

A.2.2 EQUATIONS USED FOR COMPUTER PROPERTIES

The computation of the plant loads involves the determining of moist air properties. These properties are determined using the equations of statistical mechanics, in conjunction with the best experimental results, to yield accurate evaluations of the properties of dry air and water vapour⁽³⁾. These equations were developed by taking into account the forces between like molecules, in terms of viral coefficients, and between unlike molecules in terms of interaction coefficients and are used in subroutines in the program as follows:

Sub Prop (Patm, Tdb, Rh, W, Hdb, SVOA)

This subroutine determines the humidity ratio (W), enthalpy (Hdb) and specific volume (Svoa) of moisture: given the atmospheric pressure (Patm), dry bulb temperature (Tdb) and relative humidity (Rh). The relationships are as follows:

$$T = Tdb + 273.15 \quad \dots\dots\dots (1)$$

$$Pss = (10 (30.59 051 - 8.2 \ln(T) + 0.0024804 \times T - 3142.31/T)) \quad \dots\dots\dots (2)$$

$$Ps = Rh \times Pss \quad \dots\dots\dots (3)$$

$$Z = -0.00000662 \times Tdb^3 - 0.000194 \times Tdb^2 + 1.8375 \times Tdb + 2500.83$$

$$W = 0.62198 \times 1.004 \times Ps / (Patm - 1.004 \times Ps) \quad \dots\dots\dots (4)$$

$$Hdb = 1.005 \times Tdb + W \times Z \quad \dots\dots\dots (5)$$

$$Svoa = 2.87055 \times T \times (1 + 1.607 \times W) / Patm \quad \dots\dots\dots (6)$$

where

T = absolute temperature (K)

Pss = vapour pressure of steam air water up to 100 °C (bars)

Ps = partial pressure of dry air (bars)

Sub Humid (Patm, Tdb, Twb, Rh)

This subroutine calculates the relative humidity (Rh) of moisture air given the atmospheric pressure (Patm), dry bulb temperature (Tdb) and wet bulb temperature (Twb) (3). The equations are:

$$P_s = P_{ss} - P_{atm} * E * (T_{db} - T_{wb}) \dots\dots\dots (7)$$

$$Rh = P_s / P_{ws} \dots\dots\dots (8)$$

where

Pws = vapour pressure of steam over water at the wet bulb temperature (bars)

$$E = 0.000666 \text{ if } T_{wb} > 0 \text{ or } 0.000594 \text{ if } E < 0$$

Sub Relhumid (Patm, W, Tdb, Rh)

This subroutine determines the relative humidity given the atmospheric pressure, humidity ratio and dry bulb temperature. An iterative procedure is necessary for this routine. The initial estimate is Rh = 0.8. The algorithm is :

REPEAT

$$P = P_s$$
$$P_s = W * (P_{atm} - 1.004 * P_s) / (0.62198 * 1.004) \dots\dots (9)$$

$$ERROR = ABS (P - P_s) / P_s$$

$$Rh = P_s / P_{ss}$$

UNTIL ERROR < 0.0001

where Ps and Pss are as before

Sub Wet-bulb (Patm, Tdb, Rh, Twb)

This subroutine determines the wet bulb temperature (Twb) given the atmospheric pressure, dry bulb temperature and relative humidity. An iterative solution is necessary for this subroutine. The subroutine uses equation (9) in a rearranged form solving for the dry-bulb temperature for estimates of wet-bulb temperature until a bracket stranding the known dry-bulb temperature is located⁽¹⁾.

$$W = \frac{(2501 - 2.381 \text{ Twb})}{2501 + 1.805 \text{ Tdb} - 4.186 \text{ Twb}} \cdot \frac{W_{wb} - (T_{db} - \text{Twb})}{\dots} \quad (10)$$

where

Wwb = humidity ratio at the wet-bulb temperature
W = humidity ratio at the dry-bulb temperature.

The subroutine makes use of the interval-halving method (5) to determine the wet-bulb temperature. The subroutine computes the dry-bulb temperature initially for guesses of wet-bulb temperature until a bracket is located stranding the known dry bulb temperature. This bracket is the last two calculated values. The interval halving method is then used to obtain greater accuracy.

```
SET Twbb = Tdb - 8

DO WHILE T new < Tdb
  Twbb = Twb
  Twb = Twb - 8

  T_new = (2501 * (Ws-W) + Twb * (4.186 * W - 2.381 * Ws
    + 1)) / (1.805 * W+1)
END DO

Set T1 = Twb
Set T2 = Twbb

DO WHILE |T_new - Tdb| or |T_new - Tdb| / Tdb < 0.001
  T3 = (T1 + T2) / 2

  Determine T_new

  IF T_new > Tdb
    Set T2 = T3
  ELSE
    Set T1 = T3
  END IF

  Set Twb = T3
END DO
```

SUB RETURN (NOPLANT, N2, VEC (*), PLANT (*), Rh (*), T3
(*), L, Tret (*), Rhret (*))

This subroutine determines the return air temperature and humidity for each plant based on the average room conditions of each zone.

SUB Oatemp (L, START, END, TOAMA X, TOAMIN, TOASUP,
PATM, TOA (*), HOA (*), WOA (*), SVOA (*))

This subroutine determines hourly dry-bulb temperature (Toa (*)) by approximating a sine curve through maximum and minimum passed temperature values (Toamax, Toamin). Subroutine humid is called to calculate the outdoor relative humidity (Rh) and subroutine Prop is called to calculate the hourly humidity ratio (WOA), enthalpy (HOA) and specific volume (SVOA) for the dry-bulb temperatures.

The hourly outdoor dry-bulb temperature is determined using

$$TOA = TOAM - TOAM2 \times \sin(15 \times (J+3) \times Cf)$$

WHERE:

TOA = Outdoor dry-bulb temperature peaking at 3 pm

TOAM = Average outdoor temperature

$$= (Toamax + Toamin)/2$$

TOAM2 = Average difference of the maximum and minimum outdoor dry-bulb temperature.

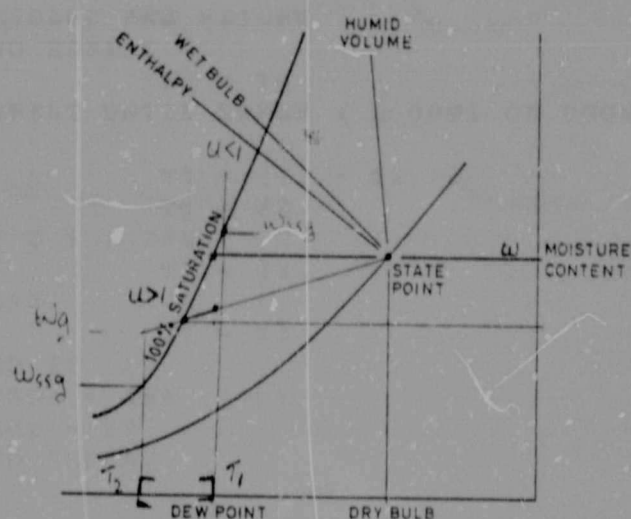
$$= (Toamax - Toamin)/2$$

CF = Radian factor

This subroutine determines the minimum fresh air quantity (Voa) required to comply with government regulations and to maintain adequate oxygen levels in the zones. Ventilation rates are calculated according to the number of occupants and the zone surface area; the selected rate is then the larger of the two.

SUB RADP (Tdiff, Patm, Tdb, Rh, Shf, Wg, Hg, Radp,
Message)

The room moisture content and enthalpy are calculated first using equations (2), (4) and (5). The program then assumes saturated temperature values until a bracket stranding the acceptable saturation temperature is located. A temperature bracket is formed when the ratio of the saturated moisture content (W_{ssg}) at the iterated dry-bulb temperature to the moisture content (W_g) at the room sensible heat ratio line at the same dry-bulb temperature exceeds 1 (i.e. when $U = W_g/W_{ssg} > 1$, see figure A2.2.1 below)



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The following equations are used:

$$H_g = H_{db} - 1.0048 * (T_{db} - T_g) / Shf \dots\dots\dots (11)$$

$$W_g = (H_g - 1.0048 * T_g) / (2501 + 1.84 * T_g) \dots\dots (12)$$

Wssg = as per eqn (4)

$$U = W_g / W_{ssg}$$

WHERE

H_g = Saturated enthalpy

H_{db} = Room enthalpy value

Shf = Room sensible heat factor

T_g = Saturated dry-bulb temperature

W_g = Humidity ratio at T_g on the Shf line

W_{ssg} = Saturated humidity ratio at T_g

The interval-halving method is then used to determine the Radp temperature within the acceptable relative error.

```
Determine W
      Hdb
      Tg
REPEAT UNTIL U > 1
      T1 = Tg
      Tg = Tg - 10
      U = Wg / Wssg
IF U ( 1 AND (Tdb - Tg) > Tdiff THEN PRINT
MESSAGE AND RETURN
END REPEAT
      T2 = Tg
REPEAT UNTIL ERROR < 0.0001 OR COUNT = 20
      T3 = (T1 + T2) / 2
      Tg = T3
IF U > 1 THEN
      T2 = T3
ELSE
      T1 = T3
END IF
ERROR = ABS (1-U)
Radp = T3
END REPEAT
```

SUP CONSVOL (Tran, (*), Hrm, Tdiff, X, Patm, Vsup, Msup,
Rload, Tdbon, Turbon, Tdboff, Twboff, Hon, Hoff, Tsup,
Tret, Adpt, Wadp, Hadp, Won, Woff, Svsup)

This subroutine determines the peak cooling parameters for the given outdoor and room conditions. The program terminates if a cooling cycle is not required. Use is made initially of subroutine Radp to determine whether the effective sensible heat factor (Eshf) intersects the saturation curve on the psychrometric chart. The message "NOT" is returned if the Eshf does not intersect the saturation curve. If the curve is intersected but the temperature drop across the cooler coil exceeds the allowable value, the message "LIMIT" is returned.

The subroutine sequence is as follows:

Terminate if $Tsh < 0$ or $Gth < = 0$ or $Eshf > = 1.0$ or

$Eshf < 0$

Call Radp

EXIT IF MESSAGE = "LIMIT" OR "NOT "

ELSE

$Msup = Tsh / (1.0048 * (Trm - Tsup) * (1-x)) \dots\dots\dots (13)$

END IF

Determine whether outdoor ventilation flowrate exceeds total flowrate.

IF $Msup < Moa$ then $Msup = Moa$; $Mret = 0$

Determine on coil states.

$Hon = (Hret * Mret + Hoa * Moa) / Msup \dots\dots\dots (14)$

$Tdbon = (Hon - Hret) * (Toa - Tret) / (Hoa - Hret) + Tret \dots (15)$

$Won = (Hon - Hret) * (Woa - Wrm) / (Hoa - Hret) + Wrm \dots\dots\dots (16)$

Determine off coil states

$Tdboff = Adpt + X * (Tdbon - Adpt) \dots\dots\dots (17)$

$Woff = Wadp + X * (Won - Wadp)$

Call Relhumid

Call Wet-bulb

Add supply fan heat gains

Determine resulting off coil states

Calculate refrigeration load

$Rload = Msup * ((Hon-Hoff)-42.11 * (Won-Woff)) \dots\dots (18)$

Sub Cv-pload (Trans (*), Des-Woff, Des-toffwb, Des-cap,
Des-toffdb, A, B, Vsup, Patm, Rload, Tdbon, Twbon,
Tdboff, Twboff, Hon, Hoff, Msup, X1, X2, Hload)

This subroutine determines the partial load operating states for the constant volume flowrate plant operating on a cooling and dehumidification with reheat control cycle.

The off coil conditions determined by subroutine consvol are fixed.

The summarized subroutine logic is as follows:

Find whether cooling or heating is required.

IF $Rsh < 0$ then heating is required.

Calculate mass flowrate for the given volume flowrate by determining the specific volumes at the room and supplying temperatures.

REPEAT

$Msup = Vsup * 2 / (Svm + Svsupw) \dots\dots\dots (19)$

UNTIL $ERROR < .001$ OR $COUNT > 10$

ELSE

Cooling cycle is required.

Calculate mass flowrate based on the design off-coil and room temperature specific volumes.

$Msup = Vsup * 2 / (Svsup + Svrn)$

Calculate return air states, Tret, Hret, Hon, Won, Tdbon, Rhon, Twbon

Determine whether refrigeration is required.

IF the resulting room temperature is within 1.5°C of design temperature. If so then exit.

Determine whether heating only is required.

IF so determine whether preheat is required then exit.

Determine whether cooling is required.

If $R_{sh} > 0$ cooling is required.

Calculate off coil conditions T_{doff} , T_{wboff} , H_{off} .

Calculate supply air conditions T_{sup} , H_{sup} .

Determine whether reheat in the cooling cycle is required.

If $T_1 > T_{sup}$ reheat cycle is required.

Calculate reheat quantity and resulting room temperature

ELSE

Determine room conditions for the design off coil states.

END IF

APPENDIX 3 : BUILDING EXAMPLE LAYOUT

A3.1 BUILDING EXAMPLE PLAN

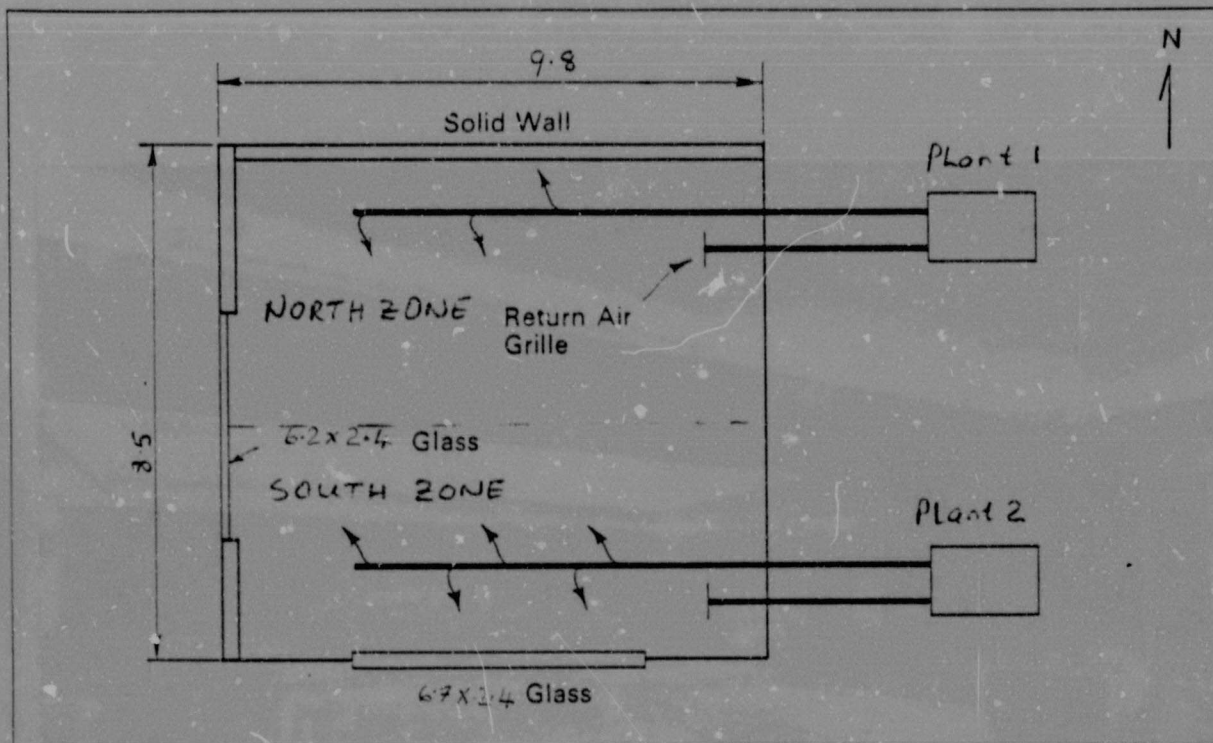


Figure A3.1 - Building example plan

A3.2 BUILDING EXAMPLE INTERIOR



Figure A3.2 .. Interior of building example

A3.3 ROOF MOUNTED AIRCONDITIONING UNIT

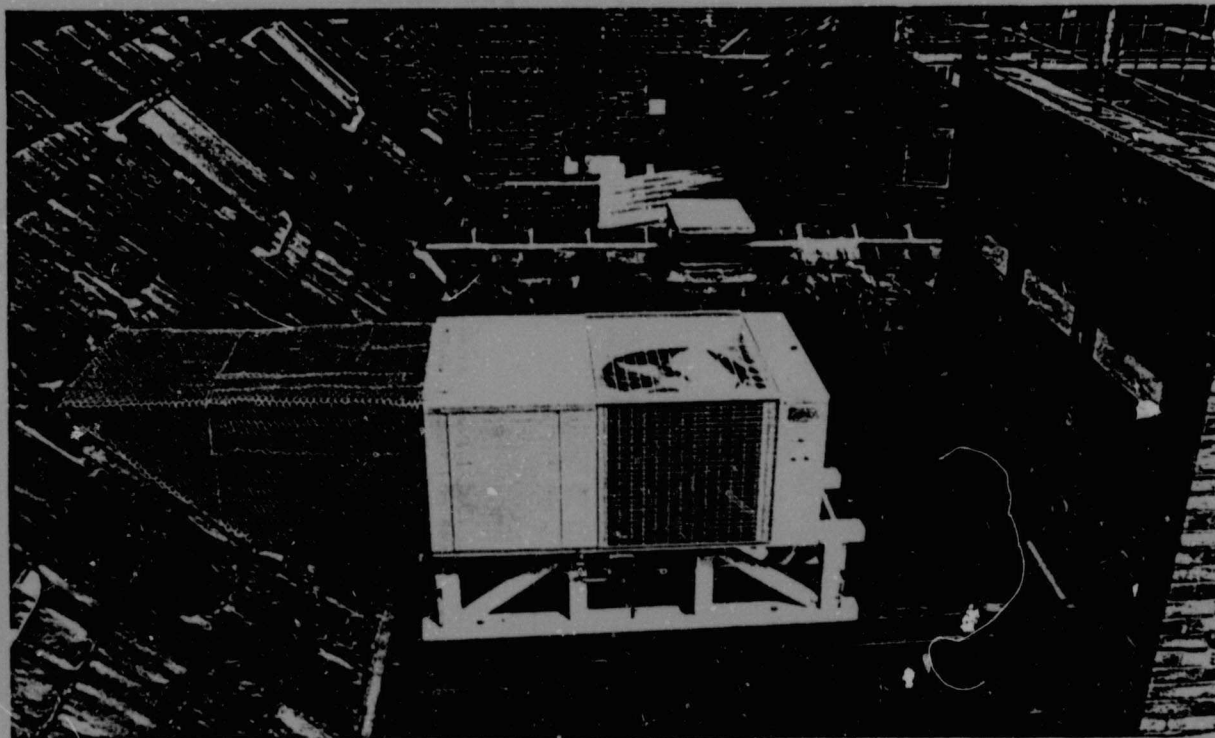


Figure A3.3 - Photograph of roof mounted unit

APPENDIX 4 : INPUT AND BUILDING LOADS FOR THE BUILDING EXAMPLE

The following tables are print-outs of the building parameters used in the building load calculation.

TABLE A4.1 OUTDOOR TEMPERATURES AND LOCATION DATA PRINT-OUT

STEYN VAN RENSBURG INC CONSULTING ENGINEERS SUNNYSIDE PRETORIA PROJECT : MSC PSY PROGRAM PROJECT LOCATION : WITS UNIVERSITY PROJECT NUMBER : MSC									
***** AIR - CONDITIONING LOADS PROGRAM BY A DA SILVA *****									
***** DETAILED PROJECT ZONE DATA PRINTOUT*****									
***** TEMPERATURE & LOCATION DATA *****									
Outdoor Max db Deg C	Outdoor Min db Deg C	Outdoor Max wb Deg C	Site Lat Deg S	Site Alt m ASL	Site Reflec Factor	Zone Design Deg C	Zone Rel h Z	Zone Int W Deg C	Zone Floor T Deg C
3.0		0.0	27.0	1552.0	200				
WINTER DESIGN TEMP									
SITE LOCATION DATA									
DECEMBER									
NORTH ZONE									
SOUTH ZONE									
						22.0	50.0	28.0	22.0
						22.0	50.0	28.0	22.0

Author Da Silva A J

Name of thesis A computer program for assessing the hourly and peak refrigeration loads of an airconditioning constant volume flow plant 1987

PUBLISHER:

University of the Witwatersrand, Johannesburg

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