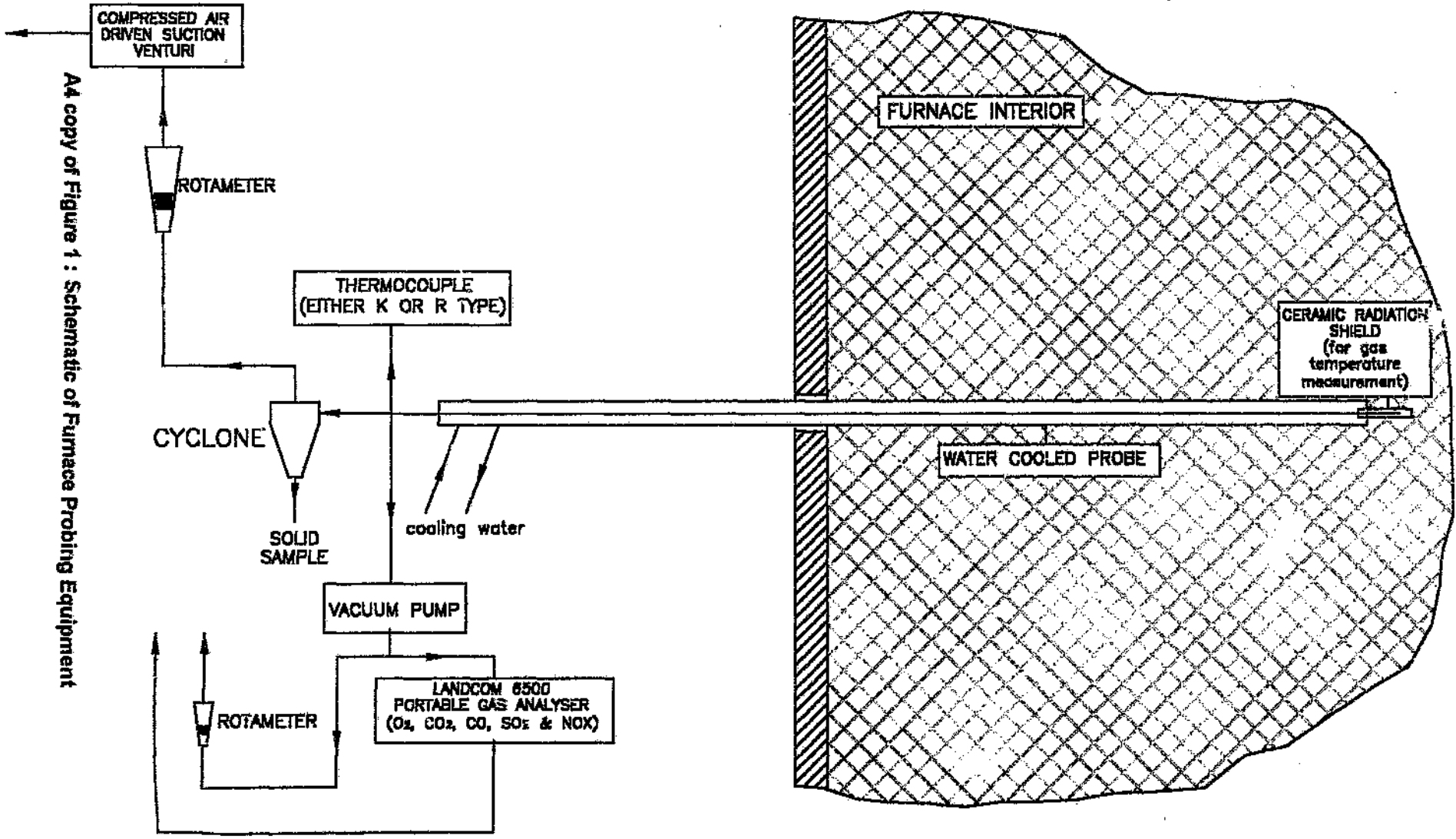
Appendix 2.2 Hendrina Washing Plant Feed

Appendix 2.2.1 Basic Analyses Datasheet
Appendix 2.2.2 Determination of Char Combustion Kinetics

Appendix 2.3 Pulverised Coal Particle Size Analyses

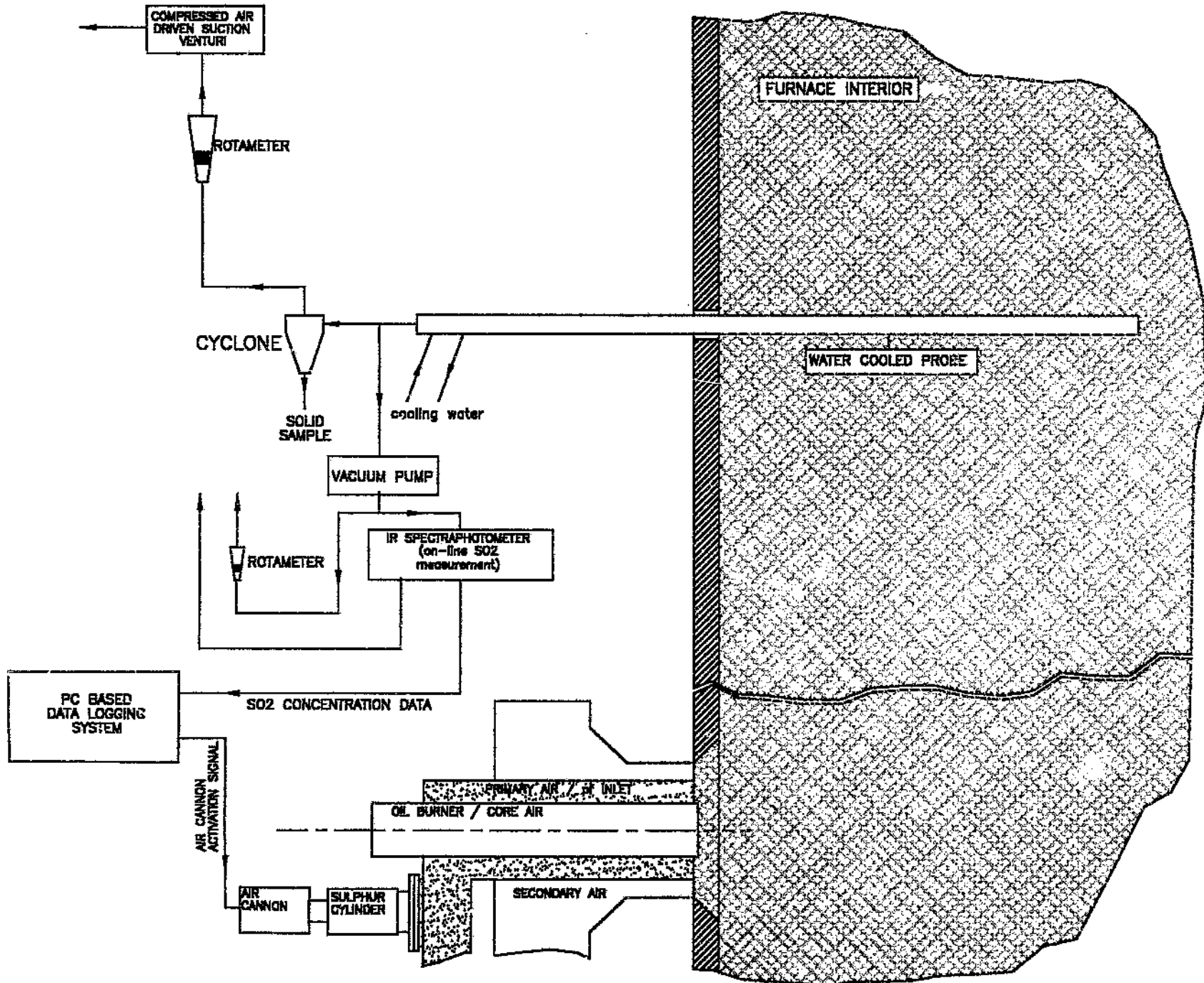
APPENDIX 3 AIOLOS PROGRAM - INPUT CODES

Appendix 3.1 Furnace Discretisation - hen9B.DAT
Appendix 3.2 Boundary Conditions Specifications - margb1.f
Appendix 3.3 Combustion Behaviour Specifications - combdat.f
Appendix 3.4 NOx Model Specifications - nodat.f



A4 copy of Figure 1 : Schematic of Furnace Probing Equipment

A4 copy of Figure 4 : Schematic of RTD Measurement Equipment

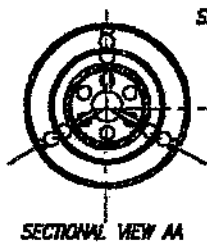


APPENDIX I

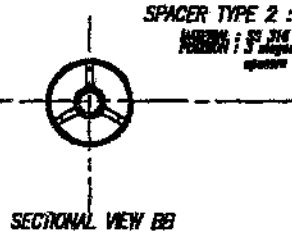
Appendix 1.1.....	Testing Equipment
Appendix 1.1.1.....	6m Probe design drawing
Appendix 1.1.2.1.....	12m Probe design drawing
Appendix 1.1.2.2.....	12m Probe design details
Appendix 1.1.3.....	Burner Door design drawing
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Appendix 1.1.6.....	Heat Flux Probe design drawing
Appendix 1.2.....	Boiler Testing Procedure
Appendix 1.3.....	Residence Time Distribution determination

Appendix 1.1.1

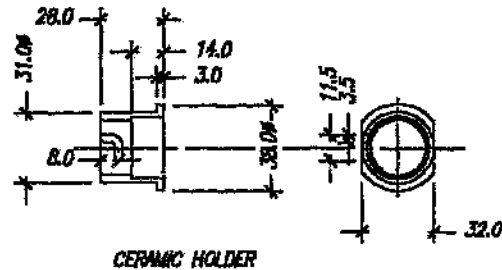
Design Sketch of the 6m water cooled suction pyrometer



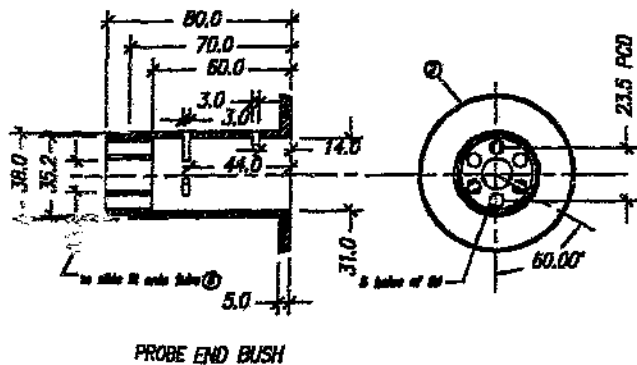
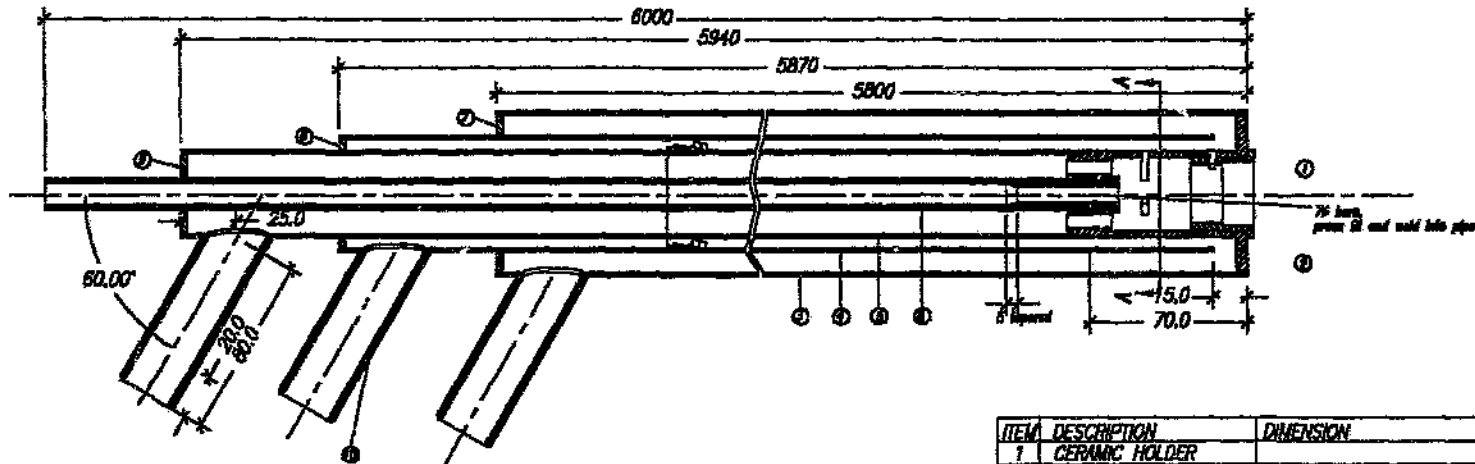
SPACER TYPE 1 :
 MATERIAL : SS 316 6.354 x 30 long
 FINISH : 3 slotted radially arranged
 spacing every 2m



SPACER TYPE 2 :
 MATERIAL : SS 316 6.354 x 30 long
 FINISH : 3 slotted radially arranged
 spacing every 2m



CERAMIC HOLDER



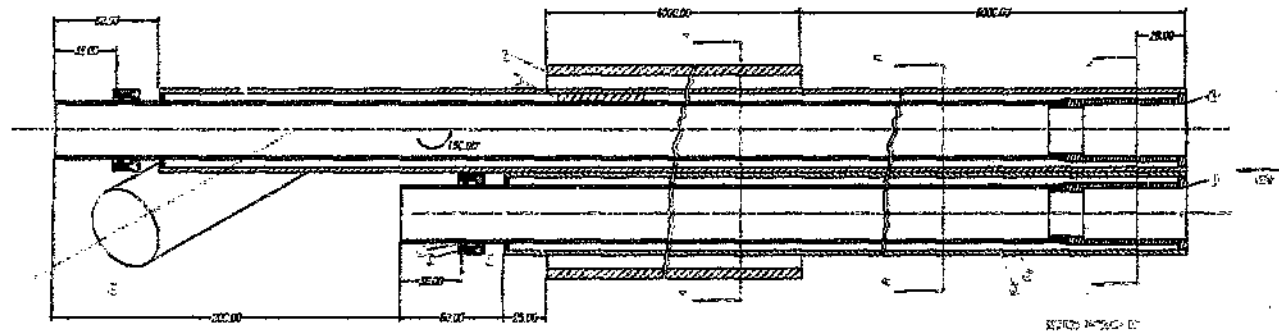
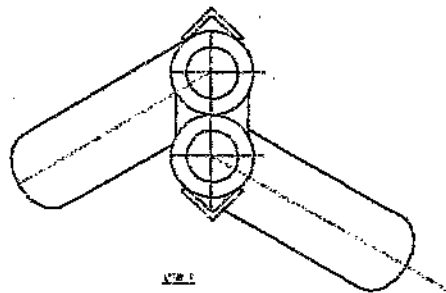
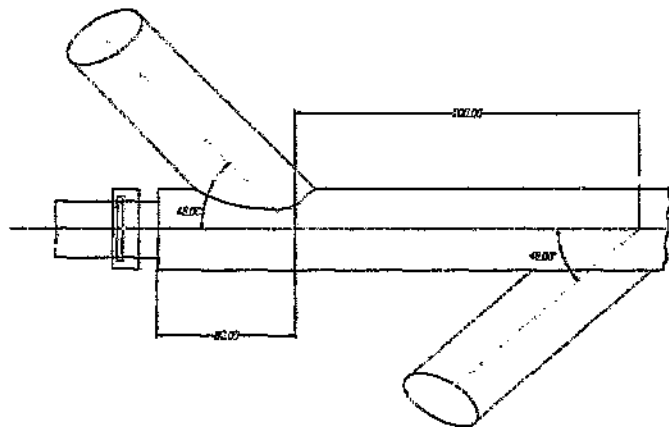
PROBE END BUSH

ITEM	DESCRIPTION	DIMENSION	MATERIAL
1	CERAMIC HOLDER		SS
2	PROBE END BUSH		SS
3	TUBE	73 OD x 69.2 ID	316 SS
4	TUBE	51 OD x 47.8 ID	316 SS
5	TUBE	38.3 OD x 35.2 ID	316 SS
6	TUBE	13.7 OD x 9.4 ID	304 SS
7	ANNULUS RING	to suit tubes	SS
8	ANNULUS RING	to suit tubes	SS
9	ANNULUS RING	to suit tubes	SS
10	GAS & WATER CONDUIT	standard with 3/4" thread	MS (galvanized)
11	SPACER TYPE 1	as specified	316 SS
12	SPACER TYPE 2	as specified	316 SS

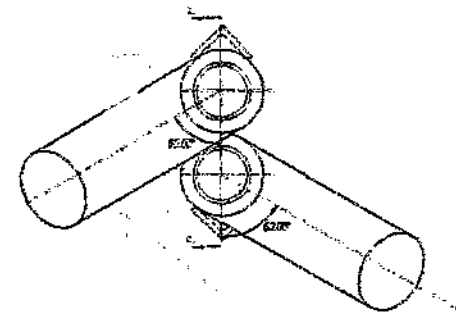
date	24/06/84	title :	BOKLER GAS SAMPLING AND TEMPERATURE PROBE - 6m	ESKOM TRI ENERGY TECHNOLOGIES tel:(011)629 7430 fac:(011)629 7226
drawn	NHE			
approved	copy			
revisions	n/a			
scale	to fit A4	drawing no :	copy of OTS 1/78	

Appendix 1.1.2.1

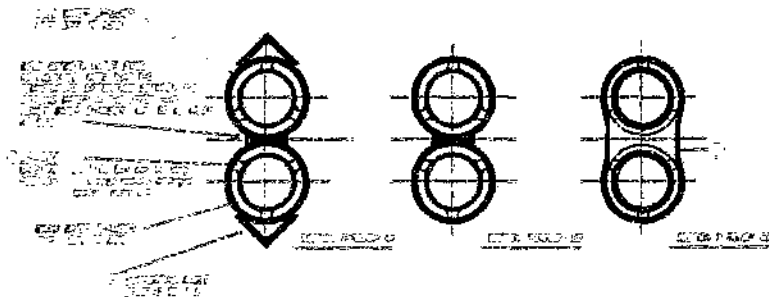
Design Sketch of 12m water cooled suction pyrometer



ITEM	DESCRIPTION	DIMENSION	MATERIAL	No. REQD
1	probe end bush	as per 12MP01	SS 316 L	2
2	seamless or welded pipe	334 OD, 1.65 WT x 6m	SS 316 L	4
3	seamless or welded pipe	48.26 OD, 2.77 WT x 6m	SS 316 L	4
4	angle	50 x 25 x 9 x 4m	SS 316 L	2
5	end cap fastener	as per 12MP05	SS 316 L	2
6	annulus ring	42.76 OD, 33.4 ID x 5	SS 316 L	2
7	annulus ring	37.06 OD, 33.4 ID x 3	SS 316 L	2
8	cooling water inlet/outlet	2" x 150 with 2" BSP male thread on one side	SS 316 L	2
9	spacer between inner and outer pipes	6 x 4 bar x 50	SS 316 L	30
10	cooling water conduit between lower and upper pipes	42.16 OD, 2.77 WT x 60	SS 316 L	1



TOLERANCES	as per DIN 7188 m			
0.5-6	>6-30	>30-120	>120	
±0.1	±0.2	±0.3	±0.5	



NOTE: ALL DIMENSIONS UNLESS OTHERWISE SPECIFIED

CAD FILENAME	12MPROBE.DWG	PROJECT:	7751E244R		
DRAWING No.	12MP03		BY CORRELATIONS IC		
SCALE	1:3 ON A3	TITLE:	WATER COOLED PROBE HOLDER		
CHECKED					
APPROVED					
This drawing is the property of ESKOM T-R-1. It is subject to copyright and may not be made available to third parties without prior written permission.					
ISSUE	DATE	ALTERATIONS	NAME		
3					
2					
1	10/08/95	ORIGINAL	NWE		



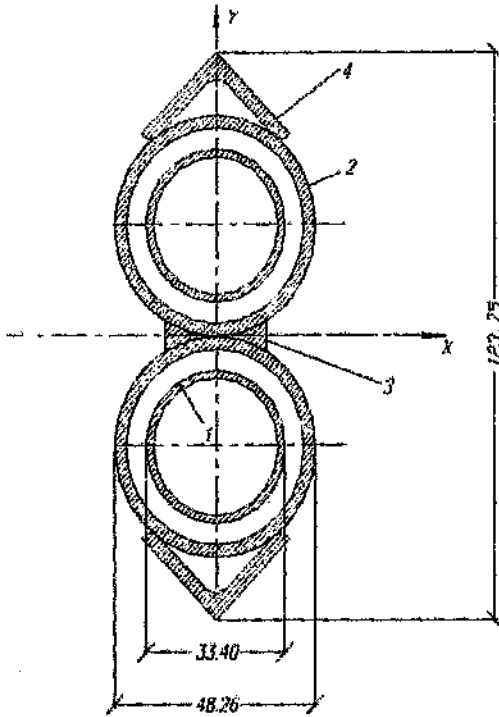
Appendix 1.1.2.2

Design Details for the 12m water cooled probe

Desig. of a water cooled suction probe :

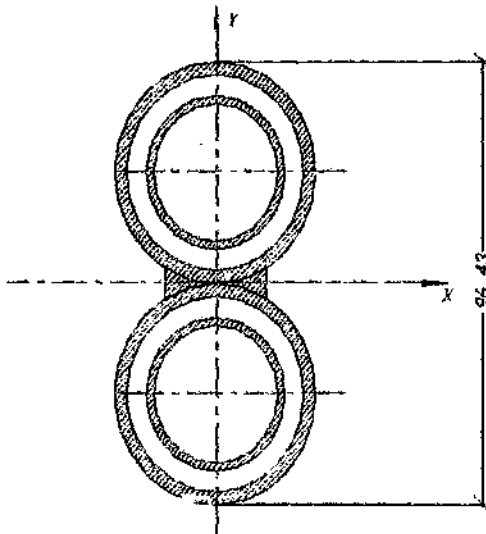
1/08/95
SvS & NWE

1. Mechanical Analysis of probe sections :



Area:	1457.433 sq mm
Perimeter:	1229.249 mm
Bounding box:	X: -24.13 -- 24.13 mm Y: 118.9821 -- 242.7354 mm
Centroid:	X: 0 mm Y: 180.8587 mm
Moments of inertia:	X: 4.933699e+007 sq mm sq mm Y: 277849.7 sq mm sq mm
Product of inertia: XY:	-5.295405e-009 sq mm sq mm
Radii of gyration:	X: 183.989 mm Y: 12.78747 mm
Principal moments (sq mm sq mm) and X-Y directions about centroid:	I: 277849.7 about [0 1] J: 1664532 about [1 0]

Item	Description
1	33.4 Ø x 1.6 WT 316L SS pipe
2	48.26 Ø x 2.77 WT 316L SS pipe
3	weld
4	25 x 25 x 3 WT 316 SS angle



Area:	1173.609 sq mm
Perimeter:	1035.687 mm
Bounding box:	X: -24.13 -- 24.13 mm Y: -19.54357 -- 76.9761 mm
Centroid:	X: 0 mm Y: 28.71621 mm
Moments of inertia:	X: 1867645 sq mm sq mm Y: 251816.3 sq mm sq mm
Product of inertia: XY:	-0.00004467853 sq mm sq mm
Radii of gyration:	X: 39.89196 mm Y: 14.64806 mm
Principal moments (sq mm sq mm) and X-Y directions about centroid:	I: 251816.3 about [6.894358e-011 -1] J: 899861.1 about [1 2.605621e-006]

Moment of Inertia for section of probe without support bar about the X axis : $I_{us,x} = 899861.1 \cdot \text{mm}^4$

Moment of Inertia for section of probe with support bar about the X axis : $I_{s,x} = 1664532 \cdot \text{mm}^4$

Moment of Inertia of probe on its side (weakest) : $I_{us,y} = 251816.3 \cdot \text{mm}^4$

Cross sectional Area of stainless steel without support bar : $A_{us} = 1173.609 \cdot \text{mm}^2$

Cross sectional Area of stainless steel with support bar : $A_s = 1457.433 \cdot \text{mm}^2$

Cross sectional area where water flows :

OD of outer tube :

$$d_0 = 48.26 \text{ mm}$$

Wall thickness of outer tube

$$wt_0 = 2.77 \text{ mm}$$

OD of inner tube :

$$d_1 = 33.4 \text{ mm}$$

$$A_w = \left[\pi \left(\frac{d_0}{2} - wt_0 \right)^2 - \pi \left(\frac{d_1}{2} \right)^2 \right] + \left[\pi \left(\frac{d_0}{2} - wt_0 \right)^2 - \pi \left(\frac{d_1}{2} \right)^2 \right] \quad A_w = 1114.38 \cdot \text{mm}^2$$

Material Properties of AISA 316L SS :

density :

$$\rho_{ss} = 8000 \frac{\text{kg}}{\text{m}^3}$$

Elastic modulus :

$$E_{ss} = 19 \cdot 10^9 \text{ Pa}$$

Mass Property of water :

density :

$$\rho_w = 1000 \frac{\text{kg}}{\text{m}^3}$$

Determining the maximum length of unsupported probe :

Divided gravity force :

$$q = (A_{us} \cdot \rho_{ss} + A_w \cdot \rho_w) \cdot g$$

Bending Moment :

$$BM(L_{\text{tube}}) = \int_{0 \cdot \text{m}}^{L_{\text{tube}}} q \cdot x \, dx$$

Calculation of the maximal bending stress occurring at a distance d_0 of the neutral axis at the insertion point

$$\sigma_{\max}(L_{\text{tube}}) = \frac{BM(L_{\text{tube}}) \cdot d_0}{I_{us,x}}$$

Calculation of the maximum deflection occurring at the end of the probe

$$\delta_{\max}(L_{\text{tube}}) = \frac{\frac{1}{8} \cdot q \cdot (L_{\text{tube}})^4}{E_{ss} \cdot I_{us,x}}$$

Safety factor n , most likely to be in the order of 2-2.5

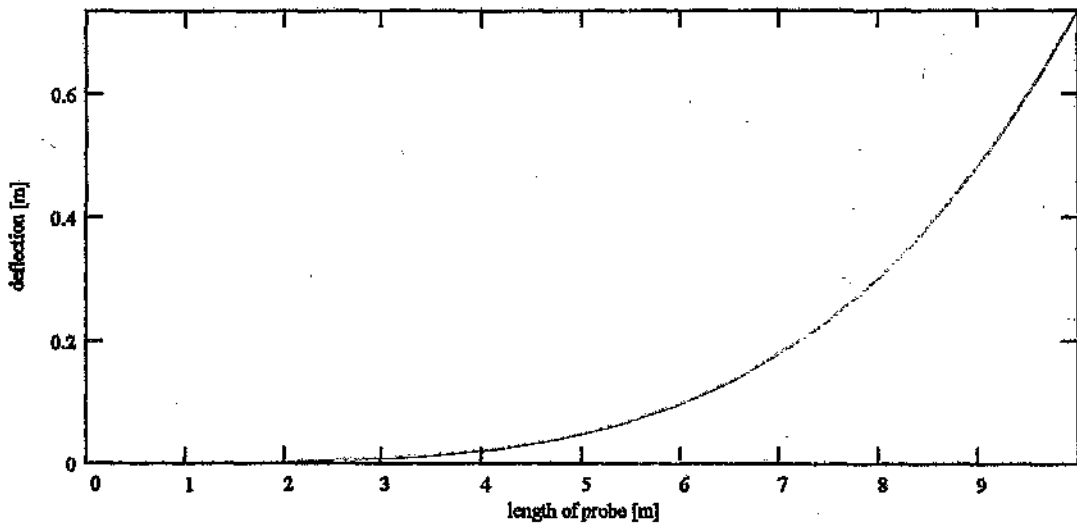
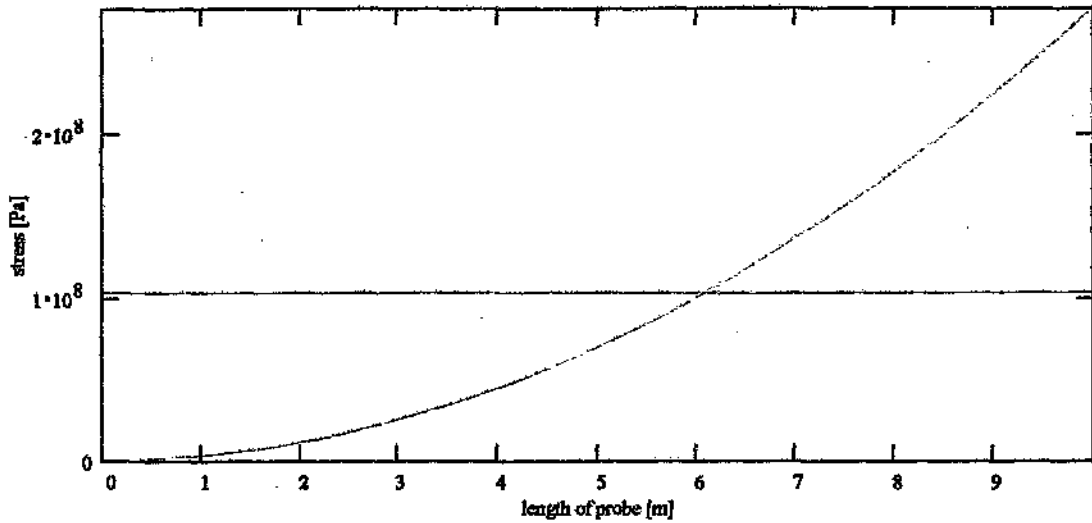
$$n = 2$$

Maximum permissible tensile stress of stainless steel

$$\sigma_{\text{maxpermissible}} = \frac{205}{n} \cdot 10^6 \text{ Pa}$$

$$\sigma_{\text{maxpermissible}} = 102500000 \text{ Pa}$$

$$L_{\text{in}} = 0 \cdot \text{m}, 0.2 \cdot \text{m}, 10 \cdot \text{m}$$



Determining the maximum length of unsupported probe :

$$l_{\max} = 8 \cdot \text{m} \quad \text{given} \quad \sigma_{\max}(l_{\max}) = \sigma_{\text{maxpermissible}}$$

$$l_{\max,1} = \text{find}(l_{\max}) \quad l_{\max,1} = 6.09 \cdot \text{m}$$

Calculating the mass of probe and water of the unsupported probe :

$$\text{Mass}(L_{\text{probe}}) = (A_{\text{us}} \cdot \rho_{\text{ss}} + A_{\text{w}} \cdot \rho_{\text{w}}) \cdot L_{\text{probe}}$$

$$\text{Mass}(l_{\max,1}) = 63.98 \cdot \text{kg} \quad \text{effecting a point load at the end of the supported probe}$$

Concentrated load at end of supported probe :

$$\text{Load}_{\text{us}} = \text{Mass}(l_{\max,1}) \cdot g$$

divided gravity force :

$$q_s = (A_s \cdot \rho_{st} + A_w \cdot \rho_w) \cdot g$$

Bending Moment :

$$BM_s(L_{tube}) = \int_{0-m}^{L_{tube}} q \cdot x \, dx + Load_{us} \cdot L_{tube}$$

Finding the maximum length of probe for which

Calculation of the maximal bending stress occurring at a distance d_0 of the neutral axis at the insertion point

$$\sigma_{max}(L_{tube}) = \frac{BM_s(L_{tube}) \cdot d_0}{I_{s,x}}$$

Calculation of the maximum deflection occurring at the end of the probe

$$\delta_{max}(L_{tube}) = \frac{\frac{1}{8} \cdot q_s \cdot (L_{tube})^4 + \frac{1}{2} \cdot Load_{us} \cdot L_{tube}^3}{E_{ss} \cdot I_{s,x}}$$

Safety factor n , most likely to be in the order of 2-2.5

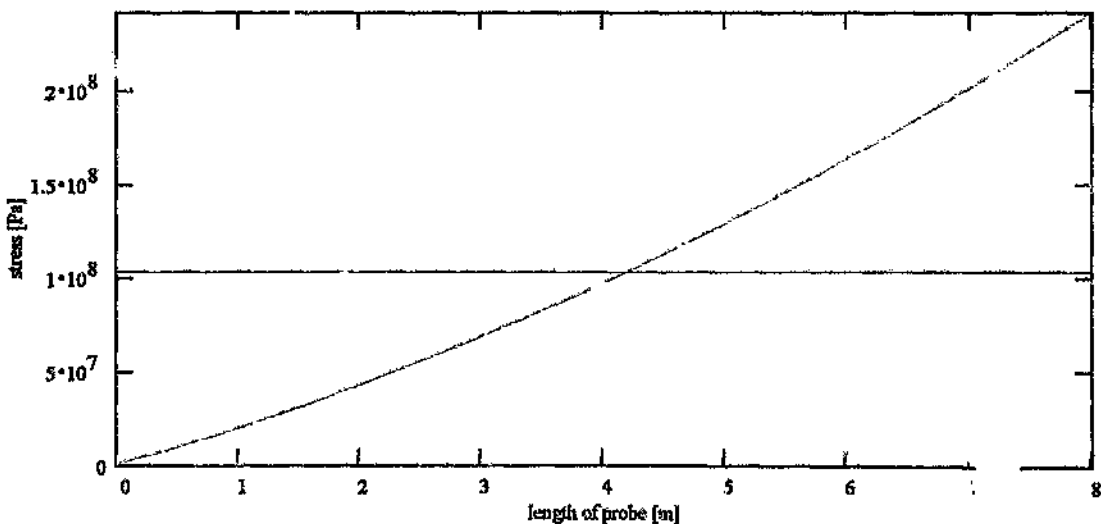
$$n = 2$$

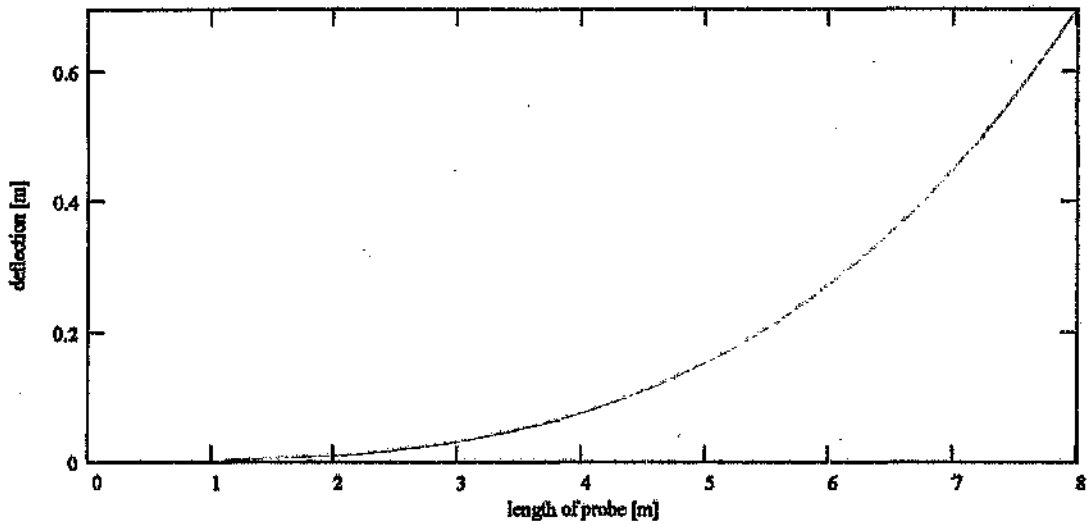
Maximum permissible tensile stress of stainless steel

$$\sigma_{maxpermissible} = \frac{205}{n} \cdot 10^6 \cdot Pa$$

$$\sigma_{maxpermissible} = 102500000 \cdot Pa$$

$$L_{in} = 0 \cdot m, 0.2 \cdot m, \dots, 8 \cdot m$$





Determining the maximum length of unsupported probe :

$$l_{\max} = 3 \cdot m \quad \text{given} \quad \sigma_{\max}(l_{\max}) = \sigma_{\max \text{ permissible}}$$

$$l_{\max.2} = \text{find}(l_{\max}) \quad l_{\max.2} = 4.19 \cdot m$$

Maximum Probe Length past cantilever point :

$$l_{\max} = l_{\max.1} + l_{\max.2}$$

$$l_{\max} = 10.28 \cdot m$$

Total mass of probe without water :

$$\text{mass}_{\text{total}} = l_{\max.1} \cdot A_{us} \cdot \rho_{ss} + (l_{\max} - l_{\max.1}) \cdot A_s \cdot \rho_{ss}$$

$$\text{mass}_{\text{total}} = 106.07 \cdot \text{kg}$$

Transporting the probe on its side :

$$\text{Load} = \text{mass}_{\text{total}} \cdot g$$

Estimated Bending Moment at centre

$$BM_c = \frac{\text{Load} \cdot l_{\max}}{8}$$

Estimating the maximum bending stress on the probe on its side :

$$\frac{BM_c \cdot \frac{d_0}{2}}{I_{us,y}} = 128.13 \cdot 10^6 \cdot \text{Pa}$$

Calculated safety factor

$$\frac{n \cdot \sigma_{\max \text{ permissible}}}{\frac{BM_c \cdot \frac{d_0}{2}}{I_{us,y}}} = 1.6$$

Heat Transfer Calculation :

Insertion Depth

$i = 0.. 100$

$$L_{\text{tube}_1} = \frac{i+1}{10} \text{ m}$$

maximum expected heat flux in furnace :

$$q_{\text{max}} = 500 \frac{\text{kW}}{\text{m}^2}$$

total area subjected to this heat flux :

$$A_{q_1} = \pi \cdot d_0 \cdot 2 \cdot L_{\text{tube}_1}$$

maximum rate of energy removal from probe holder :

$$Q_{\text{max}_1} = q_{\text{max}} \cdot A_{q_1}$$

maximum increase in cooling water temperature :

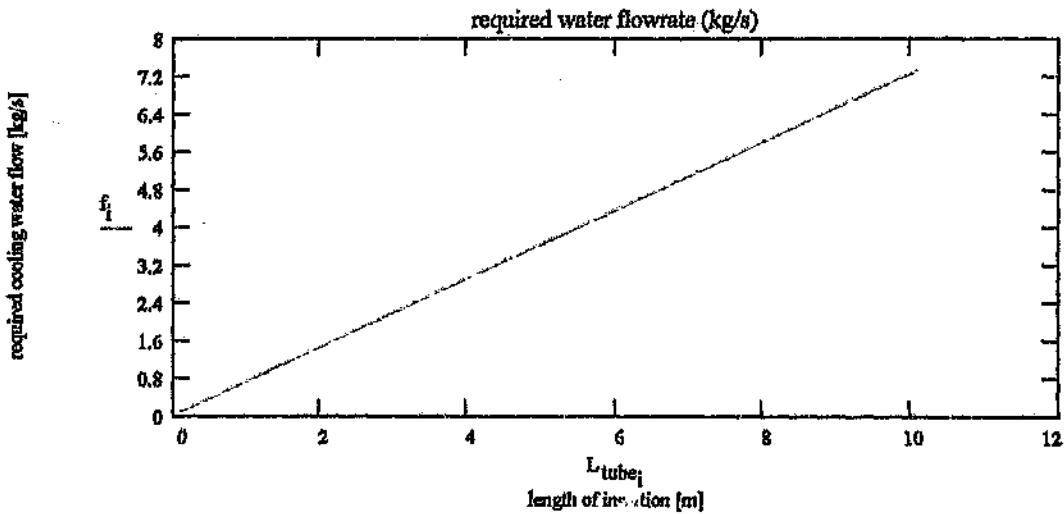
$$\Delta T_{\text{cw}} = (50) \cdot \text{K}$$

average specific heat of water :

$$C_{p \text{ cw}} = 4.186 \cdot 10^3 \frac{\text{joule}}{\text{K} \cdot \text{kg}}$$

energy balance around probe, where f = flowrate of water in kg/s :

$$f_i = \frac{Q_{\text{max}_1}}{\Delta T_{\text{cw}} \cdot C_{p \text{ cw}}}$$



Estimation of pressure drop through probe :

hydraulic mean diameter of annulus :

$$d_m = \frac{4 \cdot \left[\pi \left[\left(\frac{d_0 - wt_0 \cdot 2}{2} \right)^2 - \left(\frac{d_1}{2} \right)^2 \right] \right]}{\pi (d_0 - wt_0 \cdot 2) + \pi d_1}$$

Friction loss due to 180° bend at probe end :

$$\text{bends} = 120 \quad \text{diameters}$$

Average Viscosity of water between 280 and 360°K :

$$\mu_{cw} = 0.65 \cdot 10^{-3} \cdot \text{poise}$$

Area available for Flow :

$$A_{cw} = \frac{A_w}{2}$$

Velocity of water through pipe :

$$v_{cw1} = \frac{f_i}{A_{cw} \rho_w}$$

Reynolds Number :

$$Re_1 = \frac{d_m \cdot v_{cw1} \cdot \rho_w}{\mu_{cw}}$$

Friction factor (according to Blasius for smooth pipe):

$$f_{01} = 0.0396 \cdot (Re_1)^{-0.25}$$

Equivalent length of pipe through probe :

$$L_{eq} = 2 \cdot 12 \cdot m + \text{bends} \cdot d_m$$

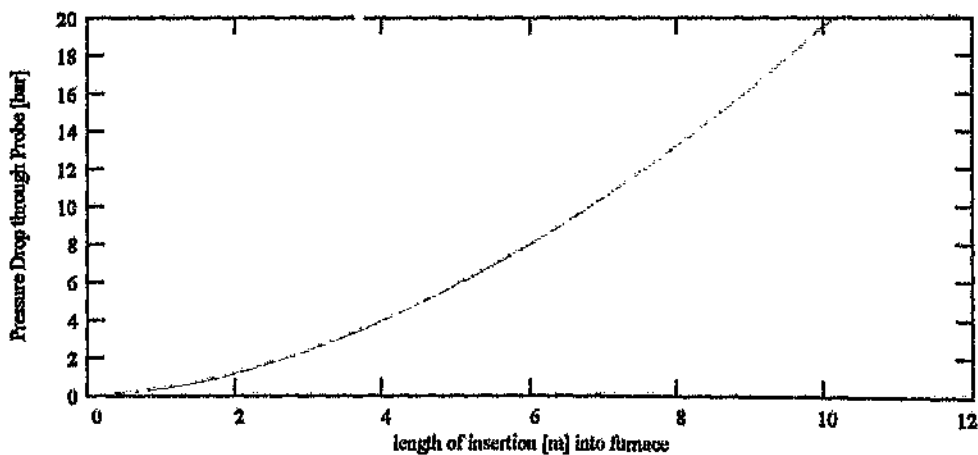
$$L_{eq} = 25.12 \cdot m$$

The energy loss due to friction in the pipe is thus :

$$F_i = f_{01} \cdot 4 \cdot \frac{L_{eq}}{d_m} \cdot (v_{cw1})^2$$

This translates into a pressure drop of :

$$\Delta P_i = F_i \cdot \rho_w \quad \text{Pa}$$



Estimating the pumping requirements :

Maximum continuous water pressure of supply at required flowrate :

$$P_{in} = 10 \cdot 101325 \cdot Pa$$

Minimum Pressure at outlet :

$$P_{out} = 2 \cdot 101325 \cdot Pa$$

Efficiency of pump :

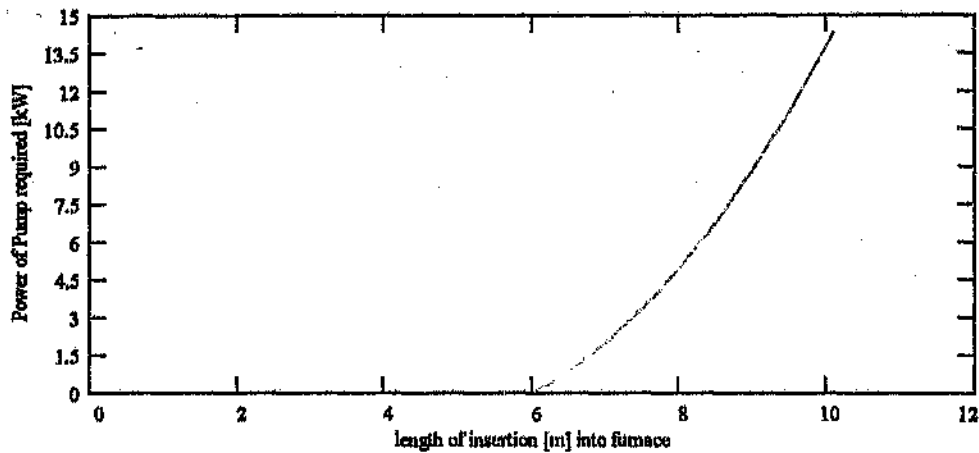
$$\text{efficiency} = 60\%$$

Power of pump :

$$\text{Power}_i = \frac{f_i \cdot \frac{\Delta P_i - P_{in} + P_{out}}{\rho_w}}{\text{efficiency}}$$

negative power requirements obviate the need for a pump :

$$\text{Power}_i := \text{if}(\text{Power}_i < 0 \cdot kW, 0 \cdot kW, \text{Power}_i)$$



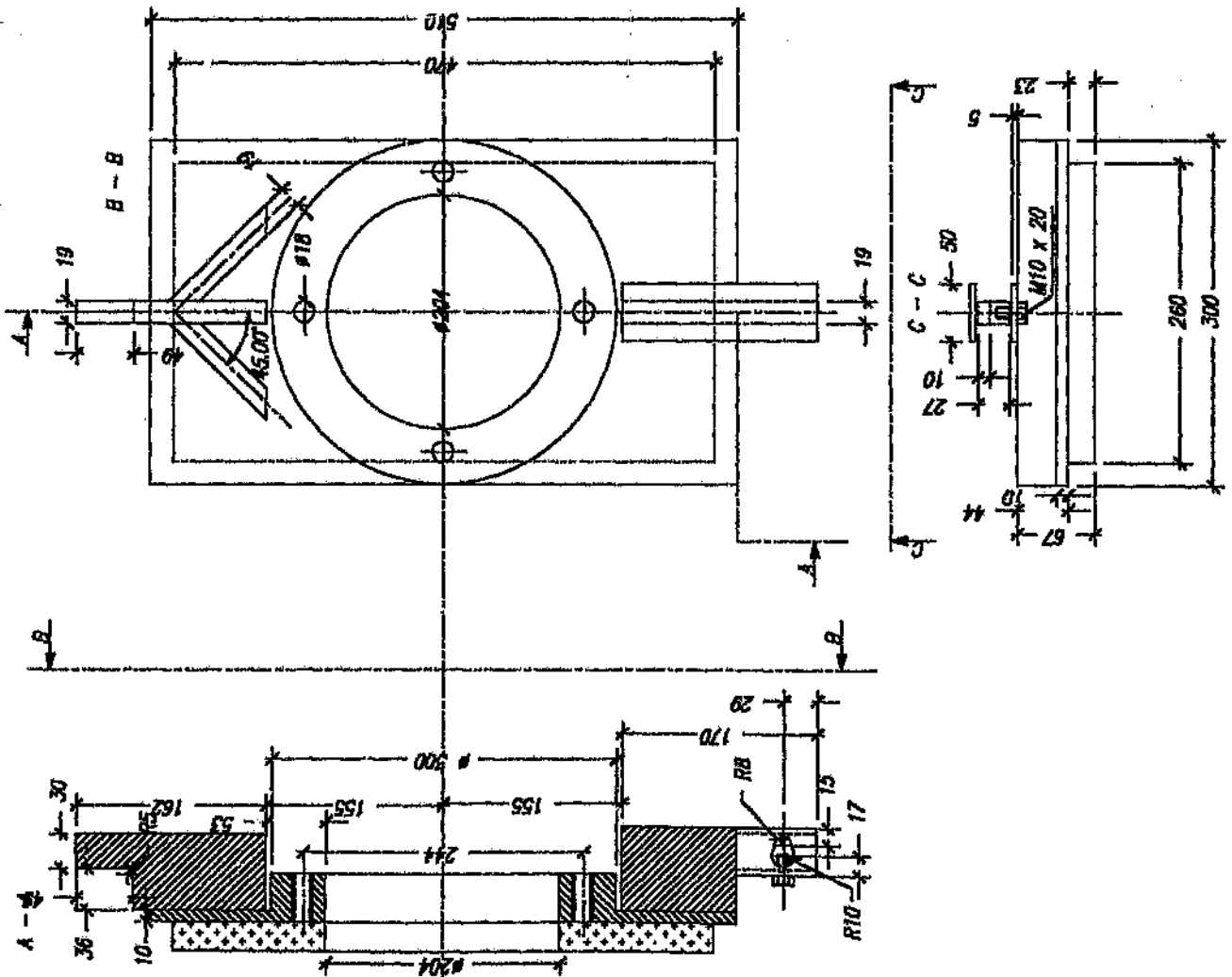
Appendix 1.1.3

Design Sketch for the Hendrina Burner Door



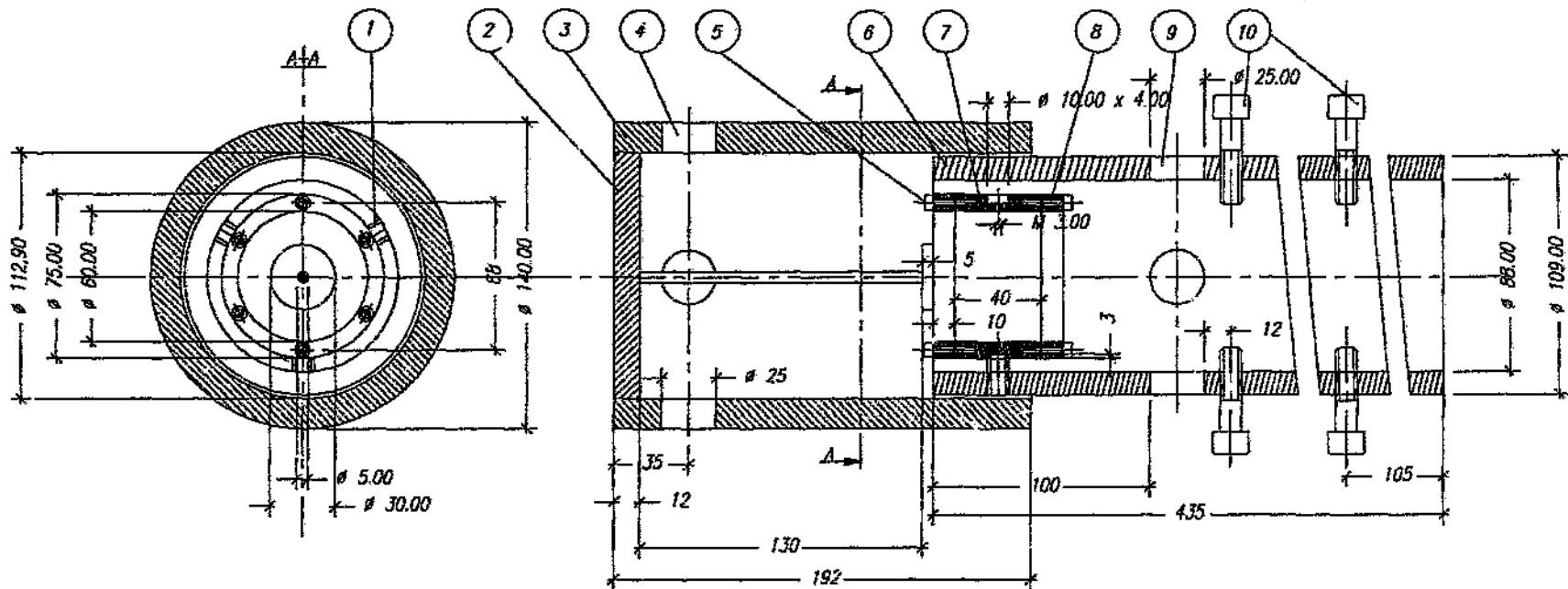


CAD FILENAME	REVISION	PROJECT: 7751 E-24-4R
DRAWING No.		PLANT SIMULATION
SCALE	related to R-A	TITLE: HEMIPHA BARRIER DOOR FOR SULPHUR INJECTION DEVICE
CHECKED		
APPROVED		
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		ISSUE DATE
		ALTERATIONS
		NO.
		DATE
		BY
		CHKD
		DATE
		BY
		CHKD
		DATE



Appendix 1.1.4

**Design Sketch for the tracer injection device
comprising the air cannon and sulphur discharge canister assembly**



ITEM NO.	NO. REQD.	DESCRIPTION	DIMENSION	MATERIAL
1	3 off	GRUB SCREW WITH ALAN CAP	M10 X 20	
2	1 off	CYLINDER COVER		PERSPEX
3	4 off	AIR INLET HOLE		
4	1 off	OUTER CYLINDER	140 od, 12,3	HDPE
5	12 off	ALAN CAP BOLT	M4 X 20	
6	1 off	PROBE HOLDER CYLINDER	100 od, 8,1	HDPE
7	1 off	SO2 CONTAINER CYLINDER	25.00	HDPE
8	2 off	SO2 CONTAINER FLANGE	25.00	HDPE
9	4 off	AIR INLET HOLE		
10	8 off	ALAN CAP BOLT	M10 X 60	

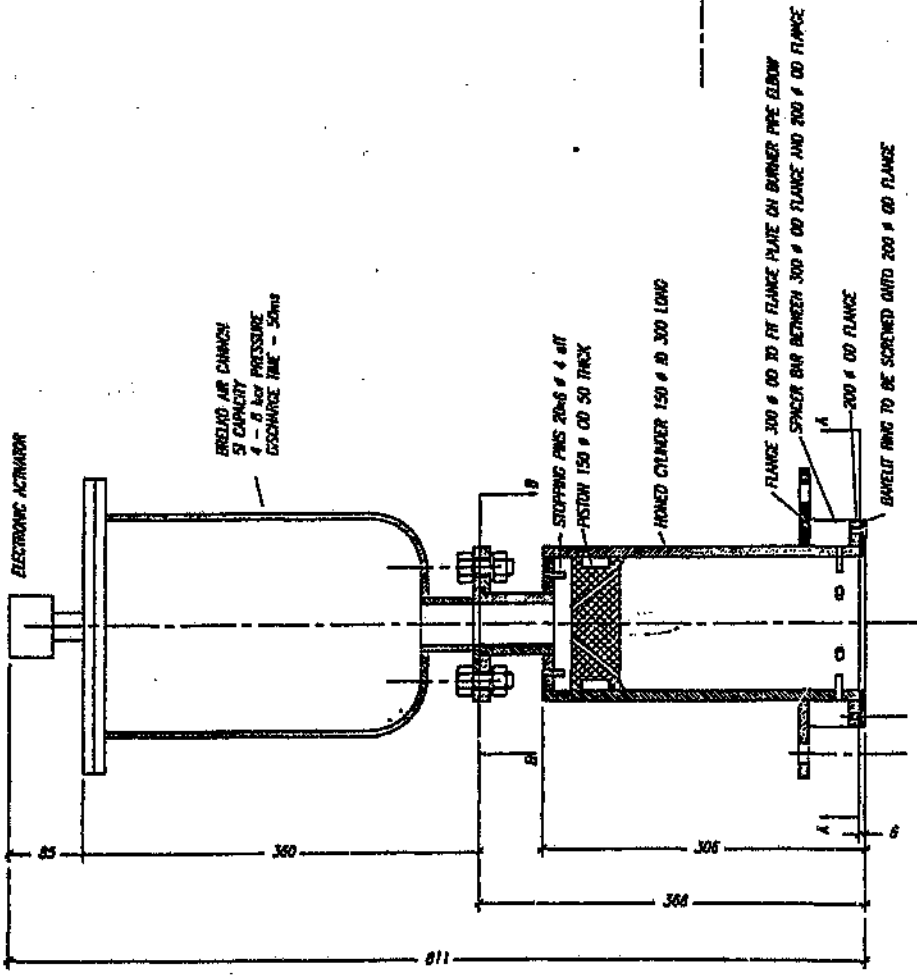
CAD FILENAME	SO2INJEC.DWG	PROJECT: 7751E244,7																
DRAWING No.	01	DIF Correlations																
SCALE	1:3 on A4	TITLE: SO2 Injection probe for RTD measurement in water cooled suction pyrometers																
CHECKED																		
APPROVED																		
This drawing is the property of ESKOM T-R-1. It is subject to copyright and may not be made available to third parties without prior written permission.		<table border="1"> <tr> <td>3</td> <td></td> <td></td> <td></td> </tr> <tr> <td>2</td> <td></td> <td></td> <td></td> </tr> <tr> <td>1</td> <td>22/04/95</td> <td>ORIGINAL</td> <td>SvS/nwe</td> </tr> <tr> <td>ISSUE</td> <td>DATE</td> <td>ALTERATIONS</td> <td>NAME</td> </tr> </table>	3				2				1	22/04/95	ORIGINAL	SvS/nwe	ISSUE	DATE	ALTERATIONS	NAME
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2																		
1	22/04/95	ORIGINAL	SvS/nwe															
ISSUE	DATE	ALTERATIONS	NAME															



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Appendix 1.1.5

SO₂ injection device for the determination of the probe RTD

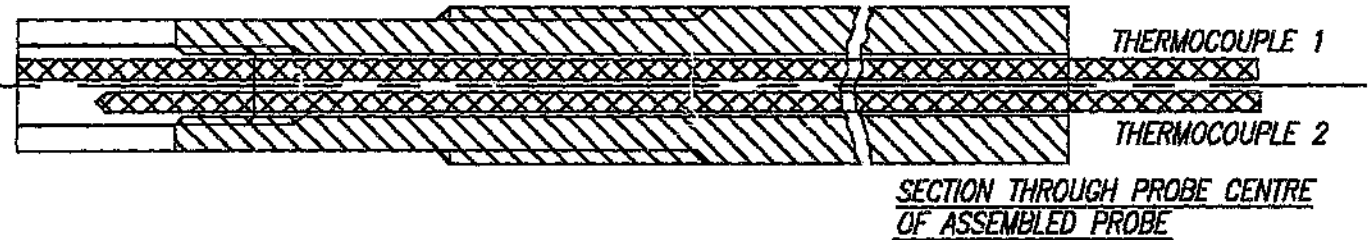
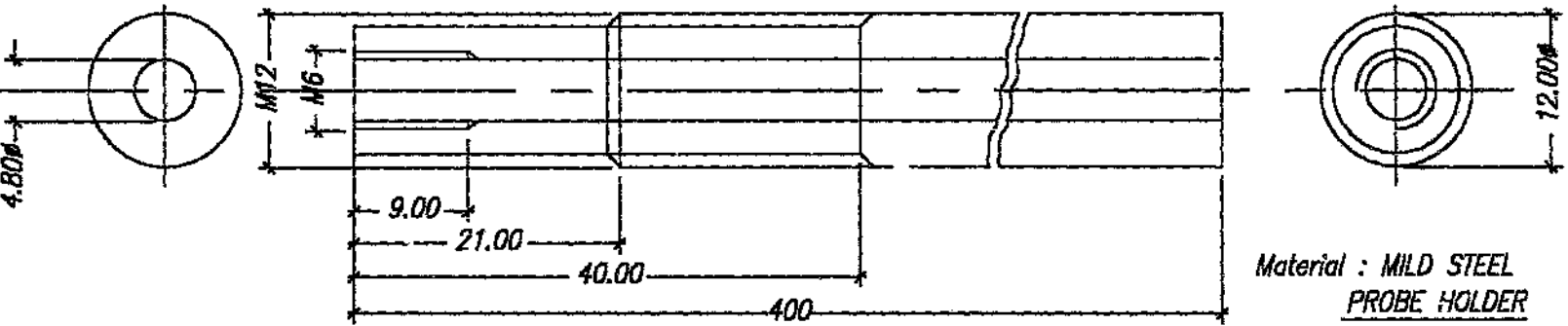
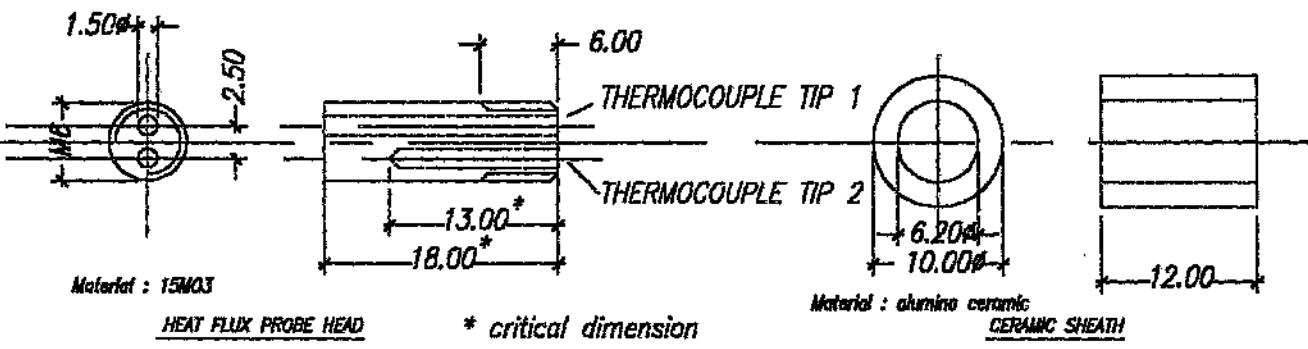


date	24/06/94	title	REVERSED TRUCKER METERING DEVICE AS USED FOR MINIMUM DRY CORRELATIONS MEASUREMENT
drawn	AMC		
approved			
revisions		to fit A4	SCALE: NOT TO SCALE - REFERENCE SKETCH
scale			

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fax:(011)629 7229

Appendix 1.1.6

Design Sketch for the heat flux probe



MATERIAL	as indicated				
No. REQD	refer enquiry				
TOLERANCES	as per DIN 7168 m				
	0.5-6	>6-30	>30-120	>120	
	±0.1	±0.2	±0.3	±0.5	

date	30/08/95
drawn	NWE
approved	copy
revisions	02
scale	2:1

title : FURNACE HEAT FLUX PROBE

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ENERGY TECHNOLOGIES
tel:(011)629 5438
fax:(011)629 5229

Appendix 1.2

Furnace Testing Procedures

1. Equipment Requirement

1. water cooled suction pyrometers
2. pyrometer trestles
3. cooling water hoses with fire hydrant fittings (source → pyrometer → drain)
4. suction line hoses (3.5 " Teflon lined)
5. cyclone filter for solids sampling (fits onto rear part of pyrometer)
6. air ejector (compressed air driven suction venturi)
7. compressed air hoses and fittings
8. silicon hose and fittings for gas sampling
9. petrol filter for gas sample line
10. membrane dryer for gas sample
11. suction pump (to draw gas sample through portable gas analyser)
12. rotameter to monitor gas flowrate through gas analyser
13. thermocouples with suitable compensating leads and displays (or datalogger)
14. ceramic shields for thermocouples (if using direct temperature measurement technique)

2. Equipment Setup

2.1 Cooling Water System

The cooling water inlet should be connected to the outer cooling jacket of the suction pyrometer and should be adjusted so that the cooling water outlet temperature is at around 60°C. Condensation in the sampled gas stream should be avoided.

2.2 Temperature Measurement - direct method using R-type thermocouple

Insert probe (with the ceramic radiation shields and thermocouple in place - refer Figure 1) into furnace with the suction turned to minimum. Once probe is at the desired furnace depth, adjust the cooling water flowrate so that the outlet water temperature is as specified in 2.1. Gradually increase the suction while observing the temperature reading. Once an increase in temperature reading is no longer apparent with an increase in suction, note the temperature reading. If the temperature reading begins to decrease, check the suction flowrate. The decrease in temperature may be attributable to the ceramic radiation shields having become blocked with molten ash deposits and as a result, the suction flowrate would be low. Remove probe and replace fouled components.

2.3 Gas Analysis and Solids Sampling

Insert probe (with cyclone and gas sample line in place - refer to Figure 1) into furnace to desired depth, adjust cooling water flowrate (this is critical in this instance) with the suction at minimum. Gradually increase suction until a steady trickle of combustion residue falls through the sampling cyclone into the glass collector bottle. The gas sample is drawn off the main sample stream before the cyclone using the sampling point on the probe. Silicon tubing takes the gas sample through a petrol filter (to remove any combustion residue solids) to a suction pump. The outlet of the suction pump passes through a T piece to split the flow into a gas sample for the portable gas analyser and bypass flow, which is monitored through a rotameter. Note that this flowrate may never be negative. Note further, that the gas sample to be analysed must first be passed through a dryer.

Appendix 1.3

Residence Time Distribution manipulations

Appendix 1.3.1 Top Right Burner to Furnace Exit - 2m into boiler

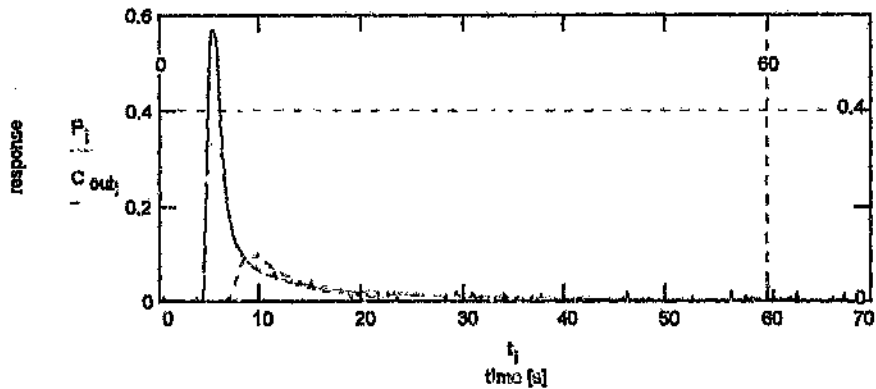
RTD Manipulations :

(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	24 m (mill B, right burner)	
Probe	2 m in the boiler	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD := READPRN(decob)	
time vector	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0 . n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) = \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\frac{\exp\left[-(t - \tau_d) \cdot \tau_1 \right] \dots}{+ \exp\left[-(t - \tau_d) \cdot \tau_2 \right]} \right] \cdot \Phi(t - \tau_d)$$

Probe Response : (visual best function fit to data)

$$\tau_{p1} := 0.77 \quad \tau_{p2} := 2.69 \quad \tau_{pd} := 4.64 \quad \text{parameters for RTD}$$

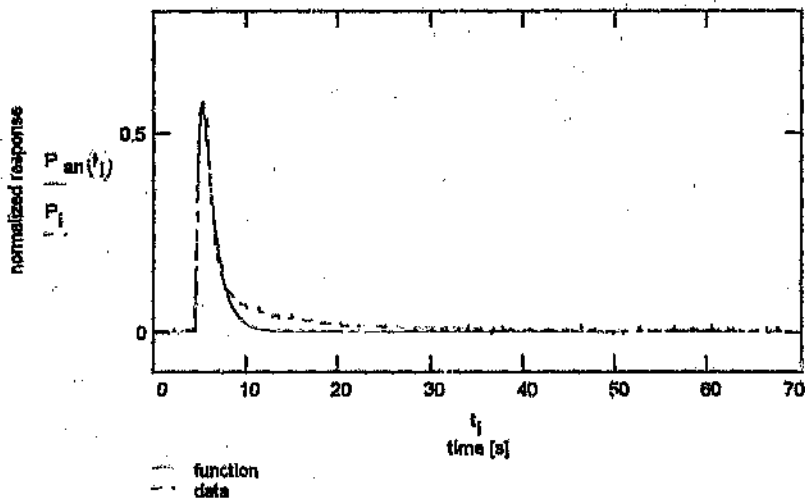
$$P_{an}(t) := C(t, \tau_{p1}, \tau_{p2}, \tau_{pd}, \text{const}_p)$$

$$\text{const}_p := 2.52$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2 \quad \text{define squared error}$$

$$E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$$

Graph comparing measured and curve fitted RTD

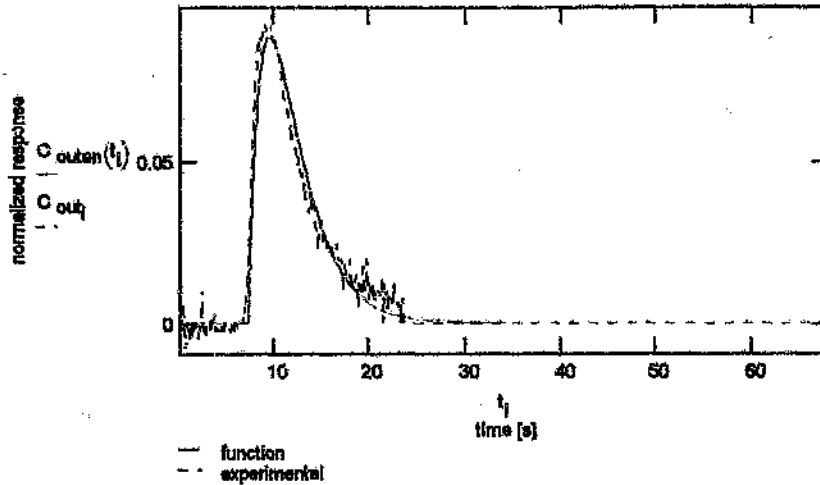


Overall Response : (best function fit to data)

$$\tau_{o1} := 0.36 \quad \tau_{o2} := 0.5 \quad \tau_{od} := 7.4 \quad \text{const}_o := 0.103 \quad \text{parameters for RTD}$$

$$C_{outan}(t) := C(t, \tau_{o1}, \tau_{o2}, \tau_{od}, \text{const}_o)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - C_{outan}(t_j))^2 \quad E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.007 \quad \text{define squared error}$$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :

$$P_{in}(t) := \text{const}_p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \left[\exp[(t - \tau_{pd}) \cdot \tau_{p1}] - \exp[(t - \tau_{pd}) \cdot \tau_{p2}] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response :

$$C_{out}(t) := \text{const}_o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \left[\exp[(t - \tau_{od}) \cdot \tau_{o1}] - \exp[(t - \tau_{od}) \cdot \tau_{o2}] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

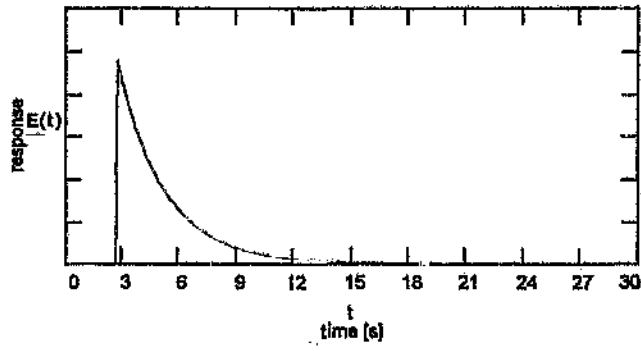
$$A := \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B := \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

$$A = 6.824 \quad B = -4.224 \quad \tau_{Ed} := \tau_{od} - \tau_{pd}$$

$$E(t) := \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp[(t - \tau_{Ed}) \cdot \tau_{o1}] - B \cdot \exp[(t - \tau_{Ed}) \cdot \tau_{o2}] \right]$$

BOILER RTD



Dead Time : $\tau_{Ed} = 2.76$ s

Appendix 1.3.2 Top Left Burner to Furnace Exit - 2m into boiler

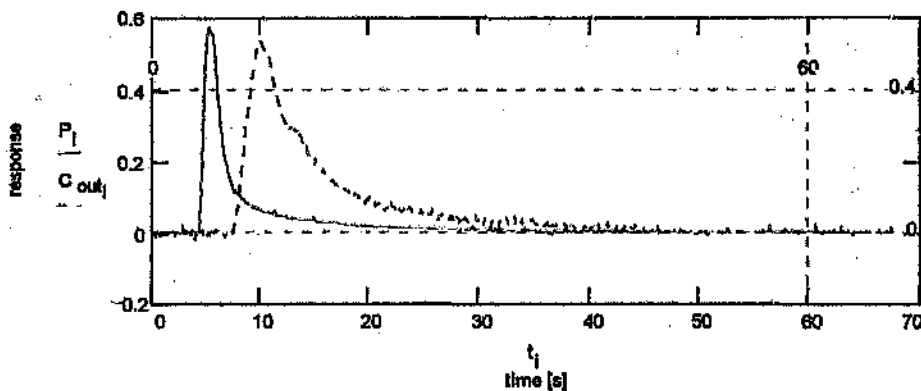
RTD Manipulations :

(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	24 m (mill B, left burner)	
Probe	2m in the boiler	
Boiler Level at Probe	31.64 m	
Read Data from File :	RTD := READPRN(decoa)	
time vector:	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0..n	

Plc. of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) = \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[-(t - \tau_d) \cdot \tau_1 \right] - \exp\left[-(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \theta(t - \tau_d)$$

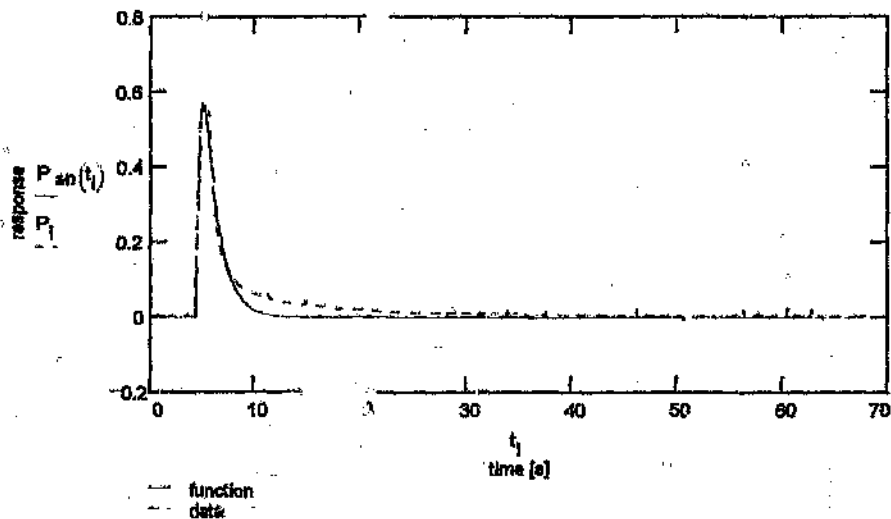
Probe Response : (best function fit to data)

$\tau_{p1} := 0.77$ $\tau_{p2} := 2.69$ $\tau_{pd} := 4.64$ $const_p := 2.52$ parameters for RTD

$P_{an}(t) := C(t, \tau_{p1}, \tau_{p2}, \tau_{pd}, const_p)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2$ define squared error $E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$

Graph comparing measured and curve fitted RTD



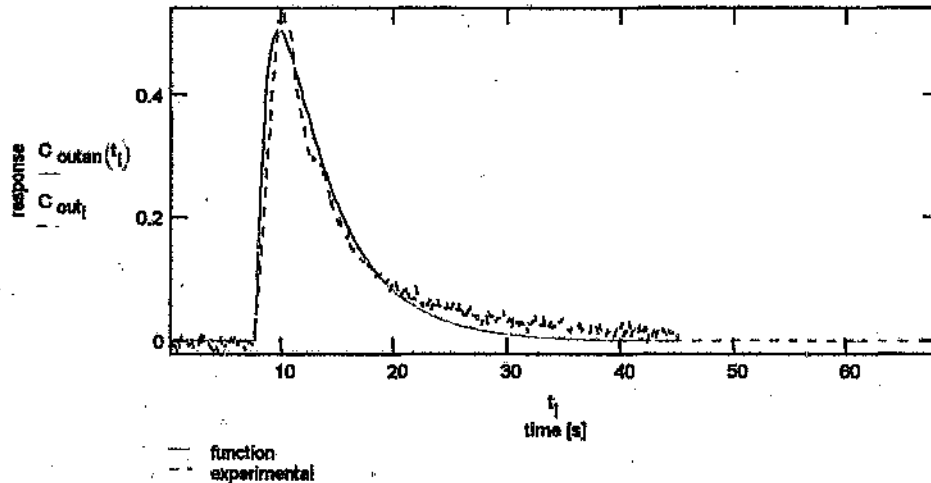
Overall Response : (best function fit to data)

$\tau_{o1} := 0.20$ $\tau_{o2} := 1.0$ $\tau_{od} := 7.8$ $const_o := 0.75$ parameters for RTD

$C_{outan}(t) := C(t, \tau_{o1}, \tau_{o2}, \tau_{od}, const_o)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{out_j} - C_{outan}(t_j))^2$ define squared error

$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.343$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) = \text{const}_p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \cdot \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response
$$C_{outan}(t) = \text{const}_o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \cdot \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

$$A = \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B = \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

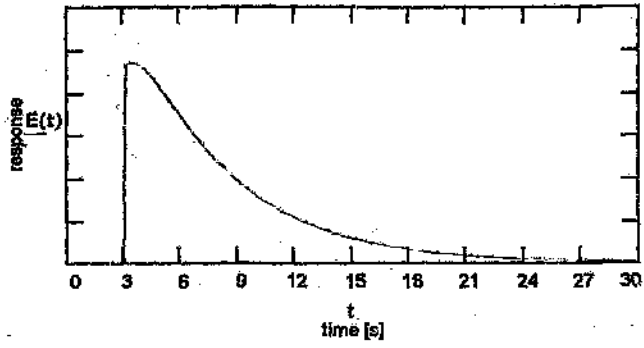
$A = 1.774$

$B = 0.486$

$\tau_{Ed} = \tau_{od} - \tau_{pd}$

$$E(t) = \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$

BOILER RTD



Dead Time : $t_{Ed} = 3.16$ s

Appendix 1.3.3 Top Right Burner to Furnace Exit - 3m into boiler

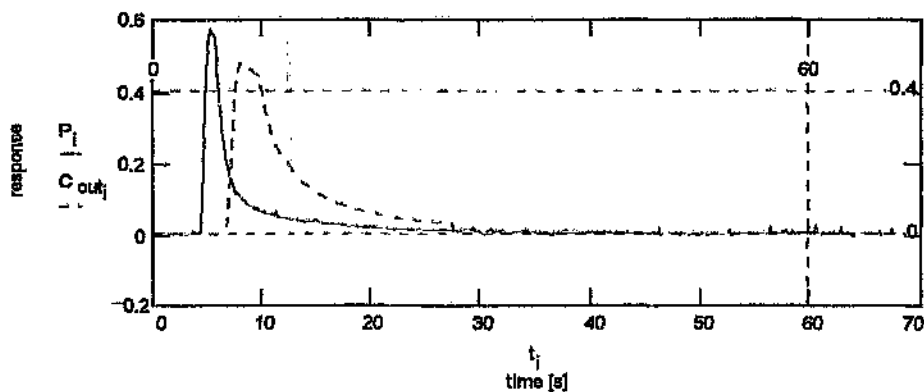
RTD Manipulations :

(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	24 m (mill B, right burner)	
Probe	3m in the furnace	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD := READPRN(decof)	
time vector	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0..n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, const) := const \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[(t - \tau_d) \cdot \tau_1 \right] - \exp\left[(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \Phi(t - \tau_d)$$

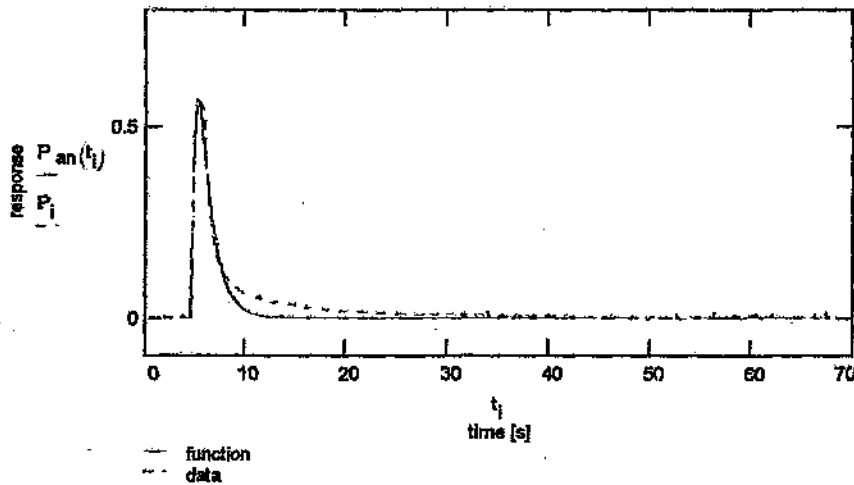
Probe Response : (visual best function fit to data)

$\tau_{p1} := 0.77$ $\tau_{p2} := 2.69$ $\tau_{pd} := 4.64$ $const_p := 2.52$ parameters for RTD

$P_{an}(t) := C(t, \tau_{p1}, \tau_{p2}, \tau_{pd}, const_p)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2$ define squared error $E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$

Graph comparing measured and curve fitted RTD



1

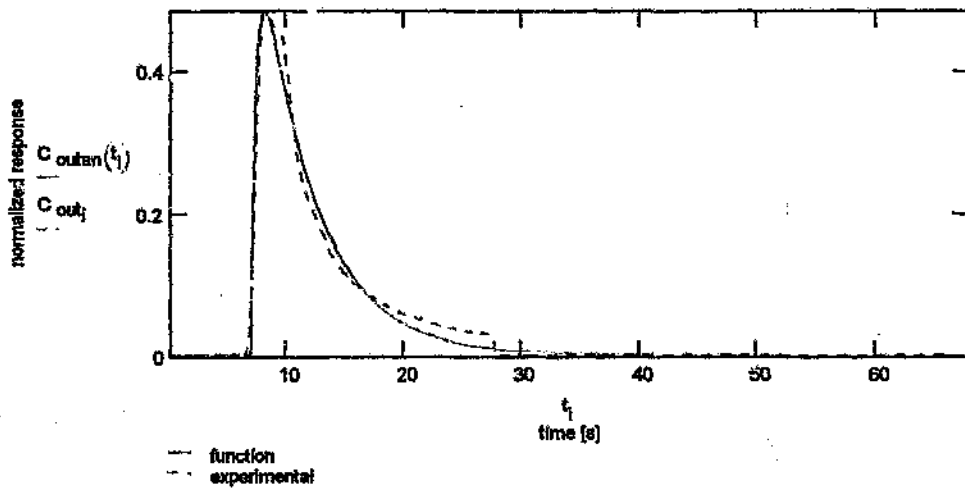
Overall Response : (best function fit to data)

$\tau_{o1} := 0.21$ $\tau_{o2} := 2.1$ $\tau_{od} := 7.0$ $const_o := 1.31$ parameters for RTD

$G_{outan}(t) := C(t, \tau_{o1}, \tau_{o2}, \tau_{od}, const_o)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - G_{outan}(t_j))^2$ define squared error

$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.113$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) := \text{const}_p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response
$$C_{outan}(t) := \text{const}_o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

$$A := \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B := \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

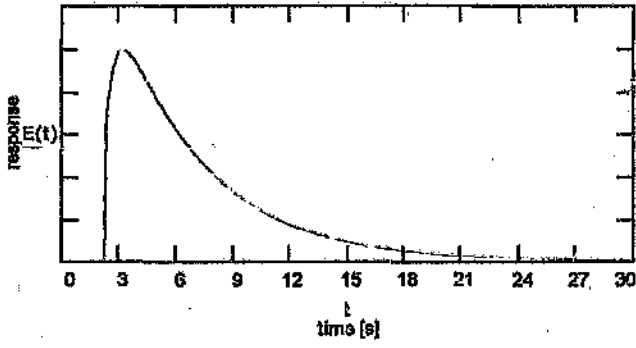
A = 0.735

B = 0.415

$\tau_{Ed} := \tau_{od} - \tau_{pd}$

$\tau_{Ed} = 2.36$

$$E(t) := \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$



Appendix 1.3.4 Top Left Burner to Furnace Exit - 4m into boiler

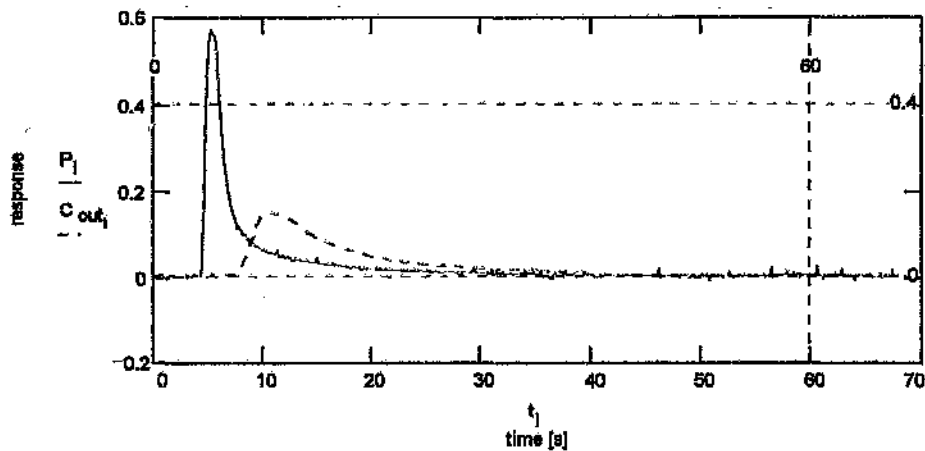


RTD Manipulations : (Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	24 m (mill B, left burner)	
Probe	4m in the boiler	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD := READPRN(decog)	
time vector	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0..n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) := \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[(t - \tau_d) \cdot \tau_1 \right] - \exp\left[(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \Phi(t - \tau_d)$$

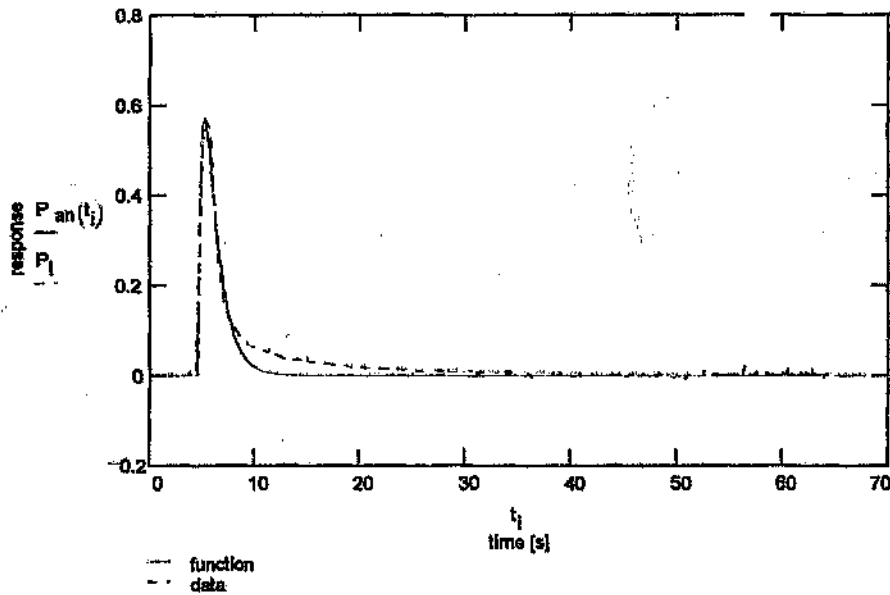
Probe Response : (visual best function fit to data)

$\tau_{p1} := 0.77$ $\tau_{p2} := 2.69$ $\tau_{pd} := 4.64$ $const_p := 2.52$ parameters for RTD

$P_{an}(t) := C(t, \tau_{p1}, \tau_{p2}, \tau_{pd}, const_p)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2$ define squared error $E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$

Graph comparing measured and curve fitted RTD



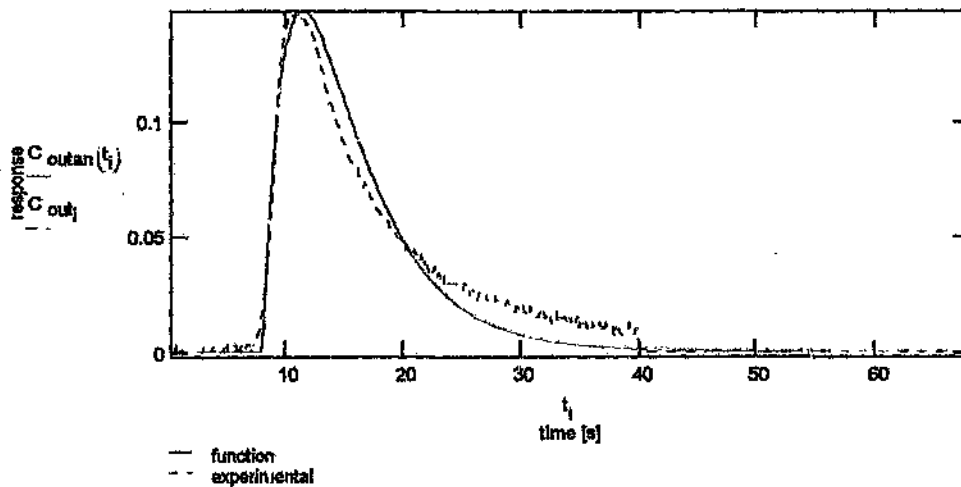
Overall Response : (best function fit to data)

$\tau_{o1} := 0.20$ $\tau_{o2} := 0.4$ $\tau_{od} := 8.01$ $const_o := 0.118$ parameters for RTD

$C_{outan}(t) := C(t, \tau_{o1}, \tau_{o2}, \tau_{od}, const_o)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - C_{outan}(t_j))^2$ define squared error

$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.032$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) = \text{const } p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response :
$$C_{outan}(t) = \text{const } o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

$$A = \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B = \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

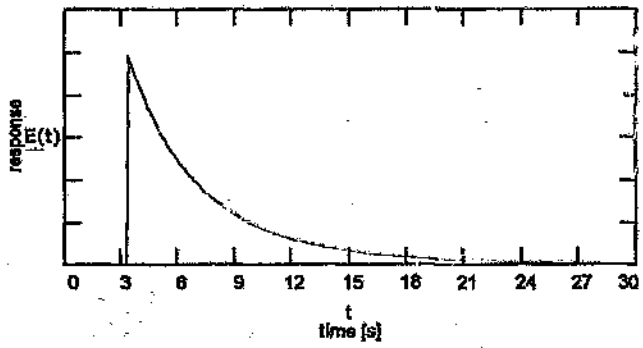
A = 7.096

B = -4.237

$\tau_{Ed} = \tau_{od} - \tau_{pd}$

$\tau_{Ed} = 3.37$

$$E(t) = \frac{\text{const } o}{\text{const } p} \cdot \Phi(t - \tau_{Ed}) \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$



Appendix 1.3.5 Top Right Burner to Furnace Exit - 4m into boiler

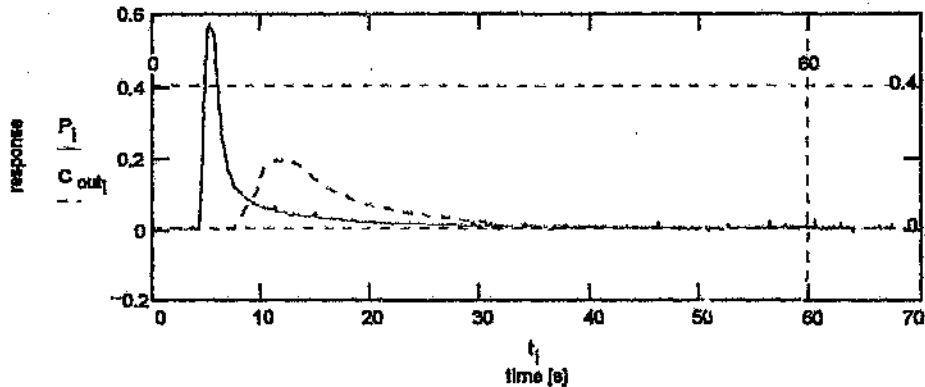
RTD Manipulations :

(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	24m (mill B, right burner)	
Probe	4 m in the boiler	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD:=READPRN(decoh)	
time vector	t:=RTD<0>	
Probe Response	P:=RTD<2>	
Overall Response	C _{out} :=RTD<1>	
# of rows in vector	n:=rows(RTD) - 1	n=679
vector pointer	i:=0..n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) := \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[(t - \tau_d) \cdot \tau_1 \right] - \exp\left[(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \Theta(t - \tau_d)$$

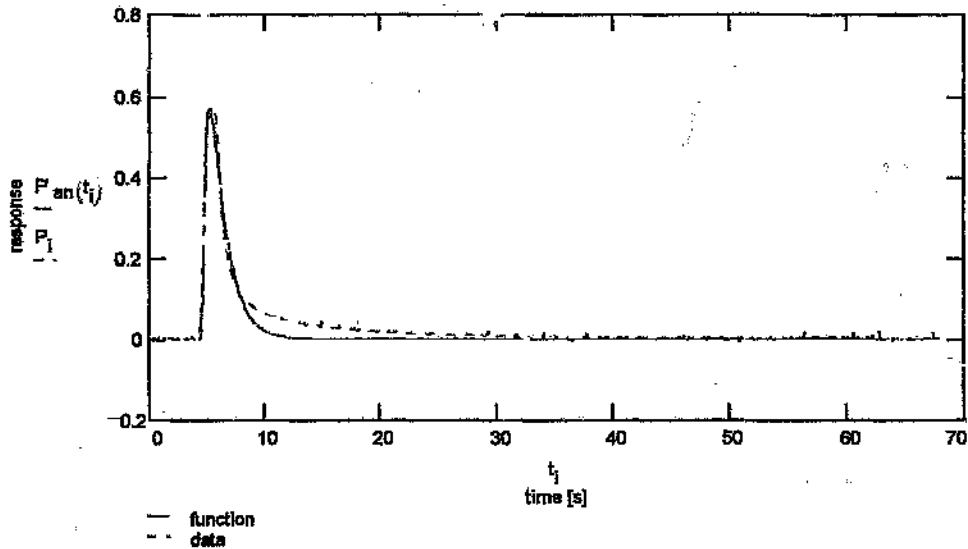
Probe Response : (visual best function fit to data)

$\tau_{p1} = -0.77$ $\tau_{p2} = -2.69$ $\tau_{pd} = 4.64$ $const_p = 2.52$ parameters for RTD

$$P_{an}(t) := C(\tau_{p1}, \tau_{p2}, \tau_{pd}, const_p)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2 \quad \text{define squared error} \quad E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$$

Graph comparing measured and curve fitted RTD



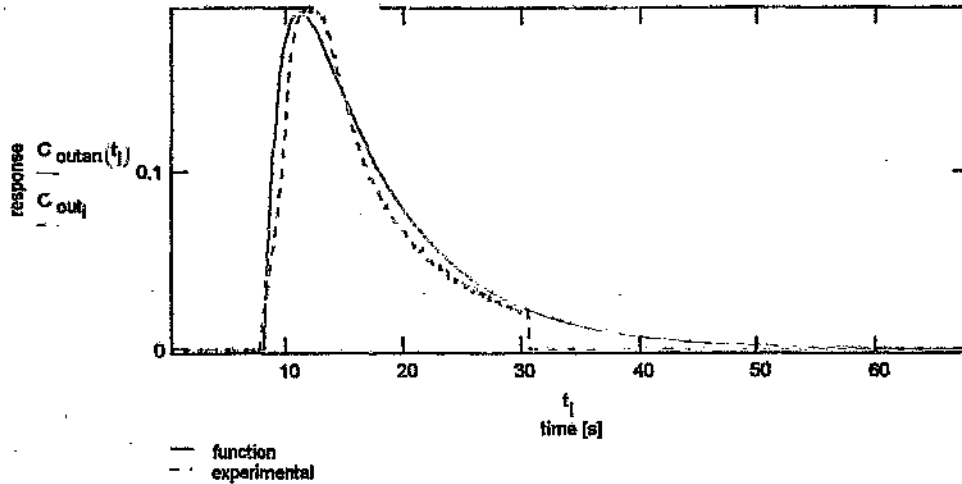
Overall Response : (best function fit to data)

$\tau_{o1} = -0.12$ $\tau_{o2} = -0.7$ $\tau_{od} = 8.1$ $const_o = 0.19$ parameters for RTD

$$C_{outan}(t) := C(\tau_{o1}, \tau_{o2}, \tau_{od}, const_o)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{out_j} - C_{outan}(t_j))^2 \quad \text{define squared error}$$

$$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.089$$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) = \text{const}_p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response
$$C_{outan}(t) = \text{const}_o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

$$A = \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B = \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

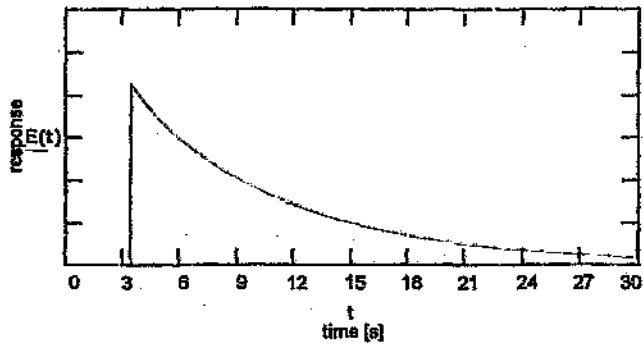
$A = 2.88$

$B = -0.24$

$\tau_{Ed} \quad \tau_{od} \quad \tau_{pd}$

$$E(t) = \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$

$\tau_{Ed} = 3.46$



$$\tau_{Ed} = 3.46 \text{ s}$$

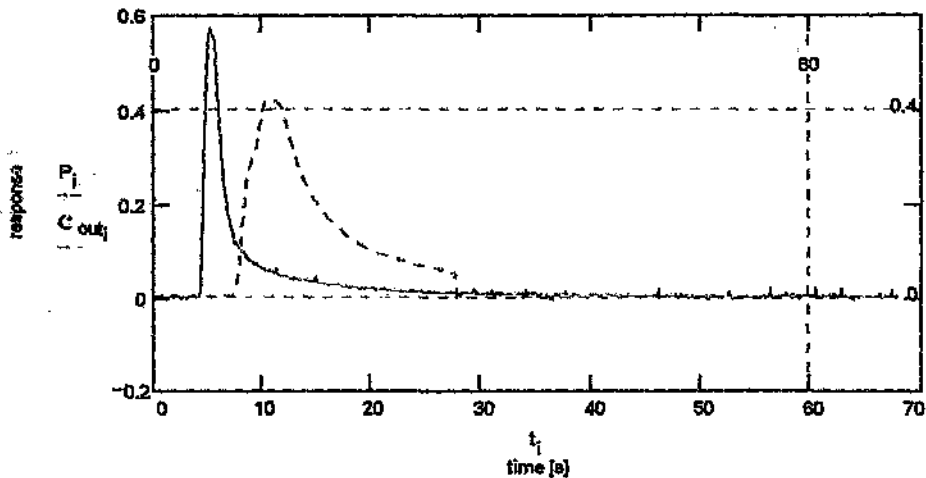
Appendix 1.3.6 Middle Left Burner to Furnace Exit - 4m into boiler

RTD Manipulations :
(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	20 m (mill C, left burner)	
Probe	4 m in the furnace	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD := READPRN(decok)	
time vector	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0..n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) := \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[(t - \tau_d) \cdot \tau_1 \right] - \exp\left[(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \Phi(t - \tau_d)$$

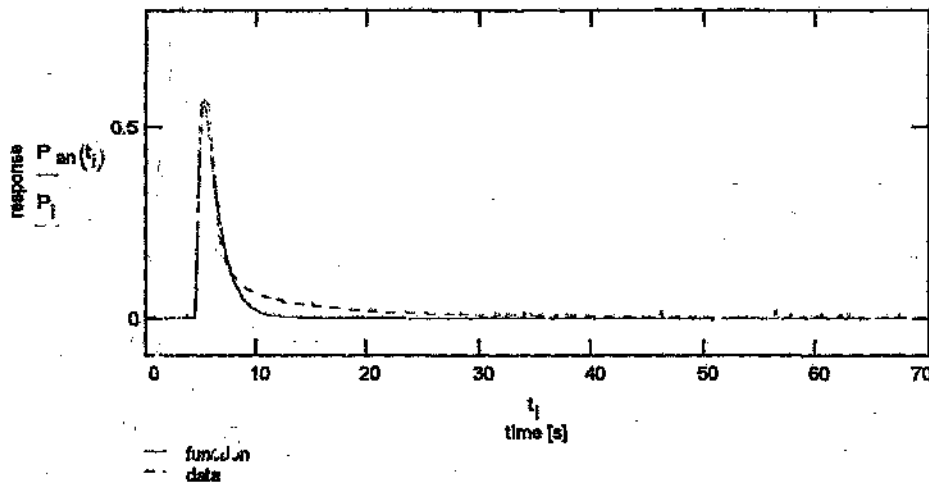
Probe Response : (visual best function fit to data)

$\tau_{p1} := 0.77$ $\tau_{p2} := 2.69$ $\tau_{pd} := 4.64$ $\text{const}_p := 2.52$ parameters for RTD

$$P_{an}(t) := C(\tau_{p1}, \tau_{p2}, \tau_{pd}, \text{const}_p)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^N (P_j - P_{an}(t_j))^2 \quad \text{define squared error} \quad E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$$

Graph comparing measured and curve fitted RTD



Overall Response : (best function fit to data)

$\tau_{o1} := 0.15$ $\tau_{o2} := 1.0$ $\tau_{od} := 8.2$ $\text{const}_o := .59$ parameters for RTD

$$C_{outan}(t) := C(\tau_{o1}, \tau_{o2}, \tau_{od}, \text{const}_o)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - C_{outan}(t_j))^2 \quad \text{define squared error}$$

$$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.212$$

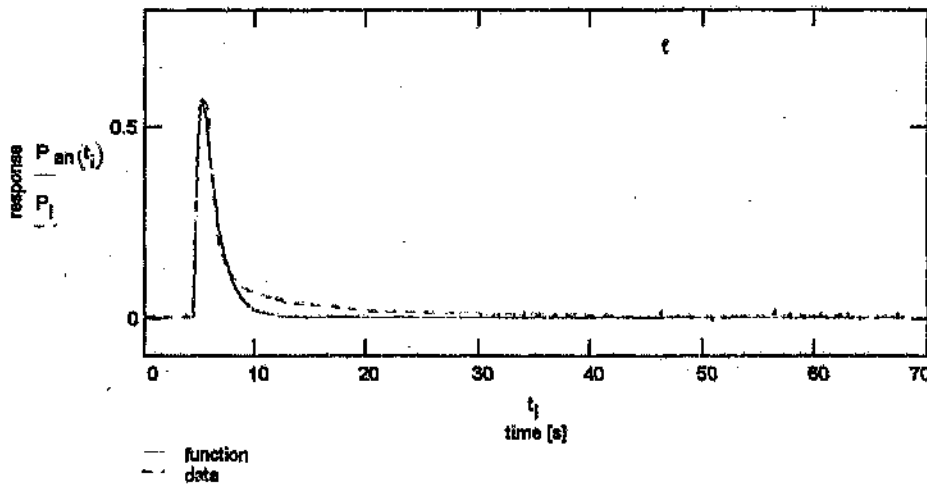
Probe Response : (visual best function fit to data)

$\tau_{p1} = 0.77$ $\tau_{p2} = 2.69$ $\tau_{pd} = 4.64$ $const_p = 2.52$ parameters for RTE

$$P_{an}(t) = C(\tau_{p1}, \tau_{p2}, \tau_{pd}, const_p)$$

$$E(\tau_1, \tau_2, \tau_d) = \sum_{j=1}^n (P_j - P_{an}(t_j))^2 \quad \text{define squared error} \quad E(\tau_{p1}, \tau_{p2}, \tau_{pd}) =$$

Graph comparing measured and curve fitted RTD



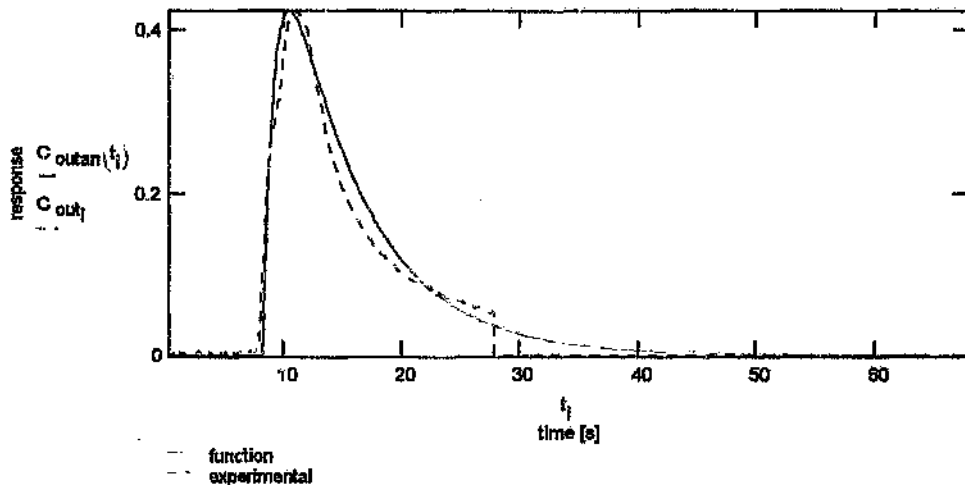
Overall Response : (best function fit to data)

$\tau_{o1} = 0.15$ $\tau_{o2} = 1.0$ $\tau_{od} = 8.2$ $const_o = .59$ parameters for RTD

$$C_{outan}(t) = C(\tau_{o1}, \tau_{o2}, \tau_{od}, const_o)$$

$$E(\tau_1, \tau_2, \tau_d) = \sum_{j=1}^n (C_{out_j} - C_{outan}(t_j))^2 \quad \text{define squared error}$$

$$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.212$$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response : $P_{an}(t) = \text{const } p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \cdot \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$

Overall Response $C_{outan}(t) = \text{const } o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \cdot \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

$$A = \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B = \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

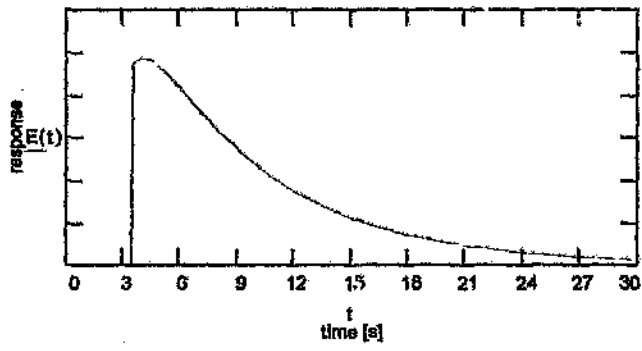
$A = 1.853$

$B = 0.457$

$\tau_{Ed} = \tau_{od} - \tau_{pd}$

$\tau_{Ed} = 3.56$

$$E(t) := \frac{\text{const } o}{\text{const } p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$



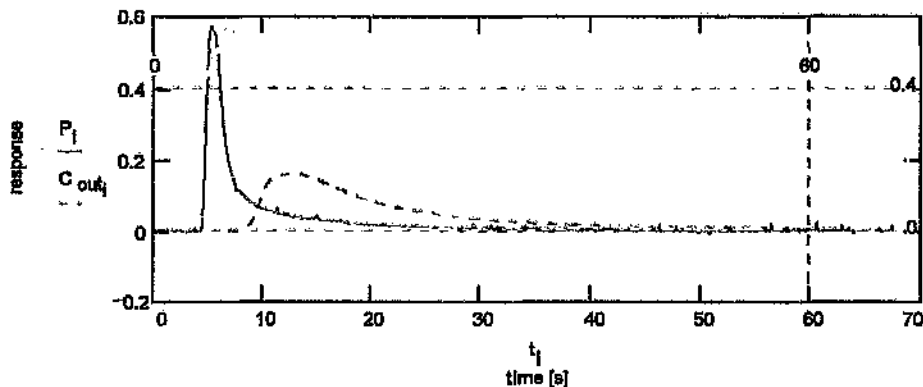
Appendix 1.3.7 : Middle Right Burner to Furnace Exit - 4m into boiler

RTD Manipulations : (Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	20 m (mill C, right burner)	
Probe	4 m in the furnace	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD :=READPRN(decol)	
time vector	t =RTD<0>	
Probe Response	P :=RTD<2>	
Overall Response	C _{out} :=RTD<1>	
# of rows in vector	n :=rows(RTD) - 1	n = 679
vector pointer	i :=0..n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) := \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[(t - \tau_d) \cdot \tau_1 \right] - \exp\left[(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \Phi(t - \tau_d)$$

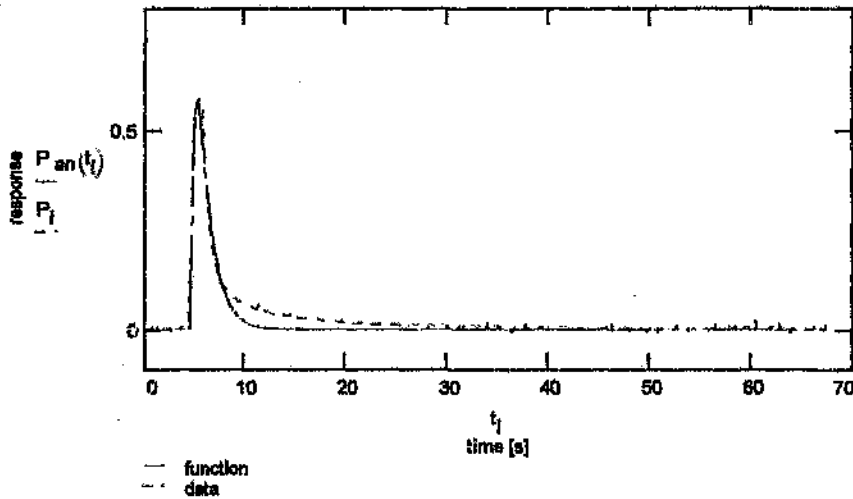
Probe Response : (visual best function fit to data)

$\tau_{p1} := 0.77$ $\tau_{p2} := 2.69$ $\tau_{pd} := 4.64$ $\text{const}_p := 2.52$ parameters for RTD

$$P_{an}(t) := C(\tau_{p1}, \tau_{p2}, \tau_{pd}, \text{const}_p)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2 \quad \text{define squared error} \quad E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$$

Graph comparing measured and curve fitted RTD



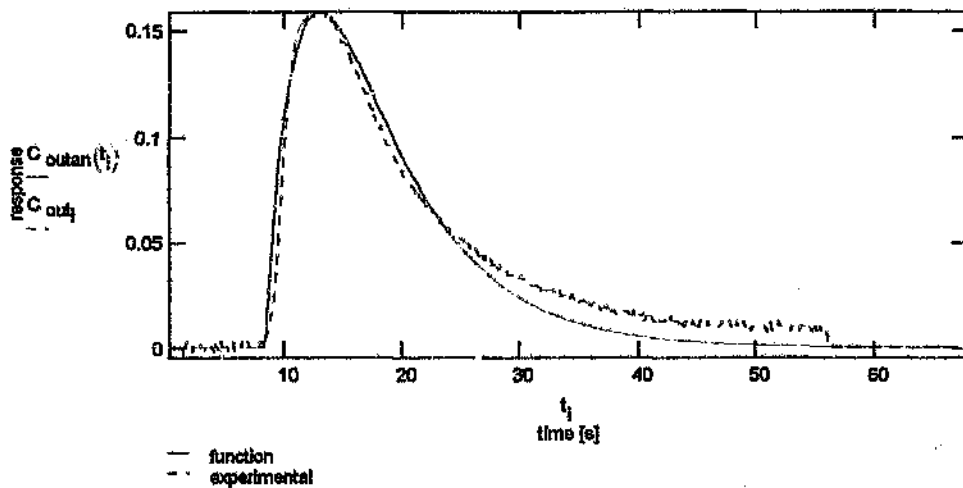
Overall Response : (best function fit to data)

$\tau_{o1} := 0.15$ $\tau_{o2} := 0.3$ $\tau_{od} := 8.32$ $\text{const}_o := 0.095$ parameters for RTD

$$C_{outan}(t) := C(\tau_{o1}, \tau_{o2}, \tau_{od}, \text{const}_o)$$

$$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - C_{outan}(t_j))^2 \quad \text{define squared error}$$

$$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.042$$



**Analysing time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) = \text{const}_p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response
$$C_{outan}(t) = \text{const}_o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the Inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace Inverting the deconvoluted expression from the Laplace domain :

$$A = \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B = \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

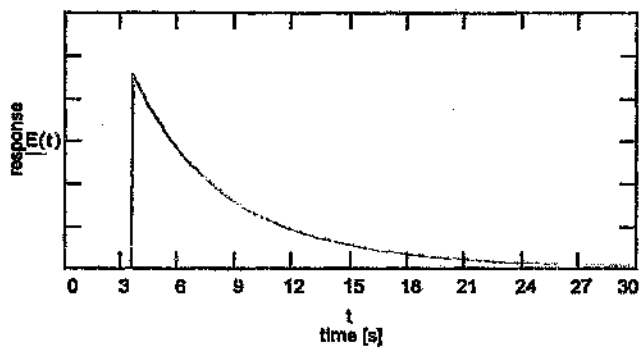
$A = 10.499$

$B = -7.489$

$\tau_{Ed} = \tau_{od} - \tau_{pd}$

$\tau_{Ed} = 3.68$

$$E(t) = \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$



Appendix 1.3.8

Bottom Left Burner to Furnace Exit -- 4m into boiler

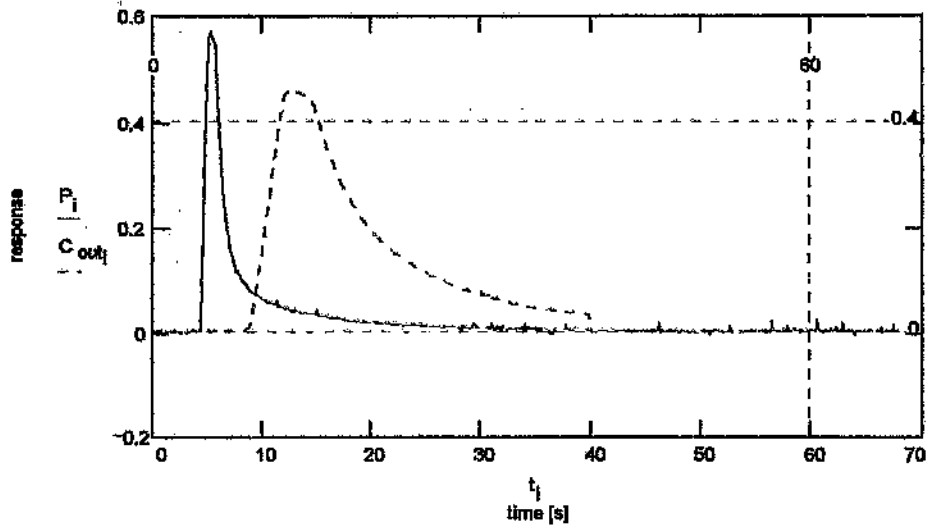
RTD Manipulations :

(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	16 m (mill D, left burner)	
Probe	4 m in the furnace	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD := READPRN(decol)	
time vector	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0..n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, const) = const \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot [\exp[(t - \tau_d) \cdot \tau_1] - \exp[(t - \tau_d) \cdot \tau_2]] \cdot \Phi(t - \tau_d)$$

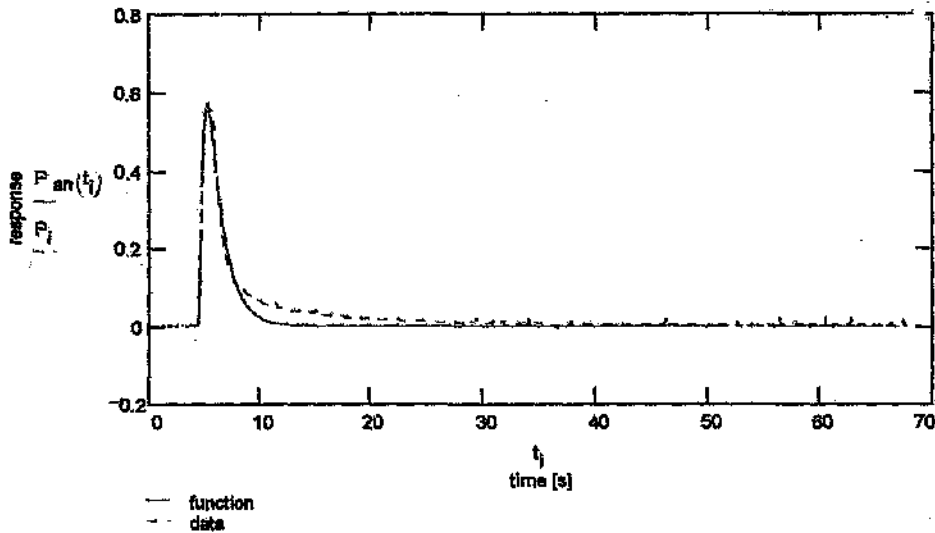
Probe Response : (visual best function fit to data)

$\tau_{p1} = 0.77$ $\tau_{p2} = 2.69$ $\tau_{pd} = 4.64$ $const_p = 2.52$ parameters for RTD

$P_{an}(t) := C(t, \tau_{p1}, \tau_{p2}, \tau_{pd}, const_p)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2$ define squared error $E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$

Graph comparing measured data curve fitted RTD



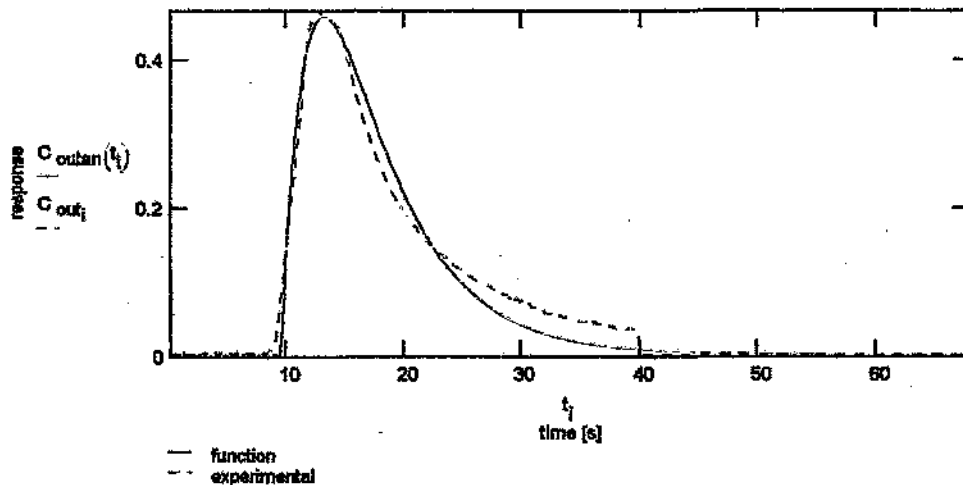
Overall Response : (best function fit to data)

$\tau_{o1} = 0.18$ $\tau_{o2} = 0.4$ $\tau_{od} = 9.6$ $const_o = 0.35$ parameters for RTD

$C_{outan}(t) := C(t, \tau_{o1}, \tau_{o2}, \tau_{od}, const_o)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - C_{outan}(t_j))^2$ define squared error

$E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.224$



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) := \text{const}_p \cdot \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \cdot \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response
$$C_{outan}(t) := \text{const}_o \cdot \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \cdot \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace inverting the deconvoluted expression from the Laplace domain :

$$A := \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}} \quad B := \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

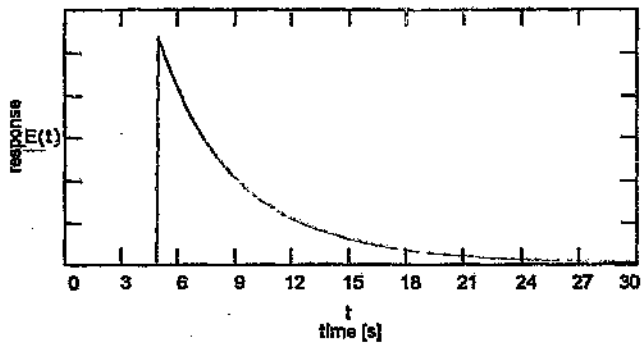
$A = 6.731$

$B = -3.851$

$\tau_{Ed} := \tau_{od} - \tau_{pd}$

$\tau_{Ed} = 4.96$

$$E(t) := \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$



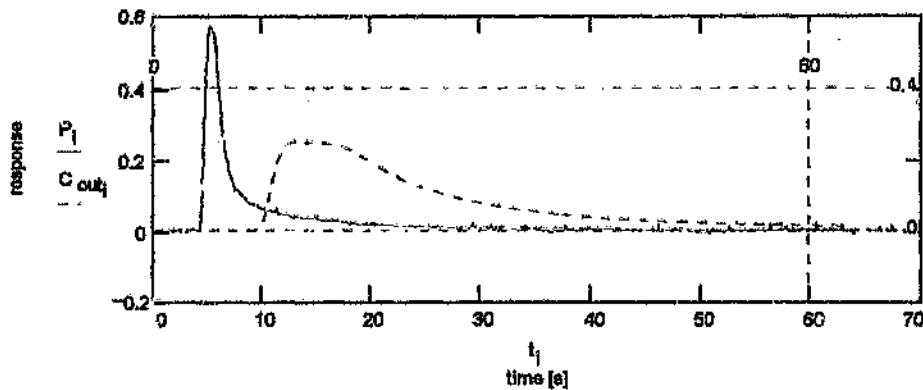
RTD Manipulations :

(Deconvolution of probe response from overall measured response)

Experimental Data

Substance injected	sulfur	
Boiler Level at injection	16 m (mill D, right burner)	
Probe	4 m in the furnace	
Boiler Level at Probe	35.64 m	
Read Data from File :	RTD := READPRN(decoj)	
time vector	t := RTD<0>	
Probe Response	P := RTD<2>	
Overall Response	C _{out} := RTD<1>	
# of rows in vector	n := rows(RTD) - 1	n = 679
vector pointer	i := 0 .. n	

Plot of experimental overall and probe response to dirac tracer injection



Time Domain Response Definitions

The function below depicts a general form of the time domain responses. This function is fitted to the measured response data by fitting of the function parameters.

$$C(t, \tau_1, \tau_2, \tau_d, \text{const}) := \text{const} \cdot \left(\frac{1}{\tau_1 - \tau_2} \right) \cdot \left[\exp\left[-(t - \tau_d) \cdot \tau_1 \right] - \exp\left[-(t - \tau_d) \cdot \tau_2 \right] \right] \cdot \Phi(t - \tau_d)$$

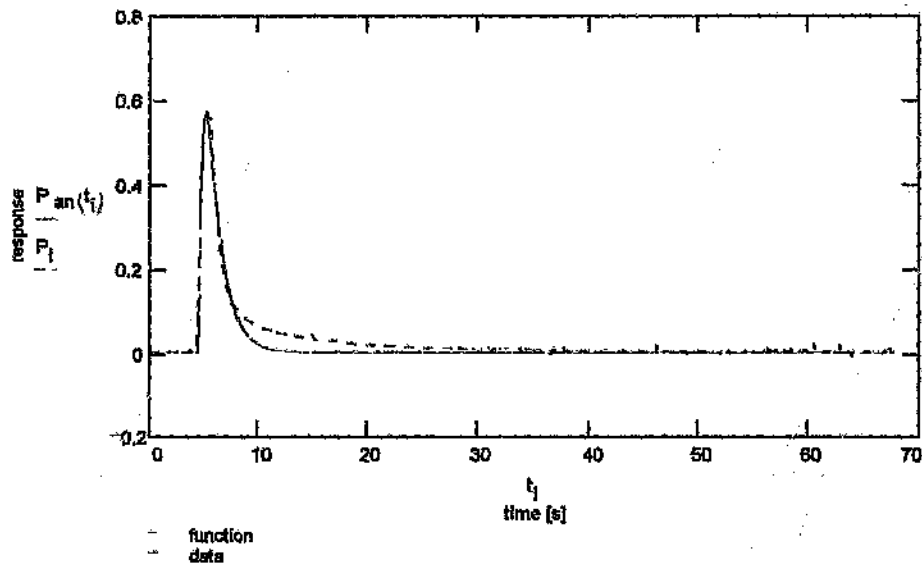
Probe Response : (visual best function fit to data)

$\tau_{p1} := -0.77$ $\tau_{p2} := -2.69$ $\tau_{pd} := 4.64$ $\text{const}_p := 2.52$ parameters for RTD

$P_{an}(t) := C(t, \tau_{p1}, \tau_{p2}, \tau_{pd}, \text{const}_p)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (P_j - P_{an}(t_j))^2$ define squared error $E(\tau_{p1}, \tau_{p2}, \tau_{pd}) = 0.2$

Graph comparing measured and curve fitted RTD

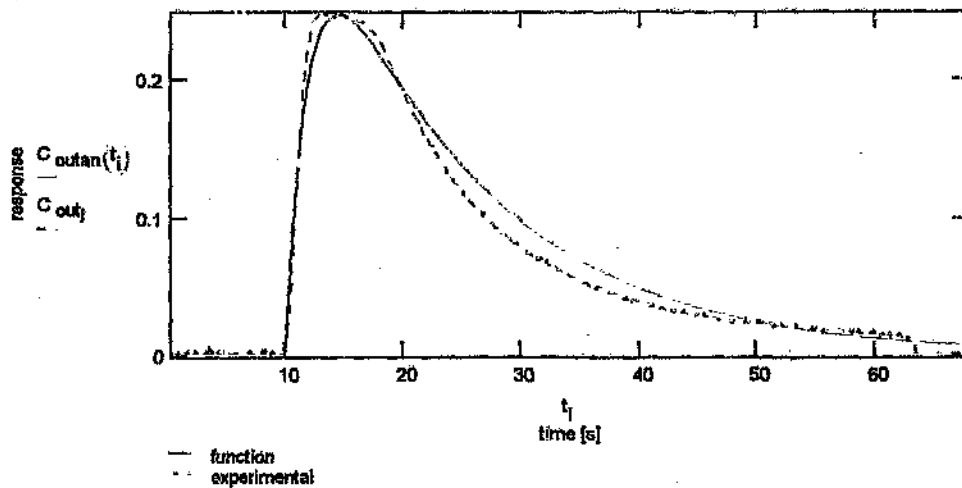


Overall Response : (best function fit to data)

$\tau_{o1} := -0.07$ $\tau_{o2} := -0.5$ $\tau_{od} := 10.0$ $\text{const}_o := 0.17$ parameters for RTD

$C_{outan}(t) := C(t, \tau_{o1}, \tau_{o2}, \tau_{od}, \text{const}_o)$

$E(\tau_1, \tau_2, \tau_d) := \sum_{j=1}^n (C_{outj} - C_{outan}(t_j))^2$ $E(\tau_{o1}, \tau_{o2}, \tau_{od}) = 0.073$ define squared error



**Analytical time domain responses expressed in the Laplace domain
(2nd order system with dead time)**

Probe Response :
$$P_{an}(t) = \text{const}_p \left(\frac{1}{\tau_{p1} - \tau_{p2}} \right) \left[\exp\left[(t - \tau_{pd}) \cdot \tau_{p1} \right] - \exp\left[(t - \tau_{pd}) \cdot \tau_{p2} \right] \right] \cdot \Phi(t - \tau_{pd})$$

Overall Response
$$C_{outan}(t) = \text{const}_o \left(\frac{1}{\tau_{o1} - \tau_{o2}} \right) \left[\exp\left[(t - \tau_{od}) \cdot \tau_{o1} \right] - \exp\left[(t - \tau_{od}) \cdot \tau_{o2} \right] \right] \cdot \Phi(t - \tau_{od})$$

Laplace Transforms can be taken of the above expressions, after which the required deconvolution can occur in the Laplace domain. By taking the inverse Laplace transform of the expression obtained by the deconvolution and normalizing the result, the RTD of the boiler can be determined.

Laplace Inverting the deconvoluted expression from the Laplace domain :

$$A = \frac{(\tau_{o1} - \tau_{p1}) \cdot (\tau_{o1} - \tau_{p2})}{\tau_{o1} - \tau_{o2}}$$

$$B = \frac{(\tau_{o2} - \tau_{p1}) \cdot (\tau_{o2} - \tau_{p2})}{\tau_{o2} - \tau_{o1}}$$

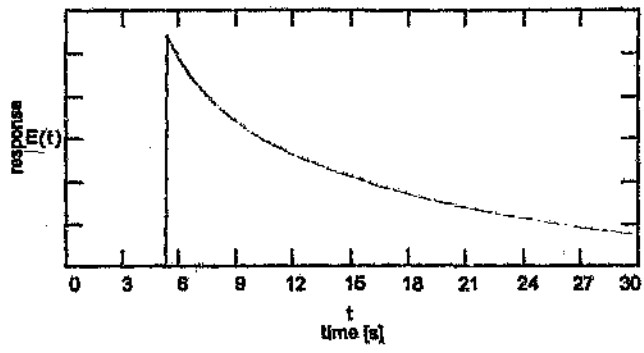
A = 4.265

B = -1.375

$\tau_{Ed} = \tau_{od} - \tau_{pd}$

$\tau_{Ed} = 5.36$

$$E(t) = \frac{\text{const}_o}{\text{const}_p} \cdot \Phi(t - \tau_{Ed}) \cdot \left[A \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o1} \right] - B \cdot \exp\left[(t - \tau_{Ed}) \cdot \tau_{o2} \right] \right]$$



APPENDIX 2

Appendix 2.1.....	Hendrina R.C. Composite Data
Appendix 2.1.1.....	Basic Analyses Datasheet
Appendix 2.1.2.....	Determination of the Devolatilisation Kinetics
Appendix 2.1.3.....	Determination of the Char Combustion Kinetics
Appendix 2.2.....	Hendrina Washing Plant Feed Data
Appendix 2.2.1.....	Basic Analyses Datasheet
Appendix 2.2.2.....	Determination of the Char Combustion Kinetics
Appendix 2.3.....	Particle Size Analyses

Appendix 2.1.1

Hendrina R.C. Composite: Basic Analysis Datasheet

COAL :

Hendrina - RC Composite

Preparation of Coal Analysis Data for specification in AIOLOS

Analysis by TRI Coal Labs

all entries in mass% except where otherwise noted

Total Moisture : (as received)	9.8			
Proximate :	air dried	as received	dry	daf
Moisture	3.5	9.8000		
	25	23.4250	25.9067	
	22.8	21.3636	23.6269	31.8881
Fixed Carbon	48.7	45.4114	50.4863	68.1119
Ultimate :				
Carbon	58.76	0.5506	60.8912	82.1818
Hydrogen	3.02	0.0283	3.1295	4.2238
Nitrogen	1.39	0.0130	1.4404	1.9441
Sulphur	0.76	0.0071	0.7876	1.0629
Carbonates	0.76	0.0071	0.7876	1.0629
Oxygen	6.81	0.0638	7.0570	9.5245
Oxygen + Carbonates		0.0667		
Calorific Value	22.69	21.28053	23.513	31.73426573
DTF DEVOL.				
Volatile Matter	28.025			
Q Factor	1.229			
DTF FULL TEST				
Size Fraction	<38 µm	38-75 µm	< 75 µm	Unit
Pre-exponential Factor	4269	70549	3212	kg/m ² .s.atm
	4213	69626	3170	kg/m ² .s.bar
Activation Energy	144.5	181.6	132.98	kJ/mol
Activation Energy /R	17380	21843	15995	K

Coal Name: HENDRINA RC COMP.
 Origin: Optimum Colliery
 Coal Preparation: as per Hendrina P/S feed

Table :

PROXIMATE ANALYSIS			METHOD
Inherent Moisture:	3.5	(m% of coal - air dried)	ESKOM 103
Ash :	25		ESKOM 101
Volatile Matter :	22.8		ESKOM 102
Fixed Carbon (by difference) :	48.7		ESKOM 128
ULTIMATE ANALYSIS			
Carbon:	58.76	(m% of coal - air dried)	ESKOM 118
Hydrogen:	3.02		ESKOM 118
Nitrogen:	1.39		ESKOM 118
Total Sulphur:	0.76		ESKOM 104
Carbonates (as CO ₂):	0.76		ESKOM 100
Oxygen (by difference):	6.81		ESKOM 132
Gross Calorific Value:	22.69	(MJ/kg)	ESKOM 105
HARDGROVE INDEX:	51		ESKOM 117
ABRASIVENESS:	477	(mg Fe/kg coal)	ESKOM 123
ASH FUSION TEMPERATURES			ESKOM 125
Initial Deformation Temperature:	1493	(°C)	
Softening Temperature:	1525		
Hemispherical Temperature:	1544		
Flow Temperature:	1552		

'-' indicates that the result of that analysis is not available
 RAW COAL PREPARATION BY SABS 0135 PART 2 - 1977

Appendix 2.1.2

Hendrina R.C. Composite: Determination of the Pyrolysis Kinetics

Determination DTF derived single step, zero order, Arrhenius pyrolysis kinetics for use with AIOLOS combustion code

Universal Gas Constant	$R := 8.314$	$\frac{J}{mol \cdot K}$
------------------------	--------------	-------------------------

Coal Parameters

Volatile Matter (DTF)	$Vols_{DTF} := 29.92\%$	
Density of coal :	$\rho_{coal} := 1642$	$\frac{kg}{m^3}$
Ash content of dry coal :	$Ash_{dry} := 29.6\%$	
Particle Diameter :	$dp_o := 22 \cdot 10^{-6}$	m
Particle Mass :	$m_p := \frac{\pi \cdot dp_o^3}{6} \cdot \rho_{coal}$	
Initial dry ash free volatile content :	$V_{daf,0} := \frac{Vols_{DTF}}{1 - Ash_{dry}}$	$V_{daf,0} = 42.5\%$

Heat Exchange Parameters and constants

Stefan Boltzmann constant :	$\sigma = 5.67 \cdot 10^{-8}$	$\frac{W}{m^2 \cdot K^4}$
average emissivity between particle and wall :	$\epsilon := 0.9$	
Nusselt Number for boundary layer convective heat exchange :	$Nu := 2$	
Convective heat exchange coefficient :	$\lambda(T_p, T_g) := 2.43 \cdot 10^{-5} \cdot \left[\frac{(T_p + T_g)}{2} \right]^{0.82}$	$\frac{W}{m \cdot K}$
Heat capacity of coal :	$Cp_{coal} := 1250$	$\frac{J}{kg \cdot K}$
Particle Surface Area :	$A_{part} := \pi \cdot dp_o^2$	

Estimating Overall Rate :

$$k_{p,k} = \frac{1 - \left(\frac{V_{daf}(DTF_{k,2})}{V_{daf,0}} \right)}{DTF_{k,1}}$$

Hendrina R.C. Composite : <38 μm

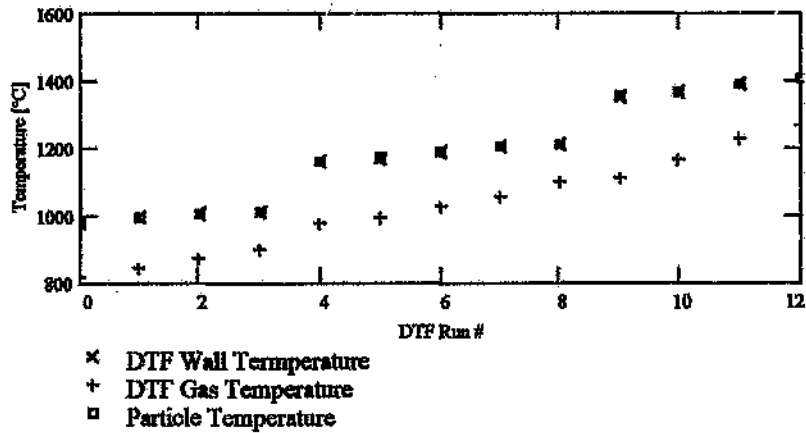
Calculating Particle Temperature:

Iterative solution of Energy Balance between particle, hot gas and reactor wall :

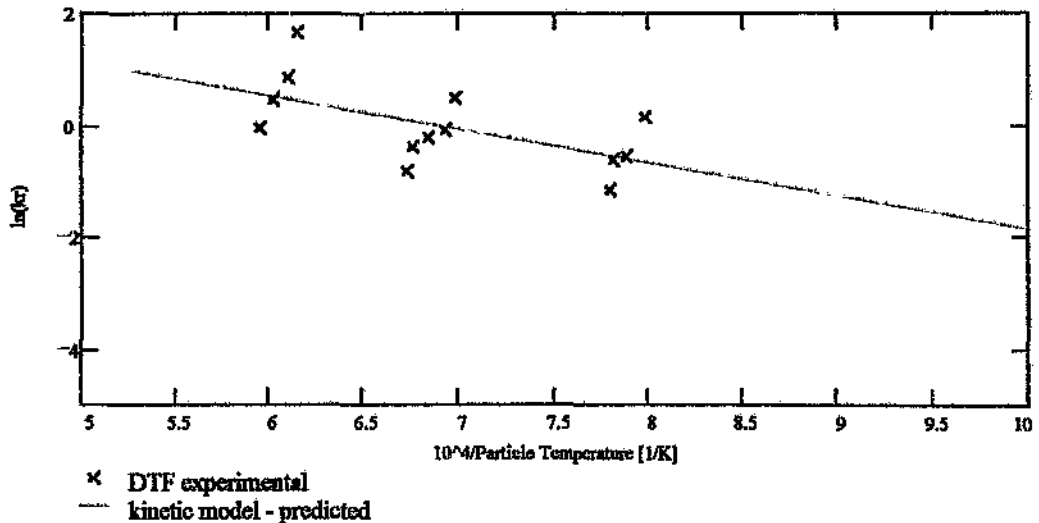
$$0 = \epsilon \sigma (T_w^4 - T_p^4) \cdot A_{part} + \frac{Nu \cdot \lambda (T_p - T_g)}{d_p} \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \cdot A_{part}$$

$$T_{particle}(T_w, T_g) := \text{find}(T_p)$$

$$DTF_{k,4} := T_{particle}(DTF_{k,0} + 273, DTF_{k,3} + 273) - 273$$



Determining the Pyrolysis Kinetics from the experimental data :



Coal pyrolysis kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 61.095 \quad \frac{1}{s}$

Activation Energy : $-\text{EonR} \cdot \frac{R}{10^3} = 49.684 \quad \frac{\text{kJ}}{\text{mol}}$

$-\text{EonR} = 5975.987 \quad \text{K}$

Function describing the pyrolysis reaction rate : $k_1(T_p) := k_{0.1} \exp\left(\frac{\text{EonR}}{T_p}\right)$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation $dT_{\text{rad}}(T_p, T_w) := \frac{\epsilon \sigma A_{\text{part}} (T_w^4 - T_p^4)}{C_p \text{coal} m_p}$

convection $dT_{\text{conv}}(T_p, T_g) := \frac{\text{Nu} \cdot A_{\text{part}} \lambda (T_p, T_g) \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_p \right]}{C_p \text{coal} m_p}$

Estimation of the initial particle heating rate at DTF setpoint temperature of 1400°C:

$T_{\text{init}} := 25 + 273 \text{ K} \quad dT_{\text{rad}}(T_{\text{init}}, 1416) + dT_{\text{conv}}(T_{\text{init}}, 1335) = 2.721 \cdot 10^4 \frac{\text{K}}{s}$

Solution of differential equations

Initial Conditions :

Volatiles (%) $x_0 := 1$

Particle Temperature $x_1 := 300$

Simultaneous Differential Equations :

$$\frac{d}{dt} V = -k_1(x_1)$$

$$\frac{d}{dt} T_p = dT_{\text{rad}}(x_0, x_1) + dT_{\text{conv}}(x_0, x_1)$$

Note on Solution Methodology :

The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature

Hendrina R.C. Composite : <38 μm

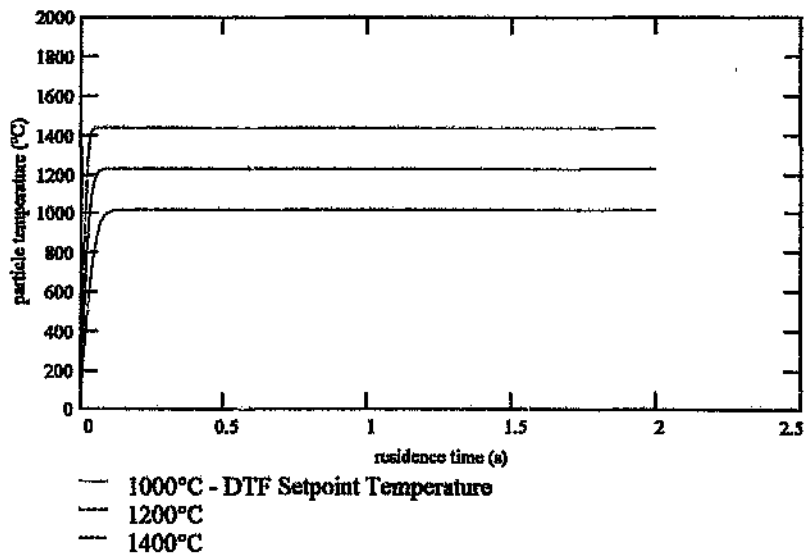
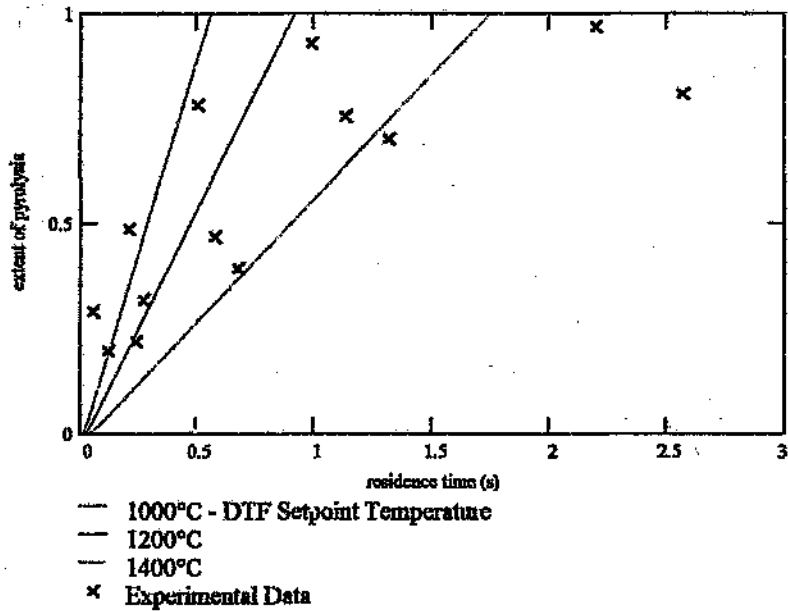
DTF Pyrolysis Evaluation Results :

Coal pyrolysis kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 61.095 \frac{1}{s}$

Activation Energy : $-\text{EonR} \cdot \frac{R}{10^3} = 49.684 \frac{\text{kJ}}{\text{mol}}$ $-\text{EonR} = 5975.987 \text{ K}$

Function describing the pyrolysis reaction rate : $k_1(T_p) := k_{0.1} \cdot \exp\left(\frac{\text{EonR}}{T_p}\right)$



Hendrina R.C. Composite : <38 μm

Determination DTF derived single step, zero order, Arrhenius pyrolysis kinetics for use with AIOLOS combustion code

Universal Gas Constant	$R := 8.314$	$\frac{J}{mol \cdot K}$
------------------------	--------------	-------------------------

Coal Parameters

Volatile Matter (DTF)	$Vols_{DTF} := 29.92\%$	
Density of coal :	$\rho_{coal} := 1642$	$\frac{kg}{m^3}$
Ash content of dry coal :	$Ash_{dry} := 29.6\%$	
Particle Diameter :	$dp_o := 59.95 \cdot 10^{-6}$	m
Particle Mass :	$m_p := \frac{\pi \cdot dp_o^3}{6} \cdot \rho_{coal}$	
Initial dry ash free volatile content :	$V_{daf,0} := \frac{Vols_{DTF}}{1 - Ash_{dry}}$	$V_{daf,0} = 42.5\%$

Heat Exchange Parameters and constants

Stefan Boltzmann constant :	$\sigma := 5.67 \cdot 10^{-8}$	$\frac{W}{m^2 \cdot K^4}$
average emissivity between particle and wall :	$\epsilon := 0.9$	
Nusselt Number for boundary layer convective heat exchange :	$Nu := 2$	
Convective heat exchange coefficient :	$\lambda(T_p, T_g) := 2.43 \cdot 10^{-5} \cdot \left[\frac{(T_p + T_g)}{2} \right]^{0.82}$	$\frac{W}{m \cdot K}$
Heat capacity of coal :	$Cp_{coal} := 1250$	$\frac{J}{kg \cdot K}$
Particle Surface Area :	$A_{part} := \pi \cdot dp_o^2$	

Estimating Overall Rate :

$$k_{Pk} := \frac{1 - \left(\frac{V_{daf}(DTF_{k,2})}{V_{daf,0}} \right)}{DTF_{k,1}}$$

Hendrina R.C. Composite : 38-75 μm

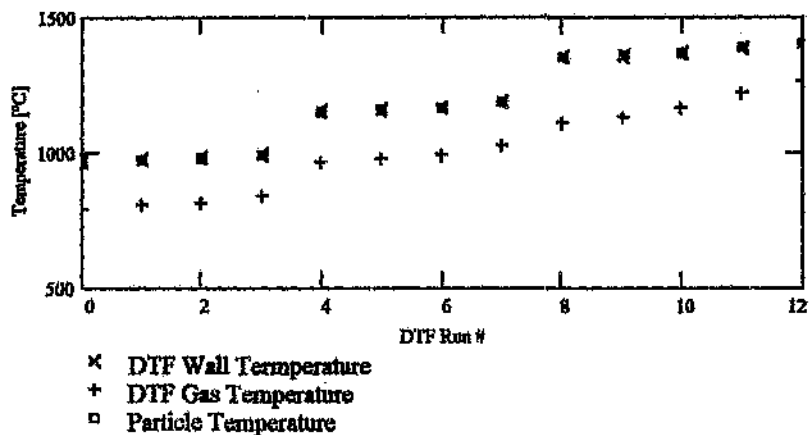
Calculating Particle Temperature:

iterative solution of Energy Balance between particle, hot gas and reactor wall :

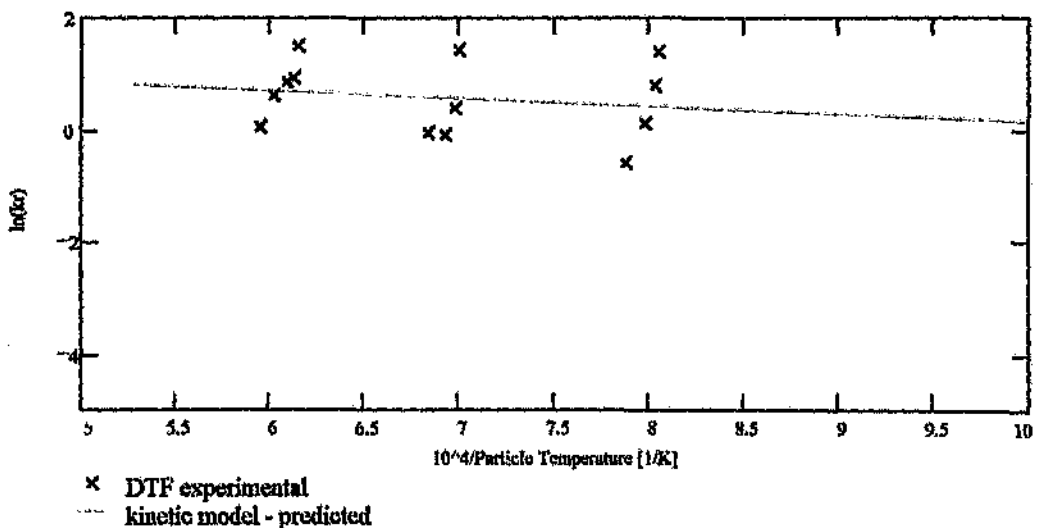
$$0 = \sigma(T_w^4 - T_p^4) \cdot A_{part} + \frac{Nu \cdot \lambda(T_p, T_g)}{dp_o} \cdot \left(\frac{T_g + T_p}{2} - T_g \right) \cdot A_{part}$$

$$T_{particle}(T_w, T_g) := \text{find}(T_p)$$

$$DTF_{k,4} := T_{particle}(DTF_{k,0} + 273, DTF_{k,3} + 273) - 273$$



Determining the Pyrolysis Kinetics from the experimental data :



Coal pyrolysis kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 4.632 \frac{1}{\text{s}}$

Activation Energy : $-E_{\text{onR}} \cdot \frac{R}{10^3} = 11.586 \frac{\text{kJ}}{\text{mol}}$

$-E_{\text{onR}} = 1393.503 \text{ K}$

Function describing the pyrolysis reaction rate : $k_1(T_p) = k_{0.1} \exp\left(\frac{E_{\text{onR}}}{T_p}\right)$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation
$$dT_{\text{rad}}(T_p, T_w) = \frac{\epsilon \sigma A_{\text{part}} (T_w^4 - T_p^4)}{C_p \text{ coal}^m p}$$

convection
$$dT_{\text{conv}}(T_p, T_g) = \frac{\text{Nu} \cdot A_{\text{part}} \lambda(T_p, T_g) \left[\left(\frac{T_g + T_p}{2} \right) - T_p \right]}{C_p \text{ coal}^m p}$$

Estimation of the initial particle heating rate at DTF setpoint temperature of 1400°C:

$T_{\text{init}} = 25 + 273 \text{ K}$ $dT_{\text{rad}}(T_{\text{init}}, 1416) + dT_{\text{conv}}(T_{\text{init}}, 1335) = 9.984 \cdot 10^3 \frac{\text{K}}{\text{s}}$

Solution of differential equations

Initial Conditions :

Volatiles (%) $x_0 = 1$

Particle Temperature $x_1 = 300$

Simultaneous Differential Equations :

$$\frac{d}{dt} V = -k_1(x_1)$$

$$\frac{d}{dt} T_p = dT_{\text{rad}}(x_0, x_1) + dT_{\text{conv}}(x_0, x_1)$$

Note on Solution Methodology :

The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature

Hendrina R.C. Composite : 38-75 μm

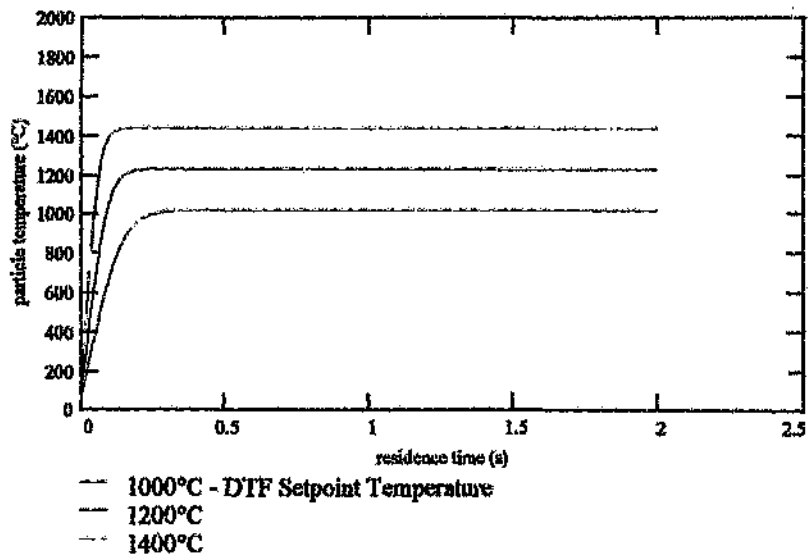
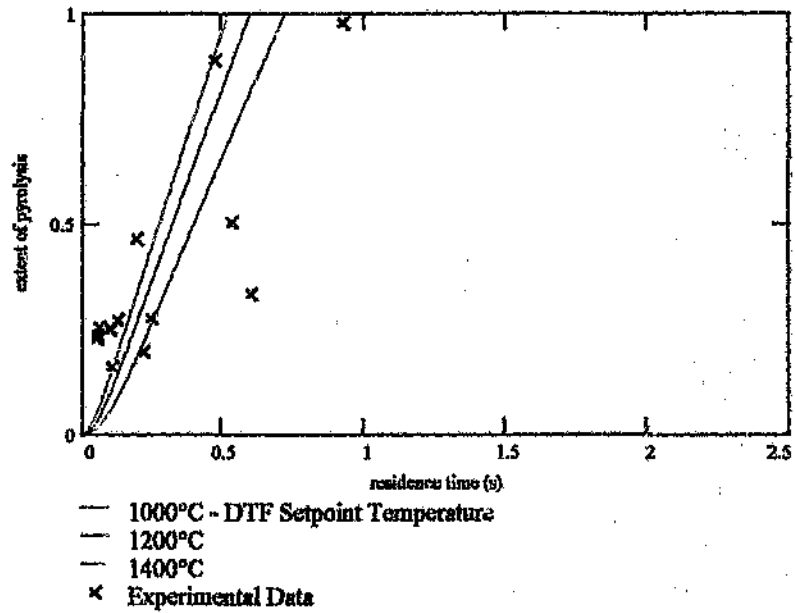
DTF Pyrolysis Evaluation Results :

Coal pyrolysis kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 4.632 \quad \frac{1}{s}$

Activation Energy : $-\text{EonR} \cdot \frac{R}{10^3} = 11.586 \quad \frac{\text{kJ}}{\text{mol}} \quad -\text{EonR} = 1393.503 \text{ K}$

Function describing the pyrolysis reaction rate : $k_1(T_p) = k_{0.1} \cdot \exp\left(\frac{\text{EonR}}{T_p}\right)$



Hendrina R.C. Composite : 38-75 μm

Appendix 2.1.3

Hendrina R.C. Composite: Determination of the Char Combustion Kinetics

Determination of DTF combustion kinetics for use with AIOLOS combustion code

Universal Gas Constant $R = 8.314 \frac{\text{J}}{\text{mol}\cdot\text{K}}$

DTF Operation and Char Description :

Ash in Coal $\text{Ash}_{\text{coal}} = \text{DTF}_{0.9} \%$

$\text{Ash}_{\text{char}} = \text{DTF}_{0.7} \%$

Ash in Char

Volatile Matter (after DTF devolatilisation)

$\text{Vols}_{\text{DTF}} = 1 - \text{Ash}_{\text{coal}} - \frac{\text{Ash}_{\text{coal}}}{\text{Ash}_{\text{char}}} (1 - \text{Ash}_{\text{char}})$

Density of coal :

$\rho_{\text{coal}} = 1678 \frac{\text{kg}}{\text{m}^3}$

Density of Char (initially) :
(shrinking core assumption)

$\rho_{\text{char},0} = (1 - \text{Vols}_{\text{DTF}}) \cdot \rho_{\text{coal}} \quad \rho_{\text{char},0} = 1.204 \cdot 10^3 \frac{\text{kg}}{\text{m}^3}$

Initial Particle Diameter :

$dp_{0,0} = \text{DTF}_{0,0} \quad \text{m} \quad dp_{0,0} = 2.18 \cdot 10^{-3} \quad \text{m}$

Pressure in DTF :

$P = 0.85 \cdot 10^5 \quad \text{Pa}$

Volume Oxygen Flowrate into DTF :

$V_{\text{O}_2} = \frac{0.6 \cdot 10^3}{60} \frac{\text{m}^3}{\text{s}} \quad M_{\text{W},\text{O}_2} = 32 \cdot 10^{-3} \frac{\text{kg}}{\text{mol}}$

Volume Nitrogen Flowrate into DTF :

$V_{\text{N}_2} = \frac{19.66 \cdot 10^3}{60} \frac{\text{m}^3}{\text{s}} \quad M_{\text{W},\text{N}_2} = 28 \cdot 10^{-3} \frac{\text{kg}}{\text{mol}}$

Mass Flowrate of Oxygen :

$m_{\text{O}_2} = \frac{V_{\text{O}_2} \cdot P \cdot M_{\text{W},\text{O}_2}}{R \cdot 298} \frac{\text{kg}}{\text{s}}$

Mass Flowrate of Nitrogen :

$m_{\text{N}_2} = \frac{V_{\text{N}_2} \cdot P \cdot M_{\text{W},\text{N}_2}}{R \cdot 298} \frac{\text{kg}}{\text{s}}$

Char Feedrate :

$m_{\text{char}} = \frac{0.1066 \cdot 10^3}{60} \frac{\text{kg}}{\text{s}}$

Initial Char Concentration

$C_{\text{char},0} = \frac{m_{\text{char}}}{m_{\text{char}} + m_{\text{O}_2} + m_{\text{N}_2}} \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

Initial Fixed Carbon Concentration

$C_{\text{c},0} = C_{\text{char},0} (1 - \text{Ash}_{\text{char}}) \quad C_{\text{c},0} = 3.4659 \cdot 10^{-3} \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

Ash Concentration :

$C_{\text{ash}} = C_{\text{char},0} (\text{Ash}_{\text{char}}) \quad C_{\text{ash}} = 1.9587 \cdot 10^{-3} \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

Heat Exchange Parameters and constants

Stefan Boltzmann constant

$$\sigma := 5.67 \cdot 10^{-8} \frac{\text{W}}{\text{m}^2 \cdot \text{K}^4}$$

average emissivity between particle and wall

$$\epsilon := 0.85$$

reaction energy : C to CO :

$$\Delta h := 9.614 \cdot 10^6 \frac{\text{J}}{\text{kg}}$$

Nusselt Number for boundary layer convective heatexchange

$$\text{Nu} := 2$$

convective heat exchange coefficient

$$h(T_p, T_g) := 2.43 \cdot 10^{11} \left[\frac{(T_p + T_g)}{273} \right]^{0.82} \frac{\text{W}}{\text{m} \cdot \text{K}}$$

heat capacity of char :

$$C_{p \text{ char}} := 1250 \frac{\text{J}}{\text{kg} \cdot \text{K}}$$

Particle Surface Area as a function of combustion extent :

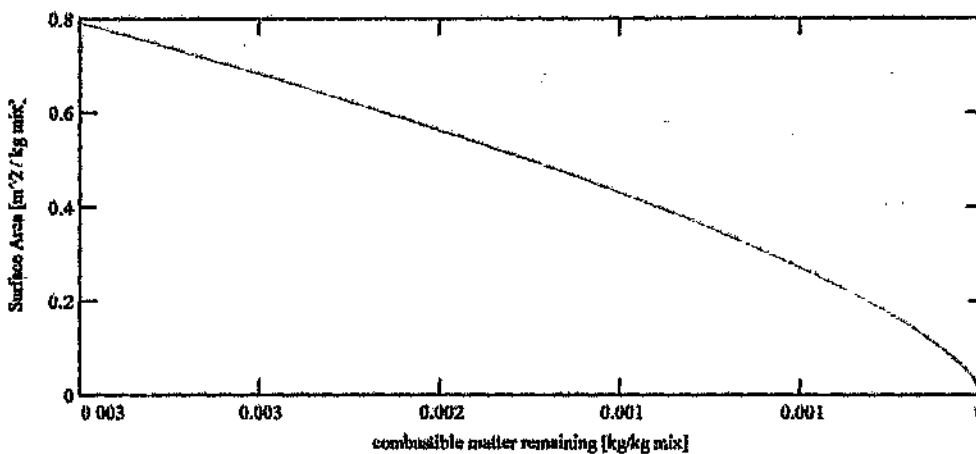
swelling Index

$$n_B := \frac{2}{3}$$

1 - swelling coals

2/3 - 'non swelling' coals

$$A_{\text{part.c}}(C_c) := \frac{6}{\rho_{\text{char.o}} \cdot d_{p_0}} \cdot C_{c,0} \cdot \left(\frac{C_c}{C_{c,0}} \right)^{n_B} \frac{\text{m}^2}{\text{kg mix}}$$



Partial Pressure of Oxygen in DTF : $P_{O_2} := \frac{V_{O_2}}{V_{O_2} + V_{N_2}} \quad P_{O_2} = 0.03 \quad \text{bar}$

Function defining mass of combustible matter remaining as a function ash content in the sampled residue

$$C_c(\text{ash}_{\text{residue}}) := \left[\frac{\text{Ash}_{\text{char}} \cdot 1 - \left(\frac{\text{ash}_{\text{residue}}}{100} \right)}{1 - \text{Ash}_{\text{char}} \cdot \left(\frac{\text{ash}_{\text{residue}}}{100} \right)} \right] \cdot C_{c,0}$$

ash_{residue}

in % by mass

Calculating Overall Rate :

$$k_{o,k} = \frac{\int_{C_{c,0}}^{C_c(DTF_{k,4})} \frac{1}{A_{part,c}(C_c)} dC_c}{-P_{O_2} DTF_{k,3}}$$

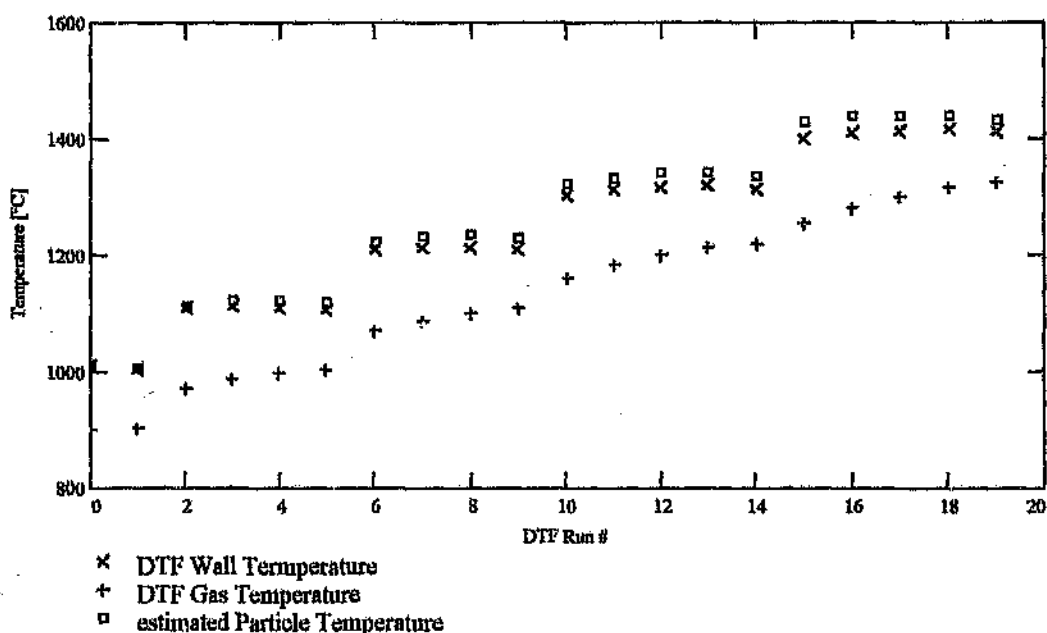
Calculating Particle Temperature:

Iterative solution of Energy Balance between particle, hot gas and reactor wall :

$$\Delta h \cdot \frac{C_c(Ash_{res}) - C_{c,0}}{t_r} = \left[\epsilon \cdot \sigma \cdot (T_w^4 - T_p^4) - \frac{Nu \cdot \lambda(T_p, T_g)}{dp_o} \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \right] \cdot A_{part,c}(C_{c,0})$$

$$T_{particle}(Ash_{res}, t_r, T_w, T_g) := \text{find}(T_p)$$

$$DTF_{k,6} := T_{particle}(DTF_{k,4}, DTF_{k,3}, DTF_{k,2} + 273, DTF_{k,5} + 273) - 273$$



Oxygen Diffusion reaction rate as a function particle Temperature

O2 diffusion constant at 1600K : $D_{O_2,1600K} = 3.39 \cdot 10^{-4} \frac{m^2}{s}$

mechanism factor : $\phi = 2$ $\phi = 2$ for CO, and 1 for CO2

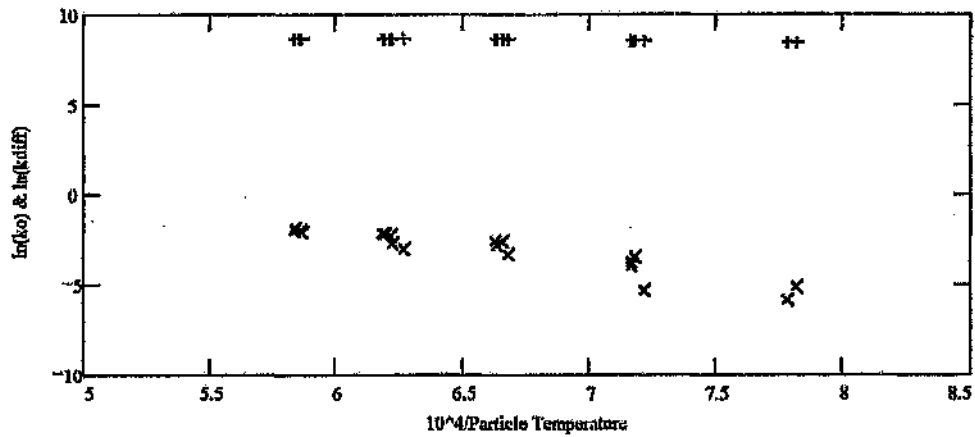
$$k_{d,char}(T_p, T_g, dp) = \frac{24 \phi D_{O_2,1600K} \left[\frac{(T_g + T_p) \cdot 0.5}{1600} \right]^{1.75}}{R \cdot 10^5 \cdot dp \cdot T_g} \frac{kg}{m^2 \cdot s \cdot bar}$$

Source : AIOLOS - combustion model

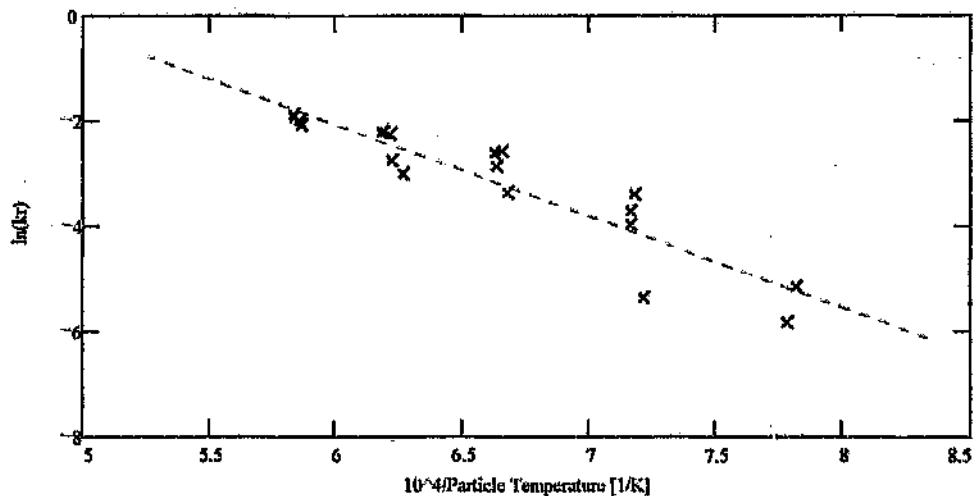
Hendrina R.C. Composite : < 38 μm

Determining the Chemical Reaction Rate from the experimental data :

$$kr_k = \left[\frac{1}{\left(\frac{1}{k_{o,k}} \right) - \frac{1}{k_{d,char}(DTF_{k,0} + 273, DTF_{k,5} + 273, DTF_{k,0})}} \right]$$



NB : if the '+'s lie beneath the 'x's, the overall reaction rate could be diffusion controlled



x DTF experimental
 - - kinetic model - predicted

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 4269.228$ $\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $E_{onR} \cdot \frac{R}{10^3} = 144.495$ $\frac{\text{kJ}}{\text{mol}}$

$E_{onR} = 17379.72$ K

Function describing chemical reaction rate : $k_{r,char}(T_p) := k_{r,char} \cdot \exp\left(\frac{E_{onR}}{T_p}\right)$

Defining the overall reaction rate function :

$$k_r(T_p, T_g, dp) = \frac{1}{\left(\frac{1}{k_{r,char}(T_p)} + \frac{1}{k_{d,char}(T_p, T_g, dp)} \right)}$$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation

$$dT_{rad}(C_c, T_p, T_w) = \frac{\varepsilon \sigma (T_w^4 - T_p^4)}{(C_c + C_{ash}) \cdot C_{p,char}} \cdot A_{part,c}(C_{c,0})$$

convection

$$dT_{conv}(C_c, T_p, T_g) = \frac{Nu \cdot \lambda(T_p, T_g) \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right]}{dp_o \cdot (C_c + C_{ash}) \cdot C_{p,char}} \cdot A_{part,c}(C_{c,0})$$

reaction

$$dT_{gen}(C_c, T_p, T_g) = \frac{-\Delta h}{(C_c + C_{ash}) \cdot C_{p,char}} \cdot (k_r(T_p, T_g, dp_o) \cdot P_{O_2} \cdot A_{part,c}(C_c))$$

estimation of initial particle heating rate at DTF setpoint Temperature of 1400°C :

$$T_{init} = 2 \cdot + 273 \text{ K} \quad dT_{rad}(C_{c,0}, T_{init}, 1416 + 273) + dT_{conv}(C_{c,0}, T_{init}, 1335 + 273) = 4.579 \cdot 10^4 \frac{\text{K}}{\text{s}}$$

Solution of differential equations

Initial Conditions : combustion extent — $x_0 = C_{c,0}$

particle temperature — $x_1 = 300 \text{ K}$

Simultaneous Differential Equations :

combustion extent — $\frac{d}{dt} C_c = -k_r(x_1, T_g, dp_o) \cdot P_{O_2} \cdot A_{part,c}(x_0)$

particle temperature — $\frac{d}{dt} T_p = dT_{rad}(x_0, x_1, T_w) + dT_{conv}(x_0, x_1, T_g) + dT_{gen}(x_0, x_1, T_g)$

Note on the Solution Methodology

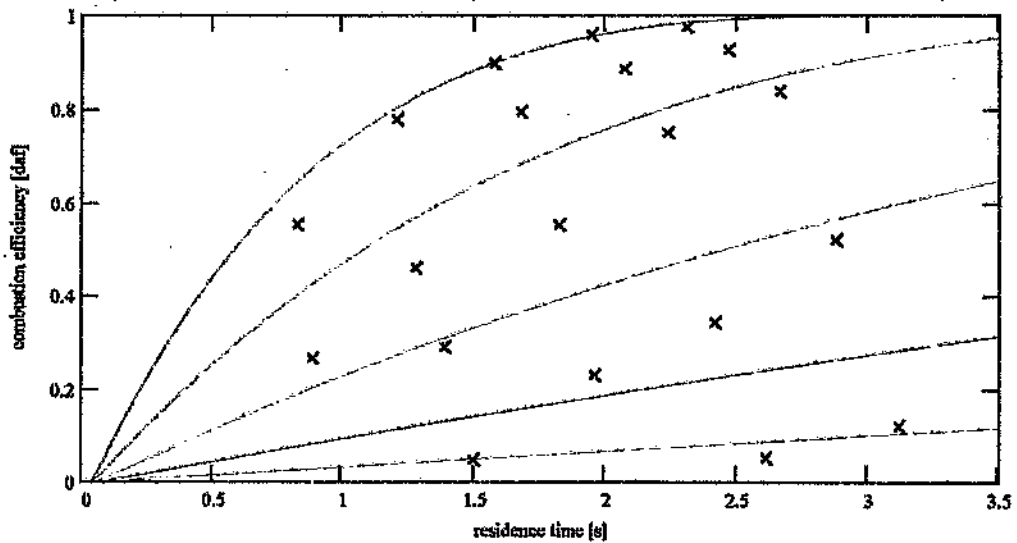
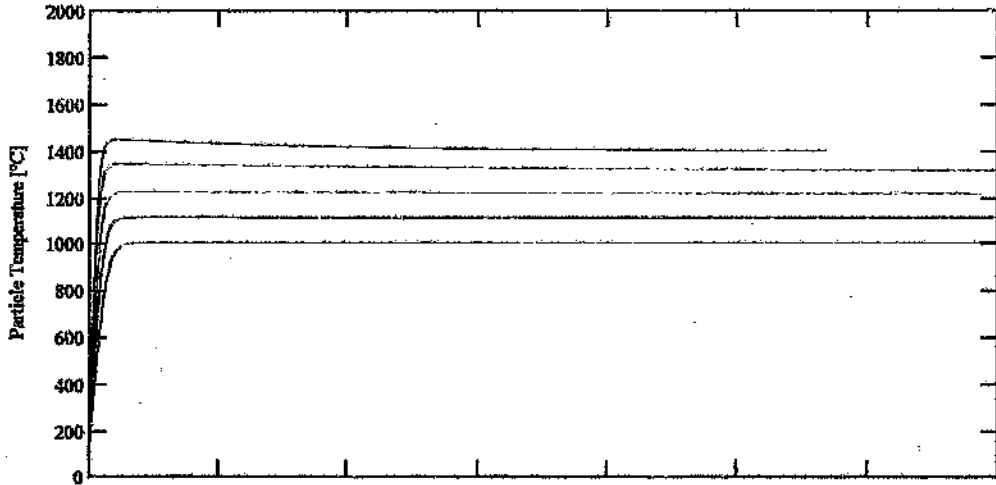
The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint Temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature.

DTF Kinetic Evaluation Results -

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 4269.228 \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $E_{\text{ouR}} \cdot \frac{R}{10^3} = 144.49 \frac{\text{kJ}}{\text{mol}}$ $E_{\text{onR}} = 17379.72 \text{ K}$



- 1000°C - DTF Setpoint Temperature
- 1100°C
- 1200°C
- 1300°C
- 1400°C
- × DTF data

Determination of DTF combustion kinetics for use with AIOLOS combustion code

Universal Gas Constant $R = 8.314 \frac{J}{mol \cdot K}$

DTF Operation and Char Description :

Ash In Coal

$$\text{Ash}_{\text{coal}} = \text{DTF}_{0,8} \cdot \%$$

$$\text{Ash}_{\text{char}} = \text{DTF}_{0,7} \cdot \%$$

Ash in Char

Volatile Matter (after DTF devolatilisation)

$$\text{Vols}_{\text{DTF}} = 1 - \text{Ash}_{\text{coal}} - \frac{\text{Ash}_{\text{coal}}}{\text{Ash}_{\text{char}}} (1 - \text{Ash}_{\text{char}})$$

Density of coal :

$$\rho_{\text{coal}} = 1678 \frac{\text{kg}}{\text{m}^3}$$

Density of Char (Initially) :
(shrinking core assumption)

$$\rho_{\text{char},0} := (1 - \text{Vols}_{\text{DTF}}) \cdot \rho_{\text{coal}} \quad \rho_{\text{char},0} = 1.189 \cdot 10^3 \frac{\text{kg}}{\text{m}^3}$$

Initial Particle Diameter :

$$dp_0 = \text{DTF}_{0,0} \quad \text{m} \quad dp_0 = 6 \cdot 10^{-5} \quad \text{m}$$

Pressure in DTF :

$$P = 0.85 \cdot 10^5 \quad \text{Pa}$$

Volume Oxygen Flowrate into DTF :

$$V_{\text{O}_2} := \frac{0.6 \cdot 10^3}{60} \frac{\text{m}^3}{\text{s}} \quad M_{\text{W},\text{O}_2} = 32 \cdot 10^{-3} \frac{\text{kg}}{\text{mol}}$$

Volume Nitrogen Flowrate into DTF :

$$V_{\text{N}_2} := \frac{19.66 \cdot 10^3}{60} \frac{\text{m}^3}{\text{s}} \quad M_{\text{W},\text{N}_2} = 28 \cdot 10^{-3} \frac{\text{kg}}{\text{mol}}$$

Mass Flowrate of Oxygen :

$$m_{\text{O}_2} := \frac{V_{\text{O}_2} \cdot P \cdot M_{\text{W},\text{O}_2}}{R \cdot 298} \frac{\text{kg}}{\text{s}}$$

Mass Flowrate of Nitrogen :

$$m_{\text{N}_2} := \frac{V_{\text{N}_2} \cdot P \cdot M_{\text{W},\text{N}_2}}{R \cdot 298} \frac{\text{kg}}{\text{s}}$$

Char Feedrate :

$$m_{\text{char}} = \frac{0.1066 \cdot 10^3}{60} \frac{\text{kg}}{\text{s}}$$

Initial Char Concentration

$$C_{\text{char},0} := \frac{m_{\text{char}}}{m_{\text{char}} + m_{\text{O}_2} + m_{\text{N}_2}} \frac{\text{kg}}{\text{kg mix}}$$

Initial Fixed Carbon Concentration

$$C_{\text{c},0} := C_{\text{char},0} (1 - \text{Ash}_{\text{char}}) \quad C_{\text{c},0} = 3.4407 \cdot 10^{-3} \frac{\text{kg}}{\text{kg mix}}$$

Ash Concentration :

$$C_{\text{ash}} := C_{\text{char},0} (\text{Ash}_{\text{char}}) \quad C_{\text{ash}} = 1.984 \cdot 10^{-3} \frac{\text{kg}}{\text{kg mix}}$$

Calculating Overall Rate :

$$k_{o,k} = \frac{\int_{C_{c,0}}^{C_c(DTF_{k,A})} \frac{1}{A_{part,c}(C_c)} dC_c}{P_{O_2} \cdot DTF_{k,3}}$$

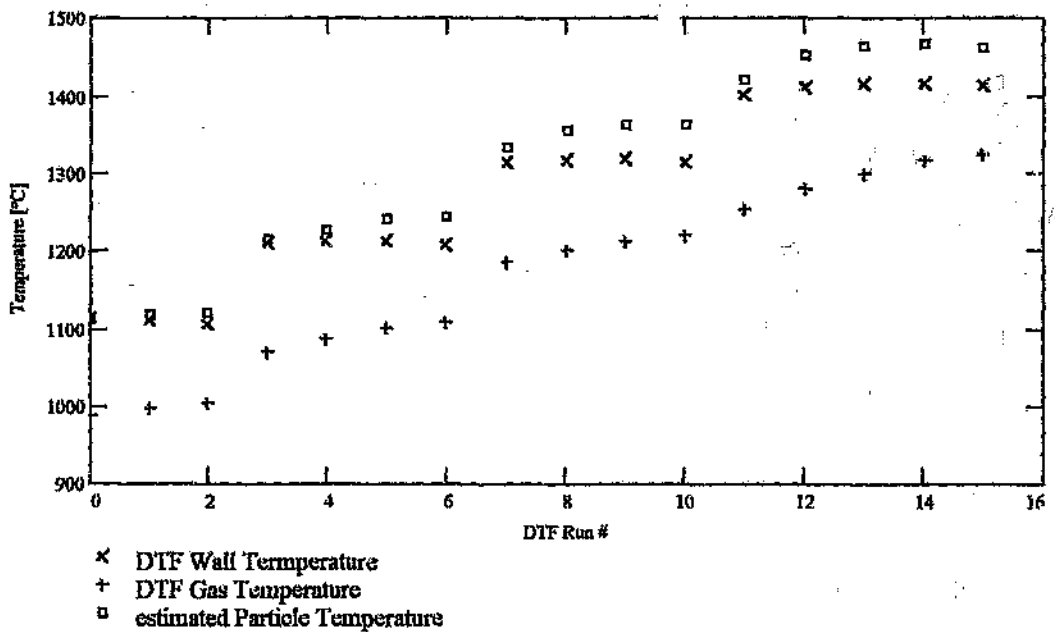
Calculating Particle Temperature:

Iterative solution of Energy Balance between particle, hot gas and reactor wall :

$$\Delta h \cdot \frac{C_c(Ash_{res}) - C_{c,0}}{t_r} = \left[\epsilon \cdot \sigma (T_w^4 - T_p^4) - \frac{Nu \cdot \lambda (T_p - T_g)}{dp_o} \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \right] \cdot A_{part,c}(C_{c,0})$$

$$T_{particle}(Ash_{res}, t_r, T_w, T_g) := \text{find}(T_p)$$

$$DTF_{k,6} := T_{particle}(DTF_{k,4}, DTF_{k,3}, DTF_{k,2} + 273, DTF_{k,5} + 273) - 273$$



Oxygen Diffusion reaction rate as a function particle Temperature

O2 diffusion constant at 1600K : $D_{O_2,1600K} = 3.39 \cdot 10^{-4} \frac{m^2}{s}$

mechanism factor : $\phi = 2$ $\phi = 2$ for CO, and 1 for CO2

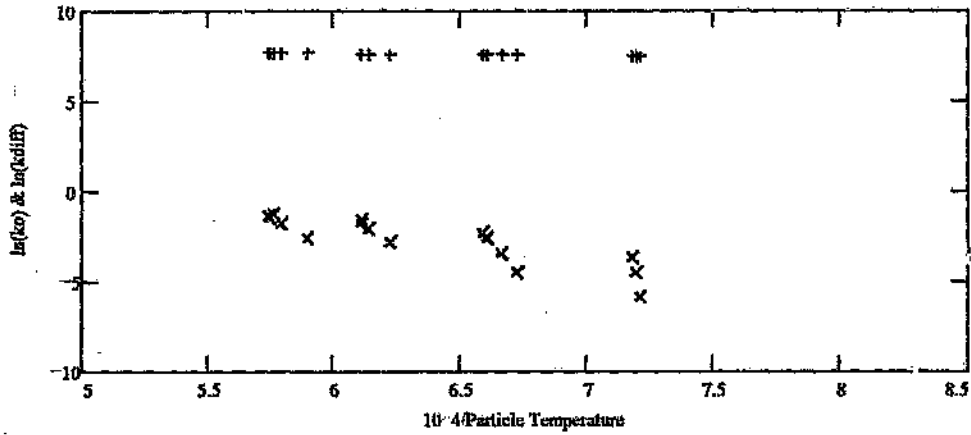
$$k_{d,char}(T_p, T_g, dp) = \frac{24 \cdot \phi \cdot D_{O_2,1600K} \cdot \left[\frac{(T_g + T_p) \cdot 0.5}{1600} \right]^{1.75}}{R \cdot 10^{-5} \cdot dp \cdot T_g} \quad \frac{kg}{m^2 \cdot s \cdot bar}$$

Source : AIOLOS - combustion model

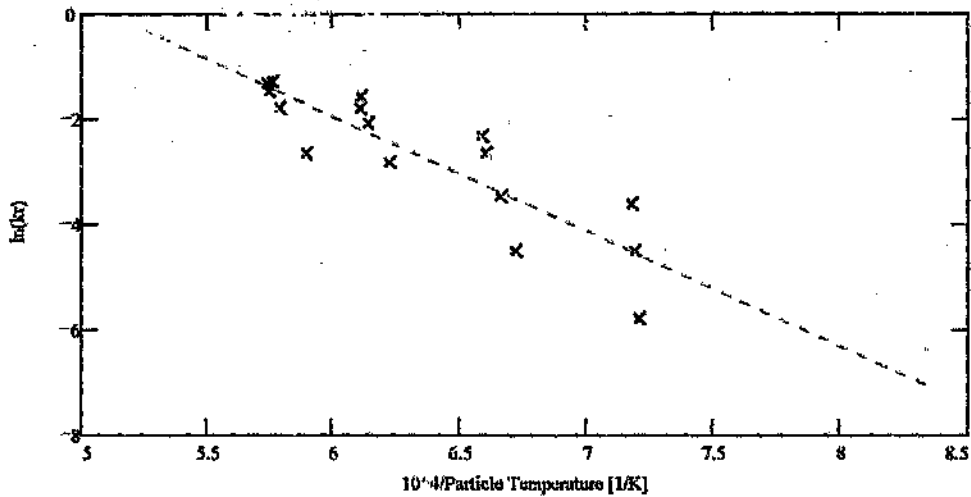
Hendrina R.C. Composite : 38-75 μm

Determining the Chemical Reaction Rate from the experimental data :

$$k_{r,k} = \left[\frac{1}{\left(\frac{1}{k_{o,k}} \right) + \frac{1}{k_{d,char}(DIF_{k,6} + 273, DIF_{k,5} + 273, DIF_{k,0})}} \right]$$



NB : if the +'s lie beneath the x's, the overall reaction rate could be diffusion controlled



x DIF experimental
 - - kinetic model - predicted

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 70549.225 \quad \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $E_{onR} \cdot \frac{R}{10^3} = 181.596 \quad \frac{\text{kJ}}{\text{mol}}$

$E_{onR} = 21842.138 \quad \text{K}$

Function describing chemical reaction rate : $k_{r,char}(T_p) = k_{r,0} \cdot \exp\left(\frac{E_{onR}}{T_p}\right)$

Defining the overall reaction rate function :

$$k_r(T_p, T_g, dp) = \frac{1}{\left(\frac{1}{k_{r,char}(T_p)} + \frac{1}{k_{d,char}(T_p, T_g, dp)} \right)}$$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation

$$dT_{rad}(C_c, T_p, T_w) = \frac{\epsilon \sigma (T_w^4 - T_p^4)}{(C_c + C_{ash}) \cdot C_{p, char}} \cdot A_{part,c}(C_{c,0})$$

convection

$$dT_{conv}(C_c, T_p, T_g) = \frac{Nu \cdot \lambda(T_p, T_g)}{dp_o} \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \cdot A_{part,c}(C_{c,0})$$

reaction

$$dT_{gen}(C_c, T_p, T_g) = \frac{-\Delta h}{(C_c + C_{ash}) \cdot C_{p, char}} \cdot (k_r(T_p, T_g, dp_o) \cdot P_{O_2} \cdot A_{part,c}(C_{c,0}))$$

estimation of initial particle heating rate at DTF setpoint Temperature of 1400°C :

$$T_{init} = 25 + 273 \text{ K} \quad dT_{rad}(C_{c,0}, T_{init}, 1416 + 273) + dT_{conv}(C_{c,0}, T_{init}, 1335 + 273) = 1.673 \cdot 10^4 \frac{\text{K}}{\text{s}}$$

Solution of differential equations

Initial Conditions : combustion extent — $x_0 = C_{c,0}$

particle temperature — $x_1 = 300 \text{ K}$

Simultaneous Differential Equations :

$$\text{combustion extent — } \frac{d}{dt} C_c = -k_r(x_1, T_g, dp_o) \cdot P_{O_2} \cdot A_{part,c}(x_0)$$

$$\text{particle temperature — } \frac{d}{dt} T_p = dT_{rad}(x_0, x_1, T_w) + dT_{conv}(x_0, x_1, T_g) + dT_{gen}(x_0, x_1, T_g)$$

Note on the Solution Methodology

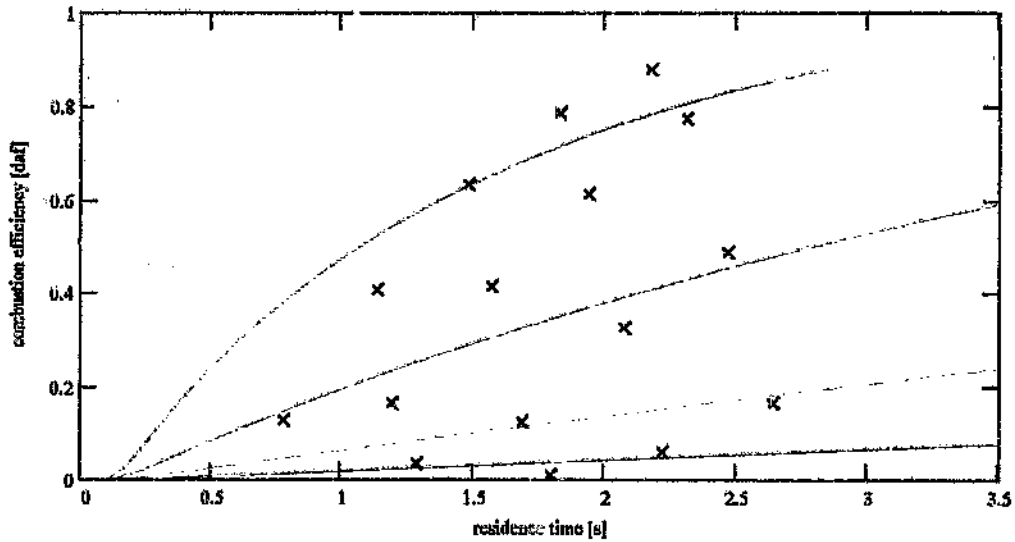
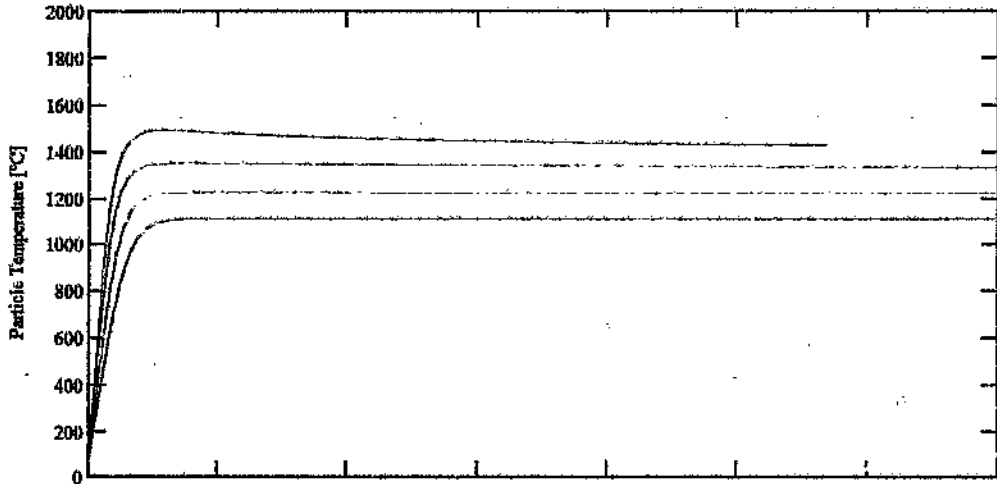
The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint Temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature.

DTF Kinetic Evaluation Results -

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 70549.225 \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $\cdot E_{onR} \cdot \frac{R}{10^3} = 181.60 \frac{\text{kJ}}{\text{mol}}$ $\cdot E_{onR} = 21842.138 \text{ K}$



- 1000°C - DTF Setpoint Temperature
- 1100°C
- 1200°C
- 1300°C
- 1400°C
- x DTF data

Determination of DTF combustion kinetics for use with AIOLOS combustion code

Universal Gas Constant $R := 8.314 \frac{\text{J}}{\text{mol}\cdot\text{K}}$

DTF Operation and Char Description :

Ash in Coal $\text{Ash}_{\text{coal}} := \text{DTF}_{0,8} \cdot \%$

Ash char $\text{Ash}_{\text{char}} := \text{DTF}_{0,7} \cdot \%$

Ash in Char

Volatile Matter (after DTF devolatilisation)

$\text{Vols}_{\text{DTF}} := 1 - \text{Ash}_{\text{coal}} - \frac{\text{Ash}_{\text{coal}}}{\text{Ash}_{\text{char}}} \cdot (1 - \text{Ash}_{\text{char}})$

Density of coal :

$\rho_{\text{coal}} := 1678 \frac{\text{kg}}{\text{m}^3}$

Density of Char (initially) :
(shrinking core assumption)

$\rho_{\text{char},0} := (1 - \text{Vols}_{\text{DTF}}) \cdot \rho_{\text{coal}} \quad \rho_{\text{char},0} = 1.189 \cdot 10^3 \frac{\text{kg}}{\text{m}^3}$

Initial Particle Diameter :

$dp_o := \text{DTF}_{0,0} \quad \text{m} \quad dp_o = 6 \cdot 10^{-5} \quad \text{m}$

Pressure in DTF :

$P := 0.85 \cdot 10^5 \quad \text{Pa}$

Volume Oxygen Flowrate into DTF :

$V_{\text{O}_2} := \frac{0.6 \cdot 10^{-3}}{60} \frac{\text{m}^3}{\text{s}} \quad M_{\text{W},\text{O}_2} := 32 \cdot 10^{-3} \frac{\text{kg}}{\text{mol}}$

Volume Nitrogen Flowrate into DTF :

$V_{\text{N}_2} := \frac{19.66 \cdot 10^{-3}}{60} \frac{\text{m}^3}{\text{s}} \quad M_{\text{W},\text{N}_2} := 28 \cdot 10^{-3} \frac{\text{kg}}{\text{mol}}$

Mass Flowrate of Oxygen :

$m_{\text{O}_2} := \frac{V_{\text{O}_2} \cdot P \cdot M_{\text{W},\text{O}_2}}{R \cdot 298} \quad \frac{\text{kg}}{\text{s}}$

Mass Flowrate of Nitrogen :

$m_{\text{N}_2} := \frac{V_{\text{N}_2} \cdot P \cdot M_{\text{W},\text{N}_2}}{R \cdot 298} \quad \frac{\text{kg}}{\text{s}}$

Char Feedrate :

$m_{\text{char}} := \frac{0.1066 \cdot 10^{-3}}{60} \quad \frac{\text{kg}}{\text{s}}$

Initial Char Concentration

$C_{\text{char},0} := \frac{m_{\text{char}}}{m_{\text{char}} + m_{\text{O}_2} + m_{\text{N}_2}} \quad \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

Initial Fixed Carbon Concentration

$C_{c,0} := C_{\text{char},0} \cdot (1 - \text{DTF}_{0,7} \cdot \%) \quad C_{c,0} = 3.4407 \cdot 10^{-3} \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

$C_{c,1} := C_{\text{char},0} \cdot (1 - \text{DTF}_{13,7} \cdot \%) \quad C_{c,1} = 3.4407 \cdot 10^{-3} \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

Ash Concentration :

$C_{\text{ash}} := C_{\text{char},0} \cdot (\text{Ash}_{\text{char}}) \quad C_{\text{ash}} = 1.984 \cdot 10^{-3} \frac{\text{kg}}{\text{kg}_{\text{mix}}}$

Heat Exchange Parameters and constants

Stefan Boltzmann constant $\sigma := 5.67 \cdot 10^{-8} \frac{W}{m^2 \cdot K^4}$

average emissivity between particle and wall $\epsilon := 0.85$

reaction energy : C to CO : $\Delta h := 9.614 \cdot 10^6 \frac{J}{kg}$

Nusselt Number for boundary layer convective heatexchange $Nu := 2$

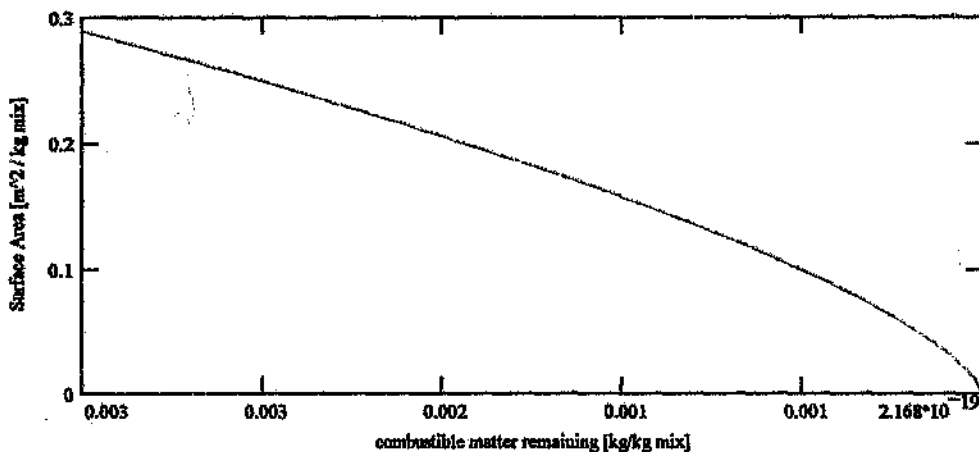
convective heat exchange coefficient $\lambda(T_p, T_g) := 2.43 \cdot 10^{-5} \cdot \left[\frac{(T_p + T_g)}{273} \right]^{0.82} \frac{W}{m \cdot K}$

heat capacity of char : $C_{p \text{ char}} := 1250 \frac{J}{kg \cdot K}$

Particle Surface Area as a function of combustion extent :

swelling index $n_B := \frac{2}{3}$ 1 - swelling coals
 2/3 - 'non swelling' coals

$$A_{\text{part.c}}(C_c) := \frac{6}{\rho_{\text{char.o}} \cdot d_{p_o}} \cdot C_{c,0} \cdot \left(\frac{C_c}{C_{c,0}} \right)^{n_B} \frac{m^2}{kg \text{ mix}}$$



Partial Pressure of Oxygen in DTF : $P_{O_2} := \frac{V_{O_2}}{V_{O_2} + V_{N_2}} \quad P_{O_2} = 0.03 \quad \text{bar}$

Function defining mass of combustible matter remaining as a function ash content in the sampled residue

$$C_c(\text{ash}_{\text{residue}}) := \left[\frac{\text{Ash}_{\text{char}} \cdot 1 - \left(\frac{\text{ash}_{\text{residue}}}{100} \right)}{1 - \text{Ash}_{\text{char}} \cdot \left(\frac{\text{ash}_{\text{residue}}}{100} \right)} \right] \cdot C_{c,0}$$

ash_{residue} in % by mass

Calculating Overall Rate :

$$k_{O_2} := \frac{\int_{C_{O_2}(DIF_{k,7})}^{C_{O_2}(DIF_{k,4})} \frac{1}{A_{part,c}(C_{O_2})} dC_{O_2}}{-P_{O_2} \cdot DIF_{k,3}}$$

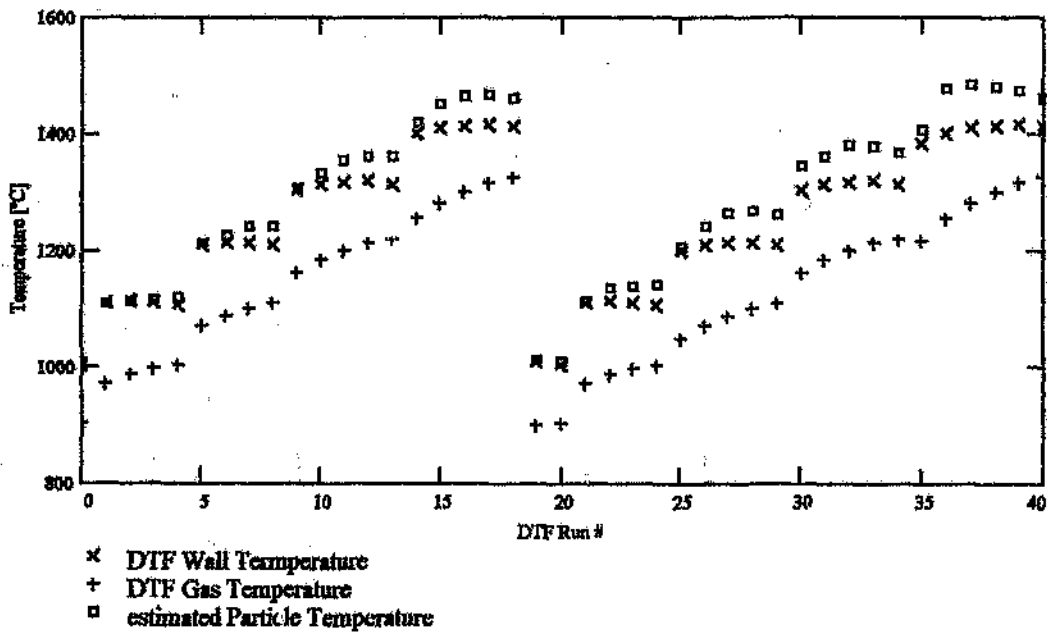
Calculating Particle Temperature:

Iterative solution of Energy Balance between particle, hot gas and reactor wall :

$$\Delta h \cdot \frac{C_{c,Ash, res} - C_{c,0}}{t_r} = \left[\varepsilon \sigma (T_w^4 - T_p^4) - \frac{Nu \cdot \lambda(T_p, T_g)}{dp} \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \right] \cdot A_{part,c}(C_{c,0})$$

$$T_{particle}(Ash_{res}, t_r, T_w, T_g, dp) := \text{find}(T_p)$$

$$DIF_{k,6} := T_{particle}(DIF_{k,4}, DIF_{k,3}, DIF_{k,2} + 273, DIF_{k,5} + 273, DIF_{k,0}) - 273$$



Oxygen Diffusion reaction rate as a function particle Temperature

O2 diffusion constant at 1600K : $D_{O_2,1600K} := 3.39 \cdot 10^{-4} \frac{m^2}{s}$

mechanism factor : $\phi = 2$ $\phi = 2$ for CO, and 1 for CO2

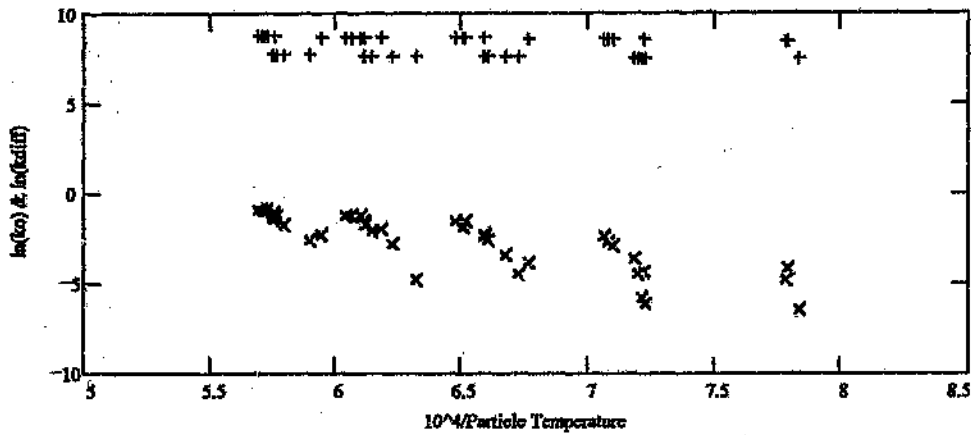
$$k_{d,char}(T_p, T_g, dp) := \frac{24 \cdot \phi \cdot D_{O_2,1600K} \cdot \left[\frac{(T_g + T_p) \cdot 0.5}{1600} \right]^{1.75}}{R \cdot 10^{-3} \cdot dp \cdot T_g} \frac{kg}{m^2 \cdot s \cdot bar}$$

Source : AIOLOS - combustion model

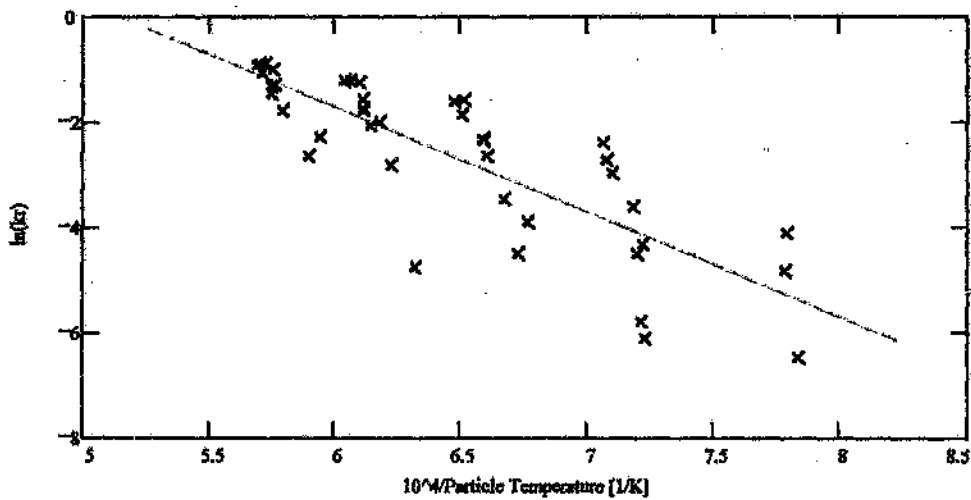
Hendrina RC Composite : <75 μm

Determining the Chemical Reaction Rate from the experimental data :

$$kr_k := \left[\frac{1}{\left(\frac{1}{k_{o,k}} \right) + \frac{1}{k_{d,char}(DTF_{k,6} + 273, DTF_{k,5} + 273, DTF_{k,0})}} \right]$$



NB : if the '+'s lie beneath the 'x's, the overall reaction rate could be diffusion controlled



× DTF experimental
 - - - kinetic model - predicted

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 28418.03 \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $-E_{onR} \cdot \frac{R}{10^3} = 165.716 \frac{\text{kJ}}{\text{mol}}$

$-E_{onR} = 19932.114 \text{ K}$

Function describing chemical reaction rate : $k_{r,char}(T_p) := k_{r,char,0} \cdot \exp\left(\frac{E_{onR}}{T_p}\right)$

DTF Kinetic Evaluation Results -

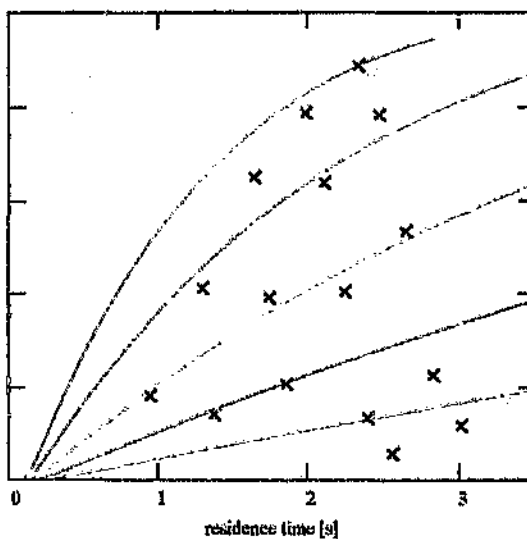
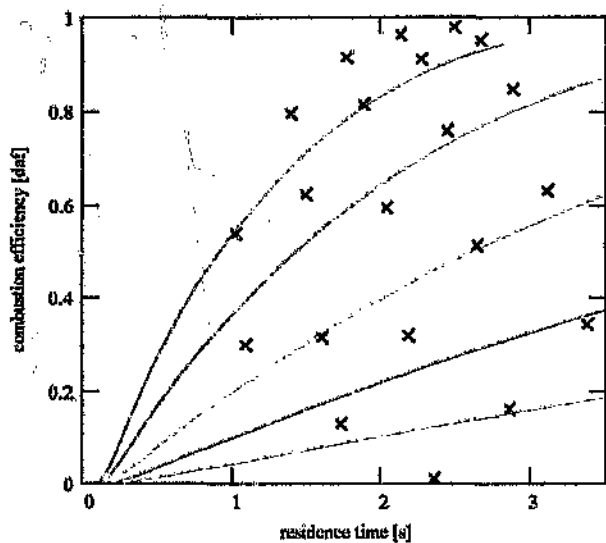
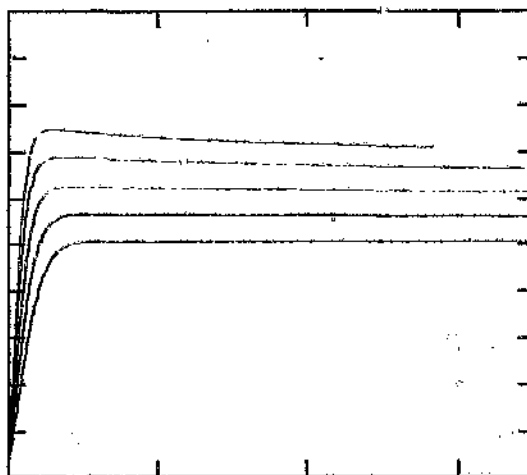
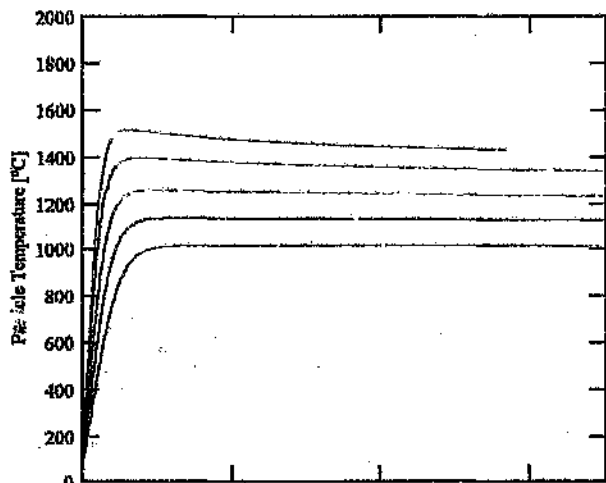
Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 282.185 \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $-E_{\text{onR}} \cdot \frac{R}{10^3} = 99.45 \frac{\text{kJ}}{\text{mol}}$ $-E_{\text{onR}} = 11961.304 \text{ K}$

<38 μm

38-75 μm



- 1000°C - DTF Setpoint Temperature
- 1100°C
- 1200°C
- 1300°C
- 1400°C
- x DTF data

Defining the overall reaction rate function :

$$k_r(T_p, T_g, dp) := \frac{1}{\left(\frac{1}{k_{r,char}(T_p)} + \frac{1}{k_{d,char}(T_p, T_g, dp)} \right)}$$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation

$$dT_{rad}(C_c, T_p, T_w, dp) := \frac{\varepsilon \sigma (T_w^4 - T_p^4)}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot \text{if}(dp > 38 \cdot 10^{-6}, A_{part,c}(C_{c,1}) \cdot A_{part,c}(C_{c,0}))$$

convection

$$dT_{conv}(C_c, T_p, T_g, dp) := \frac{Nu \cdot \lambda(T_p, T_g) \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right]}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot A_{part,c}(C_{c,0})$$

reaction

$$dT_{gen}(C_c, T_p, T_g, dp) := \frac{-\Delta h}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot (-k_r(T_p, T_g, dp) \cdot P_{O_2} \cdot A_{part,c}(C_c))$$

estimation of initial particle heating rate at DTF setpoint Temperature of 1400°C :

$$dT_{rad}(C_{c,0}, 298, DTF_{s,2} + 273, DTF_{s,0}) + dT_{conv}(C_{c,0}, 298, DTF_{s,3} + 273, DTF_{s,0}) = 1.660 \cdot 10^4 \frac{K}{s}$$

Solution of differential equations

Initial Conditions : combustion extent — $x_0 := C_c(DTF_{0,7})$

particle temperature — $x_1 := 300 \text{ K}$

Simultaneous Differential Equations :

$$\text{combustion extent} \text{ --- } \frac{d}{dt} C_c = -k_r(x_1, T_g, dp_{initial}) \cdot P_{O_2} \cdot A_{part,c}(x_0)$$

$$\text{particle temperature} \text{ --- } \frac{d}{dt} T_p = dT_{rad}(x_0, x_1, T_w) + dT_{conv}(x_0, x_1, T_g) + dT_{gen}(x_0, x_1, T_g)$$

Note on the Solution Methodology

The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint Temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature.

DTF Kinetic Evaluation Results -

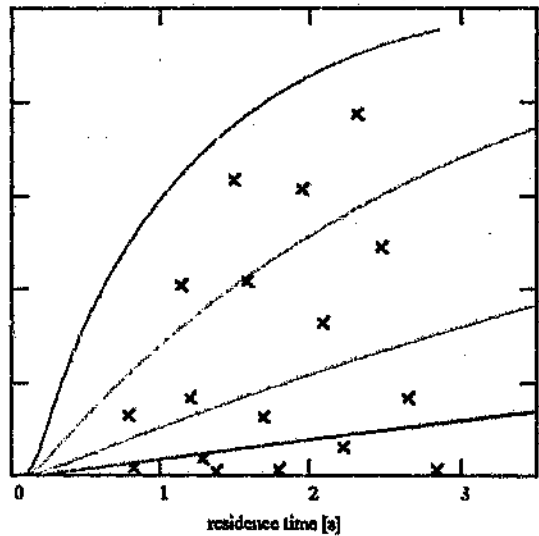
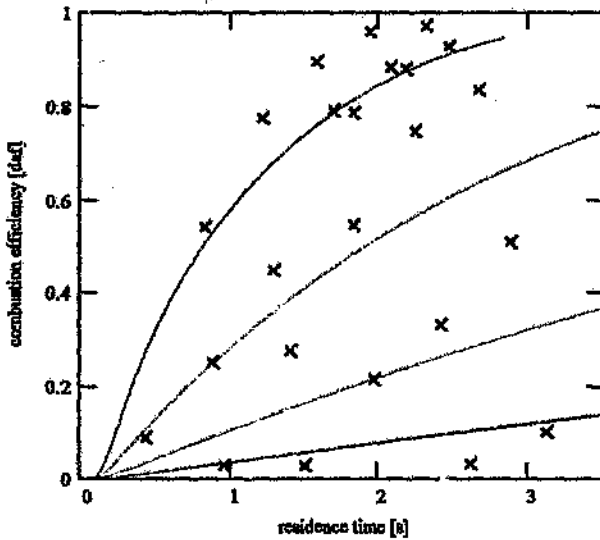
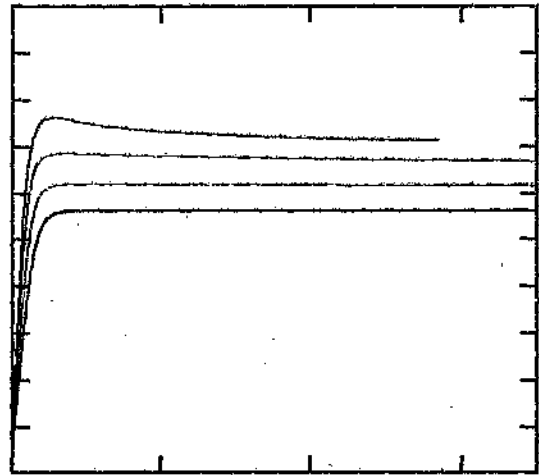
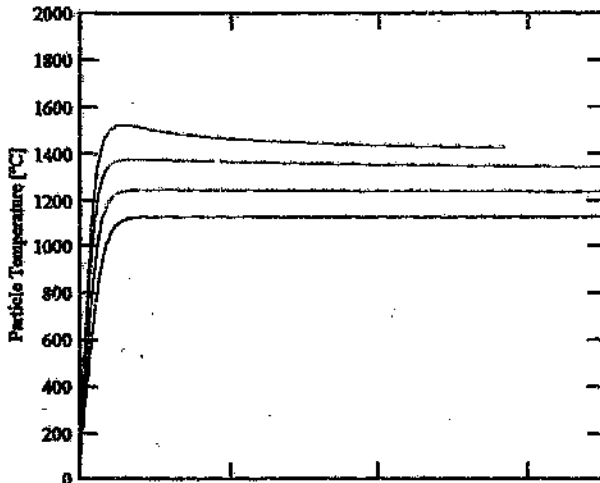
Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 28418.03$ $\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $-\text{EonR} \cdot \frac{R}{10^3} = 165.72$ $\frac{\text{kJ}}{\text{mol}}$ $-\text{EonR} = 19932.114$ K

<38 μm

38-75 μm



- 1000°C - DTF Setpoint Temperature
- 1100°C
- 1200°C
- 1300°C
- 1400°C
- x DTF data

Appendix 2.2.1

Hendrina Washing Plant Feed: Basic Analysis Datasheet

COAL :

Hendrina - Washing Plant Feed

Preparation of Coal Analysis Data for specification in AIOLOS

Analysis by TRI Coal Labs

all entries in mass% except where otherwise noted

Total Moisture : (as received)	8.7			
Proximate :	air dried	as received	dry	daf
Moisture	2	8.70		
	29.6	27.62	30.20	
	24.4	22.77	24.90	35.67
Fixed Carbon	44	40.92	44.90	64.33
Ultimate :				
Carbon	56.24	0.52	57.39	82.22
Hydrogen	3.34	0.03	3.41	4.88
Nitrogen	1.4	0.01	1.43	2.05
Sulphur	0.84	0.01	0.86	1.23
Carbonates	0.53	0.00	0.54	0.77
Oxygen	6.05	0.06	6.17	8.85
Oxygen + Carbonates		0.06		
Calorific Value	22	20.53	22.45	32.16
DTF DEVOL				
Volatile Matter	29.92			
Q Factor	1.226			
DTF FULL TEST				
Size Fraction	<38 µm	38-75 µm	< 75 µm	Unit
Pre-exponential Factor	208	1077	286	kg/m ² .s.atm
	205	1063	262	kg/m ² .s.bar
Activation Energy	101.68	124.9	99.45	kJ/mol
Activation Energy /R	12228	15023	11962	K

Coal Name: HENDRINA
 Origin: Hendrina P/S
 Coal Preparation:

Table :

PROXIMATE ANALYSIS			METHOD
Inherent Moisture:	2	(m% of coal - air dried)	ESKOM 103
Ash :	29.6		ESKOM 101
Volatile Matter :	24.4		ESKOM 102
Fixed Carbon (by difference) :	44		ESKOM 128
ULTIMATE ANALYSIS			
Carbon:	56.24	(m% of coal - air dried)	ESKOM 118
Hydrogen:	3.34		ESKOM 116
Nitrogen:	1.4		ESKOM 118
Total Sulphur:	0.84		ESKOM 104
Carbonates (as CO ₂):	0.53		ESKOM 100
Oxygen (by difference):	6.05		ESKOM 132
Gross Calorific Value:	22	(MJ/kg)	ESKOM 105
HARDGROVE INDEX:	-1		ESKOM 117
ABRASIVENESS:	-1	(mg Fe/kg coal)	ESKOM 123
ASH FUSION TEMPERATURES			ESKOM 125
Initial Deformation Temperature:	-1	(°C)	
Softening Temperature:	-1		
Hemispherical Temperature:	-1		
Flow Temperature:	-1		

'-1' indicates that the result of that analysis is not available
 RAW COAL PREPARATION BY SABS 0135 PART 2 - 1977

Appendix 2.2.2

Hendrina Washing Plant Feed: Determination of the Char Combustion Kinetics

Determination of DTF combustion kinetics for use with AIOLOS combustion code

Universal Gas Constant $R = 8.314 \frac{J}{mol \cdot K}$

DTF Operation and Char Description :

Ash in Coal $Ash_{coal} = DTF_{0.8} \cdot \%$

$Ash_{char} = DTF_{0.7} \cdot \%$

Ash in Char

Volatile Matter (after DTF devolatilisation)

$Vols_{DTF} = 1 - Ash_{coal} \cdot \frac{Ash_{coal}}{Ash_{char}} \cdot (1 - Ash_{char})$

Density of coal :

$\rho_{coal} = 1678 \frac{kg}{m^3}$

Density of Char (initially) :
(shrinking core assumption)

$\rho_{char,0} = (1 - Vols_{DTF}) \cdot \rho_{coal} \quad \rho_{char,0} = 1.179 \cdot 10^3 \frac{kg}{m^3}$

Initial Particle Diameter :

$dp_0 = DTF_{0,0} \quad m \quad dp_0 = 2.41 \cdot 10^{-3} \quad m$

Pressure in DTF :

$P = 0.85 \cdot 10^5 \quad Pa$

Volume Oxygen Flowrate into DTF :

$V_{O_2} = \frac{0.6 \cdot 10^{-3}}{60} \frac{m^3}{s} \quad M_{W,O_2} = 32 \cdot 10^{-3} \frac{kg}{mol}$

Volume Nitrogen Flowrate into DTF :

$V_{N_2} = \frac{10.66 \cdot 10^{-3}}{60} \frac{m^3}{s} \quad M_{W,N_2} = 28 \cdot 10^{-3} \frac{kg}{mol}$

Mass Flowrate of Oxygen :

$m_{O_2} = \frac{V_{O_2} \cdot P \cdot M_{W,O_2}}{R \cdot 298} \frac{kg}{s}$

Mass Flowrate of Nitrogen :

$m_{N_2} = \frac{V_{N_2} \cdot P \cdot M_{W,N_2}}{R \cdot 298} \frac{kg}{s}$

Char Feedrate :

$m_{char} = \frac{0.1066 \cdot 10^{-3}}{60} \frac{kg}{s}$

Initial Char Concentration

$C_{char,0} = \frac{m_{char}}{m_{char} + m_{O_2} + m_{N_2}} \frac{kg}{kg_{mix}}$

Initial Fixed Carbon Concentration

$C_{c,0} = C_{char,0} \cdot (1 - Ash_{char}) \quad C_{c,0} = 3.0928 \cdot 10^{-3} \frac{kg}{kg_{mix}}$

Ash Concentration :

$C_{ash} = C_{char,0} \cdot (Ash_{char}) \quad C_{ash} = 2.3318 \cdot 10^{-3} \frac{kg}{kg_{mix}}$

Calculating Overall Rate .

$$k_{o,k} = \frac{\int_{C_{c,0}}^{C_{c,0}(DTF_{k,4})} \frac{1}{A_{part} \cdot (C_c)} dC_c}{P_{O_2} \cdot DTF_{k,3}}$$

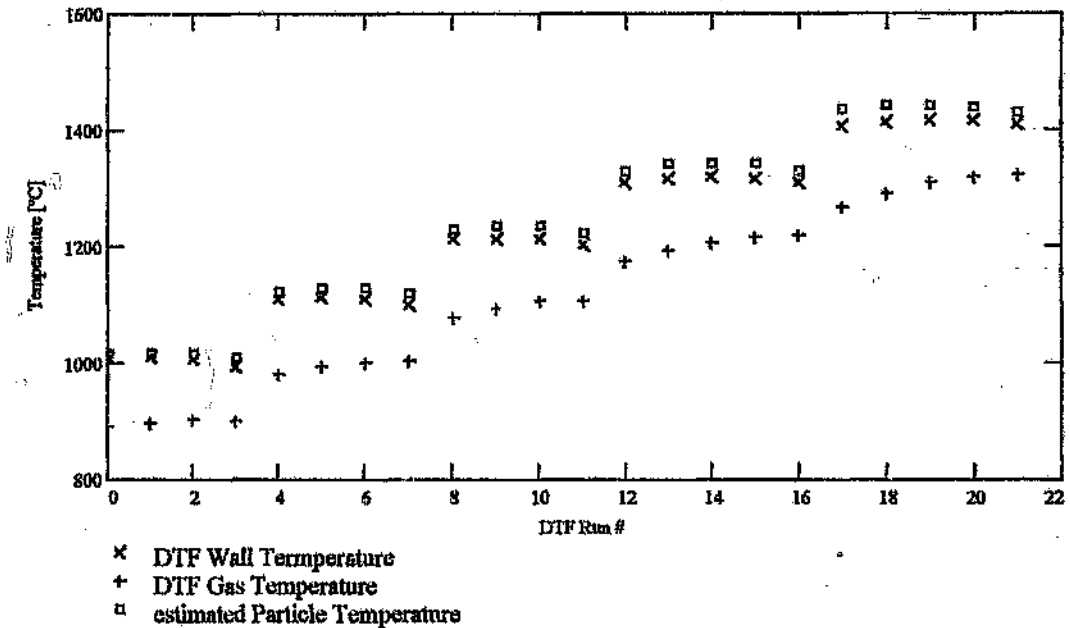
Calculating Particle Temperature:

Iterative solution of Energy Balance between particle, hot gas and reactor wall :

$$\frac{\Delta h \cdot (C_{c,Ash_{res}} - C_{c,0})}{t_r} = \left[\sigma \cdot [(T_w)^4 - (T_p)^4] + \frac{Nu \cdot \lambda(T_p, T_g)}{dp_o} \cdot \left[\left(\frac{T_g + T_p}{2} \right) - (T_g) \right] \right] \cdot A_{part,c}(C_{c,0})$$

$$T_{particle}(Ash_{res}, t_r, T_w, T_g) = \text{find}(T_p)$$

$$DTF_{k,6} := T_{particle}(DTF_{k,4}, DTF_{k,3}, DTF_{k,2} + 273, DTF_{k,5} + 273) - 273$$



Oxygen Diffusion reaction rate as a function particle Temperature

O2 diffusion constant at 1600K : $D_{O_2,1600K} := 3.39 \cdot 10^{-4} \frac{r_f^2}{s}$

mechanism factor : $\psi = 2$ $\psi = 2$ for CO, and 1 for CO2

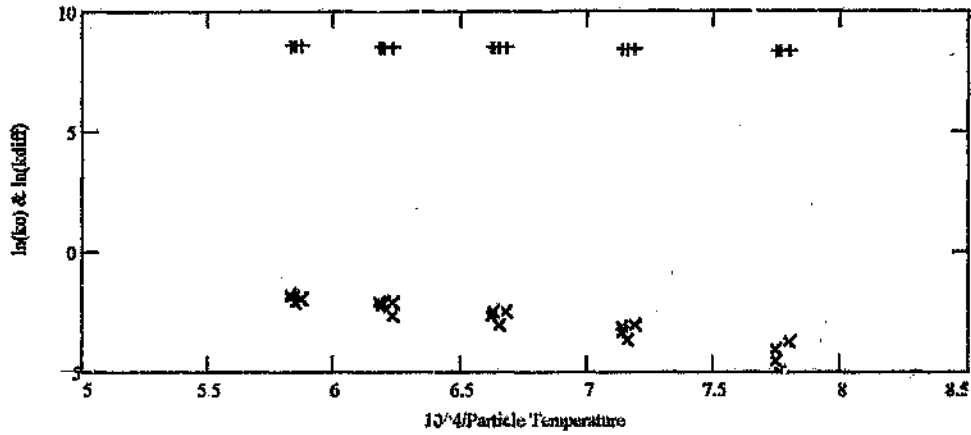
$$k_{d,char}(T_p, T_g, dp) = \frac{24 \cdot \psi \cdot D_{O_2,1600K} \cdot \left[\frac{(T_g + T_p) \cdot 0.5}{1600} \right]^{1.75}}{R \cdot 10^5 \cdot dp \cdot T_g} \quad \frac{kg}{m^2 \cdot s \cdot bar}$$

Source : AiQLOS - combustion model

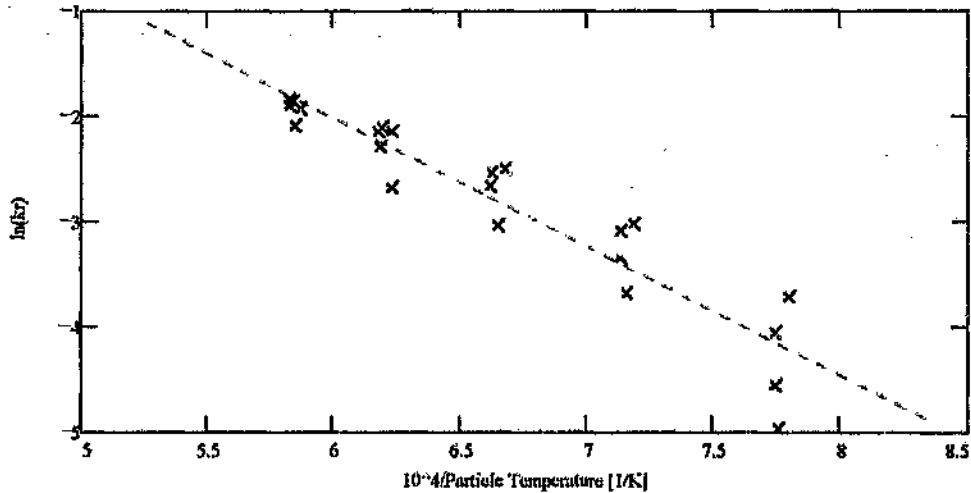
Hendrina Washing Plant Feed : < 38 μm

Determining the Chemical Reaction Rate from the experimental data :

$$k_{r,k} = \left[\frac{1}{\left(\frac{1}{k_{o,k}} \right) + \frac{1}{k_{d,char}(DTF_{k,6} + 273, DTF_{k,5} + 273, DTF_{k,0})}} \right]$$



NB : if the +'s lie beneath the x's, the overall reaction rate could be diffusion controlled



x DTF experimental
 -- kinetic model - predicted

Char chemical combustion kinetics Description :

<u>Pre-exponential factor :</u>	$\exp(\text{Preexp}) = 204.918$	$\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$
<u>Activation Energy :</u>	$-\text{EonR} \cdot \frac{R}{10^3} = 101.659$	$\frac{\text{kJ}}{\text{mol}}$
	$-\text{EonR} = 12227.406$	K
<u>Function describing chemical reaction rate :</u>	$k_{r,char}(T_p) = k_{r,char,0} \cdot \exp\left(\frac{\text{EonR}}{T_p}\right)$	

Defining the overall reaction rate function :

$$k_r(T_p, T_g, dp) = \frac{1}{\left(\frac{1}{k_{r,char}(T_p)} + \frac{1}{k_{d,char}(T_p, T_g, dp)} \right)}$$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation

$$dT_{rad}(C_c, T_p, T_w) = \frac{\epsilon \sigma (T_w^4 - T_p^4)}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot A_{part,c}(C_{c,0})$$

convection

$$dT_{conv}(C_c, T_p, T_g) = \frac{Nu \cdot \lambda(T_p, T_g) \left[\left(\frac{T_g + T_p}{2} \right) - T_p \right]}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot A_{part,c}(C_{c,0})$$

reaction

$$dT_{gen}(C_c, T_p, T_g) = \frac{-\Delta h}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot (k_r(T_p, T_g, dp_o) \cdot P_{O_2} \cdot A_{part,c}(C_{c,0}))$$

estimation of initial particle heating rate at DTF setpoint Temperature of 1400°C :

$$T_{init} = 25 + 273 \text{ K} \quad dT_{rad}(C_{c,0}, T_{init}, 1416 + 273) + dT_{conv}(C_{c,0}, T_{init}, 1335 + 273) = 3.774 \cdot 10^4 \frac{\text{K}}{\text{s}}$$

Solution of differential equations

Initial Conditions : combustion extent — $x_0 = C_{c,0}$

particle temperature — $x_1 = 300 \text{ K}$

Simultaneous Differential Equations :

combustion extent — $\frac{d}{dt} C_c = k_r(x_1, T_g, dp_o) \cdot P_{O_2} \cdot A_{part,c}(x_0)$

particle temperature — $\frac{d}{dt} T_p = dT_{rad}(x_0, x_1, T_w) + dT_{conv}(x_0, x_1, T_g) + dT_{gen}(x_0, x_1, T_g)$

Note on the Solution Methodology

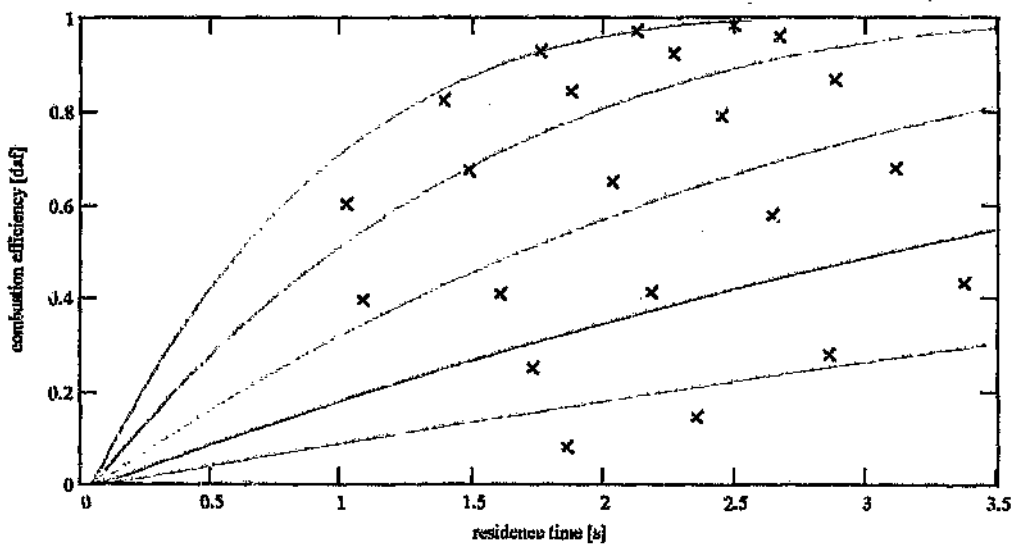
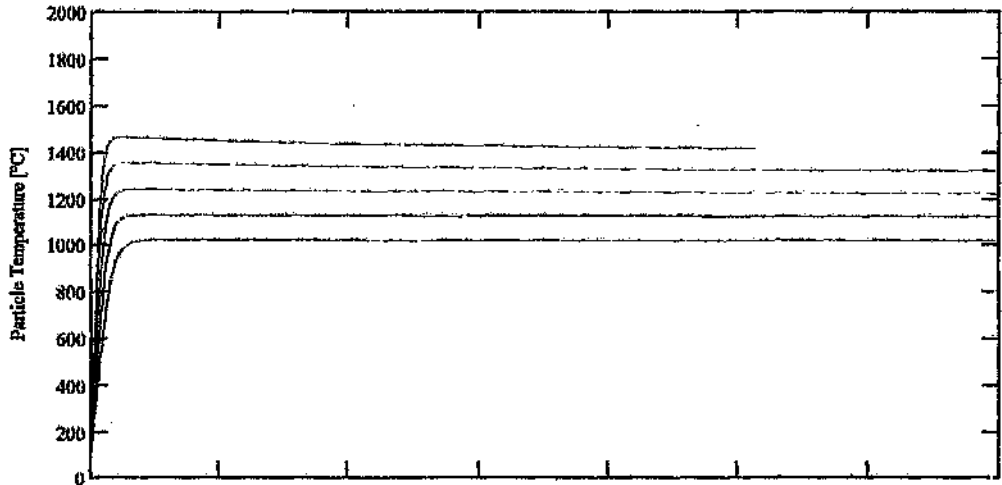
The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint Temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature.

DTF Kinetic Evaluation Results -

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 204.918 \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $-\text{EoaR} \cdot \frac{R}{10^3} = 101.66 \frac{\text{kJ}}{\text{mol}}$ $-\text{EoaR} = 12227.406 \text{ K}$



- 1000°C - DTF Setpoint Temperature
- 1100°C
- 1200°C
- 1300°C
- 1400°C
- x DTF data

Hendrina Washing Plant Feed : < 38 μm

Determination of DTF combustion kinetics for use with AIOLOS combustion code

Universal Gas Constant $R = 8.314 \frac{J}{mol \cdot K}$

DTF Operation and Char Description :

Ash in Coal $Ash_{coal} = DTF_{0.8} \cdot \%$

$Ash_{char} = DTF_{0.7} \cdot \%$

Ash in Char

Volatile Matter (after DTF devolatilisation)

$Vols_{DTF} = 1 - Ash_{coal} - \frac{Ash_{coal}}{Ash_{char}} \cdot (1 - Ash_{char})$

Density of coal :

$\rho_{coal} = 1678 \frac{kg}{m^3}$

Density of Char (initially) :
(shrinking core assumption)

$\rho_{char,0} = (1 - Vols_{DTF}) \cdot \rho_{coal} \quad \rho_{char,0} = 1.087 \cdot 10^3 \frac{kg}{m^3}$

Initial Particle Diameter :

$dp_0 = DTF_{0,0} \quad m \quad dp_0 = 6.48 \cdot 10^{-5} \quad m$

Pressure in DTF :

$P = 0.85 \cdot 10^5 \quad Pa$

Volume Oxygen Flowrate into DTF :

$V_{O_2} = \frac{0.6 \cdot 10^{-3}}{60} \frac{m^3}{s} \quad M_{W,O_2} = 32 \cdot 10^{-3} \frac{kg}{mol}$

Volume Nitrogen Flowrate into DTF :

$V_{N_2} = \frac{19.66 \cdot 10^{-3}}{60} \frac{m^3}{s} \quad M_{W,N_2} = 28 \cdot 10^{-3} \frac{kg}{mol}$

Mass Flowrate of Oxygen :

$m_{O_2} = \frac{V_{O_2} \cdot P \cdot M_{W,O_2}}{R \cdot 298} \frac{kg}{s}$

Mass Flowrate of Nitrogen :

$m_{N_2} = \frac{V_{N_2} \cdot P \cdot M_{W,N_2}}{R \cdot 298} \frac{kg}{s}$

Char Feedrate :

$m_{char} = \frac{0.1066 \cdot 10^{-3}}{60} \frac{kg}{s}$

Initial Char Concentration

$C_{char,0} = \frac{m_{char}}{m_{char} + m_{O_2} + m_{N_2}} \frac{kg}{kg_{mix}}$

Initial Fixed Carbon Concentration

$C_{c,0} = C_{char,0} \cdot (1 - Ash_{char}) \quad C_{c,0} = 2.8942 \cdot 10^{-3} \frac{kg}{kg_{mix}}$

Ash Concentration :

$C_{ash} = C_{char,0} \cdot (Ash_{char}) \quad C_{ash} = 2.5304 \cdot 10^{-3} \frac{kg}{kg_{mix}}$

Heat Exchange Parameters and constants

Stefan Boltzmann constant $\sigma := 5.67 \cdot 10^{-8} \frac{W}{m^2 \cdot K^4}$

average emissivity between particle and wall $\epsilon := 0.85$

reaction energy : C to CO : $\Delta h := 9.614 \cdot 10^6 \frac{J}{kg}$

Nusselt Number for boundary layer convective heatexchange $Nu := 2$

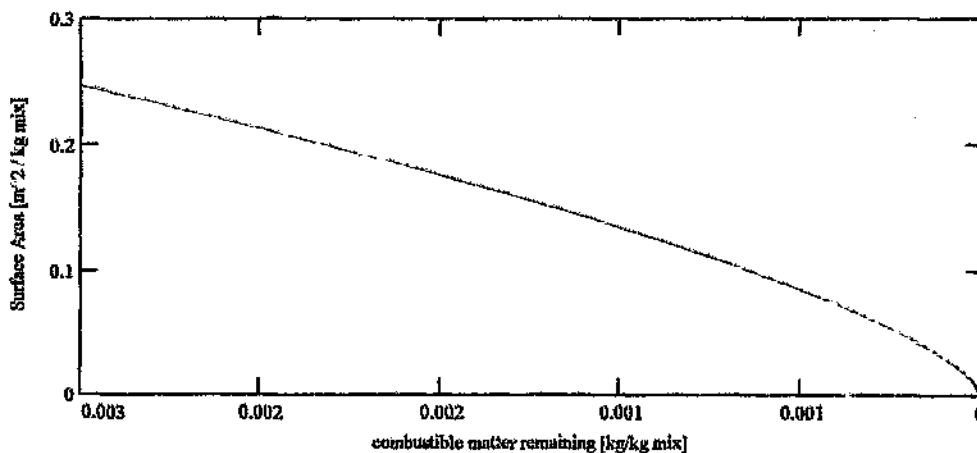
convective heat exchange coefficient $\lambda(T_p, T_g) := 2.43 \cdot 10^{-11} \left[\frac{(T_p + T_g)}{273} \right]^{0.82} \frac{W}{m \cdot K}$

heat capacity of char : $C_{p, char} := 1250 \frac{J}{kg \cdot K}$

Particle Surface Area as a function of combustion extent :

swelling index $n_B := \frac{2}{3}$ 1 - swelling coals
 2/3 - 'non swelling' coals

$$A_{part,c}(C_c) = \frac{6}{\rho_{char,0} \cdot d_{p,0}} \cdot C_{c,0} \cdot \left(\frac{C_c}{C_{c,0}} \right)^{n_B} \frac{m^2}{kg \text{ mix}}$$



Partial Pressure of Oxygen in DTF : $P_{O_2} = \frac{V_{O_2}}{V_{O_2} + V_{N_2}} \quad P_{O_2} = 0.03 \quad \text{bar}$

Function defining mass of combustible matter remaining as a function ash content in the sampled residue

$$C_c(\text{ash residue}) = \left[\frac{\text{Ash char} \cdot \left(1 - \frac{\text{ash residue}}{100} \right)}{1 - \text{Ash char} \cdot \left(\frac{\text{ash residue}}{100} \right)} \right] \cdot C_{c,0}$$

ash residue in % by mass

Calculating Overall Rate :

$$k_{o,k} \left[\frac{C_c(DTF_{k,4}) - C_{c,0}}{A_{part,c}(C_{c,0})} \frac{1}{dC_c} - P_{O_2} DTF_{k,3} \right]$$

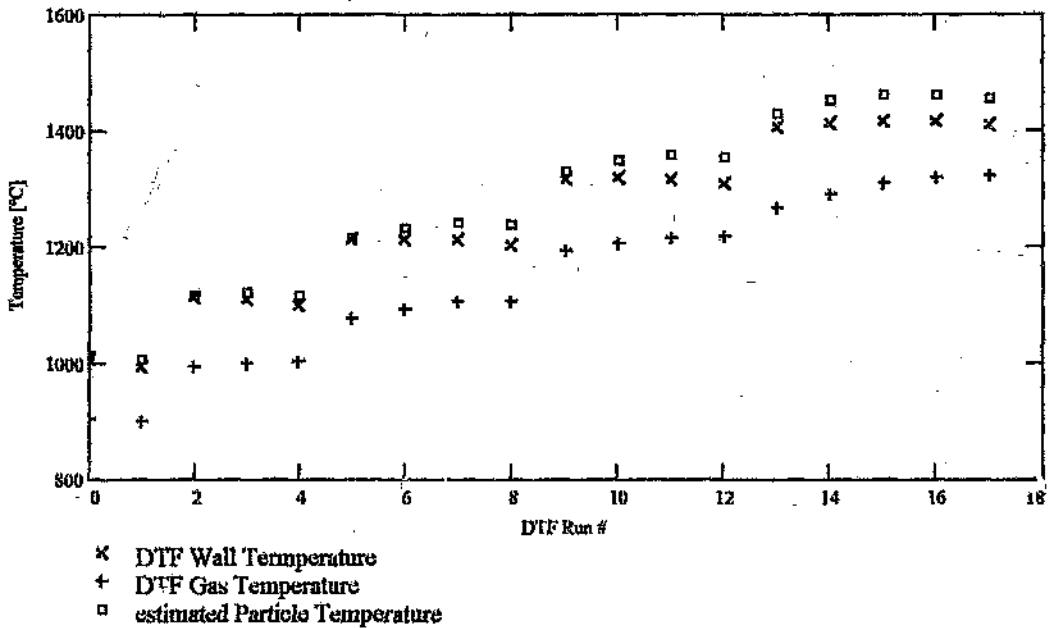
Calculating Particle Temperature:

iterative solution of Energy Balance between particle, hot gas and reactor wall :

$$\frac{\Delta h \cdot (C_c(Ash_{res}) - C_{c,0})}{t_r} = \left[\epsilon \sigma \left[(T_w)^4 - (T_p)^4 \right] + \frac{Nu \cdot \lambda(T_p, T_g)}{dp_o} \cdot \left[\left(\frac{T_g + T_p}{2} \right) - (T_g) \right] \right] \cdot A_{part,c}(C_{c,0})$$

$$T_{particle}(Ash_{res}, t_r, T_w, T_g) = \text{find}(T_p)$$

$$DTF_{k,6} = T_{particle}(DTF_{k,4}, DTF_{k,3}, DTF_{k,2} + 273, DTF_{k,5} + 273) - 273$$



Oxygen Diffusion reaction rate as a function particle Temperature

O2 diffusion constant at 1600K: $D_{O_2,1600K} = 3.39 \cdot 10^{-4} \frac{m^2}{s}$

mechanism factor: $\phi = 2$ $\phi = 2$ for CO, and 1 for CO2

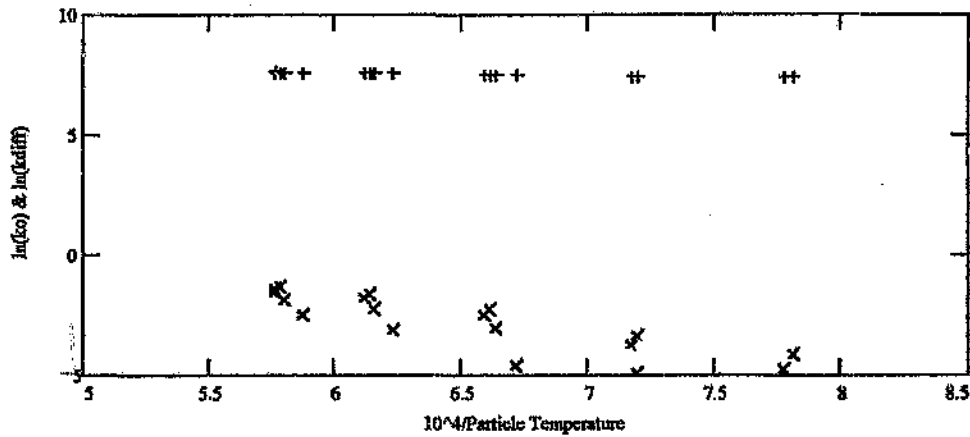
$$k_{d,char}(T_p, T_g, dp) = \frac{24 \cdot \phi \cdot D_{O_2,1600K} \cdot \left(\frac{(T_g + T_p) - 0.5}{1600} \right)^{1.75}}{R \cdot 10^5 \cdot dp \cdot T_g} \quad \frac{kg}{m^2 \cdot s \cdot bar}$$

Source : AIOLOS - combustion model

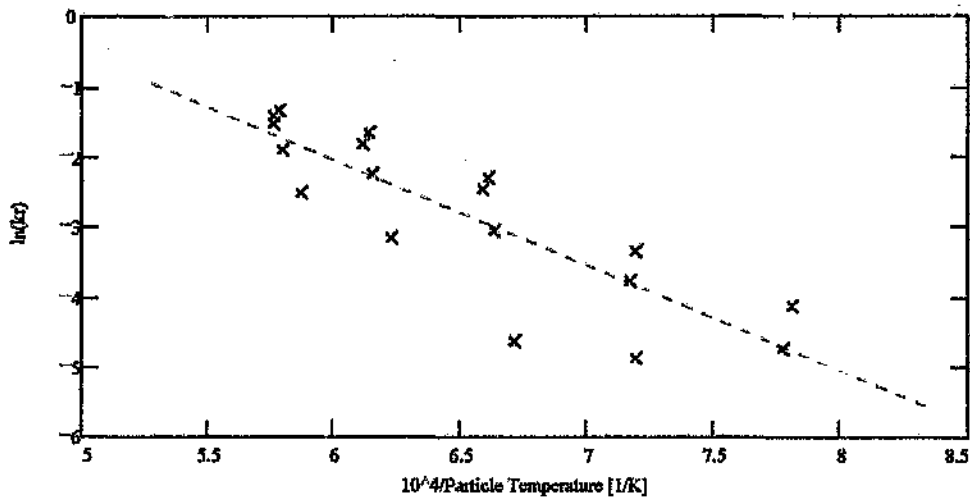
Hendrina Washing Plant Feed : 38 75 μm

Determining the Chemical Reaction Rate from the experimental data :

$$kr_k := \left[\frac{1}{\left(\frac{1}{k_{o,k}} \right) - \frac{1}{k_{d,char}(DTF_{k,6} + 273, DTF_{k,5} + 273, DTF_{k,0})}} \right]$$



NB : if the +'s lie beneath the x's, the overall reaction rate could be diffusion controlled



x DTF experimental
-- kinetic model - predicted

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 1063.044$ $\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $-\text{EonR} \cdot \frac{\text{R}}{10^3} = 124.897$ $\frac{\text{kJ}}{\text{mol}}$

$-\text{EonR} = 15022.456$ K

Function describing chemical reaction rate : $k_{r,char}(T_p) = k_{rch,o} \cdot \exp\left(\frac{\text{EonR}}{T_p}\right)$

Defining the overall reaction rate function :

$$k_r(T_p, T_g, dp) = \frac{1}{\left(\frac{1}{k_{r, \text{char}}(T_p)} + \frac{1}{k_{d, \text{char}}(T_p, T_g, dp)} \right)}$$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation

$$dT_{\text{rad}}(C_c, T_p, T_w) = \frac{\epsilon \sigma (T_w^4 - T_p^4)}{(C_c + C_{\text{ash}}) \cdot C_{p, \text{char}}} \cdot A_{\text{part}, c}(C_{c,0})$$

convection

$$dT_{\text{conv}}(C_c, T_p, T_g) = \frac{\text{Nu} \cdot \lambda (T_p - T_g) \cdot \left[\left(\frac{T_g + T_p}{2} \right) - T_p \right]}{(C_c + C_{\text{ash}}) \cdot C_{p, \text{char}}} \cdot A_{\text{part}, c}(C_{c,0})$$

reaction

$$dT_{\text{gen}}(C_c, T_p, T_g) = \frac{-\Delta h}{(C_c + C_{\text{ash}}) \cdot C_{p, \text{char}}} \cdot (-k_r(T_p, T_g, dp_0) \cdot P_{O_2} \cdot A_{\text{part}, c}(C_c))$$

estimation of initial particle heating rate at DTF setpoint Temperature of 1400°C :

$$T_{\text{init}} := 25 + 273 \text{ K} \quad dT_{\text{rad}}(C_{c,0}, T_{\text{init}}, 1416 + 273) + dT_{\text{conv}}(C_{c,0}, T_{\text{init}}, 1335 + 273) = 1.425 \cdot 10^4 \frac{\text{K}}{\text{s}}$$

Solution of differential equations

Initial Conditions : combustion extent — $x_0 := C_{c,0}$

particle temperature — $x_1 := 300 \text{ K}$

Simultaneous Differential Equations :

$$\text{combustion extent — } \frac{d}{dt} C_c = -k_r(x_1, T_g, dp_0) \cdot P_{O_2} \cdot A_{\text{part}, c}(x_0)$$

$$\text{particle temperature — } \frac{d}{dt} T_p = dT_{\text{rad}}(x_0, x_1, T_w) + dT_{\text{conv}}(x_0, x_1, T_g) + dT_{\text{gen}}(x_0, x_1, T_g)$$

Note on the Solution Methodology

The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint Temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature.

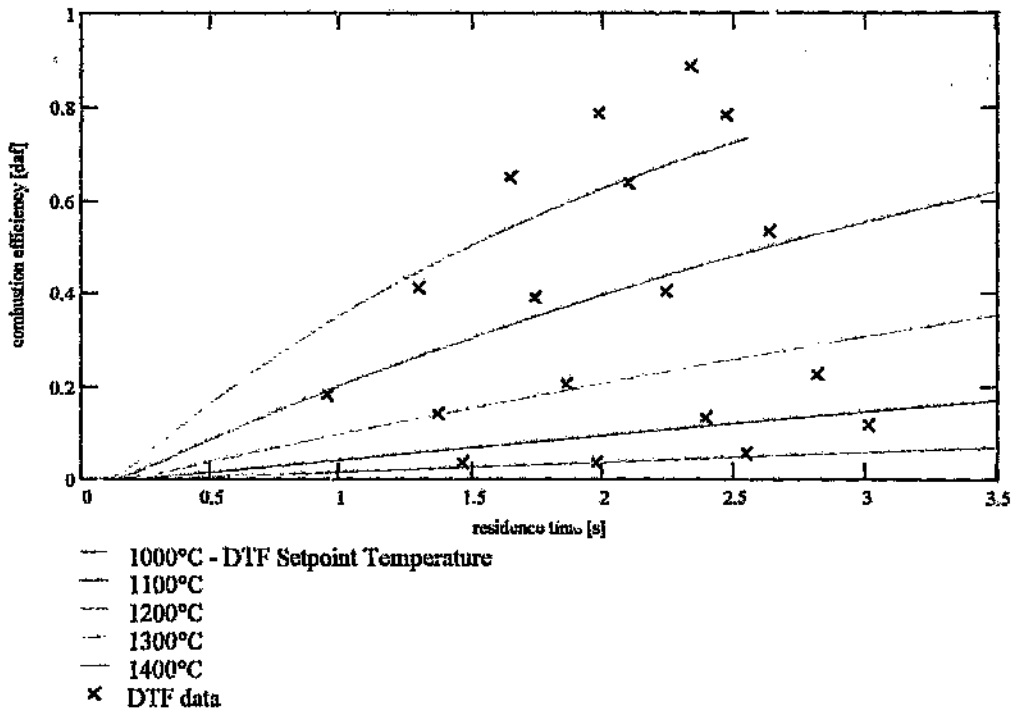
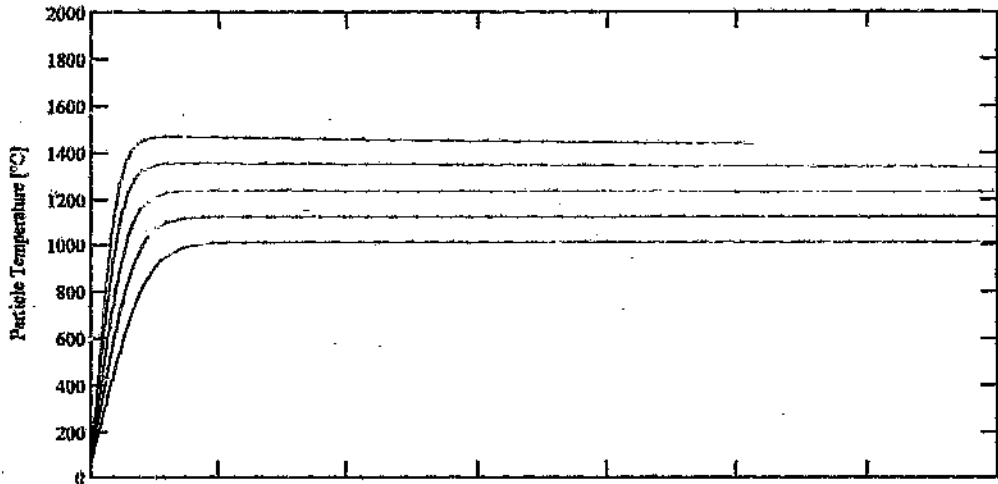
Hendrina Washing Plant Feed : 38-75 μm

DTF Kinetic Evaluation Results -

Char chemical combustion kinetics Description :

Pre-exponential factor : $\exp(\text{Preexp}) = 1063.044 \frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$

Activation Energy : $-\text{EonR} \cdot \frac{R}{10^3} = 124.90 \frac{\text{kJ}}{\text{mol}}$ $-\text{EonR} = 15022.456 \text{ K}$



Determination of DTF combustion kinetics for use with AIOLOS combustion code

Universal Gas Constant $R = 8.314 \frac{J}{mol \cdot K}$

DTF Operation and Char Description :

Ash in Coal $Ash_{coal} := DTF_{0,8} \cdot \%$

Ash in Char $Ash_{char} := DTF_{0,7} \cdot \%$

Volatle Matter (after DTF devolatilisation) $Vols_{DTF} := 1 - Ash_{coal} - \frac{Ash_{coal}}{Ash_{char}} \cdot (1 - Ash_{char})$

Density of coal : $\rho_{coal} := 1678 \frac{kg}{m^3}$

Density of Char (initially) :
(shrinking core assumption) $\rho_{char,0} := (1 - Vols_{DTF}) \cdot \rho_{coal} \quad \rho_{char,0} = 1.087 \cdot 10^3 \frac{kg}{m^3}$

Initial Particle Diameter : $dp_0 := DTF_{0,0} \quad m \quad dp_0 = 6.48 \cdot 10^{-5} \quad m$

Pressure in DTF : $P := 0.85 \cdot 10^5 \quad Pa$

Volume Oxygen Flowrate into DTF : $V_{O2} := \frac{0.6 \cdot 10^{-3}}{60} \frac{m^3}{s} \quad M_{W,O2} := 32 \cdot 10^{-3} \frac{kg}{mol}$

Volume Nitrogen Flowrate into DTF : $V_{N2} := \frac{19.66 \cdot 10^{-3}}{60} \frac{m^3}{s} \quad M_{W,N2} := 28 \cdot 10^{-3} \frac{kg}{mol}$

Mass Flowrate of Oxygen : $m_{O2} := \frac{V_{O2} \cdot P \cdot M_{W,O2}}{R \cdot 298} \frac{kg}{s}$

Mass Flowrate of Nitrogen : $m_{N2} := \frac{V_{N2} \cdot P \cdot M_{W,N2}}{R \cdot 298} \frac{kg}{s}$

Char Feedrate : $m_{char} := \frac{0.1066 \cdot 10^{-3}}{60} \frac{kg}{s}$

Initial Char Concentration $C_{char,0} = \frac{m_{char}}{m_{char} + m_{O2} + m_{N2}} \frac{kg}{kg \text{ mix}}$

Initial Fixed Carbon Concentration $C_{c,0} := C_{char,0} \cdot (1 - DTF_{0,7} \cdot \%)$ $C_{c,0} = 2.8942 \cdot 10^{-3} \frac{kg}{kg \text{ mix}}$

$C_{c,1} := C_{char,0} \cdot (1 - DTF_{13,7} \cdot \%)$ $C_{c,1} = 2.8942 \cdot 10^{-3} \frac{kg}{kg \text{ mix}}$

Ash Concentration : $C_{ash} := C_{char,0} \cdot (Ash_{char})$ $C_{ash} = 2.5304 \cdot 10^{-3} \frac{kg}{kg \text{ mix}}$

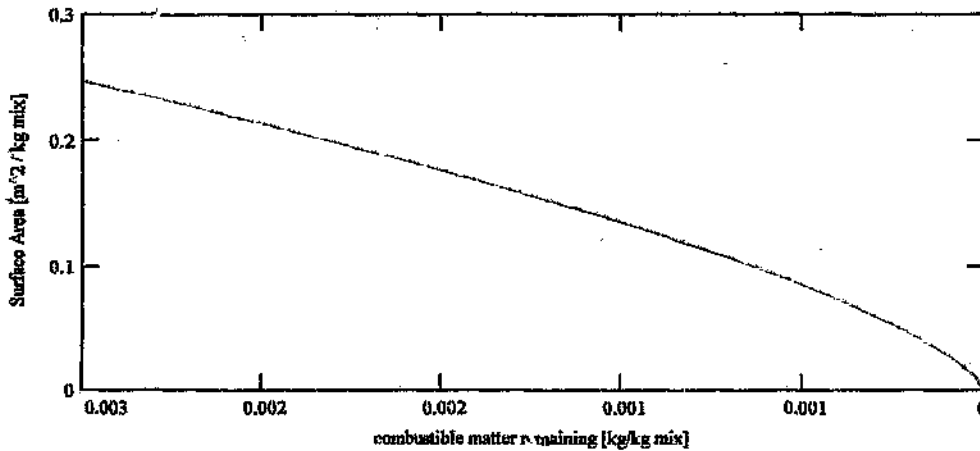
Heat Exchange Parameters and constants

Stefan Boltzmann constant	$\sigma = 5.67 \cdot 10^{-8}$	$\frac{W}{m^2 \cdot K^4}$
average emissivity between particle and wall	$\epsilon = 0.85$	
reaction energy : C to CO :	$\Delta h = 9.614 \cdot 10^6$	$\frac{J}{kg}$
Nusselt Number for boundary layer convective heat exchange	$Nu = 2$	
convective heat exchange coefficient	$\lambda(T_p, T_g) = 2.43 \cdot 10^{-11} \cdot \left[\frac{(T_p + T_g)}{273} \right]^{0.82}$	$\frac{W}{m \cdot K}$
heat capacity of char :	$C_{p \text{ char}} = 1250$	$\frac{J}{kg \cdot K}$

Particle Surface Area as a function of combustion extent :

swelling index $n_B = \frac{2}{3}$ 1 - swelling coals
 2/3 - 'non swelling' coals

$$A_{\text{part.c}}(C_c) = \frac{6}{\rho_{\text{char.o}} \cdot d_{p_o}} \cdot C_{c,0} \cdot \left(\frac{C_c}{C_{c,0}} \right)^{n_B} \quad \frac{m^2}{kg \text{ mix}}$$



Partial Pressure of Oxygen in DTF : $P_{O_2} = \frac{V_{O_2}}{V_{O_2} + V_{N_2}} \quad P_{O_2} = 0.03 \quad \text{bar}$

Function defining mass of combustible matter remaining as a function ash content in the sampled residue

$$C_{c(\text{ash residue})} = \left[\frac{\text{Ash char} \cdot \left(1 - \frac{\text{ash residue}}{100} \right)}{1 - \text{Ash char} \cdot \left(\frac{\text{ash residue}}{100} \right)} \right] \cdot C_{c,0}$$

ash residue in % by mass

Calculating Overall Rate :

$$k_{O_2} = \frac{\int_{C_c(DTF_{k,7})}^{C_c(DTF_{k,4})} \frac{1}{A_{part.c}(C_c)} dC_c}{P_{O_2} \cdot DTF_{k,3}}$$

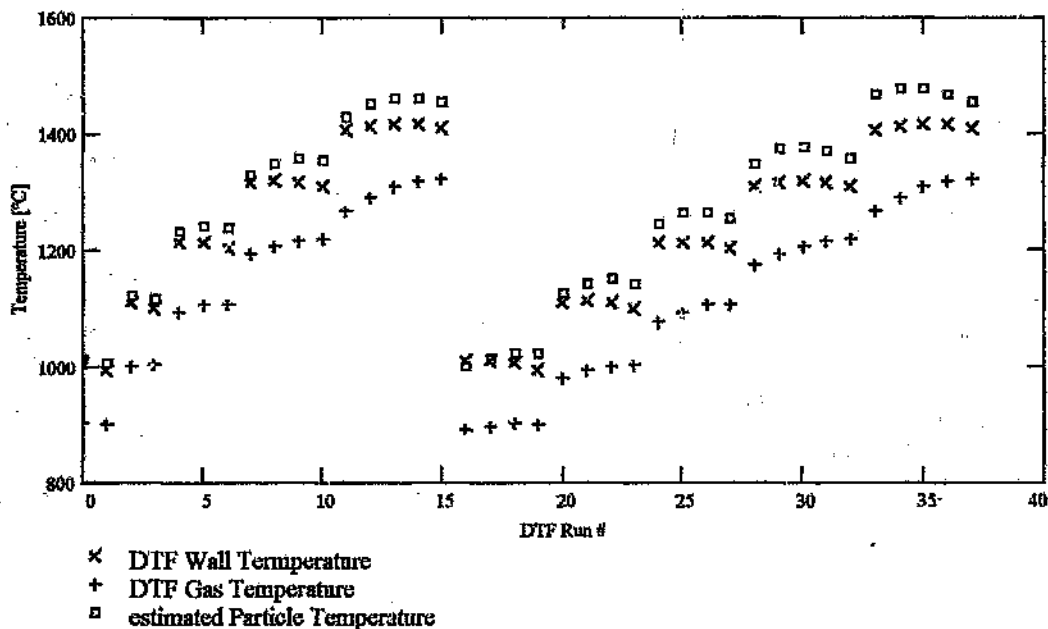
Calculating Particle Temperature:

Iterative solution of Energy Balance between particle, hot gas and reactor wall :

$$\Delta h \cdot \frac{C_c(Ash_{res}) - C_{c,0}}{t_r} = \left[\sigma \cdot (T_w^4 - T_p^4) - \frac{Nu \cdot \lambda(T_p, T_g)}{dp} \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \right] \cdot A_{part.c}(C_{c,0})$$

$$T_{particle}(Ash_{res}, t_r, T_w, T_g, dp) := \text{find}(T_p)$$

$$DTF_{k,6} = T_{particle}(DTF_{k,4}, DTF_{k,3}, DTF_{k,2} + 273, DTF_{k,5} + 273, DTF_{k,0}) - 273$$



Oxygen Diffusion reaction rate as a function particle Temperature

O2 diffusion constant at 1600K : $D_{O_2,1600K} = 3.39 \cdot 10^{-4} \frac{m^2}{s}$

mechanism factor : $\phi = 2$ $\phi = 2$ for CO, and 1 for CO2

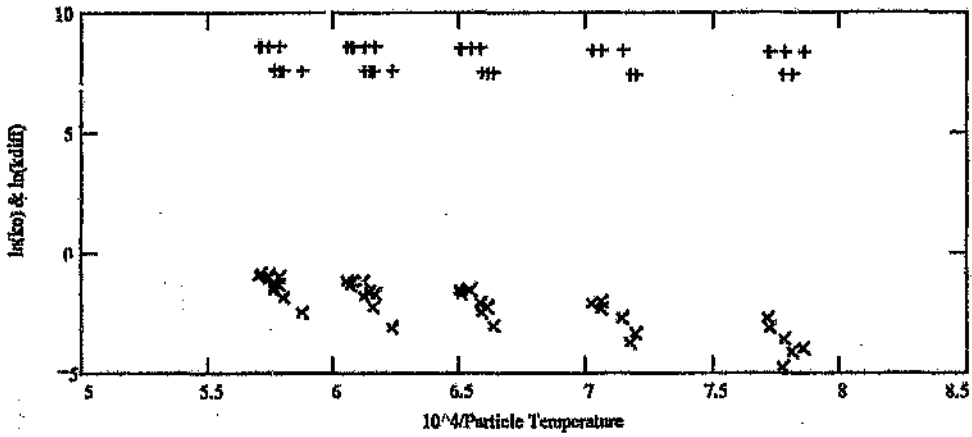
$$k_{d,char}(T_p, T_g, dp) = \frac{24 \cdot \phi \cdot D_{O_2,1600K} \left[\frac{(T_g + T_p) - 0.5}{1600} \right]^{1.75}}{R \cdot 10^5 \cdot dp \cdot T_g} \frac{kg}{m^2 \cdot s \cdot bar}$$

Source : AIOLOS - combustion model

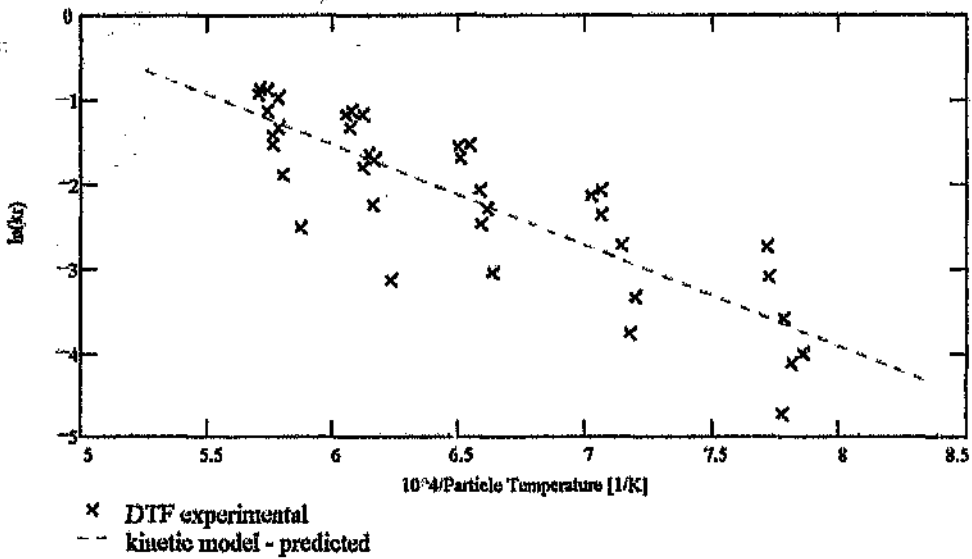
Hendrina Washing Plant Feed : <75 μm

Determining the Chemical Reaction Rate from the experimental data :

$$kr_k := \left[\frac{1}{\left(\frac{1}{k_{o,k}} \right) - \frac{1}{k_{d,char}(DTF_{k,6} + 273, DTF_{k,5} + 273, DTF_{k,0})}} \right]$$



NB : if the +’s lie beneath the x’s, the overall reaction rate could be diffusion controlled



Char chemical combustion kinetics Description :

Pre-exponential factor :	$\exp(\text{Presp}) = 282.185$	$\frac{\text{kg}}{\text{m}^2 \cdot \text{s} \cdot \text{bar}}$
Activation Energy :	$-E_{onR} \cdot \frac{R}{10^4} = 99.446$	$\frac{\text{kJ}}{\text{mol}}$
	$E_{onR} = 11961.304$	K
Function describing chemical reaction rate :	$k_{r,char}(T_p) := k_{rel,0} \cdot \exp\left(\frac{E_{onR}}{T_p}\right)$	

Defining the overall reaction rate function :

$$k_r(T_p, T_g, dp) = \frac{1}{\left(\frac{1}{k_{r,char}(T_p)} + \frac{1}{k_{d,char}(T_p, T_g, dp)} \right)}$$

Equation describing all modes of heat transfer (shrinking core assumption)

radiation

$$dT_{rad}(C_c, T_p, T_w, dp) := \frac{\epsilon \sigma (T_w^4 - T_p^4)}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot \text{if}(dp > 38 \cdot 10^{-6}, A_{part,c}(C_{c,1}) \cdot A_{part,c}(C_{c,0}))$$

convection

$$dT_{conv}(C_c, T_p, T_g, dp) := \frac{Nu \cdot h(T_p, T_g)}{dp} \left[\left(\frac{T_g + T_p}{2} \right) - T_g \right] \cdot A_{part,c}(C_{c,0})$$

reaction

$$dT_{gen}(C_c, T_p, T_g, dp) := \frac{-\Delta h}{(C_c + C_{ash}) \cdot Cp_{char}} \cdot (-k_r(T_p, T_g, dp) \cdot P_{O_2} \cdot A_{part,c}(C_c))$$

estimation of initial particle heating rate at DTF setpoint Temperature of 1400°C :

$$dT_{rad}(C_{c,0}, 298, DTF_{s_5,2} + 273, DTF_{s_5,0}) + dT_{conv}(C_{c,0}, 298, DTF_{s_5,5} + 273, DTF_{s_5,0}) = 1.401 \cdot 10^4 \frac{K}{s}$$

Solution of differential equations

Initial Conditions : combustion extent — $x_0 = C_c(DTF_{0,7})$

particle temperature — $x_1 = 300 \text{ K}$

Simultaneous Differential Equations :

combustion extent — $\frac{d}{dt} C_c = -k_r(x_1, T_g, dp_{initial}) \cdot P_{O_2} \cdot A_{part,c}(x_0)$

particle temperature — $\frac{d}{dt} T_p = dT_{rad}(x_0, x_1, T_w) + dT_{conv}(x_0, x_1, T_g) + dT_{gen}(x_0, x_1, T_g)$

Note on the Solution Methodology

The simultaneous equations are solved by a 4th order Runge Kutta solution algorithm for each DTF setpoint Temperature. An average furnace wall and furnace gas temperature have been specified for each DTF setpoint temperature.

Appendix 2.3

**Particle Size Distributions
for
HENDRINA UNIT 9**

from
31 January 1996
to
7 February 1996

The particle size analysis results presented herein are indexed as follows :

*Unit Number Mill PF Pipe
Sampling Date*

eg.

9 B P1 - 4
31/01/96

refers to Hendrina Unit 9, Mill B, PF Pipes 1 to 4, sampled on 31 January 1996



MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:04AM

PF SAMPLES :Run Number 6

9 B P1 - 4
31/01/96

Sample File Name: GENWATER, Record: 38
Measured on: Mon, Apr 01, 1996 7:45AM Last saved on: Mon, Apr 01, 1996 7:46AM

Source: Analysed

Presentation: 25\$D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.172 %

Concentration = 0.064 %

Obscuration = 24.96 %

$d(0.5) = 43.62 \mu\text{m}$

$d(0.1) = 6.89 \mu\text{m}$

$d(0.9) = 116.30 \mu\text{m}$

$D[4, \sigma] = 54.18 \mu\text{m}$

Span = 2.68

Sauter Mean ($D[3,2]$) = 15.92 μm

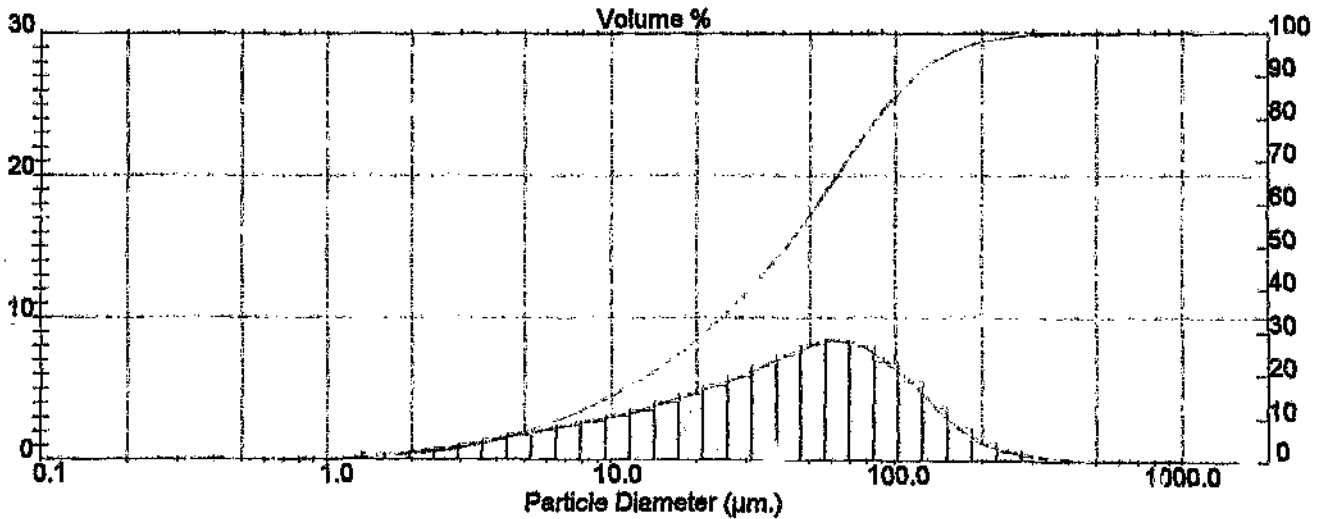
Mode = 61.45 μm

Specific Surface Area = 0.3768 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	0.50	1.32	0.50
1.32	0.41	1.60	0.91
1.60	0.42	1.95	1.33
1.95	0.59	2.38	1.92
2.38	0.82	2.90	2.74
2.90	1.09	3.53	3.84
3.53	1.42	4.30	5.26
4.30	1.76	5.24	7.02
5.24	2.10	6.39	9.12
6.39	2.41	7.78	11.53
7.78	2.74	9.48	14.27
9.48	3.10	11.55	17.37
11.55	3.53	14.08	20.90
14.08	4.03	17.15	24.93
17.15	4.59	20.90	29.52
20.90	5.20	25.46	34.72

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	5.86	31.01	40.58
31.01	6.62	37.79	47.20
37.79	7.39	46.03	54.59
46.03	8.13	56.09	62.72
56.09	8.37	68.33	71.09
68.33	8.01	83.26	79.11
83.26	6.91	101.44	86.01
101.44	5.51	123.59	91.52
123.59	3.75	150.57	95.27
150.57	2.26	183.44	97.52
183.44	1.21	223.51	98.73
223.51	0.64	272.31	99.37
272.31	0.33	331.77	99.71
331.77	0.17	404.21	99.88
404.21	0.08	492.47	99.96
492.47	0.04	600.00	100.00



Malvern MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:07AM

PF SAMPLES :Run Number 7

9 DP1-4
31/01/96

Sample File Name: GENWATER, Record: 39
Measured on: Mon, Apr 01, 1996 9:04AM Last saved on: Mon, Apr 01, 1996 9:05AM

Source: Analysed

Presentation: 235D

Very Polydisperse model

Volume :result

Focus = 300 mm.

Residual = 0.236 %

Concentration = 0.052 %

Obscuration = 23.74 %

G (0.5) = 35.40 µm

d (0.1) = 5.76 µm

d (0.9) = 126.92 µm

D [4, 3] = 56.24 µm

Span = 3.07

Sauter Mean (D[3,2]) = 13.83 µm

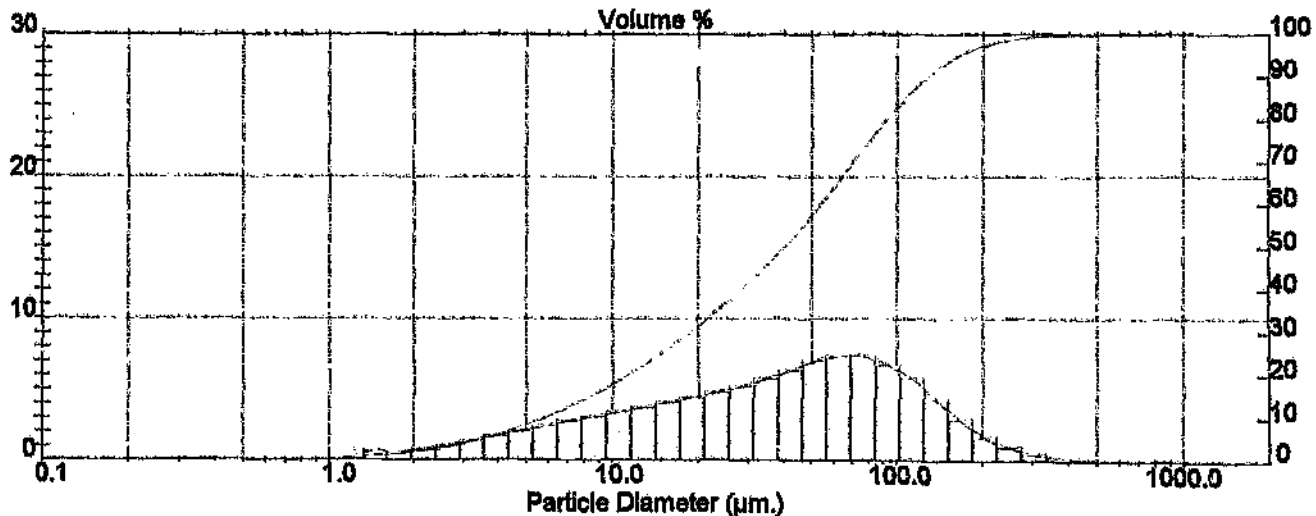
Mode = 70.26 µm

Specific Surface Area = 0.4338 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.71	1.32	0.71
1.32	0.57	1.60	1.28
1.60	0.58	1.95	1.86
1.95	0.80	2.38	2.66
2.38	1.08	2.90	3.72
2.90	1.36	3.53	5.09
3.53	1.71	4.30	6.80
4.30	2.08	5.24	8.88
5.24	2.45	6.39	11.32
6.39	2.78	7.78	14.11
7.78	3.10	9.48	17.20
9.48	3.40	11.55	20.61
11.55	3.73	14.08	24.34
14.08	4.09	17.15	28.43
17.15	4.45	20.90	32.88
20.90	4.82	25.48	37.70

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	5.23	31.01	42.93
31.01	5.78	37.79	48.71
37.79	6.34	46.03	55.05
46.03	7.02	56.09	62.07
56.09	7.36	68.33	69.43
68.33	7.37	83.26	76.80
83.26	6.72	101.44	83.53
101.44	5.80	123.59	89.32
123.59	4.30	150.57	93.62
150.57	2.86	183.44	96.48
183.44	1.67	223.51	98.15
223.51	0.94	272.31	99.09
272.31	0.50	331.77	99.59
331.77	0.25	404.21	99.84
404.21	0.11	492.47	99.95
492.47	0.05	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:09AM

PF SAMPLES :Run Number 8

9EP1-4
31/01/96

Sample File Name: GENWATER, Record: 40 Source: Analyzed
Measured on: Mon, Apr 01, 1996 9:08AM Last saved on: Mon, Apr 01, 1996 9:09AM

Preparation: 25%
Vary Polydispersity model

Residual = 0.297 %
d(0.3) = 61.47 µm
D[4,3] = 74.04 µm
Sauter Mean (D[3,2]) = 21.64 µm
Specific Surface Area = 0.2772 sq. m. / gm

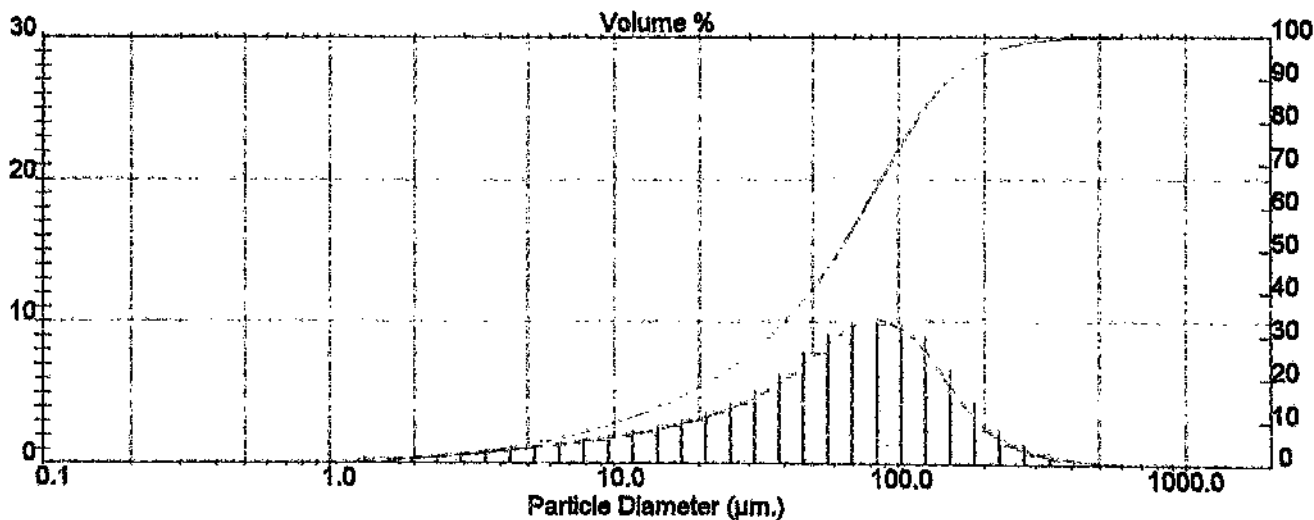
Volume Result

Concentration = 0.094 %
d(0.1) = 10.43 µm
Span = 2.24

Focus = 300 mm.
Obscuration = 26.74 %
d(0.9) = 148.03 µm
Mode = 79.07 µm
Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.41	1.32	0.41
1.32	0.32	1.60	0.73
1.60	0.32	1.95	1.05
1.95	0.43	2.38	1.48
2.38	0.56	2.90	2.04
2.90	0.71	3.53	2.75
3.53	0.88	4.30	3.63
4.30	1.06	5.24	4.69
5.24	1.26	6.39	5.95
6.39	1.47	7.78	7.41
7.78	1.68	9.48	9.10
9.48	1.94	11.55	11.04
11.55	2.24	14.08	13.28
14.08	2.61	17.15	15.89
17.15	3.04	20.90	18.93
20.90	3.58	25.46	22.51

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	4.23	31.01	26.75
31.01	5.16	37.79	31.91
37.79	6.27	46.03	38.17
46.03	7.77	56.09	45.94
56.09	9.08	68.33	55.02
68.33	10.05	83.26	65.07
83.26	9.85	101.44	74.92
101.44	8.90	123.59	83.83
123.59	6.64	150.67	90.47
150.67	4.32	183.44	94.79
183.44	2.44	223.51	97.23
223.51	1.36	272.31	98.59
272.31	0.73	331.77	99.32
331.77	0.39	404.21	99.72
404.21	0.19	492.47	99.91
492.47	0.09	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:12AM

PF SAMPLES :Run Number 9

9 FP1 - 4
31/01/96

Sample File Name: GENWATER, Record: 41 Source: Analysed
Measured on: Mon, Apr 01, 1996 9:13AM Last saved on: Mon, Apr 01, 1996 9:15AM

Presentation: 2\$\$\$
Very Polydisperse model

Volume Result Focus = 300 nm.

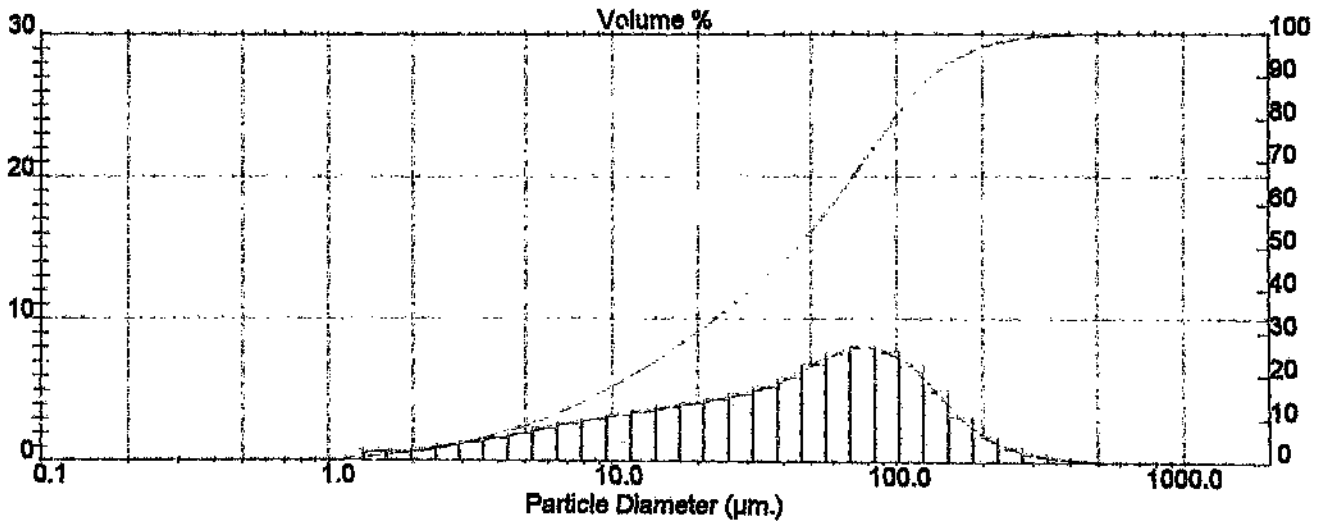
Residual = 0.247 % Concentration = 0.057 % Obscuration = 25.67 %
d(0.1) = 5.79 µm d(0.9) = 131.73 µm

D[4,3] = 59.62 µm Span = 2.87

Sauter Mean (D[3,2]) = 13.88 µm Mode = 75.64 µm
Specific Surface Area = 0.4323 sq. m. / gm Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.80	1.32	0.80
1.32	0.63	1.60	1.43
1.60	0.63	1.95	2.06
1.95	0.84	2.38	2.89
2.38	1.08	2.90	3.97
2.90	1.33	3.53	5.30
3.53	1.63	4.30	6.93
4.30	1.95	5.24	8.88
5.24	2.32	6.39	11.19
6.39	2.66	7.78	13.86
7.78	2.96	9.48	16.82
9.48	3.24	11.55	20.06
11.55	3.51	14.08	23.58
14.08	3.78	17.15	27.36
17.15	4.05	20.90	31.41
20.90	4.34	25.46	35.74

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	4.68	31.01	40.42
31.01	5.20	37.79	45.63
37.79	5.87	46.03	51.49
46.03	6.81	56.09	58.30
56.09	7.57	68.33	65.87
68.33	8.02	83.26	73.90
83.26	7.61	101.44	81.51
101.44	6.69	123.59	88.20
123.59	4.90	150.57	93.10
150.57	3.16	183.44	96.26
183.44	1.78	223.51	98.03
223.51	0.99	272.31	99.02
272.31	0.52	331.77	99.55
331.77	0.27	404.21	99.82
404.21	0.13	492.47	99.94
492.47	0.06	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:14AM

PF SAMPLES :Run Number 10

9AP1-4
31/01/96

Sample File Name: GENWATER, Record: 42 Source: Analysed
Measured on: Mon, Apr 01, 1996 9:17AM Last saved on: Mon, Apr 01, 1996 9:18AM

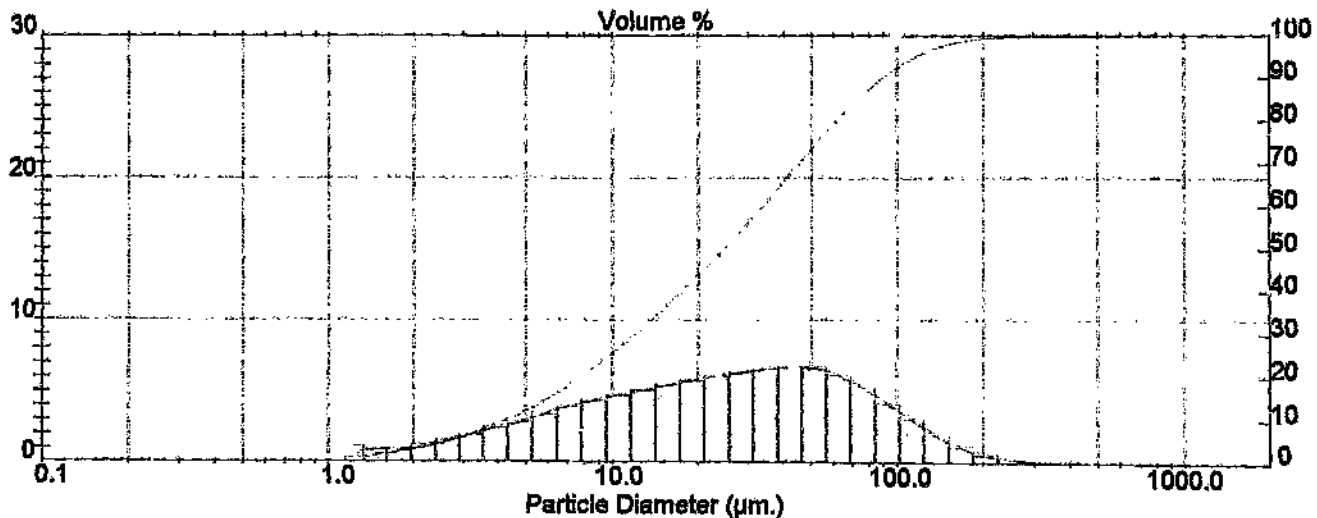
Presentation: 255D
Very Polydisperse model

Volume Result Focus = 300 mm.

Residual = 0.397 % Concentration = 0.043 % Obscuration = 26.34 %
 $d(0.5) = 24.27 \mu\text{m}$ $d(0.1) = 4.31 \mu\text{m}$ $d(0.9) = 87.21 \mu\text{m}$
 $D[4,3] = 37.22 \mu\text{m}$ Span = 3.42
Sauter Mean ($D[3,2]$) = 10.15 μm Mode = 46.67 μm
Specific Surface Area = 0.5910 sq. m. / gm Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	1.13	1.32	1.13
1.32	0.89	1.60	2.02
1.60	0.89	1.95	2.90
1.95	1.19	2.38	4.10
2.38	1.55	2.90	5.64
2.90	1.94	3.53	7.58
3.53	2.38	4.30	9.96
4.30	2.88	5.24	12.84
5.24	3.42	6.39	16.26
6.39	3.93	7.78	20.20
7.78	4.35	9.48	24.54
9.48	4.71	11.55	29.26
11.55	5.06	14.08	34.32
14.08	5.42	17.15	39.74
17.15	5.73	20.90	45.47
20.90	6.03	25.46	51.49

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	6.27	31.01	57.77
31.01	6.54	37.79	64.31
37.79	6.63	46.03	70.94
46.03	6.59	56.09	77.53
56.09	6.09	68.33	83.62
68.33	5.30	83.26	88.92
83.26	4.16	101.44	93.08
101.44	3.05	123.59	96.13
123.59	1.93	150.57	98.06
150.57	1.08	183.44	99.14
183.44	0.52	223.51	99.66
223.51	0.23	272.31	99.89
272.31	0.08	331.77	99.97
331.77	0.02	404.21	100.00
404.21	0.00	492.47	100.00
492.47	0.00	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:17AM

PF SAMPLES :Run Number 10

9A P1-4 GRAP
02/02/96

Sample File Name: GENWATER, Record: 43
Measured on: Tue, Apr 02, 1996 7:19AM Last saved on: Tue, Apr 02, 1996 7:20AM

Source: Analysed

Presentation: 255D
Vary Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.136 %

Concentration = 0.109 %

Obscuration = 23.21 %

d (0.5) = 76.64 µm

d (0.1) = 15.24 µm

d (0.9) = 169.56 µm

D [4, 3] = 89.56 µm

Span = 2.01

Sauter Mean (D[3,2]) = 29.59 µm

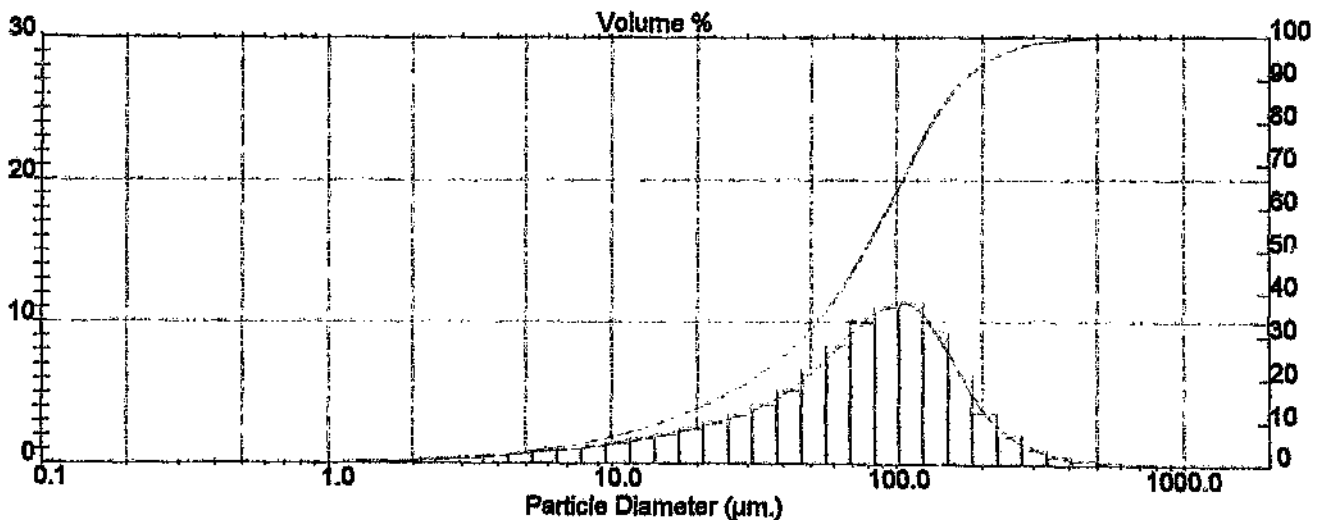
Mode = 105.60 µm

Specific Surface Area = 0.2028 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.21	1.32	0.21
1.32	0.17	1.60	0.38
1.60	0.17	1.95	0.55
1.95	0.24	2.38	0.80
2.38	0.33	2.90	1.13
2.90	0.44	3.53	1.57
3.53	0.57	4.30	2.14
4.30	0.71	5.24	2.85
5.24	0.87	6.39	3.71
6.39	1.03	7.78	4.75
7.78	1.24	9.48	5.98
9.48	1.47	11.55	7.46
11.55	1.75	14.08	9.21
14.08	2.08	17.15	11.29
17.15	2.47	20.90	13.76
20.90	2.90	25.46	16.66

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	3.43	31.01	20.09
31.01	4.18	37.79	24.27
37.79	5.22	46.03	29.50
46.03	6.64	56.09	36.14
56.09	8.27	68.33	44.41
68.33	9.92	83.26	54.33
83.26	11.02	101.44	65.35
101.44	11.21	123.59	76.56
123.59	9.25	150.57	85.81
150.57	6.37	183.44	92.19
183.44	3.68	223.51	95.87
223.51	2.02	272.31	97.90
272.31	1.08	331.77	98.97
331.77	0.58	404.21	99.55
404.21	0.30	492.47	99.85
492.47	0.15	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:19AM

PF SAMPLES :Run Number 11

9 B P1 - 4
02/02/96

Sample File Name: GENWATER, Record: 44
Measured on: Tue, Apr 02, 1996 7:23AM Last saved on: Tue, Apr 02, 1996 7:24AM

Source: Analysed

Presentation: 25\$D
Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.302 %
D[0.5] = 1.524 μ m
D[4,3] = 29.13 μ m
Sauter Mean (D[3,2]) = 8.84 μ m
Specific Surface Area = 0.6785 sq. m. / gm

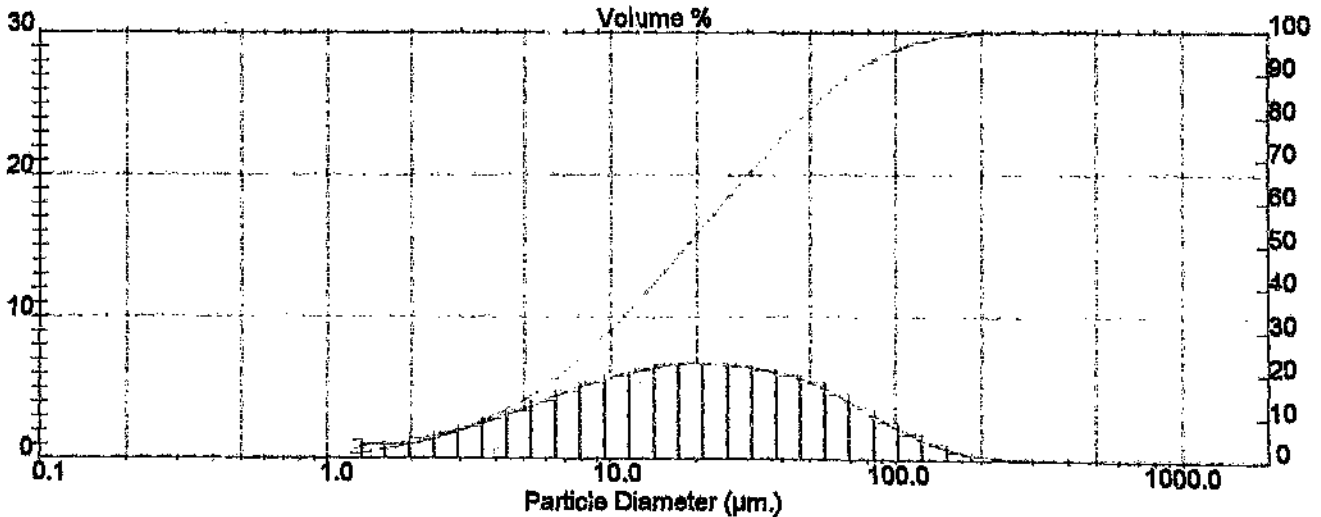
Concentration = 0.040 %
d(0.1) = 3.87 μ m
Span = 3.54

Obscuration = 27.77 %
d(0.9) = 68.47 μ m

Mode = 20.06 μ m
Density = 1.00 gm. / c.c.

Size (Lo) μ m	Result In %	Size (Hi) μ m	Result Below %
0.50	1.30	1.32	1.30
1.32	1.02	1.60	2.33
1.60	1.02	1.95	3.35
1.95	1.37	2.38	4.72
2.38	1.77	2.90	6.51
2.90	2.25	3.53	8.76
3.53	2.78	4.30	11.54
4.30	3.39	5.24	14.93
5.24	4.09	6.39	19.02
6.39	4.78	7.78	23.80
7.78	5.37	9.48	29.16
9.48	5.87	11.55	35.03
11.55	6.28	14.08	41.31
14.08	6.59	17.15	47.90
17.15	6.75	20.90	54.65
20.90	6.71	25.46	61.36

Size (Lo) μ m	Result In %	Size (Hi) μ m	Result Below %
25.46	6.54	31.01	67.90
31.01	6.29	37.79	74.18
37.79	5.90	46.03	80.08
46.03	5.34	56.09	85.42
56.09	4.53	68.33	89.96
68.33	3.60	83.26	93.56
83.26	2.64	101.44	96.20
101.44	1.79	123.59	97.99
123.59	1.07	150.57	99.06
150.57	0.56	183.44	99.61
183.44	0.25	223.51	99.86
223.51	0.10	272.31	99.96
272.31	0.03	331.77	99.99
331.77	0.01	404.21	100.00
404.21	0.00	492.47	100.00
492.47	0.00	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:22AM

PF SAMPLES :Run Number 12

9 B P1 -4 GRAP
02/02/96

Sample File Name: GENWATER, Record: 45
Measured on: Tue, Apr 02, 1996 7:28AM Last saved on: Tue, Apr 02, 1996 7:29AM

Source: Analysed

Presentation: 255D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.167 %

Concentration = 0.136 %

Obscuration = 24.03 %

$d(0.5) = 83.65 \mu\text{m}$

$d(0.1) = 24.89 \mu\text{m}$

$d(0.9) = 179.44 \mu\text{m}$

$D[4,3] = 97.48 \mu\text{m}$

Span = 1.85

Sauter Mean ($D[3,2]$) = 35.70 μm

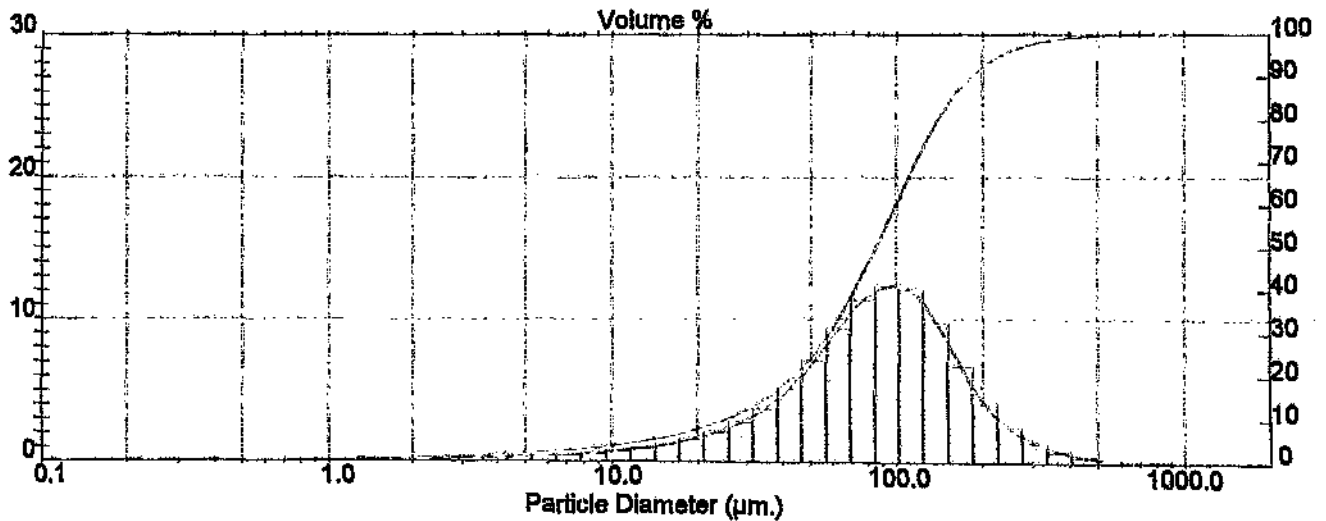
Mode = 101.50 μm

Specific Surface Area = 0.1681 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	0.27	1.32	0.27
1.32	0.20	1.60	0.47
1.60	0.19	1.95	0.63
1.95	0.22	2.38	0.88
2.38	0.25	2.90	1.14
2.90	0.28	3.53	1.42
3.53	0.31	4.30	1.72
4.30	0.35	5.24	2.07
5.24	0.41	6.39	2.48
6.39	0.50	7.78	2.99
7.78	0.61	9.48	3.60
9.48	0.76	11.55	4.36
11.55	0.96	14.08	5.33
14.08	1.24	17.15	6.57
17.15	1.61	20.90	8.17
20.90	2.10	25.48	10.27

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.48	2.77	31.01	13.05
31.01	3.77	37.79	16.82
37.79	5.17	46.03	21.99
46.03	7.11	56.09	29.09
56.09	9.30	68.33	38.39
68.33	11.32	83.26	49.71
83.26	12.31	101.44	62.03
101.44	12.07	123.59	74.09
123.59	9.76	150.57	83.85
150.57	6.76	183.44	90.62
183.44	4.10	223.51	94.72
223.51	2.42	272.31	97.14
272.31	1.39	331.77	98.53
331.77	0.80	404.21	99.33
404.21	0.44	492.47	99.77
492.47	0.23	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:24AM

PF SAMPLES :Run Number 13

9 D P1 - 4.
02/02/96

Sample File Name: GENWATER, Record: 46
Measured on: Tue, Apr 02, 1996 7:31AM Last saved on: Tue, Apr 02, 1996 7:33AM

Source: Analysed

Presentation: 25\$D
Very Polydisperse model

Volume Result

Focus = 30.0 mm.

Residual = 0.352 %

Concentration = 0.041 %

Obscuration = 24.38 %

$d(0.5) = 29.12 \mu\text{m}$

$d(0.1) = 4.50 \mu\text{m}$

$d(0.9) = 79.84 \mu\text{m}$

$D[4, 3] = 36.20 \mu\text{m}$

Span = 2.88

Sauter Mean ($D[3,2]$) = 10.49 μm

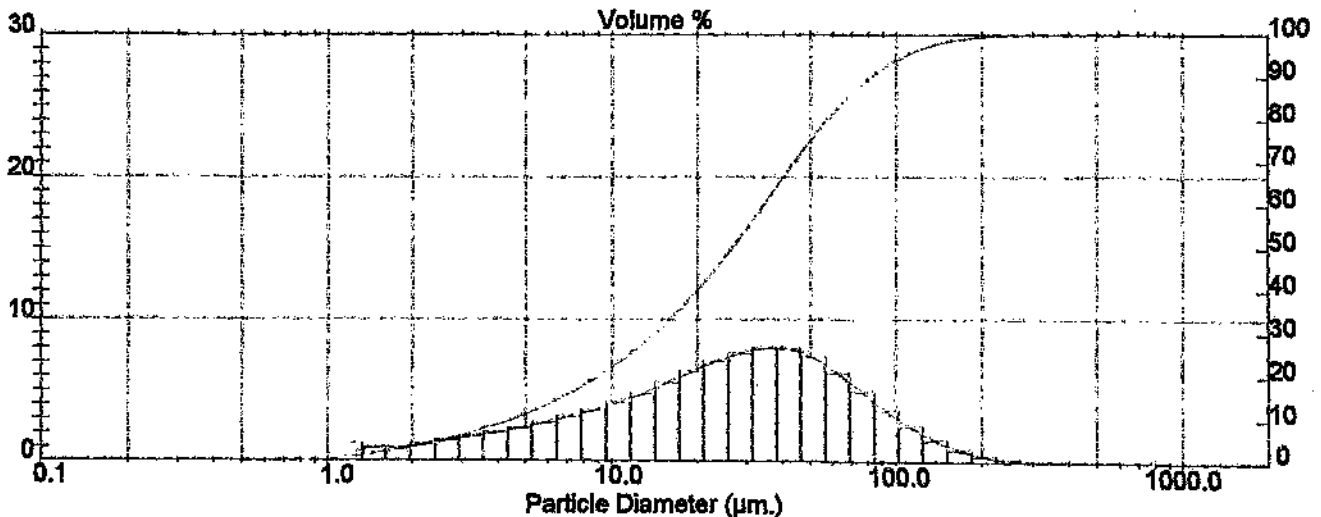
Mode = 37.09 μm

Specific Surface Area = 0.5719 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	1.24	1.32	1.24
1.32	0.95	1.60	2.19
1.60	0.94	1.95	3.13
1.95	1.19	2.38	4.32
2.38	1.46	2.90	5.77
2.90	1.72	3.53	7.49
3.53	2.00	4.30	9.49
4.30	2.31	5.24	11.80
5.24	2.71	6.39	14.51
6.39	3.15	7.78	17.66
7.78	3.59	9.48	21.25
9.48	4.13	11.55	25.38
11.55	4.78	14.08	30.15
14.08	5.53	17.15	35.69
17.15	6.32	20.90	42.01
20.90	7.04	25.46	49.05

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	7.60	31.01	56.64
31.01	7.96	37.79	64.60
37.79	7.88	46.03	72.48
46.03	7.32	56.09	79.80
56.09	6.22	68.33	86.02
68.33	4.91	83.26	90.93
83.26	3.58	101.44	94.51
101.44	2.47	123.59	96.98
123.59	1.52	150.57	98.51
150.57	0.84	183.44	99.35
183.44	0.40	223.51	99.75
223.51	0.17	272.31	99.92
272.31	0.06	331.77	99.98
331.77	0.02	404.21	100.00
404.21	0.00	492.47	100.00
492.47	0.00	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:27AM

PF SAMPLES :Run Number 14

9AP1-4
02/02/96

Sample File Name: GENWATER, Record: 47
Measured on: Tue, Apr 02, 1996 7:36AM Last saved on: Tue, Apr 02, 1996 7:37AM

Source: Analysed

Presentation: 2\$\$D
Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.150 %

Concentration = 0.062 %

Obscuration = 27.57 %

$d(0.5) = 59.50 \mu\text{m}$

$d(0.1) = 5.85 \mu\text{m}$

$d(0.9) = 106.65 \mu\text{m}$

$D[4,3] = 47.36 \mu\text{m}$

Span = 3.03

Mode = 58.57 μm

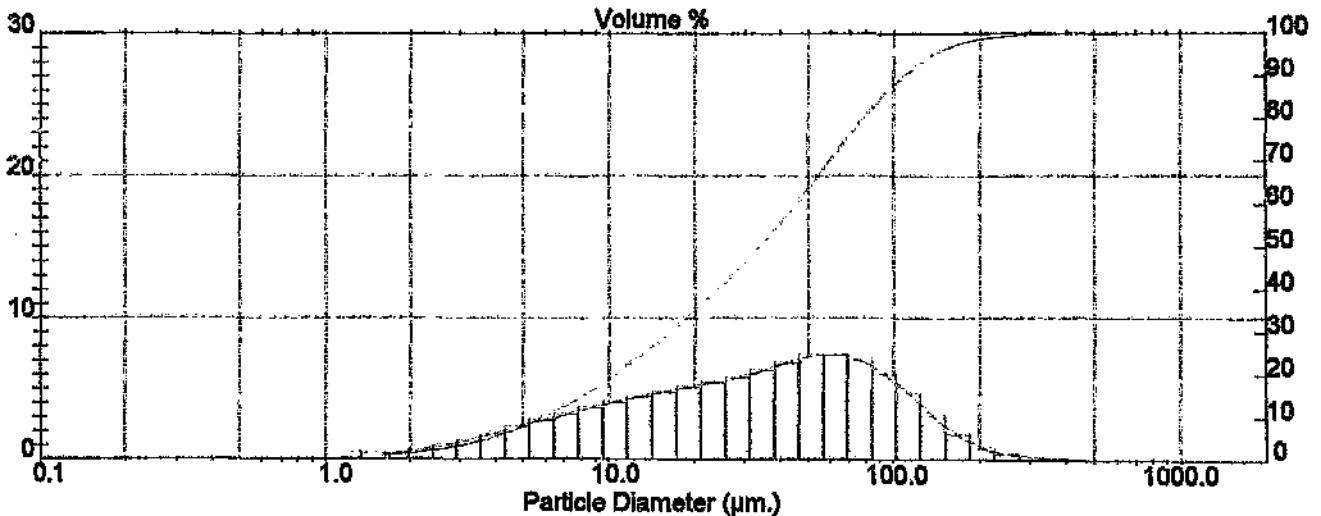
Sauter Mean ($D[3,2]$) = 13.83 μm

Specific Surface Area = 0.4337 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	0.53	1.32	0.53
1.32	0.44	1.60	0.97
1.60	0.45	1.95	1.43
1.95	0.67	2.38	2.10
2.38	0.97	2.90	3.07
2.90	1.34	3.53	4.40
3.53	1.80	4.30	6.21
4.30	2.30	5.24	8.51
5.24	2.77	6.39	11.28
6.39	3.20	7.78	14.48
7.78	3.66	9.48	18.13
9.48	4.06	11.55	22.19
11.55	4.43	14.08	26.62
14.08	4.80	17.15	31.42
17.15	5.13	20.90	36.55
20.90	5.45	25.46	42.00

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	5.80	31.01	47.80
31.01	6.28	37.79	54.08
37.79	6.82	46.03	60.90
46.03	7.32	56.09	68.22
56.09	7.41	68.33	75.62
68.33	7.01	83.26	82.64
83.26	6.03	101.44	88.66
101.44	4.73	123.59	93.40
123.59	3.12	150.57	96.52
150.57	1.79	183.44	98.31
183.44	0.89	223.51	99.20
223.51	0.43	272.31	99.63
272.31	0.20	331.77	99.83
331.77	0.10	404.21	99.93
404.21	0.05	492.47	99.98
492.47	0.02	600.00	100.00





MASTER SIZER X

Version 1.2b

Tue, Apr 02, 1996 10:29AM

PF SAMPLES :Run Number 15

9 AP1 - 4
05/02/96

Sample File Name: GENWATER, Record: 43
Measured on: Tue, Apr 02, 1996 7:39AM Last saved on: Tue, Apr 02, 1996 7:39AM

Source: Analysed

Presentation: 250D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.144 %

Concentration = 0.049 %

Obscuration = 26.68 %

d(0.5) = 25.93 µm

d(0.1) = 4.94 µm

d(0.9) = 87.38 µm

D[4, 3] = 38.65 µm

Span = 3.18

Mode = 47.23 µm

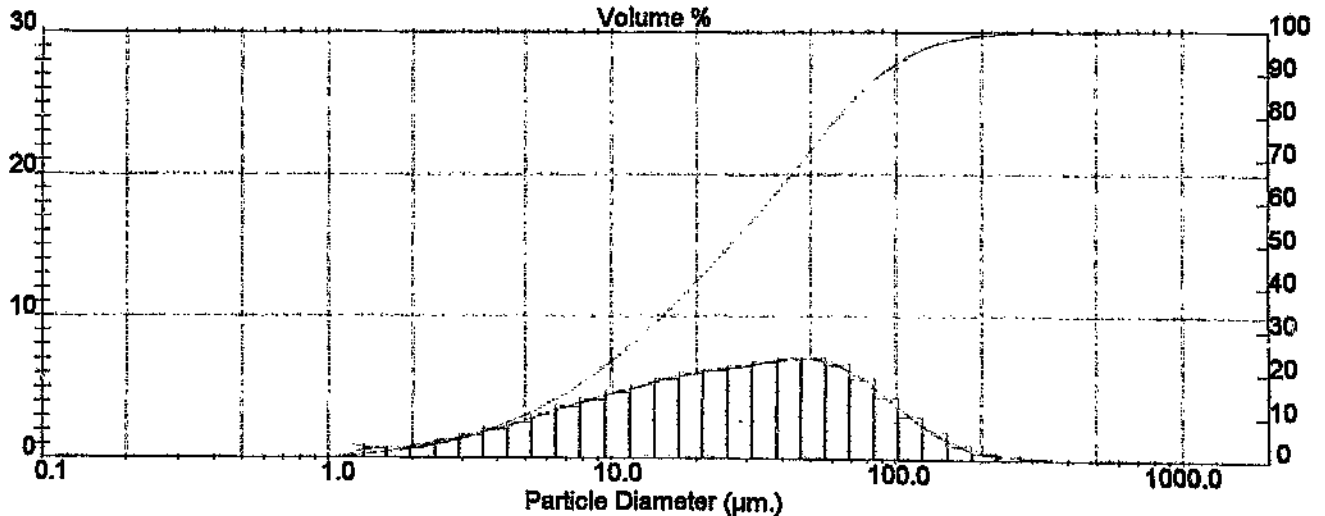
Sauter Mean (D[3,2]) = 11.29 µm

Density = 1.00 gm./c.c.

Specific Surface Area = 0.5313 sq. m./gm

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.89	1.32	0.89
1.32	0.70	1.60	1.60
1.60	0.71	1.95	2.30
1.95	0.97	2.38	3.27
2.38	1.29	2.90	4.56
2.90	1.64	3.53	6.20
3.53	2.07	4.30	8.27
4.30	2.55	5.24	10.82
5.24	3.10	6.39	13.92
6.39	3.67	7.78	17.58
7.78	4.19	9.48	21.78
9.48	4.69	11.55	26.47
11.55	5.14	14.08	31.61
14.08	5.60	17.15	37.21
17.15	5.97	20.90	43.18
20.90	6.24	25.46	49.42

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	6.46	31.01	55.88
31.01	6.76	37.79	62.64
37.79	7.00	46.03	69.64
46.03	7.03	56.09	76.65
56.09	6.55	68.33	83.22
68.33	5.60	83.26	88.82
83.26	4.30	101.44	93.13
101.44	3.03	123.59	96.15
123.59	1.84	150.57	97.99
150.57	1.01	183.44	99.00
183.44	0.50	223.51	99.50
223.51	0.26	272.31	99.76
272.31	0.13	331.77	99.89
331.77	0.07	404.21	99.95
404.21	0.03	492.47	99.99
492.47	0.01	600.00	100.00





MASTER SIZER X

Version 1.2b

Tue, Apr 02, 1996 10:32AM

PF SAMPLES :Run Number 16

9B P1-4
05/02/96

Sample File Name: GENWATER, Record: 49
Measured on: Tue, Apr 02, 1996 7:41AM Last saved on: Tue, Apr 02, 1996 7:42AM

Source: Analysed

Presentation: 25SD
Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.198 %
d(0.5) = 57.97 µm
D[4, 3] = 71.23 µm
Sauter Mean (D[3,2]) = 31.26 µm
Specific Surface Area = 0.1918 sq. m. / gm

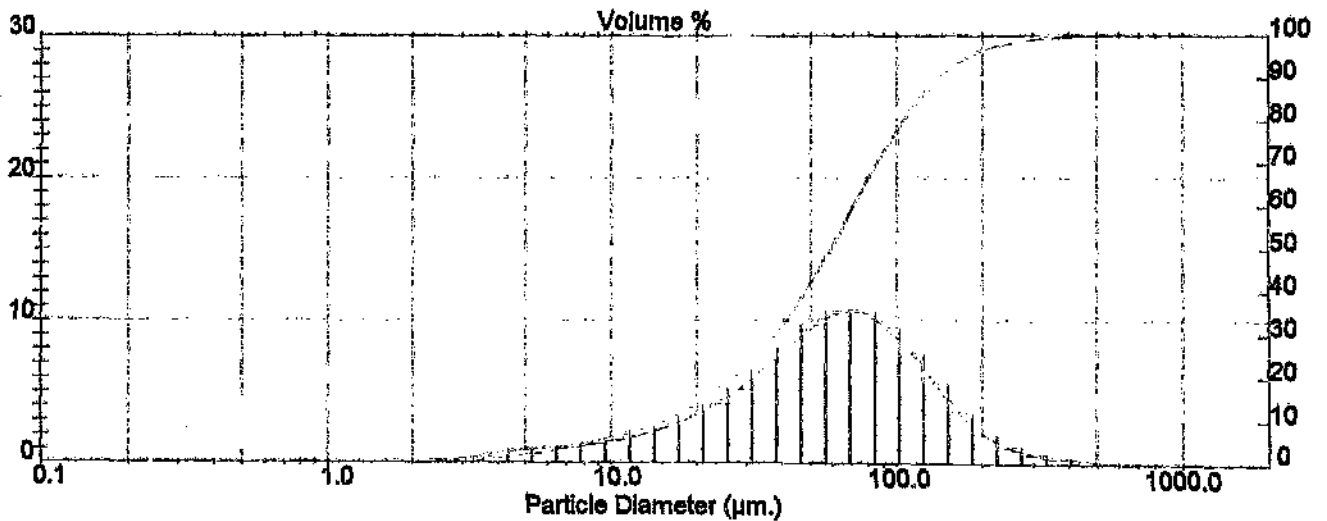
Concentration = 0.129 %
d(0.1) = 15.29 µm
Span = 2.13

Obscuration = 25.73 %
d(0.9) = 136.54 µm

Mode = 69.10 µm
Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.03	1.32	0.03
1.32	0.03	1.60	0.06
1.60	0.04	1.95	0.09
1.95	0.07	2.38	0.17
2.38	0.15	2.90	0.32
2.90	0.30	3.53	0.61
3.53	0.55	4.30	1.17
4.30	0.83	5.24	2.00
5.24	0.98	6.39	2.98
6.39	1.08	7.78	4.06
7.78	1.33	9.48	5.39
9.48	1.61	11.55	7.00
11.55	2.01	14.08	9.01
14.08	2.54	17.15	11.55
17.15	3.23	20.90	14.78
20.90	4.09	25.46	18.87

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	5.15	31.01	24.02
31.01	6.53	37.79	30.55
37.79	8.08	46.03	38.63
46.03	9.64	56.09	48.27
56.09	10.56	68.33	58.84
68.33	10.56	83.26	69.40
83.26	9.40	101.44	78.80
101.44	7.74	123.59	86.54
123.59	5.51	150.57	92.06
150.57	3.53	183.44	95.59
183.44	2.02	223.51	97.60
223.51	1.14	272.31	98.75
272.31	0.63	331.77	99.38
331.77	0.35	404.21	99.73
404.21	0.18	492.47	99.91
492.47	0.09	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:35AM

PF SAMPLES :Run Number 17

9 DP1 - 4
05/02/96

Sample File Name: GENWATER, Record: 50
Measured on: Tue, Apr 02, 1996 7:44AM Last saved on: Tue, Apr 02, 1996 7:45AM

Source: Analysed

Presentation: 2\$D
Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.187 %
d (0.5) = 52.09 µm
D [4, 3] = 86.72 µm
Sauter Mean (D[3,2]) = 18.46 µm
Specific Surface Area = 0.3251 sq. m. / gm

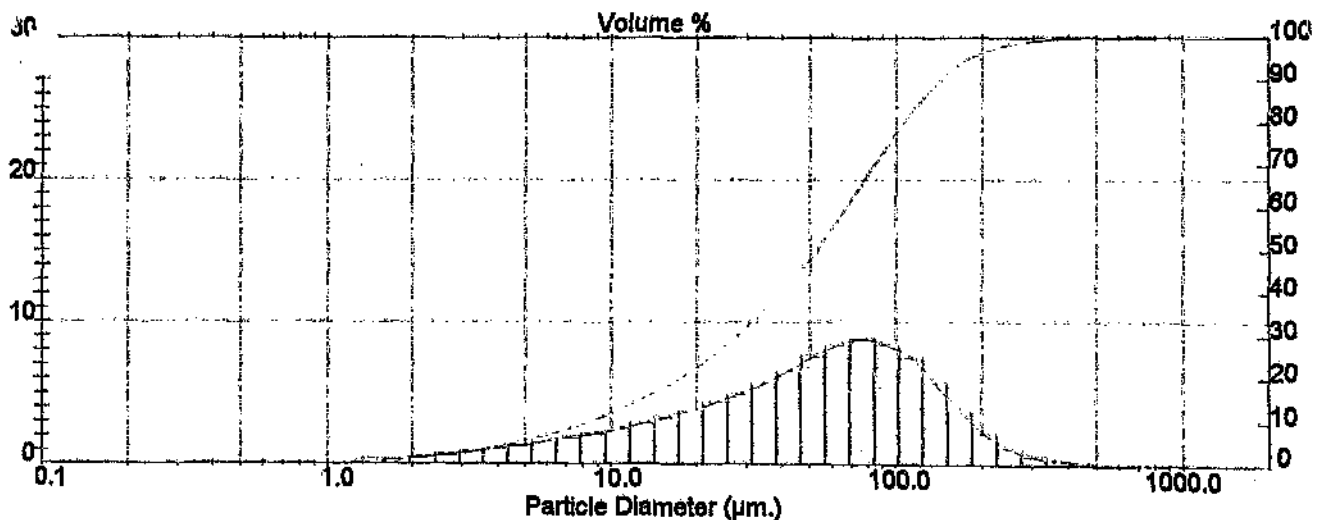
Concentration = 0.063 %
d (0.1) = 8.34 µm
Span = 2.55

Obscuration = 21.93 %
d (0.9) = 141.00 µm

Mode = 75.40 µm
Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.48	1.32	0.48
1.32	0.38	1.60	0.86
1.60	0.38	1.95	1.24
1.95	0.53	2.38	1.77
2.38	0.70	2.90	2.46
2.90	0.89	3.53	3.36
3.53	1.12	4.30	4.47
4.30	1.35	5.24	5.83
5.24	1.60	6.39	7.43
6.39	1.86	7.78	9.29
7.78	2.14	9.48	11.43
9.48	2.48	11.55	13.91
11.55	2.86	14.08	16.78
14.08	3.31	17.15	20.09
17.15	3.79	20.90	23.88
20.90	4.31	25.46	28.19

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	4.90	31.01	33.09
31.01	5.68	37.79	38.77
37.79	6.58	46.03	45.35
46.03	7.60	56.09	52.95
56.09	8.37	68.33	61.32
68.33	8.75	83.26	70.07
83.26	8.38	101.44	78.46
101.44	7.50	123.59	85.96
123.59	5.73	150.57	91.68
150.57	3.82	183.44	95.50
183.44	2.18	223.51	97.69
223.51	1.19	272.31	98.88
272.31	0.61	331.77	99.48
331.77	0.30	404.21	99.79
404.21	0.15	492.47	99.93
492.47	0.07	600.00	100.00



Malvern MASTER SIZER X

Version 1.2b

Tue, Apr 02, 1996 10:37AM

PF SAMPLES :Run Number 18

9EP1-4
05/02/96

Sample File Name: GENWATER, Record: 51
Measured on: Tue, Apr 02, 1996 7:47AM Last saved on: Tue, Apr 02, 1996 7:48AM

Source: Analysed

Presentation: 25\$D

Vary Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.193 %

Concentration = 0.062 %

Obscuration = 22.82 %

d [4, 3] = 53.20 µm

d (0.1) = 7.82 µm

d (0.9) = 137.97 µm

D [4, 3] = 65.20 µm

Span = 2.58

Mode = 73.79 µm

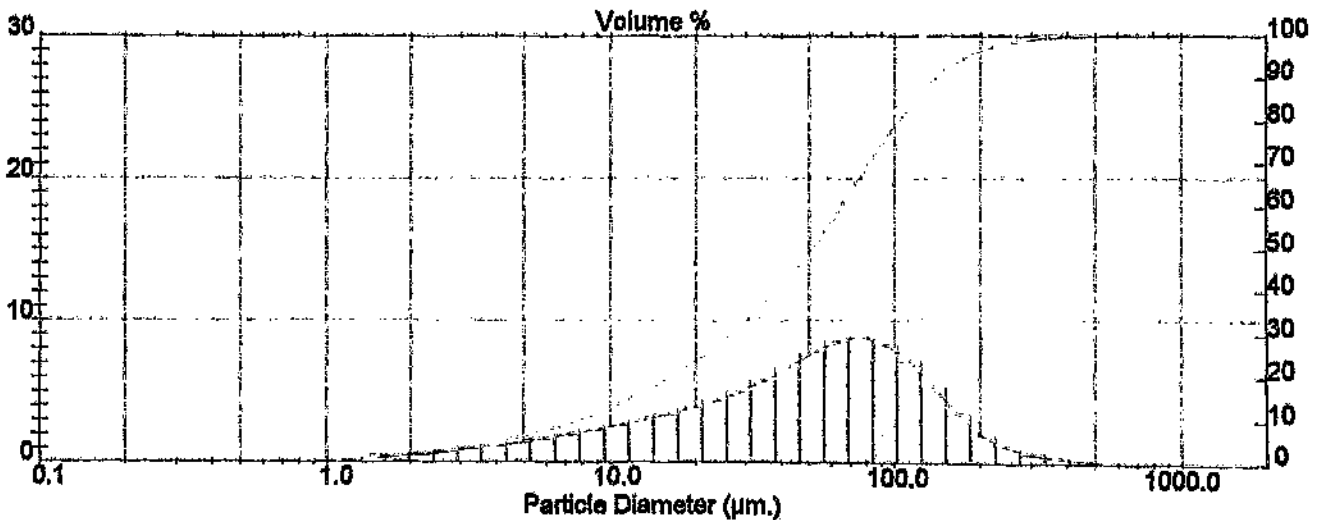
Sauter Mean (D[3,2]) = 17.23 µm

Density = 1.00 gm. / c.c.

Specific Surface Area = 0.3463 sq. m. / gm

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.59	1.32	0.59
1.32	0.46	1.60	1.06
1.60	0.46	1.95	1.51
1.95	0.60	2.38	2.11
2.38	0.77	2.90	2.88
2.90	0.94	3.53	3.82
3.53	1.14	4.30	4.97
4.30	1.37	5.24	6.34
5.24	1.65	6.39	7.99
6.39	1.95	7.78	9.94
7.78	2.25	9.48	12.19
9.48	2.60	11.55	14.79
11.55	2.98	14.08	17.75
14.08	3.39	17.15	21.15
17.15	3.84	20.90	24.99
20.90	4.34	25.46	29.33

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	4.91	31.01	34.24
31.01	5.71	37.79	39.95
37.79	6.64	46.03	46.59
46.03	7.69	56.09	54.28
56.09	8.44	68.33	62.72
68.33	8.75	83.26	71.47
83.26	8.21	101.44	79.67
101.44	7.13	123.59	86.81
123.59	5.30	150.57	92.10
150.57	3.49	183.44	95.59
183.44	2.02	223.51	97.61
223.51	1.16	272.31	98.77
272.31	0.64	331.77	99.41
331.77	0.34	404.21	99.76
404.21	0.17	492.47	99.93
492.47	0.07	600.00	100.00



Malvern MASTER SIZER X

Version 1.2b

Tue, Apr 02, 1996 10:40AM

PF SAMPLES :Run Number 19

9AP1-4
07/02/96

Sample File Name: GENWATER, Record: 52
Measured on: Tue, Apr 02, 1996 7:50AM Last saved on: Tue, Apr 02, 1996 7:51AM

Source: Analysed

Presentation: 25% D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.364 %

Concentration = 0.038 %

Obscuration = 25.91 %

$d(0.5) = 32.82 \mu\text{m}$

$d(0.1) = 3.75 \mu\text{m}$

$d(0.9) = 103.08 \mu\text{m}$

$D[4,3] = 40.72 \mu\text{m}$

Span = 4.45

Mode = 55.42 μm

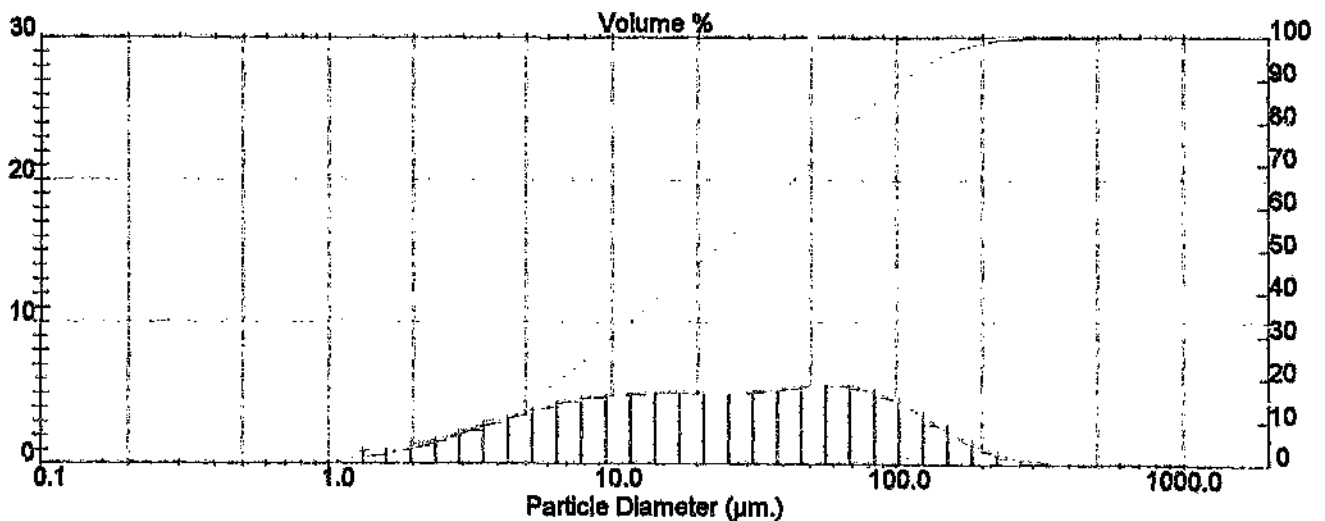
Sauter Mean ($D[3,2]$) = 9.15 μm

Density = 1.00 gm./c.c.

Specific Surface Area = 0.6561 sq. m./gm

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	1.34	1.32	1.34
1.32	1.06	1.60	2.40
1.60	1.08	1.95	3.48
1.95	1.45	2.38	4.92
2.38	1.88	2.90	6.81
2.90	2.36	3.53	9.16
3.53	2.89	4.30	12.06
4.30	3.44	5.24	15.50
5.24	3.97	6.39	19.47
6.39	4.41	7.78	23.89
7.78	4.70	9.48	28.59
9.48	4.86	11.55	33.45
11.55	4.94	14.08	38.39
14.08	4.99	17.15	43.37
17.15	4.98	20.90	48.35
20.90	4.95	25.46	53.30

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	4.96	31.01	58.25
31.01	5.10	37.79	63.36
37.79	5.29	46.03	68.64
46.03	5.50	56.09	74.14
56.09	5.49	68.33	79.63
68.33	5.30	83.26	84.93
83.26	4.72	101.44	89.65
101.44	3.94	123.59	93.58
123.59	2.84	150.57	96.43
150.57	1.81	183.44	98.24
183.44	0.98	223.51	99.22
223.51	0.49	272.31	99.71
272.31	0.21	331.77	99.92
331.77	0.07	404.21	99.98
404.21	0.01	492.47	100.00
492.47	0.00	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:42AM

PF SAMPLES :Run Number 20

9 B P1-4
07/02/96

Sample File Name: GENWATER, Record: 53
Measured on: Tue, Apr 02, 1996 7:53AM Last saved on: Tue, Apr 02, 1996 7:54AM

Source: Analysed

Presentation: 25\$D
Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.277 %

Concentration = 0.032 %

Obscuration = 25.73 %

d(0.5) = 15.96 μ m

d(0.1) = 3.37 μ m

d(0.9) = 61.77 μ m

D[4,3] = 26.19 μ m

Span = 3.68

Mode = 16.80 μ m

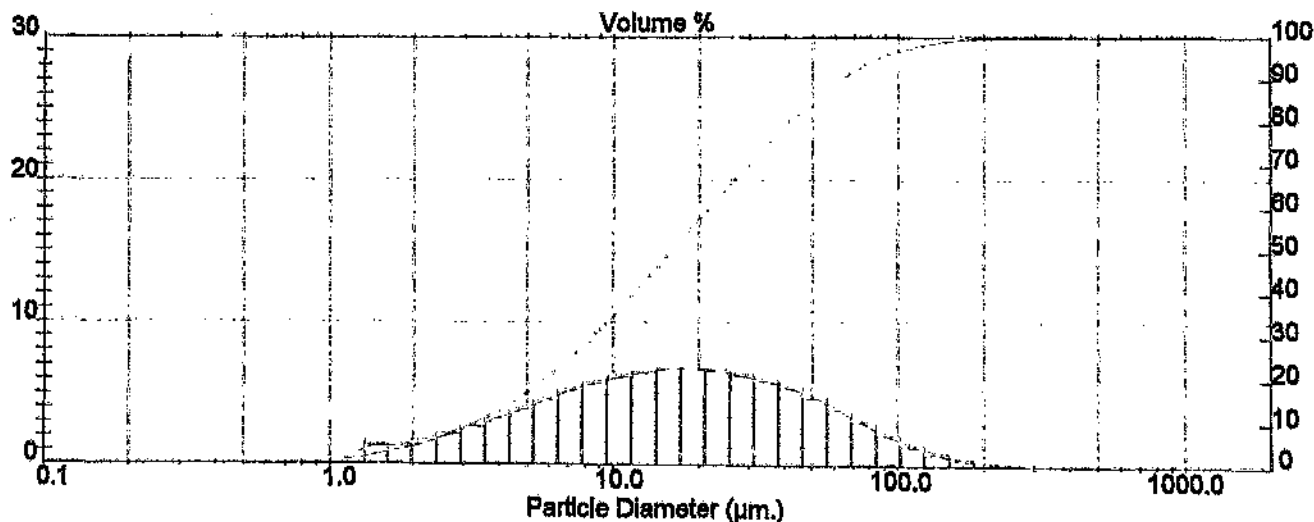
Sauter Mean (D[3,2]) = 7.83 μ m

Specific Surface Area = 0.7662 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) μ m	Result In %	Size (Hi) μ m	Result Below %
0.50	1.62	1.32	1.62
1.32	1.27	1.60	2.90
1.60	1.27	1.95	4.17
1.95	1.68	2.38	5.85
2.38	2.15	2.90	8.00
2.90	2.65	3.53	10.65
3.53	3.22	4.30	13.87
4.30	3.84	5.24	17.71
5.24	4.53	6.39	22.24
6.39	5.19	7.78	27.43
7.78	5.72	9.48	33.14
9.48	6.16	11.55	39.30
11.55	6.46	14.08	45.76
14.08	6.68	17.15	52.44
17.15	6.71	20.90	59.15
20.90	6.55	25.46	65.70

Size (Lo) μ m	Result In %	Size (Hi) μ m	Result Below %
25.46	6.25	31.01	71.95
31.01	5.90	37.79	77.85
37.79	5.40	46.03	83.25
46.03	4.74	56.09	87.99
56.09	3.88	68.33	91.88
68.33	2.99	83.26	94.86
83.26	2.12	101.44	96.98
101.44	1.41	123.59	98.39
123.59	0.84	150.57	99.22
150.57	0.45	183.44	99.67
183.44	0.20	223.51	99.87
223.51	0.09	272.31	99.96
272.31	0.03	331.77	99.99
331.77	0.01	404.21	100.00
404.21	0.00	492.47	100.00
492.47	0.00	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:45AM

PF SAMPLES :Run Number 21

9DP1-4
07/02/96

Sample File Name: GENWATER, Record: 54
Measured on: Tue, Apr 02, 1996 7:56AM Last saved on: Tue, Apr 02, 1996 7:57AM

Source: Analysed

Presentation: 25\$D
Vary Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.282 %

Concentration = 0.037 %

Obscuration = 28.24 %

$d(0.1) = 17.07 \mu\text{m}$

$d(0.1) = 3.41 \mu\text{m}$

$d(0.9) = 57.47 \mu\text{m}$

$D[4,3] = 25.32 \mu\text{m}$

Span = 3.17

Sauter Mean ($D[3,2]$) = 7.97 μm

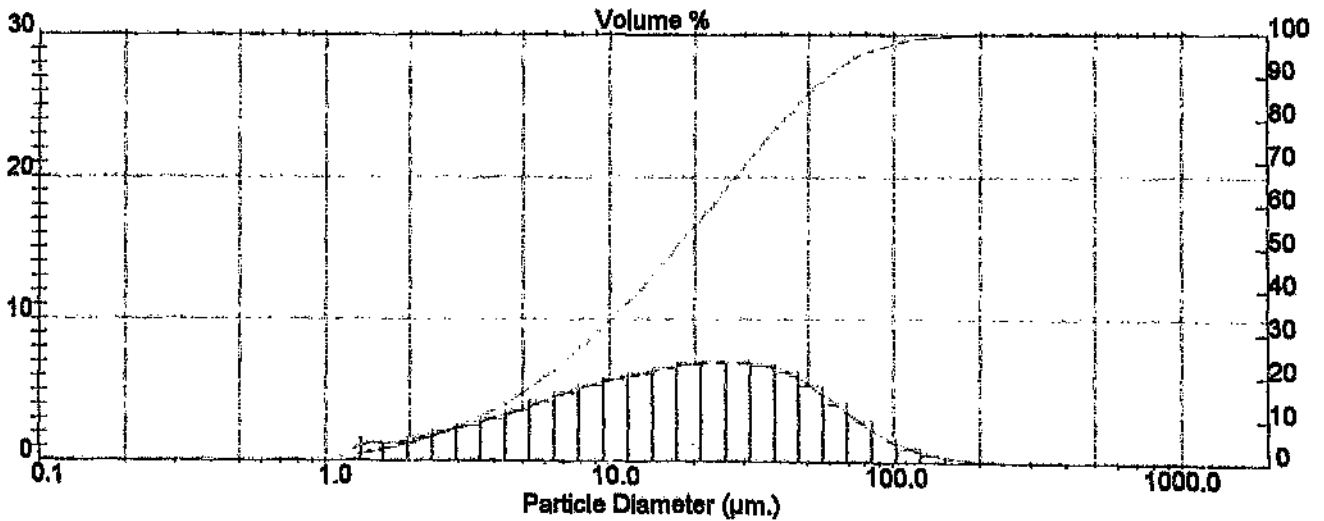
Specific Surface Area = 0.7526 sq. m. / gm

Mode = 24.42 μm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	1.65	1.32	1.65
1.32	1.28	1.60	2.93
1.60	1.27	1.95	4.20
1.95	1.66	2.38	5.86
2.38	2.09	2.90	7.96
2.90	2.54	3.53	10.50
3.53	3.05	4.30	13.55
4.30	3.59	5.24	17.14
5.24	4.23	6.39	21.37
6.39	4.85	7.78	26.22
7.78	5.35	9.48	31.58
9.48	5.81	11.55	37.39
11.55	6.20	14.08	43.58
14.08	6.58	17.15	50.16
17.15	6.86	20.90	57.02
20.90	7.00	25.46	64.03

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	6.97	31.01	71.00
31.01	6.79	37.79	77.79
37.79	6.27	46.03	84.05
46.03	5.37	56.09	89.43
56.09	4.10	68.33	93.53
68.33	2.83	83.26	96.36
83.26	1.75	101.44	98.11
101.44	1.01	123.59	99.12
123.59	0.51	150.57	99.63
150.57	0.24	183.44	99.87
183.44	0.09	223.51	99.96
223.51	0.03	272.31	99.99
272.31	0.01	331.77	100.00
331.77	0.00	404.21	100.00
404.21	0.00	492.47	100.00
492.47	0.00	600.00	100.00



MALVERN MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:47AM

PF SAMPLES :Run Number 22

9 EPI-4
07/02/96

Sample File Name: GENWATER, Record: 55
Measured on: Tue, Apr 02, 1996 7:58AM Last saved on: Tue, Apr 02, 1996 8:00AM

Source: Analysed

Presentation: 25\$D
Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.191 %

Concentration = 0.065 %

Obscuration = 28.05 %

d (0.5) = 45.65 µm

d (0.1) = 5.79 µm

d (0.9) = 127.38 µm

D [4, 3] = 58.90 µm

Span = 2.66

Mode = 75.12 µm

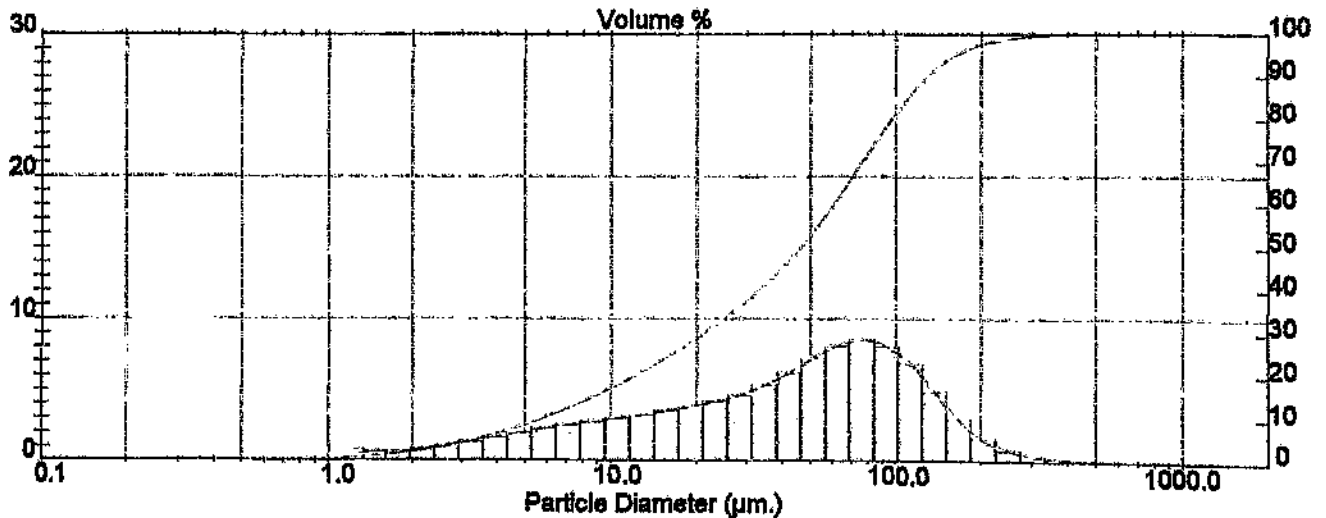
Sauter Mean (D[3,2]) = 14.12 µm

Density = 1.00 gm. / c.c.

Specific Surface Area = 0.4250 sq. m. / gm

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.78	1.32	0.78
1.32	0.62	1.60	1.40
1.60	0.63	1.95	2.03
1.95	0.84	2.38	2.86
2.38	1.09	2.90	3.95
2.90	1.35	3.53	5.30
3.53	1.65	4.30	6.95
4.30	1.85	5.24	8.90
5.24	2.24	6.39	11.14
6.39	2.51	7.78	13.65
7.78	2.75	9.48	16.40
9.48	2.99	11.55	19.39
11.55	3.23	14.06	22.62
14.06	3.52	17.15	26.13
17.15	3.82	20.90	29.96
20.90	4.18	25.46	34.14

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	4.62	31.01	38.76
31.01	5.32	37.79	44.08
37.79	6.19	46.03	50.27
46.03	7.26	56.09	57.53
56.09	8.11	68.33	65.65
68.33	8.54	83.26	74.19
83.26	8.05	101.44	82.24
101.44	6.87	123.59	89.11
123.59	4.86	150.57	93.97
150.57	2.96	183.44	96.93
183.44	1.55	223.51	98.48
223.51	0.80	272.31	99.28
272.31	0.39	331.77	99.67
331.77	0.20	404.21	99.87
404.21	0.09	492.47	99.96
492.47	0.04	600.00	100.00





MASTER SIZER X

Version 1.2b

Tue, Apr 02, 1996 10:50AM

PF SAMPLES :Run Number 23

9FP1-4
07/02/96

Sample File Name: GENWATER, Record: 56
Measured on: Tue, Apr 02, 1996 8:01AM Last saved on: Tue, Apr 02, 1996 8:02AM

Source: Analysed

Presentation: 255D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.302 %

Concentration = 0.041 %

Obscuration = 28.83 %

$d(0.1) = 17.94 \mu\text{m}$

$d(0.1) = 3.78 \mu\text{m}$

$d(0.9) = 73.98 \mu\text{m}$

$D[4,3] = 30.83 \mu\text{m}$

Span = 3.91

Mode = 20.27 μm

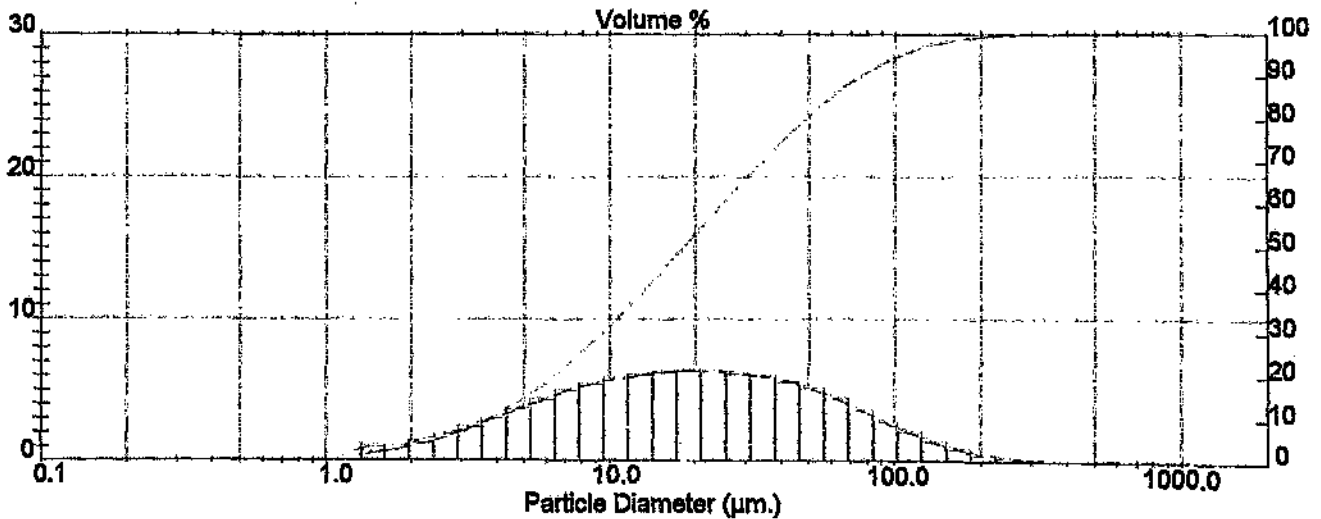
Sauter Mean ($D[3,2]$) = 8.72 μm

Specific Surface Area = 0.6884 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	1.29	1.32	1.29
1.32	1.02	1.60	2.31
1.60	1.03	1.95	3.34
1.95	1.41	2.38	4.76
2.38	1.87	2.90	6.63
2.90	2.40	3.53	9.03
3.53	3.00	4.30	12.03
4.30	3.66	5.24	15.69
5.24	4.34	6.39	20.02
6.39	4.96	7.78	24.99
7.78	5.43	9.48	30.42
9.48	5.80	11.55	36.22
11.55	6.06	14.08	42.27
14.08	6.27	17.15	48.55
17.15	6.35	20.90	54.90
20.90	6.31	25.46	61.21

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	6.16	31.01	67.38
31.01	5.96	37.79	73.34
37.79	5.61	46.03	78.95
46.03	5.11	56.09	84.06
56.09	4.39	68.33	88.45
68.33	3.61	83.26	92.06
83.26	2.79	101.44	94.85
101.44	2.09	123.59	96.94
123.59	1.41	150.57	98.34
150.57	0.86	183.44	99.21
183.44	0.45	223.51	99.66
223.51	0.22	272.31	99.88
272.31	0.09	331.77	99.96
331.77	0.03	404.21	99.99
404.21	0.01	492.47	100.00
492.47	0.00	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:52AM

PF SAMPLES :Run Number 24

9AP1-4
08/02/96

Sample File Name: GENWATER, Record: 57
Measured on: Tue, Apr 02, 1996 9:14AM Last saved on: Tue, Apr 02, 1996 9:18AM

Source: Analyzed

Presentation: 2\$\$D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.261 %

$d(0.5) = 44.25 \mu\text{m}$

$D[4, 3] = 55.03 \mu\text{m}$

Seuter Mean ($D[3, 2]$) = 18.35 μm

Specific Surface Area = 0.3271 sq. m. / gm

Concentration = 0.067 %

$d(0.1) = 8.74 \mu\text{m}$

Span = 2.31

Obscuration = 23.12 %

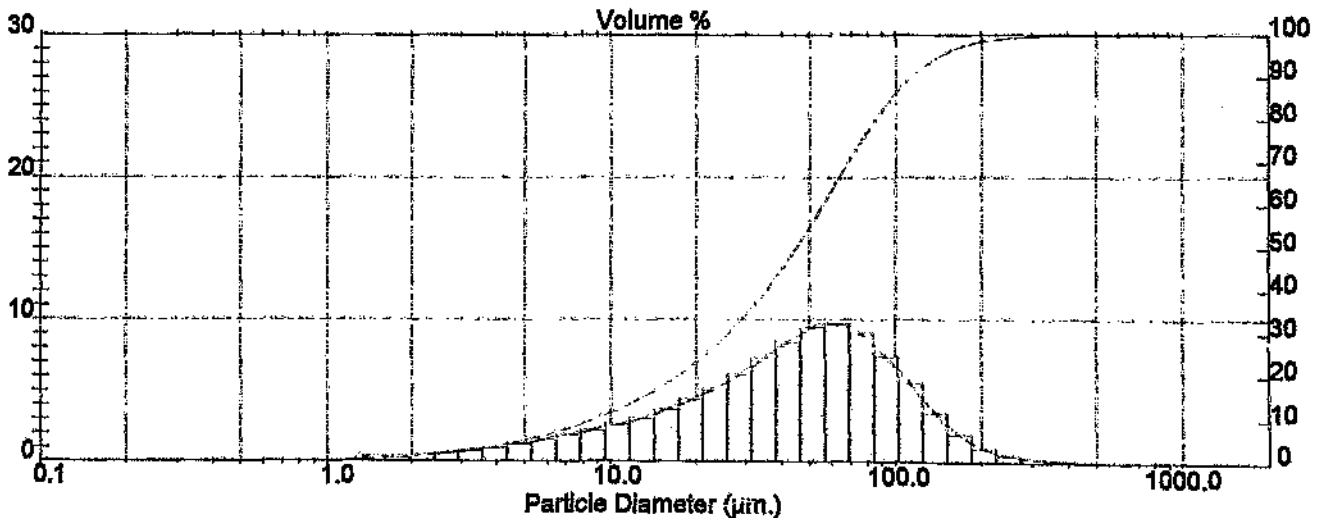
$d(0.9) = 111.64 \mu\text{m}$

Mode = 58.80 μm

Density = 1.00 gm. / c.c.

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
0.50	0.44	1.32	0.44
1.32	0.35	1.60	0.79
1.60	0.34	1.95	1.13
1.95	0.47	2.38	1.61
2.38	0.63	2.90	2.23
2.90	0.81	3.53	3.04
3.53	1.02	4.30	4.06
4.30	1.26	5.24	5.32
5.24	1.55	6.39	6.87
6.39	1.87	7.78	8.74
7.78	2.21	9.48	10.95
9.48	2.61	11.55	13.57
11.55	3.07	14.08	16.63
14.08	3.66	17.15	20.29
17.15	4.34	20.90	24.64
20.90	5.17	25.46	29.81

Size (Lo) μm	Result In %	Size (Hi) μm	Result Below %
25.46	6.09	31.01	35.90
31.01	7.26	37.79	43.16
37.79	8.41	46.03	51.57
46.03	9.46	56.09	61.03
56.09	9.67	68.33	70.70
68.33	9.01	83.26	79.72
83.26	7.37	101.44	87.09
101.44	5.49	123.59	92.58
123.59	3.47	150.57	96.05
150.57	1.96	183.44	96.01
183.44	0.99	223.51	99.00
223.51	0.51	272.31	99.51
272.31	0.26	331.77	99.77
331.77	0.13	404.21	99.90
404.21	0.07	492.47	99.97
492.47	0.03	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:55AM

PF SAMPLES :Run Number 25

9CP1-4
08/02/96

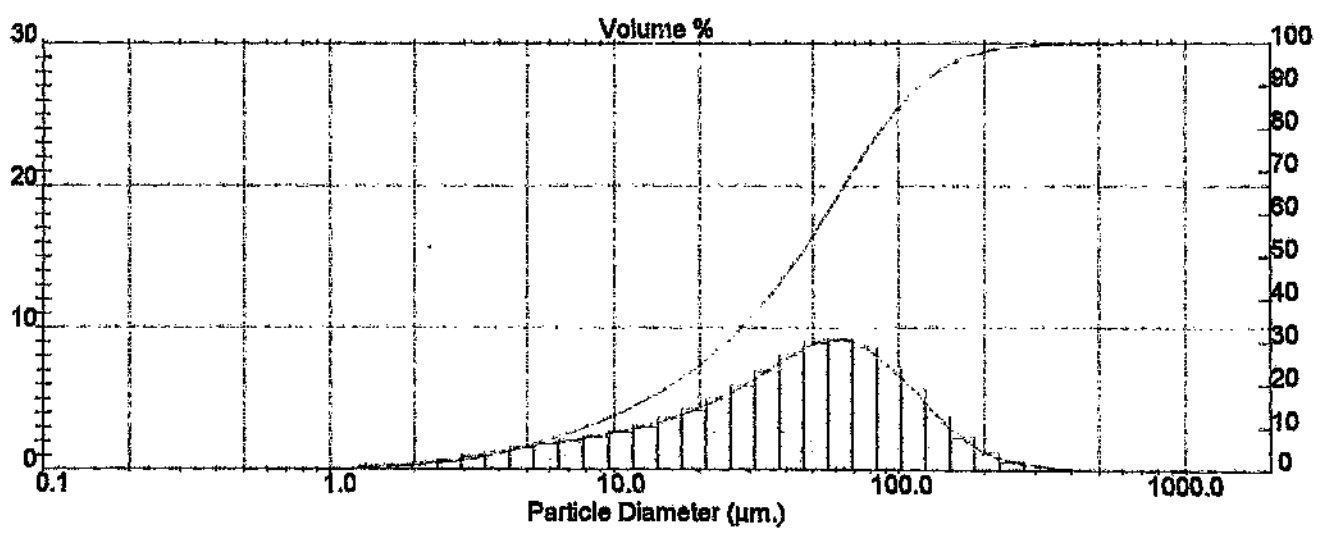
Sample File Name: GENWATER, Record: 58 Source: Analysed
 Measured on: Tue, Apr 02, 1996 9:20AM Last saved on: Tue, Apr 02, 1996 9:20AM

Presentation: 255D
 Very Polydisperse model Volume Result Focus = 300 mm.

Residual = 0.226 % Concentration = 0.068 % Obscuration = 23.92 %
 d(0.5) = 44.20 µm d(0.1) = 7.87 µm d(0.9) = 117.03 µm
 D[4,3] = 56.23 µm Span = 2.47
 Sauter Mean (D[3,2]) = 17.87 µm Mode = 59.02 µm
 Specific Surface Area = 0.3357 sq. m. / gm Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	0.39	1.32	0.39
1.32	0.32	1.60	0.70
1.60	0.33	1.95	1.03
1.95	0.48	2.38	1.52
2.38	0.69	2.90	2.20
2.90	0.94	3.53	3.14
3.53	1.24	4.30	4.38
4.30	1.56	5.24	5.94
5.24	1.84	6.39	7.78
6.39	2.09	7.78	9.87
7.78	2.37	9.48	12.24
9.48	2.70	11.55	14.94
11.55	3.09	14.08	18.03
14.08	3.62	17.15	21.65
17.15	4.24	20.90	25.89
20.90	5.00	25.46	30.89

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	5.87	31.01	36.76
31.01	6.96	37.79	43.72
37.79	8.02	46.03	51.74
46.03	8.95	56.09	60.69
56.09	9.14	68.33	69.83
68.33	8.62	83.26	78.45
83.26	7.26	101.44	85.71
101.44	5.68	123.59	91.39
123.59	3.83	150.57	95.22
150.57	2.30	183.44	97.52
183.44	1.22	223.51	98.74
223.51	0.65	272.31	99.39
272.31	0.33	331.77	99.71
331.77	0.17	404.21	99.88
404.21	0.08	492.47	99.96
492.47	0.04	600.00	100.00



Malvern MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 10:57AM

PF SAMPLES : Run Number 26

9 D P1 - 4
08/02/96

Sample File Name: GENWATER, Record: 59 Source: Analysed
Measured on: Tue, Apr 02, 1996 9:22AM Last saved on: Tue, Apr 02, 1996 9:23AM

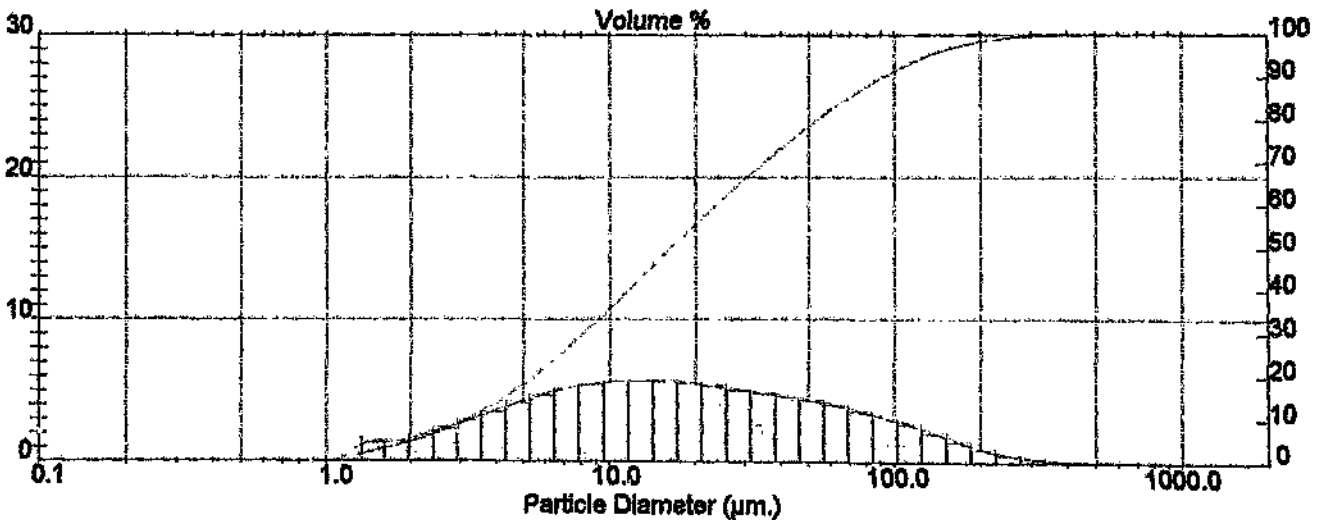
Presentation: 25\$D
Very Polydisperse model

Volume Result Focus = 300 mm.

Residual = 0.304 % Concentration = 0.033 % Obscuration = 27.02 %
d(0.5) = 16.20 µm d(0.1) = 3.18 µm d(0.9) = 89.45 µm
D[4,3] = 34.22 µm Span = 5.32
Sauter Mean (D[3,2]) = 7.63 µm Mode = 14.94 µm
Specific Surface Area = 0.7859 sq. m. / gm Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.60	1.75	1.32	1.75
1.32	1.38	1.60	3.13
1.60	1.40	1.95	4.53
1.95	1.84	2.38	6.37
2.38	2.36	2.90	8.73
2.90	2.91	3.53	11.64
3.53	3.52	4.30	15.16
4.30	4.12	5.24	19.28
5.24	4.71	6.39	23.99
6.39	5.18	7.78	29.17
7.78	5.47	9.48	34.64
9.48	5.64	11.55	40.28
11.55	5.68	14.08	45.95
14.08	5.67	17.15	51.62
17.15	5.53	20.90	57.16
20.90	5.32	25.46	62.47

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.46	5.05	31.01	67.52
31.01	4.83	37.79	72.35
37.79	4.58	46.03	76.93
46.03	4.33	56.08	81.26
56.08	3.96	68.33	85.22
68.33	3.59	83.26	88.81
83.26	3.12	101.44	91.93
101.44	2.66	123.59	94.59
123.59	2.08	150.57	96.65
150.57	1.47	183.44	98.12
183.44	0.91	223.51	99.03
223.51	0.54	272.31	99.56
272.31	0.27	331.77	99.84
331.77	0.12	404.21	99.95
404.21	0.04	492.47	99.99
492.47	0.01	630.00	100.00





MASTER SIZER X

Version 1.2b

Tue, Apr 02, 1996 11:00AM

Pr SAMPLES :Run Number 27

9EP1-4
08/0296

Sample File Name: GENWATER, Record: 60
Measured on: Tue, Apr 02, 1996 9:24AM Last saved on: Tue, Apr 02, 1996 9:25AM

Source: Analysed

Presentation: 25\$D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.251 %

Concentration = 0.072 %

Obscuration = 23.87 %

d(0.5) = 48.83 µm

d(0.1) = 7.89 µm

d(0.9) = 132.53 µm

D[4,3] = 62.83 µm

Span = 2.55

Sauter Mean (D[3,2]) = 19.12 µm

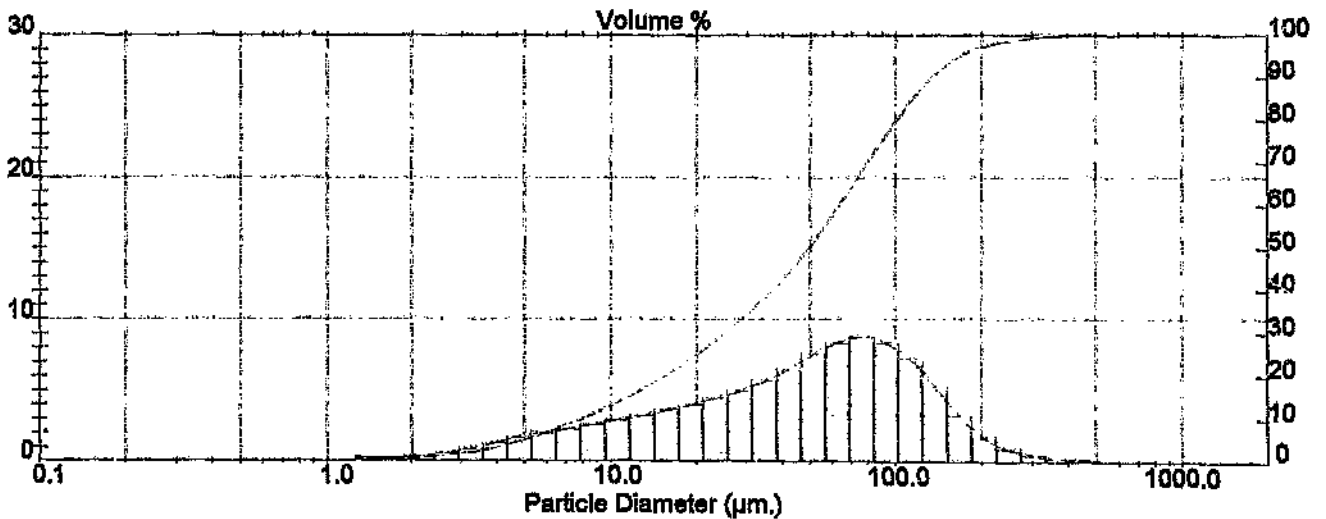
Mode = 75.37 µm

Specific Surface Area = 0.3138 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) µm	Result in %	Size (Hi) µm	Result Below %
0.50	0.26	1.32	0.26
1.32	0.22	1.60	0.48
1.60	0.24	1.95	0.72
1.95	0.38	2.38	1.09
2.38	0.59	2.90	1.68
2.90	0.88	3.53	2.57
3.53	1.28	4.30	3.84
4.30	1.69	5.24	5.53
5.24	2.01	6.39	7.54
6.39	2.28	7.78	9.83
7.78	2.61	9.48	12.44
9.48	2.92	11.55	15.36
11.55	3.23	14.08	18.59
14.08	3.61	17.15	22.21
17.15	3.99	20.90	26.20
20.90	4.44	25.46	30.64

Size (Lo) µm	Result in %	Size (Hi) µm	Result Below %
25.46	4.94	31.01	35.57
31.01	5.67	37.79	41.25
37.79	6.52	46.03	47.77
46.03	7.55	56.09	55.32
56.09	8.31	68.33	63.63
68.33	8.74	83.26	72.37
83.26	8.29	101.44	80.66
101.44	7.23	123.59	87.89
123.59	5.25	150.57	93.15
150.57	3.28	183.44	96.43
183.44	1.74	223.51	98.17
223.51	0.90	272.31	99.07
272.31	0.45	331.77	99.53
331.77	0.25	404.21	99.77
404.21	0.14	492.47	99.91
492.47	0.09	600.00	100.00





MASTERSIZER X

Version 1.2b

Tue, Apr 02, 1996 11:02AM

PF SAMPLES :Run Number 28

9FP1-4
06/02/96

Sample File Name: GENWATER, Record: 61
Measured on: Tue, Apr 02, 1996 9:27AM Last saved on: Tue, Apr 02, 1996 9:28AM

Source: Analysod

Presentation: 255D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.211 %

Concentration = 0.060 %

Obscuration = 25.97 %

d(0.5) = 35.65 µm

d(0.1) = 6.02 µm

d(0.9) = 123.20 µm

D[4,3] = 58 µm

Span = 3.03

Sauter Mean, D[3,2] = 14.39 µm

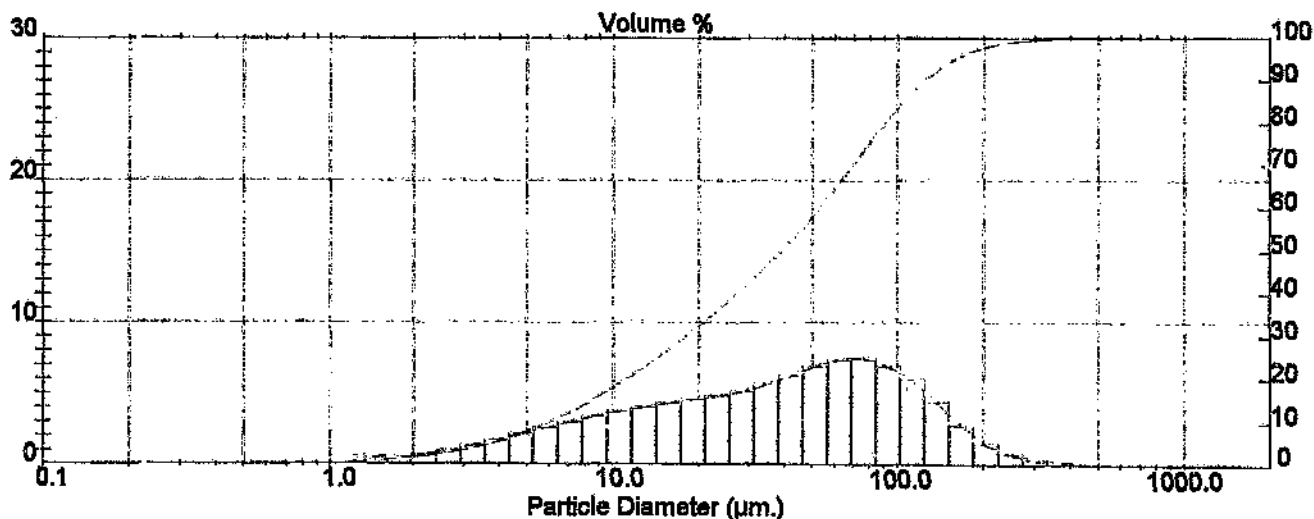
Mode = 71.83 µm

Specific Surface Area = 0.4170 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) µm	Result in %	Size (Hi) µm	Result Below %
0.50	0.57	1.32	0.57
1.32	0.46	1.60	1.04
1.60	0.48	1.95	1.51
1.95	0.68	2.38	2.20
2.38	0.95	2.90	3.15
2.90	1.28	3.53	4.43
3.53	1.69	4.30	6.12
4.30	2.13	5.24	8.25
5.24	2.57	6.39	10.82
6.39	2.99	7.78	13.80
7.78	3.38	9.48	17.18
9.48	3.72	11.55	20.91
11.55	4.01	14.08	24.92
14.08	4.31	17.15	29.23
17.15	4.55	20.90	33.77
20.90	4.81	25.46	38.58

Size (Lo) µm	Result in %	Size (Hi) µm	Result Below %
25.46	5.11	31.01	43.69
31.01	5.63	37.79	49.32
37.79	6.22	46.03	55.53
46.03	6.92	56.09	62.45
56.09	7.34	68.33	69.79
68.33	7.45	83.26	77.24
83.26	6.89	101.44	84.13
101.44	5.95	123.59	90.08
123.59	4.34	150.57	94.42
150.57	2.74	183.44	97.15
183.44	1.46	223.51	98.61
223.51	0.74	272.31	99.36
272.31	0.36	331.77	99.71
331.77	0.17	404.21	99.88
404.21	0.08	492.47	99.96
492.47	0.04	600.00	100.00





MASTERIZER X

Version 1.2b

Tue, Apr 02, 1996 11:05AM

PF SAMPLES :Run Number 29

9 C
06/02/96

Sample File Name: GENWATER, Record: 62
Measured on: Tue, Apr 02, 1996 9:30AM Last saved on: Tue, Apr 02, 1996 9:31AM

Source: Analysed

Presentation: 25\$D

Very Polydisperse model

Volume Result

Focus = 300 mm.

Residual = 0.219 %

Concentration = 0.048 %

Obscuration = 23.82 %

d(0.5) = 36.23 µm

d(0.1) = 5.18 µm

d(0.9) = 113.96 µm

D[4,3] = 50.98 µm

Span = 2.95

Mode = 64.24 µm

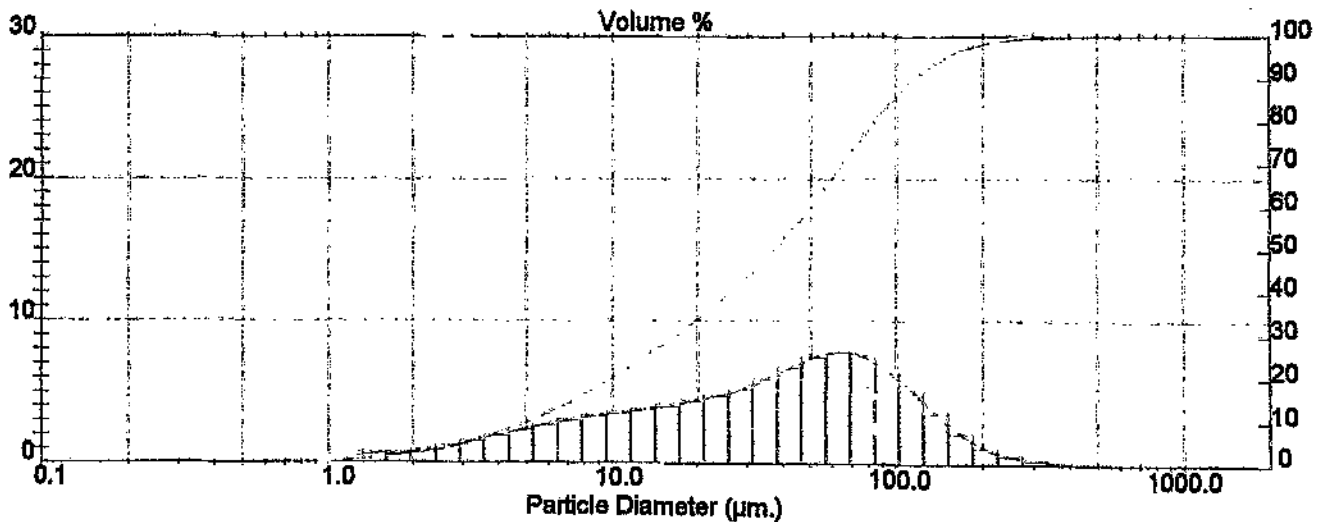
Sauter Mean (D[3,2]) = 12.65 µm

Density = 1.00 gm./c.c.

Specific Surface Area = 0.4743 sq. m./gm

Size (Lo) µm	Result in %	Size (Hi) µm	Result Below %
0.50	0.83	1.32	0.83
1.32	0.66	1.60	1.50
1.60	0.67	1.95	2.17
1.95	0.92	2.38	3.09
2.38	1.22	2.90	4.31
2.90	1.56	3.53	5.87
3.53	1.94	4.30	7.82
4.30	2.34	5.24	10.15
5.24	2.71	6.39	12.87
6.39	3.05	7.78	15.91
7.78	3.31	9.48	19.23
9.48	3.55	11.55	22.78
11.55	3.76	14.08	26.54
14.08	4.05	17.15	30.60
17.15	4.35	20.90	34.94
20.90	4.73	25.46	39.67

Size (Lo) µm	Result in %	Size (Hi) µm	Result Below %
25.46	5.19	31.01	44.86
31.01	5.90	37.79	50.77
37.79	6.67	46.03	57.44
46.03	7.47	56.09	64.91
56.09	7.77	68.33	72.67
68.33	7.54	83.26	80.21
83.26	6.53	101.44	86.74
101.44	5.22	123.59	91.96
123.59	3.57	150.57	95.53
150.57	2.17	183.44	97.70
183.44	1.15	223.51	98.85
223.51	0.60	272.31	99.45
272.31	0.30	331.77	99.75
331.77	0.15	404.21	99.90
404.21	0.07	492.47	99.97
492.47	0.03	600.00	100.00



PF SAMPLES :Run Number 35

9FP2-4 GRAP
07/02/96

Sample File Name: GENWATER, Record: 68
Measured on: Tue, Apr 02, 1996 9:47AM Last saved on: Tue, Apr 02, 1996 9:48AM

Source: Analysed

Presentation: 25SD

Very Polydispersa model

Volume Result

Focus = 300 mm.

Residual = 0.292 %

Concentration = 0.033 %

Obscuration = 24.80 %

d(0.5) = 16.97 µm

d(0.1) = 3.55 µm

d(0.9) = 73.79 µm

D[4,3] = 30.46 µm

Span = 4.14

Mode = 16.52 µm

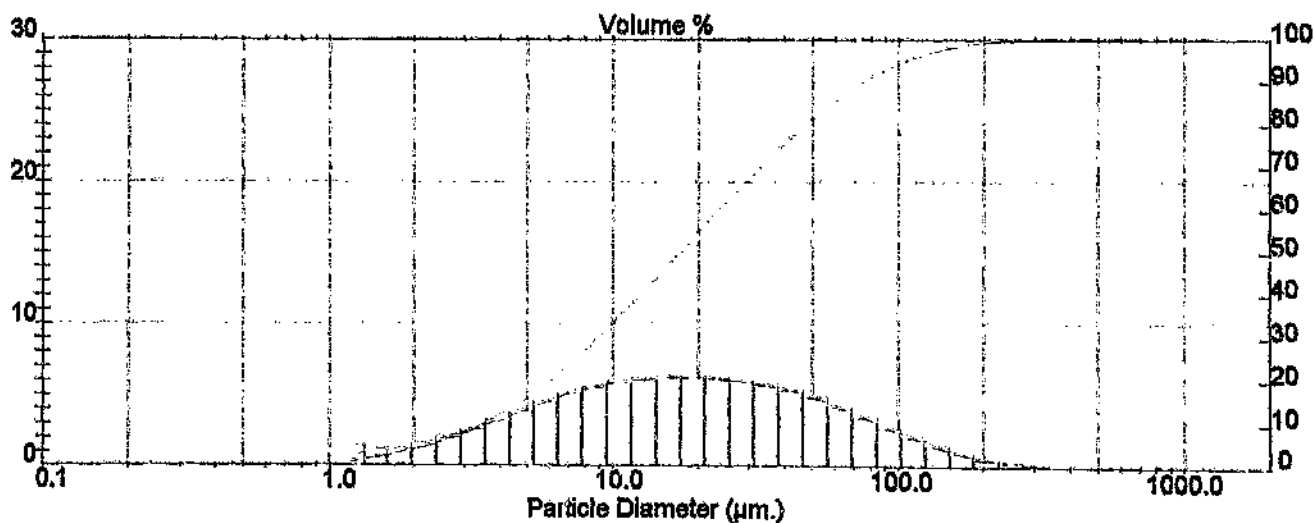
Sauter Mean (D[3,2]) = 8.25 µm

Specific Surface Area = 0.7276 sq. m. / gm

Density = 1.00 gm. / c.c.

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
0.50	1.45	1.32	1.45
1.32	1.15	1.60	2.60
1.60	1.16	1.95	3.76
1.95	1.56	2.38	5.32
2.38	2.04	2.90	7.36
2.90	2.57	3.53	9.93
3.53	3.18	4.30	13.11
4.30	3.83	5.24	16.94
5.24	4.51	6.39	21.45
6.39	5.12	7.78	26.57
7.78	5.57	9.48	32.14
9.48	5.89	11.55	38.02
11.55	6.09	14.08	44.11
14.08	6.24	17.15	50.34
17.15	6.25	20.90	56.59
20.90	6.14	25.48	62.73

Size (Lo) µm	Result In %	Size (Hi) µm	Result Below %
25.48	5.92	31.01	68.65
31.01	5.67	37.79	74.32
37.79	5.29	46.03	79.62
46.03	4.82	56.09	84.44
56.09	4.14	68.33	88.58
68.33	3.43	83.26	92.01
83.26	2.68	101.44	94.69
101.44	2.04	123.59	96.72
123.59	1.41	150.57	98.13
150.57	0.91	183.44	99.04
183.44	0.51	223.51	99.55
223.51	0.27	272.31	99.82
272.31	0.12	331.77	99.94
331.77	0.05	404.21	99.99
404.21	0.01	492.47	100.00
492.47	0.00	600.00	100.00



APPENDIX 3

Appendix 3.1.....	Grid discretisation file: Hen9b.dat
Appendix 3.2.....	Definition of inlet conditions: margbl.f
Appendix 3.3.....	Coal parameters: Composition, pyrolysis and combustion kinetics: combdat.f
Appendix 3.4.....	Specification for the NOx model: nodat.f

Appendix 3.1

Grid discretisation file: Her9b.dat

FILE: HEN9B.DAT
Grid Discretisation File

=====**Text zur Beschreibung des Gitters**=====

HEN9 - FILE fuer Gittererzeugung

=====**Daten zur Gittererzeugung**=====

HEN9 - FILE

LZYL kennzeichnet Zylinderkoordinaten

.FALSE. .FALSE.

IN=Anzahl der Gitterlinien in x-Richtung

77

JN=Anzahl der Gitterlinien in y-Richtung

48

KN=Anzahl der Gitterlinien in z-Richtung

101

Gitterlinien in x-Richtung [m]

0.0000

0.3697

0.6697

0.9697

1.2197

1.5297

1.7797

1.9397

2.0697

2.1761

2.2665

2.4375

2.6085

2.6989

2.8053

2.9353

3.0953

3.3950

3.6947

4.0047

4.2547

4.4147

4.5447

4.6511

4.7415

4.9125

5.0835

5.1739

5.2803

5.4103

5.5703

5.8203

6.1303

6.3803

6.6803

6.9803

7.1803

7.3620

7.3875

7.4130

7.5947

7.7947

8.0947

8.5947

8.6447

8.9547

9.2047

9.3647

9.4947	0.3880
9.6011	0.5080
9.6915	0.6580
9.8625	0.8380
10.0335	1.0380
10.1239	1.2680
10.2303	1.5280
10.3603	1.8280
10.5203	2.1780
10.8200	2.5780
11.1197	2.9780
11.4297	3.3780
11.6797	3.7280
11.8397	4.0780
11.9697	4.3780
12.0761	4.6780
12.1665	4.9780
12.3375	5.2780
12.5085	5.6125
12.5989	5.8755
12.7053	6.1755
12.8353	6.4385
12.9953	6.7730
13.2453	7.0730
13.5553	7.3730
13.8053	7.6730
14.1053	7.9730
14.4053	8.3230
14.7750	8.6730
Gitterlinien in y-Richtung [m]	9.0730
0.0000	9.4730
0.0900	9.8730
0.1880	10.2230
0.2880	10.5230

10.7835
11.0130
11.2130
11.3930
11.5430
11.6630
11.7630
11.8630
11.9610
12.0510

Gitterlinien in z-Richtung [m]

0.0000
0.5000
1.0000
1.5000
2.0000
2.5000
3.0000
3.5000
4.0000
4.5000
5.0000
5.5000
6.0000
6.5000
7.0660
7.2000
7.4000
7.7000
8.0500
8.3000
8.4500
8.6072
8.7136

8.8040
8.9750
9.1461
9.2365
9.3428
9.4500
9.5600
9.7600
10.1780
10.5960
11.0140
11.4320
11.8500
12.2000
12.4000
12.5000
12.6072
12.7136
12.8040
12.9750
13.1461
13.2365
13.3428
13.4500
13.5500
13.7500
14.1000
14.4500
14.8000
15.1500
15.5000
15.8500
16.2000
16.4000

16.5000
16.6072
16.7136
16.8040
16.9750
17.1461
17.2365
17.3428
17.4428
17.5628
17.6928
17.8228
18.1428
18.6428
19.1428
19.6428
20.1428
20.6428
21.1428
21.6428
22.1428
22.6428
23.1428
23.6428
24.1428
24.6428
25.1428
25.6428
26.1428
26.6428
27.1428
27.6428
28.1428
28.6428

29.1428
29.6428
30.1428
30.6428
31.0230
31.5000
32.0000
33.0000
34.0000
35.0000
***** Anzahl Einlaesse in xy-Ebenen 1 (neg. z) und 2 (pos.z)*****
0 0
***** Anzahl Auslaesse in xy-Ebenen 1 (neg. z) und 2 (pos.z)*****
0 1
Auslassnummer i,j,k - Koordinaten der Auslaesse in der xy-Ebene 2
1 2 77 2 48 101 101
***** Anzahl Einlaesse in xz-Ebenen 1 (neg. y) und 2 (pos. y)*****
96 96
Einlassnummer i,j,k - Koordinaten der Einlaesse in der xz-Ebene 1
101 11 11 2 2 24 27
101 12 13 2 2 24 24
101 14 14 2 2 24 27
101 12 13 2 2 27 27
401 10 10 2 2 23 28
401 11 14 2 2 23 23
401 15 15 2 2 23 28
401 11 14 2 2 28 28
402 29 29 2 2 23 28
402 24 24 2 2 23 28
102 28 28 2 2 24 27
102 25 25 2 2 24 27
402 25 28 2 2 23 23
102 26 27 2 2 24 24
402 25 28 2 2 28 28

102	26	27	2	2	27	27
403	55	55	2	2	23	28
403	50	50	2	2	23	28
103	54	54	2	2	24	27
103	51	51	2	2	24	27
403	51	54	2	2	23	23
103	52	53	2	2	24	24
403	51	54	2	2	28	28
103	52	53	2	2	27	27
404	69	69	2	2	23	28
404	64	64	2	2	23	28
104	68	68	2	2	24	27
104	65	65	2	2	24	27
404	65	68	2	2	23	23
104	66	67	2	2	24	24
404	65	68	2	2	28	28
104	66	67	2	2	27	27
204	65	65	2	2	42	45
204	68	68	2	2	42	45
504	64	64	2	2	41	46
504	69	69	2	2	41	46
203	51	51	2	2	42	45
203	54	54	2	2	42	45
503	50	50	2	2	41	46
503	55	55	2	2	41	46
202	25	25	2	2	42	45
202	28	28	2	2	42	45
502	24	24	2	2	41	46
502	29	29	2	2	41	46
501	15	15	2	2	41	46
501	10	10	2	2	41	46
201	14	14	2	2	42	45
201	11	11	2	2	42	45
202	26	27	2	2	42	42

502	25	28	2	2	41	41
501	11	14	2	2	41	41
201	12	13	2	2	42	42
204	66	67	2	2	42	42
504	65	68	2	2	41	41
203	52	53	2	2	42	42
503	51	54	2	2	41	41
202	26	27	2	2	45	45
502	25	28	2	2	46	46
501	11	14	2	2	46	46
201	12	13	2	2	45	45
204	66	67	2	2	45	45
504	65	68	2	2	46	46
203	52	53	2	2	45	45
503	51	54	2	2	46	46
304	65	65	2	2	61	64
304	68	68	2	2	61	64
604	64	64	2	2	60	65
604	69	69	2	2	60	65
303	51	51	2	2	61	64
303	54	54	2	2	61	64
603	50	50	2	2	60	65
603	55	55	2	2	60	65
302	25	25	2	2	61	64
302	28	28	2	2	61	64
602	24	24	2	2	60	65
602	29	29	2	2	60	65
601	15	15	2	2	60	65
601	10	10	2	2	60	65
301	14	14	2	2	61	64
301	11	11	2	2	61	64
302	26	27	2	2	61	61
602	25	28	2	2	60	60
601	11	14	2	2	60	60

301	12	13	2	2	61	61
304	66	67	2	2	61	61
604	65	68	2	2	60	60
303	52	53	2	2	61	61
603	51	54	2	2	60	60
302	26	27	2	2	64	64
602	25	28	2	2	65	65
601	11	14	2	2	65	65
301	12	13	2	2	64	64
304	66	67	2	2	64	64
604	65	68	2	2	65	65
303	52	53	2	2	64	64
603	51	54	2	2	65	65

Einlassnummer i,j,k - Koordinaten der Einlaesse in der xz-Ebene 2

307	26	27	48	48	61	61
307	28	28	48	48	61	64
307	26	27	48	48	64	64
307	25	25	48	48	61	64
308	14	14	48	48	61	64
308	12	13	48	48	61	61
308	12	13	48	48	64	64
308	11	11	48	48	61	64
207	28	28	48	48	42	45
207	26	27	48	48	42	42
207	25	25	48	48	42	45
207	26	27	48	48	45	45
208	14	14	48	48	42	45
208	12	13	48	48	42	42
208	11	11	48	48	42	45
208	12	13	48	48	45	45
107	28	28	48	48	24	27
107	26	27	48	48	24	24
107	26	27	48	48	27	27
107	25	25	48	48	24	27

108	12	13	48	48	27	27
108	12	13	48	48	24	24
108	14	14	48	48	24	27
108	11	11	48	48	24	27
305	68	68	48	48	61	64
305	66	67	48	48	61	61
305	65	65	48	48	61	64
305	66	67	48	48	64	64
306	52	53	48	48	61	61
306	54	54	48	48	61	64
306	51	51	48	48	61	64
306	52	53	48	48	64	64
205	68	68	48	48	42	45
205	66	67	48	48	42	42
205	65	65	48	48	42	45
205	66	67	48	48	45	45
206	54	54	48	48	42	45
206	52	53	48	48	42	42
206	52	53	48	48	45	45
206	51	51	48	48	42	45
105	68	68	48	48	24	27
105	66	67	48	48	24	24
105	65	65	48	48	24	27
105	66	67	48	48	27	27
106	52	53	48	48	24	24
106	54	54	48	48	24	27
106	51	51	48	48	24	27
106	52	53	48	48	27	27
405	69	69	48	48	23	28
405	64	64	48	48	23	28
406	50	50	48	48	23	28
406	55	55	48	48	23	28
407	24	24	48	48	23	28
407	29	29	48	48	23	28

408 15 15 48 48 23 28
 408 10 10 48 48 23 28
 407 25 28 48 48 23 23
 408 11 14 48 48 23 23
 405 65 68 48 48 23 23
 406 51 54 48 48 23 23
 407 25 28 48 48 28 28
 408 11 14 48 48 28 28
 405 65 68 48 48 28 28
 406 51 54 48 48 28 28
 505 64 64 48 48 41 46
 505 69 69 48 48 41 46
 506 50 50 48 48 41 46
 506 55 55 48 48 41 46
 508 10 10 48 48 41 46
 508 15 15 48 48 41 46
 508 11 14 48 48 41 41
 505 65 68 48 48 41 41
 506 51 54 48 48 41 41
 508 11 14 48 48 46 46
 505 65 68 48 48 46 46
 506 51 54 48 48 46 46
 605 64 64 48 48 60 65
 605 69 69 48 48 60 65
 606 50 50 48 48 60 65
 606 55 55 48 48 60 65
 607 29 29 48 48 60 65
 607 24 24 48 48 60 65
 608 10 10 48 48 60 65
 608 15 15 48 48 60 65
 607 25 28 48 48 60 60
 608 11 14 48 48 60 60
 605 65 68 48 48 60 60
 606 51 54 48 48 60 60

605 65 68 48 48 65 65
 606 51 54 48 48 65 65
 607 25 28 48 48 65 65
 608 11 14 48 48 65 65
 507 24 24 48 48 41 46
 507 29 29 48 48 41 46
 507 25 28 48 48 41 41
 507 25 28 48 48 46 46

***** Anzahl Auslaesse in xz-Ebenen 1 (neg. y) und 2 (pos.y)*****

0 0

***** Anzahl Einlaesse in yz-Ebenen 1 (neg. x) und 2 (pos. x)*****

0 0

***** Anzahl Auslaesse in yz-Ebenen 1 (neg. x) und 2 (pos.x)*****

0 0

***** Anzahl period. Raender in xy-Ebene 1 (neg.y) und 2 (pos.y) **

0 0

***** Anzahl period. Raender in xz-Ebene 1 (neg.y) und 2 (pos.y) **

0 0

***** Anzahl period. Raender in yz-Ebene 1 (neg.x) und 2 (pos x) **

0 0

***** min/max-Grenzen: Z-Linie, xmin, xmax, ymin, ymax

1 2 77 24 26

2 2 77 24 26

3 2 77 23 27

4 2 77 22 28

5 2 77 20 30

6 2 77 19 31

7 2 77 18 32

8 2 77 17 33

9 2 77 16 34

10 2 77 15 35

11 2 77 14 36

12 2 77 13 37

13 2 77 11 39
14 2 77 8 42
15 2 77 2 48

USW

WEITER

***** Ausblockierte Bereiche (0/1) = (nein/ja)

1

Ausblockierte Bereiche: x/y/z-Grenzen /xm...zpwax
(0=wand,1=einlass,2=auslass)

999 2 77 2 4 15 22 0 0 0 0 0 0
999 2 77 2 4 29 40 0 0 0 0 0 0
999 2 77 2 4 47 59 0 0 0 0 0 0
999 2 77 2 4 66 101 0 0 0 0 0 0
999 2 77 46 48 15 22 0 0 0 0 0 0
999 2 77 46 48 29 40 0 0 0 0 0 0
999 2 77 46 48 47 59 0 0 0 0 0 0
999 2 77 46 48 66 101 0 0 0 0 0 0
999 2 9 2 4 15 101 0 0 0 0 0 0
999 16 23 2 4 15 101 0 0 0 0 0 0
999 30 49 2 4 15 101 0 0 0 0 0 0
999 56 63 2 4 15 101 0 0 0 0 0 0
999 70 77 2 4 15 101 0 0 0 0 0 0
999 2 9 46 48 15 101 0 0 0 0 0 0
999 16 23 46 48 15 101 0 0 0 0 0 0
999 30 49 46 48 15 101 0 0 0 0 0 0
999 56 63 46 48 15 101 0 0 0 0 0 0
999 70 77 46 48 15 101 0 0 0 0 0 0
999 39 40 2 48 15 101 0 0 0 0 0 0
999 2 77 2 8 98 101 0 0 0 0 0 0
999 2 77 12 14 98 101 0 0 0 0 0 0
999 2 77 18 20 98 101 0 0 0 0 0 0
999 2 77 24 26 98 101 0 0 0 0 0 0
999 2 77 30 32 98 101 0 0 0 0 0 0
999 2 77 36 38 98 101 0 0 0 0 0 0

999 2 77 42 48 98 101 0 0 0 0 0 0
999 38 41 2 48 98 101 0 0 0 0 0 0

ENDE

***** Einblockierte Bereiche (0/1) = (nein/ja)

0

Einblockierte Bereiche: x/y/z-Grenzen /xm...zpwax
(0=wand,1=einlass,2=auslass)

6 13 8 11 3 21 0 0 0 0 0 0
2 2 2 2 2 21 0 0 0 0 0 0

ENDE

NUR 1 GITTER:DDM=0 ,GRUNDGITTEr: DDM=GITTErANZAHL.

SUBDOMAIN: DDM = GITTErNR.

0

Appendix 3.2

Definition of inlet conditions: margbl.f

FILE: margb1.f

Defines Inlet Conditions

```
C I-----I
C I      I              I
C I SR MARGBI I Besetz Brenner mit Eintrittswerten I
C I      I              I
C I-----I
```

```
SUBROUTINE MARGBI (U,V,W,TE,ED,VIS,RHO,F,H,T,G,CONC,SF^*,ZONC,RST
& ,GX,GY,GZ,FX,PY,PZ,PDX,PDY,PDZ,GDX,GDY,GDZ
& ,INLETS,NBCS,INLETSR)
```

```
C I-----I
C I INPUT :I
C I OUTPUT :I
C I-----I
C I      AUFRUF DURCH: I
C I-----I
C I usrip              I
C I-----I
```

```
INCLUDE 'CSDD.CMN'
INCLUDE 'CARRAY.CMN'
INCLUDE 'CBLANC.CMN'
INCLUDE 'CCOMB.CMN'
INCLUDE 'CDATA.CMN'
INCLUDE 'CDECOL.CMN'
INCLUDE 'CDIM.CMN'
INCLUDE 'CGEO.CMN'
INCLUDE 'COFFSET.CMN'
INCLUDE 'CPCBG.CMN'
INCLUDE 'CRADADD.CMN'
INCLUDE 'CRADGEO.CMN'
INCLUDE 'CSTEUER.CMN'
```

```
DIMENSION U(INPD,JNPD,KNPD),V(INPD,JNPD,KNPD),W(INPD,JNPD,KNPD)
DIMENSION TE(INPD,JNPD,KNPD),ED(INPD,JNPD,KNPD)
& ,VIS(INPD,JNPD,KNPD)
DIMENSION RHO(INPD,JNPD,KNPD),F(INPD,JNPD,KNPD),G(INPD,JNPD,KNPD)
DIMENSION H(INPD,JNPD,KNPD),T(INPD,JNPD,KNPD)
DIMENSION CONC(INPD,JNPD,KNPD,*),SKAL(INPD,JNPD,KNPD,9)
& ,ZONC(INPD,JNPD,KNPD,4)
```

```
& ,RST(INPD,JNPD,KNPD,6)
```

```
DIMENSION GX(0:INPD),GY(0:JNPD),GZ(0:KNPD)
& ,PX(INPD-2),PY(JNPD-2),PZ(KNPD-2)
DIMENSION PDX(INPD,JNPD,KNPD),PDY(INPD,JNPD,KNPD)
& ,PDZ(INPD,JNPD,KNPD)
DIMENSION GDX(INPD,JNPD,KNPD),GDY(INPD,JNPD,KNPD)
& ,GDZ(INPD,JNPD,KNPD)
```

```
DIMENSION INLETS(NBCS),INLETSR(NBCSR)
```

```
PARAMETER (MAXDAT=999)
```

```
DIMENSION UDAT(MAXDAT),VDAT(MAXDAT),WDAT(MAXDAT)
DIMENSION RHODAT(MAXDAT),FDAT(MAXDAT),TDAT(MAXDAT)
DIMENSION CDAT(MAXDAT,15)
DIMENSION DHYD(MAXDAT),TURINL(MAXDAT)
DIMENSION DRALL(MAXDAT)
```

```
IDEBUG = 0
```

```
NLI = INPD * JNPD
```

```
N_BRENNER = 24
```

```
PI = 4. * ATAN(1.)
```

```
GMPRK = 27.103/20 ! Gesamt-Massenstrom Brennstoff
GMPPL = 48.659/20 ! Gesamt-Massenstrom Primaerluft
GMPSL = 178.775/20 ! Gesamt-Massenstrom Sekundaerluft
```

```
BEDARF = SUMLB * GMPRK * 0.233
ANGEBOT = (GMPPL + GMPSL) * 0.233
QUOTE = ANGEBOT / BEDARF
```

```
AREASL = 0.2677 ! Sekundaerluft Eintrichtsflaeche pro Rringspalt
AREAPL = 0.1564 ! Primaerluft " " "
```

```
TPL = 80. + TREF
TSL = 248. + TREF
TWAND = 670. + TREF
```

RHOSL = PREF/(288.*TSL)

CNE Alle Grossen (ausser C) aller Einlaesse nullen

```
DO I=1,MAXDAT
  UDAT(I) = 0.0
  VDAT(I) = 0.0
  WDAT(I) = 0.0
  FDAT(I) = 0.0
  TDAT(I) = 0.0
  DRYD(I) = 0.0
  TURINL(I) = 0.0
ENDDO
```

CNE Alle Einlasszellen mit Luft belegen

```
DO I=1,MAXDAT
  CDAT(LISCOAL) = 0.
  CDAT(LISCHAR) = 0.
  CDAT(LISCH) = 0.
  CDAT(LISO2) = 0.233
  CDAT(LISCO2) = 0.
  CDAT(LISH2O) = 0.
  CDAT(LISCO) = 0.
  CDAT(LISASH) = 0.
  CDAT(LISN2) = 1.0 - CDAT(LISCOAL) - CDAT(LISCHAR)
& - CDAT(LISCH) - CDAT(LISO2) - CDAT(LISCO2)
& - CDAT(LISH2O) - CDAT(LISCO)
& - CDAT(LISASH)
ENDDO
```

CNE Brenner Primaereinlaesse in Betrieb

CNE =====

DO J=1,N_BRENNER

```
  IF (JEQ.1) I = 101
  IF (JEQ.2) I = 102
  IF (JEQ.3) I = 103
  IF (JEQ.4) I = 104
  IF (JEQ.5) I = 201
  IF (JEQ.6) I = 202
  IF (JEQ.7) I = 203
  IF (JEQ.8) I = 204
  IF (JEQ.9) I = 301
  IF (JEQ.10) I = 302
  IF (JEQ.11) I = 303
```

```
  IF (JEQ.12) I = 304
  IF (JEQ.13) I = 105
  IF (JEQ.14) I = 106
  IF (JEQ.15) I = 107
  IF (JEQ.16) I = 108
  IF (JEQ.17) I = 205
  IF (JEQ.18) I = 206
  IF (JEQ.19) I = 207
  IF (JEQ.20) I = 208
  IF (JEQ.21) I = 305
  IF (JEQ.22) I = 306
  IF (JEQ.23) I = 307
  IF (JEQ.24) I = 308
```

FAKON = 1. Idefiniert ob Brenner in Betrieb sind

IF (JGT.20) FAKON = 0.0

```
  CDAT(LISCOAL) = FAKON*GMPRK/(GMPRK+GMPPL)*ESUB
  CDAT(LISCHAR) = 0.
  CDAT(LISCH) = 0.
  CDAT(LISO2) = FAKON*GMPPL/(GMPRK+GMPPL)*0.233
  CDAT(LISCO2) = 0.
  CDAT(LISH2O) = FAKON*GMPRK/(GMPRK+GMPPL)*XSIF
  CDAT(LISCO) = 0.
  CDAT(LISASH) = FAKON*GMPRK/(GMPRK+GMPPL)*XSIA
  CDAT(LISN2) = FAKON*(1.0 - CDAT(LISCOAL) - CDAT(LISCHAR)
& - CDAT(LISCH) - CDAT(LISO2) - CDAT(LISCO2)
& - CDAT(LISH2O) - CDAT(LISCO)
& - CDAT(LISASH))
```

FAK = 1.

IF (JEQ.12) FAK = -1.

```
  UDAT(I) = 0.
  VDAT(I) = FAKON*FAK*(GMPPL*288.0+GMPRK*XSIF*RALLG**VMR2O)*TFL
& (PRE**AREAPL)
  WDAT(I) = 0.
  FJL(JE) RHODAT(I) = FAKON*(GMPRK + GMPPL)
& (ABS(VDAT(I)) * AREAPL)
  FDAT(I) = FAKON*1.0
  IF (FAKON.EQ.1.0) THEN
    TDAT(I) = TFL
  ELSE
```

```

TDAT(I) = TWAND
ENDIF
DHYD(I) = FAKON*0.1808
TURINL(I) = FAKON * 0.07
VANEP = 0. ! 'Schaufelteilung Primaerluft'
DP = 0.90 ! "Druckverlust' Drallschaufeln bis Muffel
DRAI L(I) = DP * TAN(VANEP*PI/180)
ENDDO

```

```

CNE BRENNER SEKUNDAEREINLAESSE
CNE =====

```

```

DO J=1,N BRENNER
IF (JEQ. 1) I = 401
IF (JEQ. 2) I = 402
IF (JEQ. 3) I = 403
IF (JEQ. 4) I = 404
IF (JEQ. 5) I = 501
IF (JEQ. 6) I = 502
IF (JEQ. 7) I = 503
IF (JEQ. 8) I = 504
IF (JEQ. 9) I = 601
IF (JEQ.10) I = 602
IF (JEQ.11) I = 603
IF (JEQ.12) I = 604
IF (JEQ.13) I = 405
IF (JEQ.14) I = 406
IF (JEQ.15) I = 407
IF (JEQ.16) I = 408
IF (JEQ.17) I = 505
IF (JEQ.18) I = 506
IF (JEQ.19) I = 507
IF (JEQ.20) I = 508
IF (JEQ.21) I = 605
IF (JEQ.22) I = 606
IF (JEQ.23) I = 607
IF (JEQ.24) I = 608

```

FAKON = 1. !definiert ob Brenner in Betrieb sind

IF (JGT.20) FAKON = 0.0

FAK = 1.
IF (JGT.12) FAK = -1.

```

UDAT(I) = 0.
VDAT(I) = FAKON * FAK * GMPSL / (RHO*SL*AREASL)
WDAT(I) = 0.
RHODAT(I) = FAKON * RHOSL
FDAT(I) = 0.
IF (FAKONEQ.1.0) THEN
TDAT(I) = TSL
ELSE
TDAT(I) = TWAND
ENDIF
DHYD(I) = FAKON* 0.2127
TURINL(I) = FAKON * 0.07
VANEM = 45. ! 'Schaufelteilung Mantelluft'
DP = 0.90 ! "Druckverlust' Drallschaufeln bis Muffel
DRALL(I) = DP * TAN(VANEM*PI/180)

```

ENDDO

NR = 0

```

DO JDO=7,12
CDIRS JVDEP
DO IDO=IPTD(JDO),IPTD(JDO+1)-1
INC = JA(IDXBCSD+IDO-1)
INC = INC + NND(JDO)
K = INT((INC-1)/NLI)-1
J = INT((INC-1-(K-1)*NLI)/INPD)+1
I = INC-(J-1)*INPD-(K-1)*NLI

```

NR = NR +1
NGRP = INLETS(NR)
V(L,K) = VDAT(NGRP) ! fuer Brenner in der i-Ebene

```

IF (K.LT.29.AND.ISDD.EQ.1) THEN
ZMITTE = 8.9750
ELSEIF (K.LT.51.AND.ISDD.EQ.1) THEN
ZMITTE = 12.9750
ELSEIF (ISDD.EQ.1) THEN
ZMITTE = 16.9750
ENDIF

```

```

IF (ISDD.EQ.2) THEN
  ZMITTE = 8.5750
ELSEIF (ISDD.EQ.3) THEN
  ZMITTE = 12.9750
ELSEIF (ISDD.EQ.4) THEN
  ZMITTE = 16.9750
ENDIF
BDZ = GZ(K) - ZMITTE - GDZ(L,J,K)/2

IF (LLT.20.AND.ISDD.EQ.1) THEN
  XMITTE = 2.4375
  FAK = 1
ELSEIF (LLT.30.AND.ISDD.EQ.1) THEN
  XMITTE = 4.9125
  FAK = -1
ELSEIF (LLT.60.AND.ISDD.EQ.1) THEN
  XMITTE = 9.8625
  FAK = 1
ELSEIF (ISDD.EQ.1) THEN
  XMITTE = 12.3375
  FAK = -1
ENDIF
IF (LLT.30.AND.ISDD.GT.1) THEN
  XMITTE = 2.4375
  FAK = 1
ELSEIF (ISDD.GT.1) THEN
  XMITTE = 4.9125
  FAK = -1
ENDIF
BDX = GX(I) - XMITTE - GDX(L,J,K)/2

BETA = ATAN2 (BDX,BDZ)
L(L,J,K) = ABS(V(L,K)) * COS(BETA) * DRALL(NGRP) * FAK
W(L,J,K) = -ABS(V(L,K)) * SIN(BETA) * DRALL(NGRP) * FAK

VEL = SQRT( U(L,K)**2 + V(L,K)**2 + W(L,K)**2 )

RHO(L,K) = RHODAT(NGRP)
T(L,K) = TDAT(NGRP)
IF (LOCX(FF)) F(L,K) = FDAT(NGRP)

IF (LPROP) THEN
  CONC(L,J,K,ISCOAL) = CDAT(NGRP,ISCOAL)
  CONC(L,J,K,ISCHAR) = CDAT(NGRP,ISCHAR)

```

```

DO JC = LNCHAR
  CONC(L,J,K,ISCHAR+JC) = WSIZE(JC)*CDAT(NGRP,ISCHAR)
ENDDO
CONC(L,J,K,ISCH) = CDAT(NGRP,ISCH)
CONC(L,J,K,ISO2) = CDAT(NGRP,ISO2)
CONC(L,J,K,ISCO2) = CDAT(NGRP,ISCO2)
CONC(L,J,K,ISH2O) = CDAT(NGRP,ISH2O)
CONC(L,J,K,ISN2) = CDAT(NGRP,ISN2)
CONC(L,J,K,ISH2) = CDAT(NGRP,ISH2)
CONC(L,J,K,ISCO) = CDAT(NGRP,ISCO)
CONC(L,J,K,ISASH) = CDAT(NGRP,ISASH)
CALL STC*K(INC,T(L,J,K),CONC,CP,CP0)
H(L,K) = CP*T(L,K) - CP0*TREF
ENDIF

IF (LOCX(KK).AND.(VEL.GT.0.)) THEN
  TEDUM = 1.5*TURINL(NGRP) * VEL)**2
  EDDUM = (CMUE**4*TEDUM**1.5)/(0.03*DHYD(NGRP)**2)
  TE(L,K) = TEDUM
  ED(L,K) = EDDUM
  VIS(L,K) = CMUE*RHO(L,K)*TEDUM**2/EDDUM + RMUE
  IF (NTURBD.GE.2) THEN
    RST(L,K,1) = 2./3.*TE(L,K)
    RST(L,K,2) = 2./3.*TE(L,K)
    RST(L,K,3) = 2./3.*TE(L,K)
    RST(L,K,4) = 0.
    RST(L,K,5) = 0.
    RST(L,K,6) = 0.
  ENDIF
  ELSE
    VIS(L,K) = 7.3E-03
  ENDIF

ENDDO
ENDDO

CBR -----
CBR Einlasstemperatur im Strahlungsgitter
CBR -----
IF (ISDD.EQ.1.AND.LDD) THEN
  C .AND.LRAYD) THEN
  NR = 0

```

```
DO JDO=7,12
CDIRS INDEP
DO IDO=IPTR(JDO),IPTR(JDO+1)-1
INC = JA(IDXBCSR+IDO-1)
INC = INC + NNR(JDO)
L = INT((INC-1)/NIJ)+1
J = INT((INC-(K-1)*NIJ)/INPD)+1
I = INC-(L-1)*INPD-(K-1)*NIJ
```

```
NR = NR + 1
NGRP = INLETSR(NR)
```

```
T(L,K) = TDAT(NGRP)
ENDDO
ENDDO
ENDIF
```

```
IF(IDEBUG.EQ.1)THEN
WRITE(31,10) ( (U(L,K),I=2,20),K=7,21 )
WRITE(32,10) ( (U(L,K),I=21,39),K=7,21 )
WRITE(33,10) ( (W(L,K),I=2,20),K=7,21 )
WRITE(34,10) ( (W(L,K),I=21,39),K=7,21 )
10 FORMAT(19(F6.2,2X))
ENDIF
```

```
END
```

Appendix 3.3

Coal parameters: Composition, pyrolysis and combustion kinetics: combdat.f

FILE: combdat.f

Coal Parameters (composition, pyrolysis and combustion kinetics)

```
C |-----|
C |   |   |   |
C |ISR COMBDAT | Parameter der Kohleverbrennung |   |
C |   |   |   |
C |-----|
```

SUBROUTINE COMBDAT

```
C |-----|
C |           | AUFRUF DURCH: |
C |-----|
C | usrip           |
C |-----|
```

INCLUDE 'CSDD.CMN'
INCLUDE 'CARRY.CMN'
INCLUDE 'CCOMB.CMN'
INCLUDE 'CSTEUER.CMN'

```
C -----
CCM Berechnung der Fluechtigenzusammensetzung
C -----
```

CCM Formel des Kohlenwasserstoffes
CNE Chemical Formula of hydrocarbon
CCM -----

DATA RNCX /1.0/
DATA RNHY /2.0/

WMCXHY = RNCX*WMC + RNHY*WMH2/2.

IF (LCOAL) THEN

CRS Heizwert [J/kg]
CNE Calorific Value (daf - derive from gross cal. value)
CRS -----

DATA HUDAF
& 32188782.

CRS Fluechtige laut Analyse (waf)
CNE Volatile Matter Content (daf) according to DTF Analysis
CRS -----

VMDAFA = 0.356725*1.226

CCM Elementaranalyse (roh !!!)
CNE Elemental Analysis (as received)
C -----

XSIC = 0.5247
XSIO = 0.0601 IO
XSIH = 0.0312 IH
XSIH = 0.0078 IS
XSIN = 0.0131 IN
XSIA = 0.2762 Ash
XSIF = 0.08696 Total Moisture

ELSE
VMDAFA = 1.0

XSIC = WMC/WMCXHY
XSIO = 0.
XSIH = 2.*WMH2/WMCXHY
XSIH = 0.
XSIN = 0.
XSIA = 0. Ash
XSIF = 0. Feuchte

ENDIF

CHK = XSIC + XSIO + XSIH + XSIH + XSIN + XSIA + XSIF

IF (CHK.NE.1.) THEN
WRITE(*,*) '>>> COMBDAT <<< '
WRITE(*,*) 'Summation der Elementaranalyse ungleich 1 '
WRITE(*,*) 'Summe :',CHK
WRITE(*,*) ''
ENDIF

CRS Bezug auf System ohne Schwefel und Stickstoff
CNE Recalc of Ultimate Elemental Analysis on a N and S free basis
CRS -----

XSIC = XSIC + XSIS + XSIN
 XSIH = XSIH
 XSIA = XSIA
 XSIF = XSIF
 XSIO = XSIO

CRS brennbare Substanz
 CNE combustible matter
 CRS
 BSUB = 1. - XSIA - XSIF

CCM Umrechnung der Elementaranalyse auf DAF (entspricht waf)
 CNE Recalc of Ultimate/Elemental on a daf basis

C
 SUMVM = XSIC + XSIH + XSIO
 XSICDAF = XSIC / SUMVM
 XSIHDAF = XSIH / SUMVM
 XSIODAF = XSIO / SUMVM

CCM minimal bzw maximal moeglicher Fluechtigenanteil (waf)

CNE possible max/min volatile matter contents on daf basis

CCM !! ggf Anpassung vom VMDAF !!

CCM
 VMCH4 = XSIODAF + XSIHDAF
 & + (WMC/(0.5*WMO2)) * XSIODAF
 & - (WMC/WMH2) * XSIHDAF

VMCO = VMCH4
 & - (WMH2/(0.5*WMO2) * XSIODAF - XSIHDAF)
 & + WMC/(0.5*WMH2)*(RNCX + RNHY/2.)/RNHY

VMH2O = VMCH4
 & - (WMC/(0.5*WMH2)) * XSIHDAF * (RNCX + RNHY/2.)/RNHY

VMMIN = MAX(VMCH4, VMCO)
 VMMAX = VMH2O

VMDAF = MAX(VMMIN, VMDAFA)
 VMDAF = MIN(VMMAX, VMDAF)

CCM C, H, O in Fluechtigen
 CNE C, H, O content in volatile matter

CCM
 XSICVM = (VMDAF - XSIHDAF - XSIODAF) / VMDAF
 XSIHVM = XSIHDAF / VMDAF
 XSIOVM = XSIODAF / VMDAF

IF (LCOAL) THEN
 CXHYNU1 = (XSICVM/WMC - XSIOVM/(0.5*WMO2) + XSIHVM/WMH2)
 & * (WMCXHY / (RNCX + RNHY/2.))

H2ONU1 = (XSIHVM/WMH2 - (RNHY/2.)*(CXHYNU1/WMCXHY)) * WMH2O

CONU1 = 1.0 - CXHYNU1 - H2ONU1

RNIRK = -1.0
 RNIC = 1. - VMDAF
 RNICO = CONU1 * VMDAF
 RN1H2O = H2ONU1 * VMDAF
 RN1CH4 = CXHYNU1 * VMDAF

RN1VM = VMDAF

CRS
 CRS massenbezogene stoichiometrische Koeffizienten
 CNE mass related stoichiometric coefficients
 CRS

DATA RN2C , RN2O2 , RN2CO
 & / -0.42857, -0.57143, 1. /
 DATA RN4CO , RN4O2 , RN4CO2
 & / -0.63636, -0.36364, 1. /
 DATA RN5H2 , RN5O2 , RN3H2O
 & / -0.1119, -0.8881, 1. /

CNE Kinetics of Devolatilisation
 c_standard_ivd DATA RK01,E1R / 1.5E05, 2900. /
 DATA RK01,E1R / 1.5E05, 8900. /
 CNE Kinetics of Char Combustion
 c_standard_ivd DATA RK02,E2R / 208. , 9553. /
 DATA RK02,E2R / 271. , 12817. /
 CNE Diffusion Constant

DATA DO2REF /3.49E-04/
 DATA FMP /0.875/
 DATA RHOS0 /1609./

CCM NCHAR Einfuehrung
 PHO_CA0 = RHOS0*(RN1C*(1.-XSIA-XSIF)+XSIA)
 CSR Faktor fuer Koksmodell (vgl.combcoal.f)
 RKASH = BSUE/XSIA

DATA NCHAR /3/
 DATA WSIZE(1) /0.3/
 DATA WSIZE(2) /0.4/
 DATA WSIZE(3) /0.3/
 DATA DPIJ(1) /50.E-06/
 DATA DPIJ(2) /50.E-06/
 DATA DPIJ(3) /50.E-06/

IF(NCHAR.GT.3) STOP 'NCHAR > 3 !!!!! COMBDAT STOP'

CCM Kohleparameter
 CRS _____
 DATA RN_B /0.67/ ! Ausbrandexponent (1-Bca)**n_B

CRS _____
 CRS Heizwerte [J/kg]
 CRS _____
 HUCH4 = HUDAF - RN1CO*HUCO
 & - (XSICDAF - (VMDAF-XSIIIDAF-XSIODAF))*HUKOKS
 HUCH4 = HUCH4/RNICH4
 HUR2 = HUKOKS - HUCO*WMCO/WMCO
 HUR3 = HUCH4 - HUCO*RNXC*WMCO/WMCXHY
 HUR4 = HUCO

ELSE
 RN1RK = 0.
 RN1C = 0.
 RN1CO = 0.
 RN1H2O = 0.
 RN1CH4 = 0.
 RN2C = 0.

RN2O2 = 0.
 RN2CO = 0.

CRS _____
 CRS Heizwerte [J/kg]
 CRS _____
 HUR2 = 0.
 HUR3 = HUCH4-HUCO*RNXC*WMCO/WMCXHY ! HUCH4 - HUCO*28/16
 HUR4 = HUCO

ENDIF

CRS _____
 CRS spezifischer Luftbedarf (nur Info)
 CRS _____
 CBO2 = XSIC * WMO2/WMCO
 HBO2 = XSII * WMO2/(2.*WMH2)
 SUMBO2 = CBO2 + HBO2 - XSIO
 SUMLB = SUMBO2 / 0.233

CCM _____
 CCM EDC-Modell
 CCM _____

CCM FUEL/AIR RATIO Reaktion von CxHy,CO,H2
 CRS _____

CRS stoech. Koeff. fuer CxHy vollstaendig bis zu CO2 + H2O
 ZAE3 = WMCXHY + (RNXC+RNHY/4)*WMO2
 RN3CXHYVOLL = - WMCXHY / ZAE3
 RN3O2VOLL = -(RNXC+RNHY/4)*WMO2 / ZAE3

RFU3 = ABS(RN3O2VOLL/RN3CXHYVOLL)
 RFU4 = ABS(RN4O2/RN4CO)
 RFU5 = ABS(RN5O2/RN5H2)

END

XSIC = XSH
XSIH = XSH
XSIA = XSL
XSIF = XSH
XSIO = XSH

CRS bremsare
CNE combustit
CRS _____
BSUB = 1. -

CCM Umroctm
CNE Recalc of
C _____
SUMVM =
XSICDAF =
XSHDAF =
XSIODAF =

CCM minimal
CNE possible n
CCM !! g
CCM _____
VMCH4 = X
& +(WM
& -(WAR

VMCO = V
& -(WMI
& +WM0

VMH2O = V
& +(WM

VMAN = A
VMAN = '

VMDAF = A
VMDAF = A

```

DATA DO2REF /3.49E-04/
DATA FMP /0.875/
DATA RHOSH /1609./

CCM NCHAR Einfuehrung
RHO_CA0 = RHOSH*(RNIC*(1.-XSLA-XSIF)+XSLA)
CSR Faktor fuer Koksmodell (vgl.combcoal.f)
RKASH = BSUB/XSLA

DATA NCHAR /3/
DATA WSIZE(1) /0.3/
DATA WSIZE(2) /0.4/
DATA WSIZE(3) /0.3/
DATA DPII(1) /50.E-06/
DATA DPII(2) /50.E-06/
DATA DPII(3) /50.E-06

IF(NCHAR.GT.3) STOP 'NCHAR > 3 !!!!! COMBDAT STOP'

CCM Koksparameter
CSR
DATA RN_B /0.67/ ! Anbrandexponent (1-Bca)**n_B

CSR
CSR Heizwerte [J/kg]
CSR
HUCH4 = HUDAF - RNICO*HUCO
& -(XSICDAF - (VMDAF-XSIHDAF-XSIODAF))*HUKOKS
HUCH4 = HUCH4/RN1CH4
HUR2 = HUKOKS - HUCO*WMCO*WMC
HUR3 = HUCH4 - HUCO*RNCX*WMCO*WMCXHY
HUR4 = HUCO

ELSE
RNIRK = 0.
RNIC = 0.
RNICO = 0.
RN1H2O = 0.
RN1CH4 = 0.

RN2C = 0.

```

```

RN2O2 = 0.
RN2CO = 0.

CSR
CSR Heizwerte [J/kg]
CSR
HUR2 = 0.
HUR3 = HUCH4-HUCO*RNCX*WMCO*WMCXE / HUCH4 - HUCO*28/16
HUR4 = HUCO

ENDIF

CSR
CSR spezifischer Luftbedarf (nur Info)
CSR
CBO2 = XSIC * WMO2/WMC
HBO2 = XSIH * WMO2/(2.*WMH2)

SUMBO2 = CBO2 + HBO2 - XSIO
SUMLB = SUMBO2 / 0.233

CCM
CCM EDC-Modell
CCM
CCM FUEL/AIR RATIO Reaktion von CxHy,CO,H2
CSR
CSR stoech. Koeff. fuer CxHy vollstaendig bis zu CO2 + H2O
ZAE3 = WMCXHY + (RNCX+RNHY/4.)*WMO2
RN3CXHYVOLL = - WMCXHY / ZAE3
RN3O2VOLL = -(RNCX+RNHY/4.)*WMO2 / ZAE3

RFU3 = ABS(RN3O2VOLL/RN3CXHYVOLL)
RFU4 = ABS(RN4O2/RN4CO)
RFU5 = ABS(RN5O2/RN5H2)

END

```

Appendix 3.4

Specification for the NOx model: nodat. f

FILE: nodat.f

Specifications for the NO_x model

```
C |-----|
C I      I      I
C I SR NODAT I Parameter der Kohleverbrennung      I
C I      I      I
C |-----|
```

SUBROUTINE NODAT

```
C |-----|
C I      AUFBRUF DURCH:      I
C |-----|
C I usrup      I
C |-----|
```

```
INCLUDE 'CSDD.CMN'
INCLUDE 'CARRY.CMN'
INCLUDE 'CCOMB.CMN'
INCLUDE 'CSTEUER.CMN'
```

```
CCM *****
CCM          NO-Modell
CCM *****
CCM -----
CCM NO Parameter
CCM -----
DATA XSINDAF ! N-Gehalt der Kohle
& / 0.0199 /
```

```
CCM Anteil der Kohle-N Freisetzung
CCM Pyrolyse , Koksabbrand , nicht verwendet!
DATA RL1HCN , RL2HCN , RL3HCN ! %eN von Nges als HCN ueber
& / 0.54 , 0.46 , 0.00 / ! RLx austretend
```

```
CCM -----
CCM 1 = original DeSoete- Modell nach Schnell
CCM 2 = neues Reburn-NO-Modell von C. Magel
CCM -----
```

NO_MODEL = 1

```
CCM -----
CCM Berechnung von NO Konstanten fuer Quellkernbestimmung
CCM -----
CCM XSINDAF schon von Kohle daf
RN1HCN = 1.0 *RL1HCN*XSINDAF/1.0 *27./14.
RN2HCN = ABS(RN2C) *RL2HCN*XSINDAF/ABS(RN1C) *27./14.
RN3HCN = 0.0 (ABS(RN3CH4) *RL3HCN*XSINDAF/ABS(RN1CH4)*27./14.
```

```
CCM -----
CCM Felder ausnullen
CCM -----
DO NMF=1,NCNOMAX
DO NRL=1,NRLNOMAX
VNO(NMF,NRL,1)=0.0
VNO(NMF,NRL,2)=0.0
ENDDO
DO NRL=1,6
VNSORC(NMF,NRL)=0.0
ENDDO
ENDDO
```

```
CCM -----
IF(NO_MODEL.EQ. 1)THEN
CCM -----
CCM original DeSoete- Modell nach Schnell
CCM Z2 = HCN
CCM Z3 = NO gesamt
CCM Z4 = NO thermisch Kontrollgroesse
CCM -----
VNSORC(1,1) = RN1HCN
CCM KOKS-N
VNSORC(1,2) = RN2HCN * 1.0 ! als HCN
VNSORC(2,2) = RN2HCN * 0.0 *30./27. ! als NO

VNO(1,2,2) = -1.
VNO(1,3,2) = -1.
VNO(2,2,1) = 30./27.
VNO(2,3,2) = -30./27.
VNO(2,4,1) = 1.
VNO(3,4,1) = 1.
```

ELSEIF(NO MODEL.EQ 2)THEN

CCM neues Reburn-Modell C Magel

CCM /2 HCN

CCM /3 NO gasmt

CCM /4

CCM

VNOSRC(1,1) RN1HCN

VNOSRC(1,2) RN2HCN*0.3

VNOSRC(2,2) RN2HCN*0.7*30 27

CCM --HCN-----

VNO(1,1,2) -1. !! Oxidation Mitchell

VNO(1,4,1) = 1. !! Reburning Wei Rate

C VNO(1,6,1) = 27. 30.!! Reburning proportional zu RI.3 (C Magel)

CCM --NO-----

VNO(2,2,1) = 30 17. !! Oxidation

VNO(2,3,2) = -30. 17 !! Reduktion

VNO(2,4,2) = -30. 27. !! Reburning Wei Rate

VNO(2,6,2) -1. !! Reburning proportional zu RI.3 (C. Magel)

VNO(2,5,2) -1. !! Charreduktion Levy Rate

CCM --NH3-----

VNO(3,1,1) 17 27.

VNO(3,2,2) = -1

VNO(3,3,2) = -1

ENDIF

RETURN

END

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